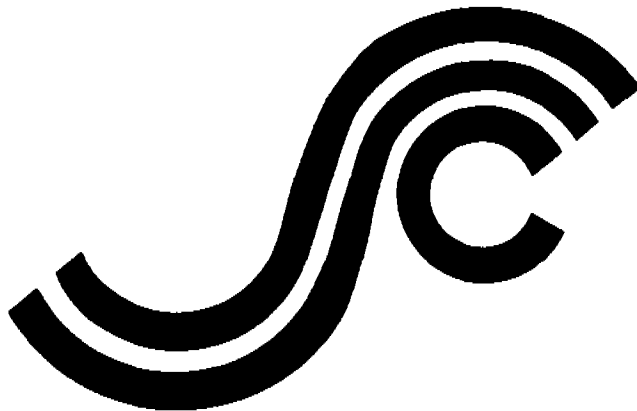


**SSC-333**

**ADVANCED METHODS FOR  
SHIP MOTION AND WAVE  
LOAD PREDICTION**



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**1990**

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SSC-333  
SR-1277

**ADVANCED METHODS FOR SHIP  
MOTION AND WAVE LOAD PREDICTION**

Advanced numerical methods are needed by ship designers to better predict and simulate ship motions and hull girder loads. Complex structural loading problems such as bottom slamming, bow flare impact, and green water on deck cannot be satisfactorily analyzed using linear strip theory.

This report provides a numerical method for predicting transient three-dimensional hydrodynamic pressures and resulting loads. This work is based on an initial level of investigation and development, and will require further testing, validation, and refinement of the numerical methods and computer programs.

A handwritten signature in black ink, appearing to read 'J. D. Sipes', is written over the printed name.

J. D. SIPES  
Rear Admiral, U. S. Coast Guard  
Chairman, Ship Structure Committee

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## ABSTRACT

A numerical method for the simulation of ship motions, hull girder loads and transient three-dimensional hydrodynamic pressures is developed. A time-step integration of the equations of motion is performed with sectional hydrodynamic forces computed using strip theory and buoyancy forces evaluated over the instantaneous immersed hull. In addition to hydrostatic and dynamic buoyancy forces, ship motion induced and wave diffraction forces, the method can also account for "flare" force and quadratic damping forces. The numerical model has several features which may provide for more realistic simulations, including the ability to evaluate sectional hydrodynamic coefficients at the instantaneous sectional draft, and a crude scheme for adjusting the frequency at which sectional coefficients are evaluated for use in ship motion induced force calculations. Linear wave theory is assumed for the representation of the incident wave field, and regular and irregular, unidirectional or multidirectional wave systems can be generated.

A method for the prediction of transient three-dimensional hydrodynamic pressures extending the technique developed by R.B. Chapman is presented. Initial and boundary values for the pressure simulation are simultaneously generated from the solution of the equations of motion using sectional force computations. An option exists in the program to substitute the three-dimensional pressures for the two-dimensional hydrodynamic forces.

Computer programs have been developed to perform the computations -- the main simulator is called SSX. There are also two preprocessors -- HYDREX2 and HYDREX3. A Frank close-fit computation of sectional hydrodynamic coefficients is performed by HYDREX2. HYDREX3 performs computations to derive geometric data and several large arrays necessary for the transient pressure simulation.

The work presented herein represents an initial level of investigation and development. The numerical methods and computer programs require further testing, validation and improvement.

## NOMENCLATURE

### Equations of Motion and Strip Theory

|             |   |  |
|-------------|---|--|
| $A_{jk}$    | = | components of added mass matrix                    |
| $B_{km}$    | = | transformation matrix for Eulerian angles          |
| $BM_y(x_0)$ | = | vertical bending moment at $x_0$                   |
| $BM_z(x_0)$ | = | horizontal bending moment at $x_0$                 |
| $F_j$       | = | external forces                                    |
| $I_{jk}$    | = | components of moment and product of inertia matrix |
| $K$         | = | roll moment  |
| $M$         | = | pitch moment                                       |
| $M_{jk}$    | = | components of mass matrix                          |
| $M_s$       | = | ship's mass  |
| $N$         | = | yaw moment   |
| $R_G$       | = | vector position of ship c.g. in space system       |
| $SF_y(x_0)$ | = | vertical shear force at $x_0$                      |
| $SF_z(x_0)$ | = | horizontal shear force at $x_0$                    |

NOMENCLATURE (Continued)

|                       |  |
|-----------------------|--|
| $S_i, C_i$            | = sine and cosine components of force fluctuation for wave component $i$ , the relative magnitudes of which provide the proper phase relationship of all wave-related forces |
| $T_j$                 | = external moments   |
| $TM_x(x_0)$           | = torsional moment at $x_0$  |
| $U$                   | = ship's speed   |
| $V_{G_k}$             | = translational velocities of center of mass in space fixed system   |
|                       | = $\frac{d}{dt} (R_G)$   |
| $X$                   | = longitudinal force   |
| $Y$                   | = vertical force   |
| $Z$                   | = lateral force  |
| $X_b^S \dots M_b^S$   | = hydrostatic force components of the Froude-Krylov force  |
| $X_{3D} \dots M_{3D}$ | = force components calculated from the three-dimensional pressure computations   |
| $a_i$                 | = amplitude of $i^{\text{th}}$ wave component  |
| $a_{22}$              | = heave added mass   |
| $a_{33}$              | = sway added mass  |

NOMENCLATURE (Continued)

|   |   |
|---|---|
| $a_{34}$  | = section roll added mass moment of inertia due to sway motion            |
| $a_{44}$  | = sectional roll added mass moment of inertia                             |
| $b_{22}$  | = heave damping coefficient   |
| $b_{33}$  | = sway damping coefficient  |
| $b_{34}$  | = sectional roll damping moment coefficient due to sway motion            |
| $b_{44}$  | = sectional roll damping coefficient                                      |
| $b$   | = Froude-Krylov force subscript   |
| $\frac{dY_h}{dx}, \frac{dZ_h}{dx}, \frac{dK_h}{dx}$ | = sectional ship motion-induced hydrodynamic force in heave sway and roll |
| $\frac{dY_w}{dx}, \frac{dZ_w}{dx}, \frac{dK_w}{dx}$ | = sectional diffraction force in heave, sway and roll                     |
| $f$   | = "flare" force subscript   |
| $h$   | = ship motion-induced force subscript                                     |
| $h_{cg}$  | = vertical distance from waterline to ship c.g. (+ up).                   |
| $i, N$  | = wave index, total number of wave components                             |
| $i_x$   | = local section's mass moment of inertia                                  |
| $\vec{k}_i$   | = horizontal vector wave number   |

NOMENCLATURE (Continued)

|                       |   |  |
|-----------------------|---|--|
| $\vec{k}_i \vec{x}_i$ | = | $xk_i \cos\beta - zk_i \sin\beta$                                      |
| $\hat{k}_i$           | = | projection of the wave number onto the x-axis of ship                  |
| q                     | = | quadratic damping force subscript                                      |
| t                     | = | total force  |
| w                     | = | wave diffraction force subscript                                       |
| x                     | = | longitudinal distance to section from ship's c.g.                      |
| $\vec{x}_i$           | = | horizontal cartesian coordinates                                       |
| $\bar{y}$             | = | local section's center of gravity relative to ship c.g. (positive up). |
| $y_G, z_G$            | = | heave and sway velocity of center of mass of ship                      |
| $\beta$               | = | direction of wave propagation  |
| $\omega_i$            | = | circular frequency   |
| $\epsilon_i$          | = | phase angle  |
| $\omega_k$            | = | rotational velocities = $B_{km} \frac{d\alpha_m}{dt}$                  |
| $\psi, \dot{\psi}$    | = | Eulerian pitch angle and velocity in ship system                       |
| $\phi, \dot{\phi}$    | = | Eulerian yaw angle and velocity in ship system                         |

## NOMENCLATURE (Continued)

|                            |  |
|----------------------------|--|
| $\theta, \dot{\theta}$     | = Eulerian roll angle and roll velocity in ship system |
| $\xi_i, \dot{\xi}_i$       | = average vertical water velocity, acceleration        |
| $\zeta_i, \dot{\zeta}_i$   | = average lateral water velocity, acceleration         |
| $\kappa_i, \dot{\kappa}_i$ | = average roll water velocity, acceleration            |
| $\xi_i, \dot{\xi}_i$       | = average vertical water velocity, acceleration        |
| $\zeta_i, \dot{\zeta}_i$   | = average lateral water velocity, acceleration         |
| $\kappa_i, \dot{\kappa}_i$ | = average roll water velocity, acceleration            |
| $\mu$                      | = vertical sectional added mass gradient               |

### Three-dimensional Analysis

|                         |  |
|-------------------------|--|
| $A_{nm}, B_{nm}$        | = complex functions of wave numbers and time   |
| $A_{nm}^*, B_{nm}^*$    | = complex conjugates   |
| $A_i(t)$                | = area of panel $i$  |
| $E_{ij}$                | = source - velocity influence matrix   |
| $L_x, L_y, l_x, l_y, T$ | = parameters for computing spectral free-surface representation wave numbers and spacing |
| $\hat{P}_{ij}$          | = source - potential influence matrix  |

NOMENCLATURE (Continued)

|                            |  |
|----------------------------|--|
| $P_T, P_B, P_{FS}, P_A$    | = pressure at panel center, total, body, free-surface, ambient                           |
| $X_{ij}$                   | = source - x velocity influence matrix   |
| $a_T, a_B, a_{FS}, a_A$    | = acceleration vector  |
| $k_{nm}$                   | = $k_{nm}^2 = kx_n^2 + ky_m^2$   |
| $kx_n, ky_m$               | = wave numbers in x and y; e.g., $\frac{\omega_n^2}{g}$                                  |
| $s_i(t)$                   | = strength of source at panel i  |
| $v_T, v_B, v_{FS}, v_A$    | = velocities vector at panel centers, total, body-induced, free-surface and ambient wave |
| $v_X, v_Z, a_X, a_Z$       | = x and z components of ambient wave-induced velocity and acceleration at panel center   |
| $\phi_T$                   | = total time varying potential   |
| $\phi_B$                   | = instantaneous effect on the body   |
| $\phi_{FS}$                | = free surface disturbance due to previously radiated and diffracted waves               |
| $\phi_A$                   | = ambient wave potential   |
| $\sigma_i$                 | = source density of panel i  |
| $\Delta kx_n, \Delta ky_m$ | = wave number spacing; e.g., $\Delta kx_n = kx_{n+1} - kx_n$                             |

NOMENCLATURE (Continued)

$$\omega_{nm} = \omega_{nm}^2 = g [kx_n^2 + ky_m^2]^{1/2}$$

$$\Delta A_{nm}^{BODY}, \Delta A_{nm}^{BODY*}, \Delta B_{nm}^{BODY}, \Delta B_{nm}^{BODY*}$$

= influence of body on free surface spectral component



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## 1.0 INTRODUCTION

While the established ship motion and loading methods currently used in design practice remain indispensable, there is need for a more advanced tool that can carry the ship designer a step closer toward a fully rational ship design procedure. Although the application of linear strip theory in the frequency and probabilistic domains has met with remarkable success, there are a number of practical seakeeping and structural loading problems not satisfactorily analyzed by such techniques. Green water, bow flare impact and bottom slamming are particularly obvious examples. Reported large differences in dynamic sagging and hogging moments in some type of ships is a consequence of nonlinear loading [1,2]. The accurate prediction of rolling amplitudes requires the use of a nonlinear restoring moment at a large angle and nonlinear viscous damping estimates. These examples tend to highlight the generally nonlinear character of the ship motion and loads problem. An equally important part of the problem is its three-dimensionality. For example, the prediction of hydrodynamic pressures on the ship's hull below the mean waterline using linear two- and three-dimensional methods is not at a consistent level of accuracy. Clearly, there are a number of areas where improvements can be made in ship motion and loading prediction.

Based on an extensive review of the reported advanced methods in body-wave hydrodynamics and ship motion/loads prediction, a time-domain simulation of ship motion, hull girder loads and hydrodynamic pressures has been developed. Strip theory with certain nonlinear enhancements is used to predict sectional hydrodynamic forces for motion and hull girder loading prediction, and a three-dimensional source distribution of the hull, combined with a spectral representation of the free surface is used for the evaluation of hydrodynamic hull pressures. Linear wave theory is used and some optional numerical features have been included that may improve the predictions in some cases. The time-domain numerical simulation technique using strip theory-derived forces expands upon the capsizing simulation work of Oakley, Paulling, Wood and others [3]. In that method, the hydrodynamic forces from ship motion and diffraction are assumed to be small due to low encounter frequency in following and quartering waves -- allowing for approximate formulations. On the other hand, the Froude-Krylov forces (hydrostatic and dynamic buoyancy) are computed very accurately for the instantaneous immersed hull. In the present model, this accurate approach to Froude-Krylov forces has been used, while the ship motion and diffraction-induced forces are formulated according to standard strip theory equations [4].

Three-dimensional hull pressures and resultant loads are computed by a separate method based on Chapman's techniques for the simulation of arbitrary linearized motions of a floating body [5,6]. The present method extends Chapman's formulation to include ambient wave field and associated diffraction effects. Although the three-dimensional technique can be generalized to include a time-dependent hull shape, the computational method has been developed here in as simple a form as possible. Thus, as presently established, the hull is specified by surface panels describing the hull portion directly beneath the static-waterline only. The computational method can be easily expanded to include the entire hull; however, at present the computational effort to run a program with such an extension would be prohibitively expensive.

Several optional features have been incorporated into the numerical simulation procedures. Sectional added mass and damping coefficients may be evaluated for the instantaneous average sectional draft at each time step. Also the frequencies at which the added mass and damping coefficients are evaluated (for ship motion-induced forces) can be selected to correspond to either (a) the peak frequencies of motion response spectra, or (b) allowed to vary based on the measured response of the vessel during the previous two cycles of motion. Additionally, a vertical velocity-squared term is included in the strip formulations to partially account for the effect of flare impact loading. Finally, the simulation can incorporate the effects of velocity-squared damping using quadratic damping coefficients selected by the user.

It should be pointed out that because the majority of the testing and validation efforts described in this report use regular waves, the reader may tend to form the impression that the method is limited to regular waves. This is not the case. The incident wave system may be irregular and multidirectional. It is constructed from the superposition of regular waves, the amplitudes, frequencies, directions and phases of which may be specified by the user. They can thus be selected to represent the time history associated with any given sea spectrum.

This document (Volume I) describes the method and presents some test and validation results. Several computer programs were written to implement the approach and a user's manual (Volume II) is provided as a companion report.

## 1.1 BACKGROUND

In recent years, a number of impressive advances have been made in body-wave hydrodynamics. In particular, the fast-growing field of numerical ship hydrodynamics is producing a wide range of promising techniques with the potential to handle difficult nonlinear three-dimensional hydrodynamic phenomena. It is enlightening to follow the progress made by various investigators as presented in the First, Second and Third International Conferences on Numerical Ship Hydrodynamics in 1975, 1977 and 1981, respectively. Some of those efforts and others pertinent to the ship motion problem will be noted.

Efforts towards quasi-analytical solutions to Laplace's equation using singularity distributions for the ship motion problem have been reported by Chapman [6] and Chang [7]. M.S. Chang represents the ship hull by source/dipole distributions and solves the fluid domain and free surface equations using a mean hull boundary condition. It is a frequency-domain solution and appears to be an excellent way to find the correct linear wave excitation load distribution along the ship for any heading, speed and wave length. Solutions in the frequency domain assume that the response of the body to excitations of varied frequency content may be represented as the linear summation of responses to a series of discrete frequencies. Thus the equations of motion need to be solved only for the component frequencies. In contrast, in a time-domain solution, a time-history of the excitation is decomposed into a series of time steps with the time history of the response constructed by re-solving the equations of motion at each time step. R.B. Chapman's method is a time-domain computation somewhat analagous to Chang's approach. However, it does

not use Green's functions representing solutions of the free surface equations for oscillating singularities moving with uniform speed as does Chang. Simple sources and their image are used to represent the hull. Although the problem is linearized, the method is theoretically capable of accounting for a time-dependent hull shape. It thus provides the flexibility of arbitrary motions.

Another effort using singularity distributions includes that of Pettersen [8]. Pettersen's work is particularly interesting. As the problem is now formulated, he solves the integral equation at zero frequency for three-dimensional sources over the bow and stern and two-dimensional sources using strip theory for the middlebody. The approach has applications for ship maneuvering analysis but some work is being done for  $\omega \neq 0$ . Methods of finite difference potential field solutions to a free surface and wave problems have been reported by von Kerczek [9] and McCormick and J.W. Thomas [10], to mention only a few.

Bai [11] has continued to develop a finite element method applied to a three-dimensional harmonic ship-motion problem. The computational domain is reduced to a very small local domain using an eight-node three-dimensional element. The work is a direct extension of the earlier work by Bai and Yeung [12]. Other studies of finite element method applied to time harmonic ship motion problems have been presented by Chen and Mei [13], Seto and Yamamoto [14], Yue, Chen and Mei [15] and Chowdbury [16]. Emphasis in these cases is on the solution of regular harmonic motions, although finite element techniques have also been applied to transient problems by J.T. Beale [17] and C. Licht [18]. A. Jami and M. Lenoir [19] present a method for coupling finite elements and an integral representation.

Limited improvements to strip theory in the area of wave excitation and diffraction effects have been reported by S.O. Skjoldal [20], and Troesch [21].

Research in the area of incompressible viscous flow has yielded several interesting techniques. Bourianoff and Penumalli [22] have developed a three-dimensional time-dependent ship motion simulation using a method they call the Inertial Marker Particle (IMP) technique. It is a Eulerian approach. A Eulerian approach uses a space-fixed reference system through which the fluid flows. In the alternate Lagrangian description, the reference frame moves with the fluid. A Lagrangian approach to transient-free surface flows was reported by Fritts and Boris [23]. Nichols and Hirt [24] report on progress being made with variants and improvements of the original Marker-and-Cell method in free-surface computations.

In the area of ship motion and loads prediction, the more advanced methods incorporate a time-simulation approach using some form of strip theory. The primary motivation for the development of this class of simulation programs is to account for nonlinearities from various sources. Paulling, Oakley and Wood [3] were interested in the capsizing problem with emphasis on accurate computation of buoyancy forces. Chang [25] developed a motion simulator emphasizing non-rigid body structural response to regular head seas. Meyerhoff and Schlacter [26] emphasized the dynamic whipping response due to bottom and flare impact in unidirectional irregular head seas. Borresen and

Telsgard [27] emphasized the nonlinear effects of bow flare on motions and rigid body hull girder loads. In the last three references, the relative motion hypothesis is employed in each case, and the frequencies at which the hydrodynamic coefficients are evaluated are usually the frequency of encounter ( $\omega_e$ ) or infinite frequency ( $\omega_\infty$ ). Also, the methods in the last three references treat only pitch and heave in unidirectional regular waves.

In view of these reported successes with the time-step solution to the equations of motion and the method's general capability of incorporating loads of a nonlinear character, this approach will be used as the basis of the present model. It is also intended that some type of three-dimensional hydrodynamic formulation cast as an initial value problem would be integrated with the motion simulator from which initial and boundary values would be specified. As noted before, a finite difference solution to the Euler equation was initially chosen based on the reported successes of Bourianoff and Penumalli [22]. The technique developed to apply this finite difference solution is described in [28]. The fundamental difficulty with using this technique for ship motion and transient-free surface simulation was the conflicting demands of numerical stability vs. realistic fluid dynamics. The finite difference Euler equation solution method was abandoned after it became apparent that some of the numerical difficulties could not be overcome within the scope of this project. Alternative methods were investigated, and the technique developed by R.B. Chapman reported in [6] was selected because of its ability to predict three-dimensional flow and handle arbitrary motions and time-dependent hull shapes. Its principal disadvantage is the fact that it is computationally demanding, requiring long run times.\* Nevertheless, in view of the ever-increasing capabilities of modern computational hardware, the use of the Chapman method and others like it should become practical in the near future.

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\* Prescribed motion of a 120-panel body requires 7 CPU minutes per time step ( $0.1 \text{ sec} < t < 0.25 \text{ sec}$ ) on a DEC VAX 11/750 or roughly 20-25 CPU secs on a so-called supercomputer (e.g., CRAY or CDC STAR).

## 2.0 THEORETICAL MODEL

### 2.1 GENERAL FORMULATION OF EQUATIONS OF MOTION

The equations of motion for the ship can be derived from Newton's second law and formulated as four first-order ordinary differential equations, the solution of which gives translational and rotational displacements and velocities. Solution is accomplished in a time-step fashion using a fourth-order Runge-Kutta integration scheme. The external forces and moments called for in the equations of motion are calculated using an extension of basic strip theory to derive ship motion-induced and diffraction forces. Froude-Krylov forces are computed using a volume integral approach over the instantaneous immersed hull. The incident wave system is represented by one or more linear waves of arbitrary amplitude, direction, frequency and phase relationship.

The ship is considered to be advancing with forward speed with an arbitrary heading in a system of sinusoidal waves of various amplitudes, frequencies and directions. Let  $(x,y,z)$  be a Cartesian coordinate system fixed in space. Let  $(x',y',z')$  be a Cartesian coordinate system fixed with respect to the ship, whose axes coincide with the principal axes of inertia of the ship and whose origin corresponds to the ship's center of mass. The direction of the axes are illustrated in Figure 1a.

The position of the ship in space is completely characterized by the position of the ship's mass center and the angular displacement of the ship. Eulerian angles will be used to describe angular displacement. Figure 1b shows the convention for the Eulerian angles.

The motion of the ship in an inertial frame of reference, according to Newton's second law, can be formulated using subscript notation as

$$\frac{d}{dt} M_{jk} V_{G_k} = F_j \quad (j = 1,2,3) \quad (1)$$

$$\frac{d}{dt} I_{jk} \omega_k = T_j \quad (j = 1,2,3) \quad (2)$$

in which

$M_{jk}$  = components of mass matrix

$I_{jk}$  = components of moment and product of inertia matrix

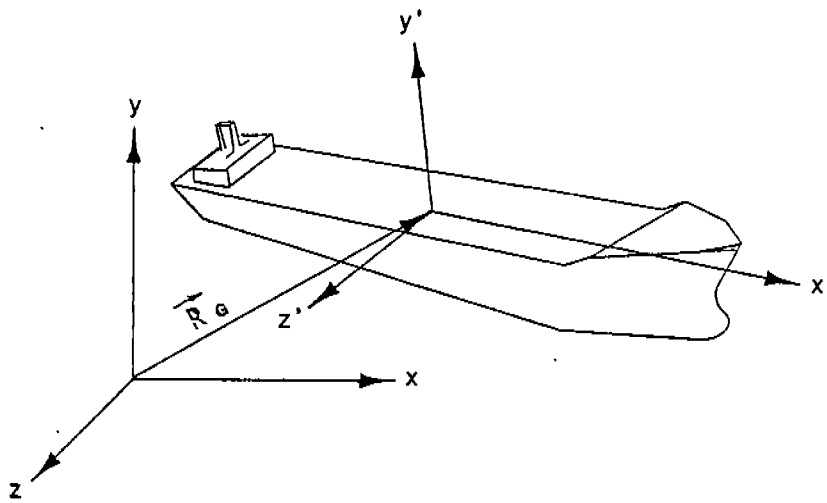


Figure 1a Axes for Solution of Equations of Motion

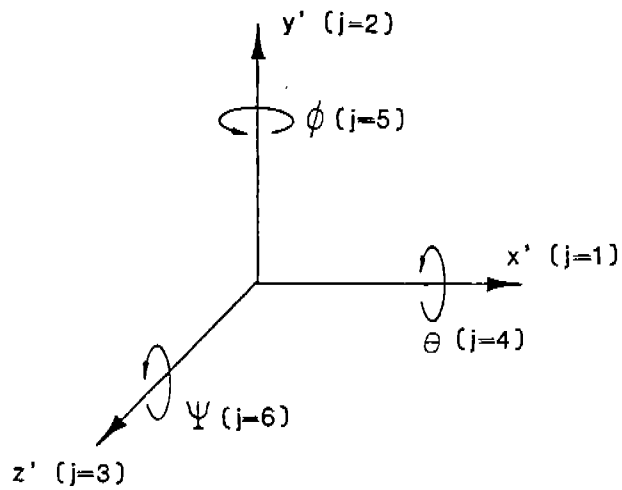


Figure 1b Sign Convention for Translational and Rotational Motions

$$V_{G_k} = \text{translational velocities of center of mass in space fixed system}$$

$$= \frac{d'}{dt} R_G$$

$\omega_k$  = rotational velocities

$$= B_{km} \frac{d\alpha_m}{dt}$$

and

$$B_{km} = \begin{bmatrix} 1 & \sin \psi & \theta \\ 0 & \cos \theta \cos \psi & \sin \theta \\ 0 & -\sin \theta \cos \psi & \cos \theta \end{bmatrix}$$

$$\alpha_m = \{ \theta, \phi, \psi \}$$

$\theta$  = Eulerian roll angle

$\phi$  = Eulerian yaw angle

$\psi$  = Eulerian pitch angle

$F_j$  = external forces

$T_j$  = external moments

In the ship coordinate system, the left side of equation (2) takes the following form:

$$\frac{d}{dt} (I_{jk} \omega_k) = I_{jk} \frac{d}{dt} \omega_k + \omega_k \times I_{jk} \omega_k \quad (3)$$



This is the Eulerian equation of motion whose application makes possible the use of constant moments and products of inertia. In the inertial frame of reference, these values are continuously changing.

Equations (1), (2) and (3) can be manipulated to yield the following two sets of two first-order ordinary differential equations:

$$\frac{d}{dt} X_{G_k} = V_{G_k} \quad (k = 1,2,3) \quad (4)$$

$$\frac{d}{dt} V_{G_k} = (M_{jk})^{-1} F_j \quad (k = 1,2,3) \quad (5)$$

$$\frac{d}{dt} \alpha_m = (B_{km})^{-1} \omega_k \quad (m = 1,2,3) \quad (6)$$

$$\frac{d}{dt} \omega_k = (I_{jk})^{-1} \{T_j - \omega_k \times I_{jk} \omega_k\} \quad (k = 1,2,3) \quad (7)$$

Assuming that the ship has lateral symmetry, the mass matrix is given by

$$M_{jk} = \begin{bmatrix} M_s & 0 & 0 \\ 0 & M_s & 0 \\ 0 & 0 & M_s \end{bmatrix} + A_{jk} \quad (j = 1,2,3) \quad (8)$$

where  $A_{jk}$  represents added mass. The ship's mass is shown as  $M_s$  within the brackets.

The moment and product of inertia matrix is given as

$$I_{jk} = \begin{bmatrix} I_{11} & I_{21} & I_{31} \\ I_{12} & I_{22} & I_{32} \\ I_{13} & I_{23} & I_{33} \end{bmatrix} + A_{mn} \quad \begin{matrix} (j = 1,2,3) \\ (m = 4,5,6) \end{matrix} \quad (9)$$

where  $A_{mn}$  represents added mass coefficients, and the values within the brackets represent ship moment and products of inertia.

If we can evaluate the external forces ( $F_j$ ) and moments ( $T_j$ ) required by expressions (5) and (7), we can solve for the ship translational ( $V_G$ ) and rotational ( $\omega$ ) velocities. These, in turn, are used in equations (4) and (6) to yield the ship translational ( $X_G$ ) and rotational ( $\alpha$ ) displacements. The integration of these four first-order ordinary differential equations is performed numerically with the fourth-order Runge-Kutta scheme.

Thus, the prediction of ship motions by this method then becomes a matter of predicting the external forces and moments. The various types of forces and moments which comprise the total force and moment used in equations (5) and (7) are outlined in the following section.

## 2.2 EVALUATION OF EXTERNAL FORCES AND MOMENTS

The basic method used to evaluate the external forces will be based on strip theory. This approach is used to evaluate forces throughout the simulation of ship motions and loads. An additional and important feature is the prediction of pressures on the hull using a three-dimensional hull source representation. The resultant force on the ship from these pressures can be substituted for the hydrodynamic forces predicted from the strip approach into the equations of motion for a selected interval of time. The interval of time for which this force substitution may be made must be limited in order to minimize the considerable computing effort associated with a three-dimensional solution in the time domain. Figure 2 depicts this procedure. It should be emphasized that the formulations for the two methods (strip and three-dimensional) are separate and distinct and each can provide force information for the solution of the equations of motion.

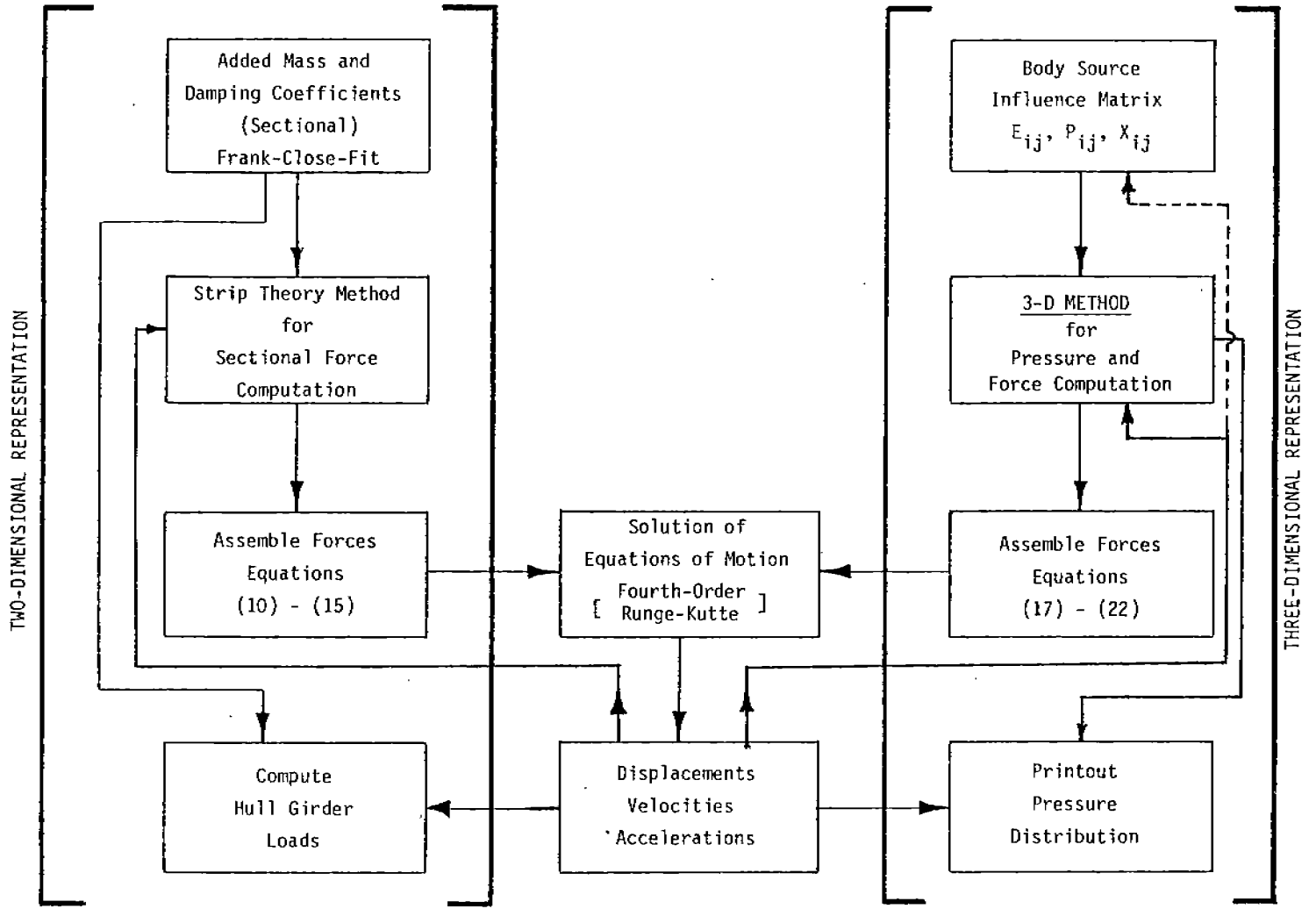
A brief outline of the force components that comprise the total force used in the ship motion equations follows. We first examine the strip method followed by a brief discussion of the three-dimensional method.

### 2.2.1 Strip Approach Forces

Following strip theory, we will assume that the hydrodynamic force acting on the ship can be decomposed into three parts:

- (a) Froude-Krylov Force - The resultant force from the pressure in the waves which acts on the hull surface with the assumption that the pressure field is not affected by the presence of the ship.
- (b) Diffraction Force - A correction to the Froude-Krylov force that accounts for the disturbance of the ambient wave field by the presence of the ship.

Figure 2 Diagram of Computational Scheme



- (c) Ship Motion-Induced Force - The force resulting from the motion of the ship with the assumption that it undergoes this motion in calm water.

The evaluation of the Froude-Krylov forces is obtained by integrating the pressure existing in the undisturbed waves over the wetted surface of the hull. The evaluation of the diffraction force and ship motion-induced force will generally be according to the elementary strip theory (see Raff [4]). Linear wave theory is employed and it will be assumed that the Froude-Krylov pressure and diffraction force evaluated in an irregular wave system is equivalent to the summation of the pressures and forces from the individual waves that comprise the irregular system.

The calculation of added mass and damping coefficients used in the evaluation of diffraction and ship motion-induced forces is by the Frank close-fit method. Because these coefficients are frequency dependent, a problem exists as to which frequency to use in the ship motion-induced force calculations since the ship motion may be irregular (non-harmonic). One approach taken by several investigators [26, 27] is to select a constant set of coefficients associated with one frequency.

We propose two other approaches and briefly examine the effect of each in Section 3.3. The first and simpler method is for a user-specified fixed value associated with the highest peak of the response spectra obtained from a frequency domain analysis or sea spectrum. The second approach is based on a scheme that samples the previous two zero-crossing periods of motion in sway, heave and roll. The two values are averaged, and three characteristic frequencies are derived for evaluation of the sectional hydrodynamic coefficients. The sway-roll or roll-sway coefficients are (arbitrarily) linked to the sway characteristic frequency.

In frequency domain strip theory formulations, the added mass and damping coefficients are evaluated at the still waterline. In the present formulation, at each time step, two-dimensional hydrodynamic coefficients can be evaluated for each section at either the mean still water draft or at the instantaneous depth of immersion. When it is desired that the hydrodynamic coefficients be draft dependent, the coefficients are obtained by linear interpolation from an array of six values corresponding to six drafts covering a range from near the baseline to the deck edge. Since the hydrodynamic coefficients are also frequency dependent, they are selected using linear interpolation from an array of twelve values associated with a range of twelve frequencies specified by the user. Thus, when the two-dimensional sectional coefficients are to be both draft and frequency dependent, two-dimensional linear interpolation is performed.

phenomena as bottom slamming, flare slamming, and the shipping of green water; as well as wind forces, mooring forces, rudder and propeller forces, viscous and appendage related damping, etc. The incorporation of methods to evaluate most of these forces will be left as the subject of subsequent investigations. We will, however, include here a flare force prediction method and also the means to incorporate a quadratic damping force. For flare impact force, we will use the approach described by Meyerhoff [26].

### 2.2.2 Three-Dimensional Pressure and Forces

The ship hull is modeled by quadrilateral panels. At the center of each panel, the pressure is evaluated, and resultant forces may be obtained. The pressure computed by this method includes the Froude-Krylov pressure (excluding the hydrostatic part), the diffraction pressure and the ship motion-induced pressure. Further description of this method will be left to Section 2.5, where it is discussed in detail.

### 2.2.3 Total Force Equations

The total force equations are given using the notation shown below. The force for each degree of freedom is indicated by:

X = longitudinal force  
Y = vertical force  
Z = lateral force  
K = roll moment  
N = yaw moment  
M = pitch moment

Subscripts to these force symbols specify the type of force according to:

b = Froude-Krylov  
h = ship motion-induced  
w = wave diffraction  
f = "flare" force  
q = quadratic damping force  
t = total force

Two sets of equations will be given; the first showing the force components normally used during the simulation, and the second showing the force components when the three-dimensional method is invoked. The external forces (F) and moments (M) used in the equations of motion (5) and (7) can be assembled from various sectional force components according to:

$$F_1 = \int \frac{X_b}{dx} dx \quad (10)$$

$$F_2 = \int \frac{dY_h}{dx} dx + \int \frac{dY_w}{dx} dx + \int \frac{dY_f}{dx} dx + \int \frac{dY_b}{dx} dx + Y_q \quad (11)$$

$$F_3 = \int \frac{dZ_h}{dx} dx + \int \frac{dZ_w}{dx} dx + \int \frac{dZ_b}{dx} dx + Z_q \quad (12)$$

$$T_1 = \int \frac{dK_h}{dx} dx + \int \frac{dK_w}{dx} dx + \int \frac{dK_b}{dx} dx + K_q \quad (13)$$

$$T_2 = \int x \frac{dZ_h}{dx} dx + \int x \frac{dZ_w}{dx} dx + \int \frac{dN_b}{dx} dx + N_q \quad (14)$$

$$T_3 = \int x \frac{dY_h}{dx} dx + \int x \frac{dY_w}{dx} dx + \int x \frac{dY_f}{dx} dx + \int \frac{dM_b}{dx} dx + M_q \quad (15)$$

All integrations are performed over the length of the ship. The strip theory equations used to evaluate the sectional forces are given in Section 2.4. As will be seen in that section, in the equations for the sectional ship motion-induced forces there is a part related to acceleration. These acceleration-related forces cannot be computed directly at any particular time step, since the acceleration values at that time are not a priori known. Consequently, the acceleration force terms in those equations are extracted and transferred to the mass moment and product of inertia matrices as the second terms in the right-hand side of expressions (8) and (9) given earlier. These matrices are assembled according to the following expressions:

$$A_{22} = \int a_{22} dx$$

$$A_{33} = \int a_{33} dx$$

$$A_{43} = \int a_{34} dx$$

$$A_{44} = \int a_{44} dx \quad (16)$$

$$A_{45} = \int x a_{34} dx$$

$$A_{53} = \int x a_{33} dx$$

$$A_{55} = \int x^2 a_{33} dx$$

$$A_{62} = \int x a_{22} dx$$

$$A_{66} = \int x^2 a_{22} dx$$

with the remaining values of  $A_{jk} = 0$ . All integrations are over the length of the ship.

When the three-dimensional pressure calculational procedure is run concurrently with the stripwise force calculations, three-dimensional pressure forces may be substituted for the stripwise hydrodynamic forces. During such an interval, the force expressions (10) - (15) would instead appear as:

$$X_T = X_{3D} + X_b^S + X_q \quad (17)$$

$$Y_T = Y_{3D} + Y_b^S + \int \frac{dY_f}{dx} dx + Y_q \quad (18)$$

$$Z_T = Z_{3D} + Z_b^S + Y_q \quad (19)$$

$$K_T = K_{3D} + K_b^S + K_q \quad (20)$$

$$N_T = N_{3D} + N_b^S + N_q \quad (21)$$

$$M_T = M_{3D} + M_b^S + \int x \frac{dY_f}{dx} dx + M_q \quad (22)$$

where  $X_b^S \dots M_b^S$  are the hydrostatic force components of the Froude-Krylov forces, and  $X_{3D} \dots M_{3D}$  are the force components calculated from the three-dimensional pressure computations.

This completes the formulation of the equations of motion and the force equations. The next several sections will present the computation of the Froude-Krylov force, the sectional diffraction and ship motion-induced forces, the flare impact force, and the three-dimensional pressures and resultant forces.

### 2.3 FROUDE-KRYLOV FORCES

We will assume that the elevation of the sea surface  $\eta(\vec{x}, t)$  can be described as the superposition of a number of sinusoids of the form:

$$\eta(\vec{x}, t) = \sum_{i=1}^N a_i \cos(\vec{k}_i \cdot \vec{x} - \omega_i t + \epsilon_i) = \sum_{i=1}^N \eta_i \quad (23)$$

The expression for hydrostatic and dynamic pressure can be derived from Bernoulli's equation and the definition of velocity potential for the incident wave field. If we include only the linear terms, and consider only deep water, the expression for pressure may be given as

$$p(t, x, y, z) = -\rho g \left[ y - \sum_{i=1}^N e^{k_i y} \eta_i \right] \quad (24)$$

The Froude-Krylov force may be obtained by integrating the pressure over the immersed surface of the ship. This surface integral may be replaced by a volume integral,\* so that the expressions for total Froude-Krylov forces and moments can be given by the integral of the pressure gradients. Sectional forces can be evaluated according to:

$$\frac{dX_b}{dx} = - \iint \frac{\partial p}{\partial x} dy dz$$

---

\* Gauss Theorem -

$$F = \iint \mathbf{n} p dA = \iiint \nabla p dV$$



$$\frac{dY_b}{dx} = - \iint \frac{\partial p}{\partial y} dydz$$

$$\frac{dZ_b}{dx} = - \iint \frac{\partial p}{\partial z} dydz$$

$$\frac{dK_b}{dx} = \iint (z \frac{\partial p}{\partial y} - y \frac{\partial p}{\partial z}) dydz$$

$$\frac{dN_b}{dx} = \iint (x \frac{\partial p}{\partial z} - z \frac{\partial p}{\partial x}) dydz \quad (25)$$

$$\frac{dM_b}{dx} = \iint (y \frac{\partial p}{\partial x} - x \frac{\partial p}{\partial y}) dydz$$

For each section, vertical integration extends to the wave surface rather than the mean water level. This fact necessitates some sort of approximation to evaluate pressures in the free surface zone since linear wave theory implies infinitesimal amplitudes which do not extend measurably above or below the mean water level.

From a review of the relevant literature, there appear to be three basic approaches for approximating the velocities and pressures in the free surface zone:

- (a) Unmodified Formula - Use the same formulas (for pressure) above mean sea level that apply below sea level. (Eq. 26 with no restrictions on y.)
- (b) Stretching Correction - Stretch the still water level in the formula to the sea surface.
- (c) Hydrostatic - Assume the pressure in the wave above the mean water level is hydrostatic.

There is not sufficient experimental data to support the use of one scheme over the next. The CAPSIZE program [29] uses the unmodified formula. There are also some limited experimental data obtained by Chakrabarti [30] to indicate that a hydrostatic pressure assumption is accurate in regular waves. If we select the hydrostatic assumption, the pressure equation may then be written as

$$p(t,x,y,z) = -\rho g \left[ y - \sum_{i=1}^N e^{k_i y} \eta_i \right] \quad y < 0 \quad (26)$$

$$p(t,x,y,z) = -\rho g \left( y - \sum_{i=1}^N \eta_i \right) \quad y > 0 \quad (27)$$

The differences in ship response as a result of using the unmodified pressure formula above the mean waterline were investigated. Comparisons for heave and pitch response of the SL-7 containership in 15-foot waves indicated that the differences were negligible. In any event, both methods are approximations. Their use, however, is necessitated by the employment of linear wave theory, and we therefore cannot escape the use of such simplifications. The computer code, as presently written, allows the user to select either approximation.

## 2.4 SECTIONAL HYDRODYNAMIC FORCES

Following elementary strip theory, local sectional hydrodynamic forces induced by the ship's motion and by diffraction force for heave, sway and roll are given in this section. The equations given below are equivalent to the basic equations given by Raff [4] in SSC-230. Flare impact force and quadratic roll damping are also discussed.

### Vertical Sectional Ship Motion-Induced Force

$$\begin{aligned} \frac{dY_h}{dx} = & -a_{22} (\ddot{y}_G + x\ddot{\psi} - 2U\dot{\psi}) \\ & - b_{22} (\dot{y}_G + x\dot{\psi} - U\psi) \\ & + U \frac{da_{22}}{dx} (\dot{y}_G + x\dot{\psi} - U\psi) \end{aligned} \quad (28)$$

### Vertical Sectional Diffraction Force

$$\frac{dy_w}{dx} = \sum_{i=1}^N \left[ \dot{\xi}_i a_{22} + \xi_i \left( b_{22} - U \frac{da_{22}}{dx} \right) \right] \quad (29)$$

Lateral Sectional Ship Motion-Induced Force

$$\begin{aligned}
 \frac{dZ_h}{dx} = & - a_{33} (\ddot{z} - x \ddot{\phi} + 2U \dot{\phi}) \\
 & + (U \frac{da_{33}}{dx} - b_{33}) (\dot{z} - x \dot{\phi} + U\phi) \\
 & + (a_{43} + h_{cg} a_{33}) \ddot{\theta} \\
 & + [b_{43} + h_{cg} b_{33} - U (\frac{da_{43}}{dx} + h_{cg} \frac{da_{33}}{dx})] \dot{\theta}
 \end{aligned} \tag{30}$$

Lateral Sectional Diffraction Force

$$\begin{aligned}
 \frac{dZ_w}{dx} = & \sum_{i=1}^N \{ [\zeta_i \dot{a}_{33} - \zeta_i (U \frac{da_{33}}{dx} - b_{33})] \\
 & - [\kappa_i h_{cg} \dot{a}_{34} - \kappa_i h_{cg} (U \frac{da_{34}}{dx} - b_{34})] \}
 \end{aligned} \tag{31}$$

Roll Sectional Ship Motion-Induced Moment

$$\begin{aligned}
 \frac{dK_h}{dx} = & - (a_{44} + h_{cg} a_{34}) \ddot{\theta} \\
 & - [b_{44} + h_{cg} b_{34} - U (\frac{da_{44}}{dx} + h_{cg} \frac{da_{34}}{dx})] \dot{\theta} \\
 & + a_{34} (\ddot{z} - x \ddot{\phi} + 2U \dot{\phi}) \\
 & + (b_{34} - U \frac{da_{34}}{dx}) (\dot{z} - x \dot{\phi} + U\phi) - h_{cg} \frac{dZ_h}{dx}
 \end{aligned} \tag{32}$$

## Roll Sectional Diffraction Force

$$\begin{aligned} \frac{dK_w}{dx} = & \sum_{i=1}^N \{ \kappa_i a_{44} - \kappa_i (U \frac{da_{44}}{dx} - b_{44}) \} \\ & - [ \zeta_i h_{cg} a_{34} - \zeta_i (U \frac{da_{34}}{dx} + b_{34}) ] - h_{cg} \frac{dZ_w}{dx} . \end{aligned} \quad (33)$$

Following the procedure used in the program CAPSIZE [29], each ship section is defined as a closed polygon made up of straight line segments between offsets. The integrals of vertical, horizontal and "roll" wave velocities and accelerations, as well as pressure gradients over each section, are evaluated exactly at each time step. Longitudinal integrations of sectional diffraction forces and Froude-Krylov forces (excluding hydrostatic component) are performed assuming the sectional forces are functions of the following form:

$$f(x) = \sum_{i=1}^N [ C_i \cos(k_i' x) + S_i \sin(k_i' x) ]$$

where

$S_i, C_i$  = sine and cosine components of force fluctuation for wave component  $i$ , the relative magnitudes of which provide the proper phase relationship of all wave-related forces.

$k_i'$  = projection of the wave number onto the x-axis of ship.

Longitudinal integrations are performed assuming  $C_i$  and  $S_i$  vary linearly along the ship's length.

## Flare Force

The approach for evaluating flare force is actually an extension of the strip theory expressions. Its incorporation into the present methodology serves to highlight the capability of the time domain method to account for the effects of the above-waterline hull on motions and loads. Furthermore, earlier work by Kaplan [31], Borreson and Tellsgaard [27], and Meyerhoff [22] have already provided the basic method to compute flare forces.

Following the approach described by Meyerhoff, the time derivative of the relative wave elevation for any strip may be given as

$$\dot{\eta}_r = \frac{\omega_0}{\omega_e} \dot{\eta} - \dot{y}_G - x\dot{\psi} + \psi U \quad (34)$$

where  $\dot{\eta}$  denotes the time-derivative of the wave elevation relative to the ship-fixed coordinate system,  $\omega_0$  is wave frequency and  $\omega_e$  is frequency of encounter. The convective derivative of relative velocity is accounted for by the factor  $\omega_0 / \omega_e$ . In the derivation of the expression for hydrodynamic inertia forces in the strip method, it can be shown that a nonlinear term results which represents an impact term. It contains the square of the relative velocity, and according to Meyerhoff [26] appears as:

$$\frac{dY_f}{dx} = \mu (\dot{\eta}_r)^2$$

where

$$\begin{aligned} \mu &= \left\{ \frac{\partial a_{22}(\infty)}{\partial \eta_r} \right\} && \text{if } \dot{\eta}_r > 0 \\ &= 0 && \text{if } \dot{\eta}_r \leq 0 \end{aligned}$$

The limiting value of added mass at infinite frequency is used in accordance with the theory of normal-symmetric impact upon a calm water surface. (It should also be pointed out that this formulation does not account for diffracted free surface motion, and the name "Froude-Krylov flare force" might more accurately represent the effect.)

### Quadratic Damping

The expressions for total forces and moments (10) - (15) and (17) - (22) each contain a term which represents a force or moment which is proportional to velocity-squared. The user of the computer program may specify, as input, his own values for quadratic damping coefficients. Accurate roll prediction often requires some quadratic roll damping. The damping moment associated with roll, for example, may be written in the form

$$K_q = B_{44}^* \left| \dot{\theta} \right| \dot{\theta}$$

where  $B_{44}^*$  is the quadratic roll damping coefficient for the ship, and  $\dot{\theta}$  is roll angle velocity.

## 2.5 THREE-DIMENSIONAL DYNAMIC PRESSURES AND FORCES

### 2.5.1 Introduction

A numerical method for calculating the transient three-dimensional flow induced by the motion in waves of a floating body of arbitrary shape with forward speed is presented in this section. Dynamic pressures and the resultant forces and moments acting on the hull surface can be evaluated in the time domain using this technique. The solution is generated in terms of a source distribution representation of the body and a spectral representation of the free surface. The approach presented here expands upon the work of R.B. Chapman [5,6]. Although Chapman had provided the basic method to solve the problem of ship motions in waves, his formulation did not explicitly include the ambient wave field effects and associated diffraction flow.

Following Chapman, the general formulation is linearized, assuming that the individual potentials and their derivatives associated with forward speed, body motion, free surface disturbance caused by radiated and diffracted waves, and the ambient wave flow are each sufficiently small to ignore higher order terms and cross-products. The free surface equations are also linearized. The formulation partially accounts for the "exact" hull boundary by evaluating hydrodynamic pressures on the actual immersed portion of the body surface at each time step below the still waterline. Thus, the effect of large-amplitude motions is included to some extent.

It should be emphasized that this three-dimensional approach is a separate procedure quite distinct from the strip formulation and equations of motion solution techniques that form the basic framework. It is best viewed as a "parallel process," interfacing with the main solution procedures in the following ways:

- (a) It requires, as input, the specification of body position and velocities at each time step.
- (b) It provides as output forces and moments on the hull, as well as additional information regarding distributed dynamic pressures.

In the formulation that follows, for the convenience of the reader who wishes to refer to Chapman's work, we will use Chapman's coordinate system throughout Section 2.5.

### 2.5.2 Formulation

A Cartesian coordinate system is fixed in space with the x-axis in the direction of the ship's forward advance, positive forward; the z-axis positive

downward and y-axis positive to starboard. The still mean water level corresponds to  $z=0$ . The free surface elevation is specified by  $(x,y,t)$  and the ship's hull is defined by  $S(x,y,z,t) = 0$  for  $z \geq 0$ . If the fluid is assumed to be inviscid, incompressible and homogeneous and the flow irrotational, then the fluid velocity  $\vec{v}(\vec{x}, t)$  can be represented by the gradient of the velocity potential  $\phi(\vec{x}, t)$ . If we linearize the free surface equations, the velocity potential must satisfy the following conditions:

$$\nabla^2 \phi = 0 \quad z > 0 \quad (35)$$

$$\frac{\partial \phi}{\partial t} = -g \eta \quad z = 0 \quad (36)$$

$$\frac{\partial \phi}{\partial z} = \frac{\partial \eta}{\partial t} \quad z = 0 \quad (37)$$

$$\frac{\partial \phi}{\partial \vec{n}} - \vec{v}_s \cdot \vec{n} = 0 \quad - \text{ on } S(x,y,z,t) = 0 \quad (38)$$

where  $\vec{v}_s$  is the velocity at any point on the hull surface  $S$ , and  $\vec{n}$  is the unit normal to the boundary surface at the point, pointing outward. The velocity potential can be decomposed into two parts:

$$\phi(x,y,z,t) = [-Ux + \phi_S(x,y,z) + \phi_T(x,y,z,t)] \quad (39)$$

where  $-Ux + \phi_S$  is the steady part and  $U$  is the forward speed of the ship, and  $\phi_T(x,y,z,t)$  is the time-varying potential. In the present formulation, we will neglect the steady part and consider only the time-varying potential.

Separating the time-varying potential into three parts, we obtain

$$\phi_T = \phi_B + \phi_{FS} + \phi_A \quad (40)$$

where  $\phi_B$  represents the instantaneous effect of the body,  $\phi_{FS}$  represents the existing free surface disturbance due to previously radiated and diffracted waves, and  $\phi_A$  is the ambient wave potential.

The body potential will be represented by a distribution of simple sources over the wetted hull and its negative image. A spectral representation of the free surface potential will be used so that the diffracted and radiated wave fields are represented by two series, harmonic in space and time. The potential for the ambient wave system will be given according to linear theory for progressive waves. It should be noted that, although the method can be extended to include sources distributed over the entire body, the present procedure only distributes sources over the body below the mean waterline.

The objective of this analysis is to compute the hydrodynamic pressure on the hull, as determined from the potential field  $\phi_T(x,y,z,t)$ . The dynamic pressure can be obtained from Bernoulli's equation applied to  $\phi_T(x,y,z,t)$ . If the flow is assumed to be slow enough that the nonlinear term can be neglected, Bernoulli's equation is:

$$p_T = -\rho \frac{\partial \phi_T}{\partial t} = -\rho \frac{\partial}{\partial t} [\phi_B + \phi_{FS} + \phi_A] \quad (41)$$

By linearizing the problem, we can evaluate the total pressure at any time or point as the sum of the individual pressure components, as given by:

$$p_B(x,y,z,t) = -\rho \left[ \frac{\partial \phi_B}{\partial t} - \vec{V}_s \cdot \vec{\nabla} \phi_B \right] \quad (42)$$

$$p_{FS}(x,y,z,t) = -\rho \frac{\partial \phi_{FS}}{\partial t} \quad (43)$$

$$p_A(x,y,z,t) = -\rho \frac{\partial \phi_A}{\partial t} \quad (44)$$

The convective derivative for the body pressure expression is included because the Bernoulli equation given here is for fixed points in space, yet the body potential is associated with the moving hull surface. We will assume, however, that products of the body velocity and potential gradients are negligible with the exception of  $U \frac{\partial \phi_B}{\partial x}$ . Equation (42) then becomes:



$$p_B(x,y,z,t) = -\rho \left[ \frac{\partial \phi_B}{\partial t} - U \frac{\partial \phi_B}{\partial x} \right]. \quad (45)$$

Prior to describing the individual potential representations in detail, it will be advantageous to first describe the numerical solution procedure on a step-by-step basis.

### 2.5.3 Solution Procedure

This is an initial value problem that ideally should start from a condition at rest. Prior to starting the simulation, the following arrays relating to the body source representation are calculated:

- (a)  $E_{ij}$  - gives the normal velocity component at the center point of panel  $i$  induced by a uniform source distributed over panel  $j$  and its image. (As will be explained, the hull surface is modeled by quadrilateral panels; at the center of each is a simple source.)
- (b)  $\hat{p}_{ij}$  - gives the potential at the center of panel  $i$  induced by uniform source density of unit strength acting over panel  $j$ .
- (c)  $X_{ij}$  - gives the x-direction velocity component at the center point of panel  $i$  induced by a uniform source distributed over panel  $j$  and its image.

The simulation progresses by a series of small time steps, according to the following sequence:

#### Step 1.

As we begin each time step, we will have already computed the velocities, accelerations and pressures induced by the free-surface disturbance and ambient wave field at the center of each panel. (These will be known from steps (5) and (6) of the previous time cycle.) The body velocity and acceleration will have been obtained from the solution to the equations of motion.

First, for time  $t = t_n$ , compute the resultant total normal velocity  $(\vec{v}_T \cdot \vec{n})$  \* at the center of each panel.\* This resultant velocity represents the sum of the velocity from the body motion, the velocity induced by the existing free surface disturbance, and the velocity induced by the ambient wave field:

$$\vec{v}_T \cdot \vec{n} = (\vec{v}_B + \vec{v}_{FS} + \vec{v}_A) \cdot \vec{n} \text{ at panel centers.} \quad (46)$$

According to Chapman [6], it is more convenient to use accelerations instead of velocities. Equation (46) can then be expressed as:

$$\vec{a}_T \cdot \vec{n} = (\vec{a}_B + \vec{a}_{FS} - U \frac{\partial}{\partial x} \vec{v}_{FS} + a_A - U \frac{\partial}{\partial x} v_A) \cdot \vec{n}. \quad (47)$$

(The convective derivatives are required in this case for  $a_{FS}$  and  $a_I$  because we are in a hull-fixed system when computing the resultant normal acceleration on the body.)

### Step 2.

Knowing  $\vec{a}_T$  from step (1) and having precalculated  $E_{ij}$ , determine the time derivative of the source strengths  $\dot{\sigma}_j$  according to

$$\sum_{i=1}^{N_B} E_{ij} \dot{\sigma}_j = \vec{a}_T \cdot \vec{n} \text{ at } t = t_n. \quad (48)$$

### Step 3.

Calculate the body-induced component of pressure at the center point of each panel induced by the computed time derivative of the source strength in a hull fixed system according to

\* In the equations that follow, it is understood that all normal vectors, areas, positions or coordinates on the hull are functions of time, even though they may not be so specified.

$$p_B = -\rho \frac{\partial \phi_B}{\partial t} + \rho U \frac{\partial \phi_B}{\partial x} \quad (49)$$

or

$$p_B = -\rho \sum_{j=1}^{N_B} \hat{p}_{ij} \dot{\sigma}_j(t_n) + \rho U \chi_{ij} \sigma_j(t_n) \quad (50)$$

Step 4.

Compute total pressure at each panel

$$p_T(t_n) = p_B(t_n) + p_{FS}(t_n) + p_A(t_n) .$$

Step 5.

Update the source strengths for this time step ( $t_n$ ) according to

$$\sigma_i(t_n) \approx \sigma_i(t_n - \Delta t) + \Delta t \dot{\sigma}_i(t_n) \quad (51)$$

Step 6.

As described in Section 2.5.5, update the free surface representations. Modify the free surface to account for the effect of the hull source strengths on the free surface. Calculate acceleration induced by the free surface at the center of each panel  $\vec{a}_{FS}(t_n + \Delta t)$  and the pressure induced by the free surface disturbance at the center of each panel  $p_{FS}(t_n + \Delta t)$  .

Step 7.

Update the ambient wave field. Calculate the acceleration  $\vec{a}_A(t_n + \Delta t)$  and pressure  $p_A(t_n + \Delta t)$  induced by the ambient wave field at the center of each panel.

### Step 8.

Go to Step 1 with the new  $\vec{a}_{FS}$ ,  $\vec{a}_A$  and newly prescribed  $\vec{a}_B$ .

#### 2.5.4 Body Representation

As shown in Figure 3, the hull is modeled by a set of  $N_B$  quadrilateral panels. A simple source and its image is distributed over each panel. The expression for the body potential can then be expressed as:

$$\phi_B(x,y,z) = \sum_{i=1}^{N_B} \sigma_i \iint_{S_i} G(x,y,z,x',y',z') dS' \quad (53)$$

in which

$$G(x,y,z,x',y',z') = \left[ \frac{1}{\sqrt{(x-x')^2 + (y-y')^2 + (z-z')^2}} - \frac{1}{\sqrt{(x-x')^2 + (y-y')^2 + (z+z')^2}} \right] \quad (54)$$

$\sigma_i$  is the strength on the  $i^{\text{th}}$  source panel

$S_i$  is the  $i^{\text{th}}$  panel surface.

Panel source strengths are evaluated at each time step using a linear system of equations relating resulting total normal velocities  $(\vec{v}_T \cdot \vec{n})$  at each panel center to the source strength  $\sigma_j$  at every other panel center and by the satisfaction of the hull boundary condition at the center of each panel. The exact method of Hess and Smith [32] is used here to determine the normal velocity component at the center point of panel  $i$  induced by a plane quadrilateral source element with a unit value of source density at the center point of panel  $j$ . A body coefficient scalar matrix  $E_{ij}$  is thus computed which can be used to determine unknown source densities when given panel center normal velocities, according to

$$\vec{v}_T \cdot \vec{n}_i = \sum_{j=1}^{N_B} E_{ij} \sigma_j \quad (55)$$

As was shown in equation (46), the panel center total normal velocity is the sum of components from the hull motion and induced velocities from radiated and diffracted free surface disturbances and ambient wave field.

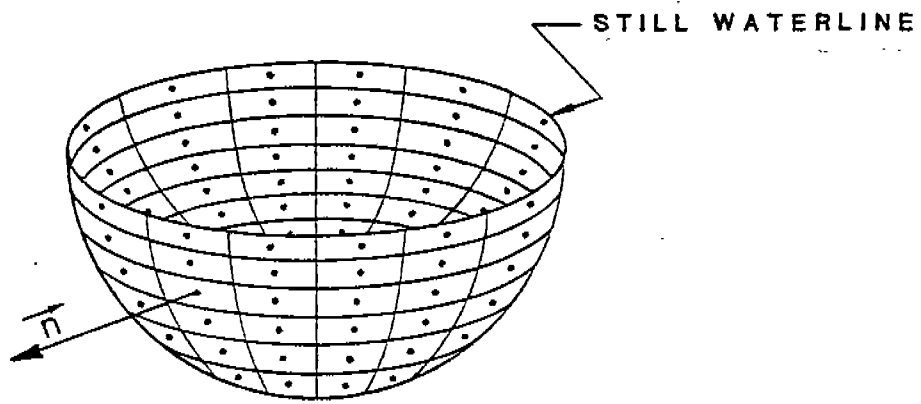


Figure 3    Quadrilateral panels and source representation of semisubmerged sphere.

Also, in Step 3 of the computational sequence, equations (49) and (50) show that the calculation of body-induced pressure at panel centers requires the use of two other arrays  $\hat{P}_{ij}$  and  $X_{ij}$ . The  $X_{ij}$  array gives the x-direction velocity component at the center point of panel  $i$  induced by a uniform source (of unit strength) distributed over panel  $j$  and its image. It is thus very similar to the  $E_{ij}$  array, relating the x-component instead of the normal component of panel velocities. In practice, it is evaluated directly from the information used to evaluate  $E_{ij}$ .

The  $\hat{P}_{ij}$  array gives the potential at the center of panel  $i$  induced by a uniform source (of unit strength) distributed over panel  $j$  and its image, or

$$\phi_{B_i} = \sum_{j=1}^{N_B} \hat{P}_{ij} \sigma_j \quad (56)$$

Looking back at equation (53) for a moment, it can be seen the  $\hat{P}_{ij}$  can be computed by relating each panel center to every other panel center using the expression for  $G(x,y,z,x',y',z')$ , or in the indexed notation,

$$\hat{P}_{ij} = \iint_{S_i} dS_i \{ [(x_j - x_i)^2 + (y_j - y_i)^2 + (z_j - z_i)^2]^{-1/2} - [(x_j - x_i)^2 + (y_j - y_i)^2 + (z_j + z_i)^2]^{-1/2} \} \quad (57)$$

These integrals are evaluated numerically by dividing the surface  $S_i$  into many small elements and evaluating the above integrand at the center of each smaller element.

### 2.5.5 Free Surface Representation

A spectral representation of the free surface will be used. Let the free surface elevation and velocity potential be represented by  $\eta(x,y,t)$  and  $\phi_{FS}(x,y,z,t)$ , respectively. The elevation and potential satisfy the dynamic and kinematic linearized free-surface conditions at  $z = 0$ .

$$\frac{\partial}{\partial t} \phi(x,y,0,t) = -g \eta(x,y,t) \quad (58)$$

$$\frac{\partial \phi}{\partial z} \eta(x,y,t) = - \frac{\partial \eta}{\partial t} (x,y,0,t) \quad (59)$$

Now assume that both the elevation and deep water potential can be represented by the real part of a finite summation of harmonics in the following forms:

$$\eta(x,y,t) = \sum_{n=1}^{N_{kx}} \Delta kx_n \sum_{m=1}^{N_{ky}} \Delta ky_m [A_{nm}(t) e^{i(kx_n \cdot x + ky_m \cdot y)} + A_{nm}^*(t) e^{i(kx_n \cdot x - ky_m \cdot y)}] \quad (60)$$

$$\phi_{FS}(x,y,z,t) = \sum_{n=1}^{N_{kx}} \Delta kx_n \sum_{m=1}^{N_{ky}} \Delta ky_m [B_{nm}(t) \omega_{nm} k_{nm}^{-1} e^{i(kx_n \cdot x + ky_m \cdot y)} + B_{nm}^*(t) \omega_{nm} k_{nm}^{-1} e^{i(kx_n \cdot x - ky_m \cdot y)}] e^{-k_{nm} \cdot z} \quad (61)$$

where

$A_{nm}, B_{nm}$  - complex functions of wave numbers and time  
(to be further defined below)

$A_{nm}^*, B_{nm}^*$  - complex conjugates

$kx_n, ky_m$  - wave numbers in x and y; e.g.,  $\frac{\omega_n^2}{g}$

$\Delta kx_n, \Delta ky_m$  - wave number spacing; e.g.,  $\Delta kx_n = kx_{n+1} - kx_n$

$k_{nm}$  -  $k_{nm}^2 = kx_n^2 + ky_m^2$

$\omega_{nm}$  -  $\omega_{nm}^2 = g [kx_n^2 + ky_m^2]^{1/2}$

Substitution of equations (60) and (61) into the free surface equations (58) and (59) would confirm that the above elevation and potential expressions do indeed satisfy the linearized free surface conditions. We would also obtain, from those substitutions, the following relationships between the complex amplitude functions:

$$\frac{d}{dt} A_{nm}(t) = \omega_{nm} B_{nm}(t) \quad (62)$$

$$\frac{d}{dt} B_{nm}(t) = -\omega_{nm} A_{nm}(t) \quad (63)$$

(The complex conjugates are described by identical expressions).

As part of the free surface elevation representation, we further define the complex amplitude function  $A_{nm}$  as

$$A_{nm}(t) = a_{nm} \cos \omega_{nm} t + b_{nm} \sin \omega_{nm} t \quad (64)$$

where  $a_{nm}$  and  $b_{nm}$  are complex constants. Substitution of (64) into (62) yields the expression for  $B_{nm}(t)$ .

$$B_{nm}(t) = b_{nm} \cos \omega_{nm} t - a_{nm} \sin \omega_{nm} t \quad (65)$$

Similar expressions also describe the complex conjugates.

### Updating the Free Surface

Equations (62), (63), (64) and (65) allow us to develop the expressions for the time evolution of  $A_{nm}(t)$  and  $B_{nm}(t)$ . Consider  $A_{nm}(t)$  first. If we increment the time in equation (64) by  $\Delta t$ , we obtain

$$A_{nm}(t + \Delta t) = a_{nm} \cos \omega_{nm} (t + \Delta t) + b_{nm} \sin \omega_{nm} (t + \Delta t) \quad (66)$$



Using the following trigonometric identities,

$$\cos (x \pm y) = \cos x \cos y \mp \sin x \sin y \quad (67)$$

$$\sin (x \pm y) = \sin x \cos y \pm \cos x \sin y$$

we can rewrite (66) as

$$\begin{aligned} A_{nm}(t + \Delta t) = & a_{nm} \cos \omega_{nm} t \cos \omega_{nm}(\Delta t) - a_{nm} \sin \omega_{nm} t \sin \omega_{nm}(\Delta t) \\ & + b_{nm} \sin \omega_{nm} t \cos \omega_{nm}(\Delta t) + b_{nm} \cos \omega_{nm} t \sin \omega_{nm}(\Delta t) \end{aligned} \quad (68)$$

Rearranging (68), we can write

$$\begin{aligned} A_{nm}(t + \Delta t) = & [a_{nm} \cos \omega_{nm} t + b_{nm} \sin \omega_{nm} t] \cos \omega_{nm}(\Delta t) \\ & + [b_{nm} \cos \omega_{nm} t - a_{nm} \sin \omega_{nm} t] \sin \omega_{nm}(\Delta t) \end{aligned} \quad (69)$$

It can be seen that the quantities in the brackets are equivalent to the right-hand sides of equations (64) and (65), so that we can now obtain

$$A_{nm}(t + \Delta t) = A_{nm}(t) \cos \omega_{nm}(\Delta t) + B_{nm}(t) \sin \omega_{nm}(\Delta t) \quad (70)$$

The expression for  $B_{nm}(t)$  can be similarly obtained and is given as

$$B_{nm}(t + \Delta t) = B_{nm}(t) \cos \omega_{nm}(\Delta t) - A_{nm}(t) \sin \omega_{nm}(\Delta t) \quad (71)$$

Identical expressions are used for updating  $A_{nm}^*(t)$  and  $B_{nm}^*(t)$ . Thus, expressions (70) and (71) provide the means to update the coefficients which define the wave field. To these updated values must be added any contribution to the wave field from the body motion between times  $t$  and  $t + \Delta t$ , as described next.

### Body Effect on the Free Surface

Next we consider the effect of the body source distribution on the wave elevation field over the time interval  $\Delta t$ . Assume that the change in  $A_{nm}(t)$  induced by the body between  $t_n$  and  $t_n + \Delta t$  is expressed by the sum of the effects of the individual sources and their images. For convenience, we will first replace a uniform source density acting over a panel of area  $A_i$  by a single point source with strength

$$s_i(t) = A_i [\sigma_i(t_n) + (t-t_n) \dot{\sigma}(t_n)] \quad (72)$$

which is located at the panel center. The time,  $t$ , is defined to be  $t = t_n + 1/2\Delta t$ , the midpoint of the time interval.

At any given point on the mean water level plane  $(x,y,0)$ , the vertical velocity induced by the source points and their images is given by

$$\frac{\partial \phi_B(t)}{\partial z} = - \sum_{i=1}^{N_B} \frac{2z_i s_i(t)}{[(x-x_i)^2 + (y-y_i)^2 + z_i^2]^{3/2}} \quad (73)$$

According to Chapman [6], equation (73) can also be expressed in an integral form given as

$$\frac{\partial \phi_B(t)}{\partial z} = \frac{2}{\pi} \sum_{i=1}^{N_B} s_i(t) \int_0^{\infty} dkx e^{ikx(x-x_i)} \int_{-\infty}^{\infty} dky \cos ky(y-y_i) \cdot e^{-(kx^2 + ky^2)z_i} \quad (74)$$

Substituting (74) into the linearized kinematic free surface condition (59), and integrating over the time interval  $\Delta t$ , we can rewrite (74) to show how the elevation changes over  $\Delta t$  due to the body,

$$\Delta \eta_B(x,y) = \frac{2}{\pi} \int_{t_n}^{t_n + \Delta t} dt \sum_{i=1}^{N_B} s_i(t) \int_0^{\infty} dk_x e^{ikx(x-x_i)} \int_{-\infty}^{\infty} dk_y \cos ky(y-y_i) \cdot e^{-(kx^2 + ky^2)z_i} \quad (75)$$

We may also write  $\Delta \eta_B(x,y)$  in a different form, following the same form of the original expression (60) defining the free surface elevation. In that form, we can write

$$\Delta \eta_B(x,y) = \sum_{n=1}^{N_{kx}} \Delta k_{x_n} \sum_{m=1}^{N_{ky}} \Delta k_{y_m} [ \Delta A_{nm}^{BODY} (t) e^{i(kx_n \cdot x + ky_m \cdot y)} + \Delta A_{nm}^{BODY*} (t) e^{i(kx_n \cdot x - ky_m \cdot y)} ] \quad (76)$$

Then, if we represent the wave number integrals in (75) numerically by a finite summation, and also evaluate the time integral, it should be evident by direct comparison with (76) that the  $A_{nm}$  coefficient increments can be written as

$$\Delta A_{nm}^{BODY} = 2 \frac{\Delta t}{\pi} \sum_{i=1}^{N_B} s_i(t_n + 1/2 \Delta t) e^{-i(kx_n \cdot x_i + ky_m \cdot y_i)} e^{-k_{nm} z_i} \quad (77)$$

$$\Delta A_{nm}^{BODY*} = 2 \frac{\Delta t}{\pi} \sum_{i=1}^{N_B} s_i(t_n + 1/2 \Delta t) e^{-i(ky_n \cdot x_i - ky_m \cdot y_i)} e^{-k_{nm} z_i} \quad (78)$$

The changes in  $B_{nm}$  induced by the body can be derived from (78) and (79) using the relationship between  $A_{nm}(t)$  and  $B_{nm}(t)$  given in (63). Multiplying both sides of (63) by  $\Delta t$  and then integrating both sides over time, we obtain

$$\Delta B_{nm}^{BODY} = -1/2 \frac{\partial}{\partial t} A_{nm}^{BODY} (t) \omega_{nm} (\Delta t)^2 \quad (79)$$

or

$$\Delta B_{nm}^{BODY} = - \frac{(\Delta t)^2}{\pi} \sum_{i=1}^{N_B} s_i(t_n) e^{-i(kx_n \cdot x_i + ky_m \cdot y_i)} e^{-k_{nm} z_i} \omega_{nm} \quad (80)$$

with a similar expression for  $\Delta B_{nm}^{BODY*}$ .

The effect of the body on the free surface can now be easily included by adding  $\Delta A_{nm}^{BODY}$ ,  $\Delta A_{nm}^{BODY*}$ ,  $\Delta B_{nm}^{BODY}$ , and  $\Delta B_{nm}^{BODY*}$  to the time evolution equations given earlier as (70) and (71) and the corresponding (complex conjugate) expressions.

#### Free Surface-Induced Pressure and Acceleration

Finally, we need to evaluate the pressure and acceleration induced by the free surface disturbance at the center of each panel. These two quantities are required in steps (4) and (1), respectively, of the simulation procedure. The pressure at any point (x,y,z) is given by

$$\begin{aligned} p_{FS} &= -\rho \frac{\partial \phi_{FS}}{\partial t} \\ &= \rho \sum_{n=1}^{N_{kx}} \Delta kx_n \sum_{m=1}^{N_{ky}} \Delta ky_m e^{(ikx_n \cdot x - k_{nm} \cdot z)} \\ &\quad [ A_{nm}(t) e^{iky_m \cdot y} + A_{nm}^*(t) e^{-iky_m \cdot y} ] \end{aligned} \quad (81)$$

the acceleration at any point (x,y,z) coincident with a unit normal vector  $(n_x, n_y, n_z)$  may be written as

$$\vec{a}_{FS} \cdot \vec{n} = -g \sum_{n=1}^{N_{kx}} \Delta kx_n \sum_{m=1}^{N_{ky}} \Delta ky_m e^{(ikx_n \cdot x - k_{nm} \cdot z)}$$

$$\begin{aligned}
& [ A_{nm}(t) \cdot e^{iky_m \cdot y} (iky_n \cdot n_x + iky_m \cdot n_y - k_{nm} \cdot n_z) \\
& + A_{nm}^*(t) \cdot e^{-iky_m \cdot y} \cdot (ikx_n \cdot n_x - iky_m \cdot n_y - k_{nm} \cdot n_z) ] \quad (82)
\end{aligned}$$

### 2.5.6 Ambient Wave Field Representation

The ambient wave field is represented by a linear wave system of amplitude A. Consider unidirectional, regular waves and define the velocity potential as

$$\phi_A(x,z,t) = -\frac{Ag}{\omega} e^{-kz} \sin(kx-\omega t) \quad (83)$$

The x and z components of velocity can then be expressed as

$$V_x(t) = -\frac{\partial \phi_A}{\partial x} = \frac{Agk}{\omega} e^{-kz} \cos(kx-\omega t) \quad (84)$$

$$V_z(t) = -\frac{\partial \phi_A}{\partial z} = -\frac{Agk}{\omega} e^{-kz} \sin(kx-\omega t) \quad (85)$$

The x and z components of acceleration can be given as

$$a_x(t) = \frac{\partial V_x}{\partial t} = Agk e^{-kz} \sin(ky-\omega t) \quad (86)$$

$$a_z(t) = \frac{\partial V_z}{\partial t} = Agk e^{-kz} \cos(kx-\omega t) \quad (87)$$

The dynamic pressure may be expressed as

$$p_A(t) = -\rho \frac{\partial \phi_A}{\partial t} = -Ae^{-kz} \cos(kx-\omega t) \quad (88)$$

We can now evaluate the normal acceleration and pressure induced at the center of each panel by the ambient wave field. These quantities are used in Steps 1 and 4 of the simulation procedure. The pressure is given in (88) and the normal acceleration is simply

$$\vec{a}_A \cdot \vec{n} = \vec{a}_x \cdot \vec{n}_x + \vec{a}_z \cdot \vec{n}_z \quad (89)$$

where the velocity components are given in (84) and (85). We will assume that in an irregular wave system, the total velocity, acceleration and pressure components can be described as the superposition of individual single frequency components.

## 2.6 HULL GIRDER LOADS

Hull girder loads are computed in the conventional manner of strip theory where the dynamic shear force at a cross section is the difference between the inertia force and the sum of external forces acting on the portion of the hull forward of the section. The vertical shear force and bending moment at any location  $x_0$  along the ship's length is, in the original coordinate system,

$$SF_y(x_0) = \int_{x_0} \frac{df_y}{dx} dx \quad (90)$$

and

$$BM_y(x_0) = \int_{x_0} (x-x_0) \frac{df_y}{dx} dx \quad (91)$$

where

$$\frac{df_y}{dx} = -\delta m (\ddot{y} - x \ddot{\psi}) + \frac{dY_h}{dx} + \frac{dY_w}{dx} + \frac{dY_f}{dx} + \frac{dY_b}{dx} + y_q \quad (92)$$

and  $\delta m$  = local sectional mass. Integrations are performed from the location  $x_0$  forward to the bow.

The lateral shear force, bending moment and torsional moment are given as

$$SF_z(x_0) = \int_{x_0}^{\text{bow}} \frac{df_z}{dx} dx \quad (93)$$

$$BM_z(x_0) = \int_{x_0}^{\text{bow}} (x-x_0) \frac{df_z}{dx} dx \quad (94)$$

$$TM_x(x_0) = \int_{x_0}^{\text{bow}} \frac{dm_x}{dx} dx \quad (95)$$

where

$$\begin{aligned} \frac{df_z}{dx} &= -\delta m(\ddot{z} - x\ddot{\phi} - \bar{y}\ddot{\theta}) \\ &\quad + \frac{dZ_h}{dx} + \frac{dZ_w}{dx} + \frac{dZ_b}{dx} + Z_q \end{aligned} \quad (96)$$

$$\begin{aligned} \frac{dm_x}{dx} &= -i_x\ddot{\theta} - \delta m\bar{y}(\ddot{z} - x\ddot{\phi}) \\ &\quad + \frac{dK_h}{dx} + \frac{dK_w}{dx} + \frac{dK_b}{dx} + K_q \end{aligned} \quad (97)$$

and  $\bar{y}$  = local section's center of gravity relative to ship c.g. (positive up).

$i_x$  = local section's mass moment of inertia.

## 2.7 NUMERICAL ASPECTS

### 2.7.1 Solution Procedures - Two-Dimensional Approach

The equations of motion as represented by expressions (4) - (7) are numerically integrated for each time to yield velocities and displacements. The fourth-order Runge-Kutta scheme is used for this purpose. At each time step, two-dimensional hydrodynamic coefficients are evaluated for each section for either the still water draft or at the instantaneous depth of immersion, at the option of the user. When it is desired that the hydrodynamic coefficients be draft dependent, the coefficients are obtained by linear interpolation from an array of six values corresponding to six drafts which cover a

range specified by the user in the input data. Since the hydrodynamic coefficients are also frequency dependent, they are selected using linear interpolation from an array of twelve values associated with a range of twelve frequencies specified by the user. Thus, when the two-dimensional sectional coefficients are to be both draft and frequency dependent, two-dimensional linear interpolation is performed.

There are two methods for the selection of frequencies at which to evaluate the hydrodynamic coefficients. One is based on a scheme that samples the previous two zero-crossing periods of sectional motion in sway, heave and roll. The two values are averaged, and three characteristic frequencies are derived for evaluation of the sectional hydrodynamic coefficients. The sway-roll or roll-sway coefficients are (arbitrarily) linked to the sway characteristic frequency. The other allows the user to specify a frequency, such as the peaks of the response spectra or peak of the wave energy spectrum.

The sectional hydrodynamic coefficients are computed using the Frank close-fit method. Spatial derivatives of added mass and damping coefficients in the x-direction are approximated by center-space finite differences. In the calculation of the derivative of added mass with respect to the relative wave elevation used in the computation of vertical flare force, the added mass is evaluated at each time step at the instantaneous immersed draft of the section. For each time step interval  $\Delta t$ , the change in added mass is computed and divided by the change in instantaneous draft of the section. During water exit, the calculation of the impact term is not performed and the sectional flare force is set to zero.

## 2.7.2 Solution Procedures - Three-Dimensional Analysis

### Specification of Parameters for Free Surface Harmonics

There are three parameters which must be chosen. These parameters dictate the wave numbers and wave number spacing for the free surface representation. The following parameters are to be specified for the three-dimensional transient analysis:

- $L_x, L_y$  - maximum distances in  $\pm x$  and  $\pm y$  from body origin defining the physical region over which the wave field is required.
- $T$  - maximum time interval for transient simulation.
- $l_x, l_y$  - minimum half-wavelength related to the smallest scale of disturbance or physical feature of the hull that needs to be represented.

There are no set rules for the selection of these parameters, and their choice must be based on an understanding of the problem. There are, however, some



guidelines which generally seem to work well. The values of  $L_x$  and  $L_y$  should be 2 to 2.5 times the length and beam of the body, respectively. The values of  $l_x, l_y$  should be on the order of the average dimensions of the hull panels. The parameter  $T$  can be set to about 60% of the actual time for which a transient analysis is desired.

The wave numbers and wave number spacing are then specified according to the following relations:

(a) Maximum step sizes for  $kx_n$  and  $ky_m$  are:

$$\begin{aligned} kx_{n+1} - kx_n &\leq \frac{2\pi}{L_x} \\ ky_{m+1} - ky_m &\leq \frac{2\pi}{L_y} \end{aligned} \quad (98)$$

(b) The minimum upper bounds of  $kx$  and  $ky$  are:

$$\begin{aligned} kx_{N_{kx}} &\geq \frac{2\pi}{l_x} \\ ky_{N_{ky}} &\geq \frac{2\pi}{l_y} \end{aligned} \quad (99)$$

(c) Another set of conditions for wave number spacing is:

$$\begin{aligned} \sqrt{kx_{n+1}} - \sqrt{kx_n} &\leq \frac{2\pi}{(T\sqrt{g})} \\ \sqrt{ky_{m+1}} - \sqrt{ky_m} &\leq \frac{2\pi}{(T\sqrt{g})} \end{aligned} \quad (100)$$

A physical interpretation of the wave number spacing required by the above conditions has been pointed out by Chapman [5]. The wave numbers defined by (98), (99), and (100) correspond in the physical domain to "vertical walls"

which must be placed sufficiently far away from the body so that the free surface disturbances generated near the body do not reflect back over the time interval. Associated with the faster propagating, longer wavelengths are vertical walls placed further away - the shorter wavelength walls are nearer. The result is that the time interval for each wave to reflect is equal. For further discussion of the reasoning behind this method of wave number spacing and selection criteria, see [5].

### Integration with Equations of Motion

At a time specified by the user, the three-dimensional transient analysis can start, using accelerations and velocities generated by the solution of the equations of motion at that time step. Then, data generated (pressures and loads) from the three-dimensional analysis is not used for a period of time specified by the user, so-called "start-up" time. This allows the free surface disturbance to fully develop since, initially, there exist no radiated or diffracted waves within the three-dimensional simulation. Based on the present level of experience with the program, the minimum start-up time for ship forms has not been fully defined; however, for a sphere, two full cycles of motion appear sufficient.



### 3.0 NUMERICAL RESULTS

Several computer codes have been developed according to the predictive techniques discussed in the previous sections. The main program is entitled SSX (Ship Simulator, Experimental). It performs the simulation of ship motions, hull girder loads and hull pressures. There are two preprocessing programs - HYDREX2 and HYDREX3. The program HYDREX2 computes sectional added mass and damping coefficients for an array of twelve frequencies and six drafts using the Frank close-fit technique. The program also calculates general hydrostatic data for each of the six drafts. The program HYDREX3 computes areas, normals and panel center coordinates for the panel (either quadrilateral or triangular) representation of the hull. It also evaluates three computationally demanding arrays used for the simulation of hull pressures. For further information about these programs, refer to Volume II (Program Manual).

Because the debugging and testing of SSX or any time simulation program, for that matter, is an extremely time-consuming process, the extent of the validation effort has been limited to the minimum number of cases needed to test the capabilities of the computer programs and verify the basic theory and numerical techniques when possible.

#### 3.1 TWO-DIMENSIONAL HYDRODYNAMIC COEFFICIENTS

Sectional added mass and damping coefficients computed by Program HYDREX2 for the SL-7 containership were checked against values generated by a conformal mapping method, as given by Zielinski [33]. Agreement was excellent in most cases. As an example, Figure 4 compares added mass and damping coefficients for heave and sway at midships with a draft of 32.8 feet.

#### 3.2 MOTION RESPONSE AND HULL GIRDER LOADS IN SMALL AMPLITUDE REGULAR WAVES

A few comparisons between computed and experimental values for heave and pitch motions in one-foot regular waves will first be presented to check the time-domain solution methods using strip theory computed forces where the added mass and damping coefficients are calculated for the mean still water sectional drafts. Figure 5 shows the heave and pitch amplitudes and phases for the Series 60 standard hull form with block coefficient 0.70 at Froude numbers 0.15 and 0.20. The points in the figures represent experimental results by Gerritsma and Beukelman [34]; the solid line is computed by the present method, and the broken line by modified strip theory (frequency domain) of Gerritsma and Beukelman. Note that the pitch amplitude is scaled by wave slope. It is seen that agreement is generally good with the exception of heave amplitude in the vicinity of  $\lambda/L = 1.0$ . The reason for this disagreement has not been identified.

Figure 6 gives the theoretical and experimental pitch and heave values for the SL-7 containership at 25 knots in a full load condition. The experimental values shown are by Dalzell [35]; the solid line represents the present

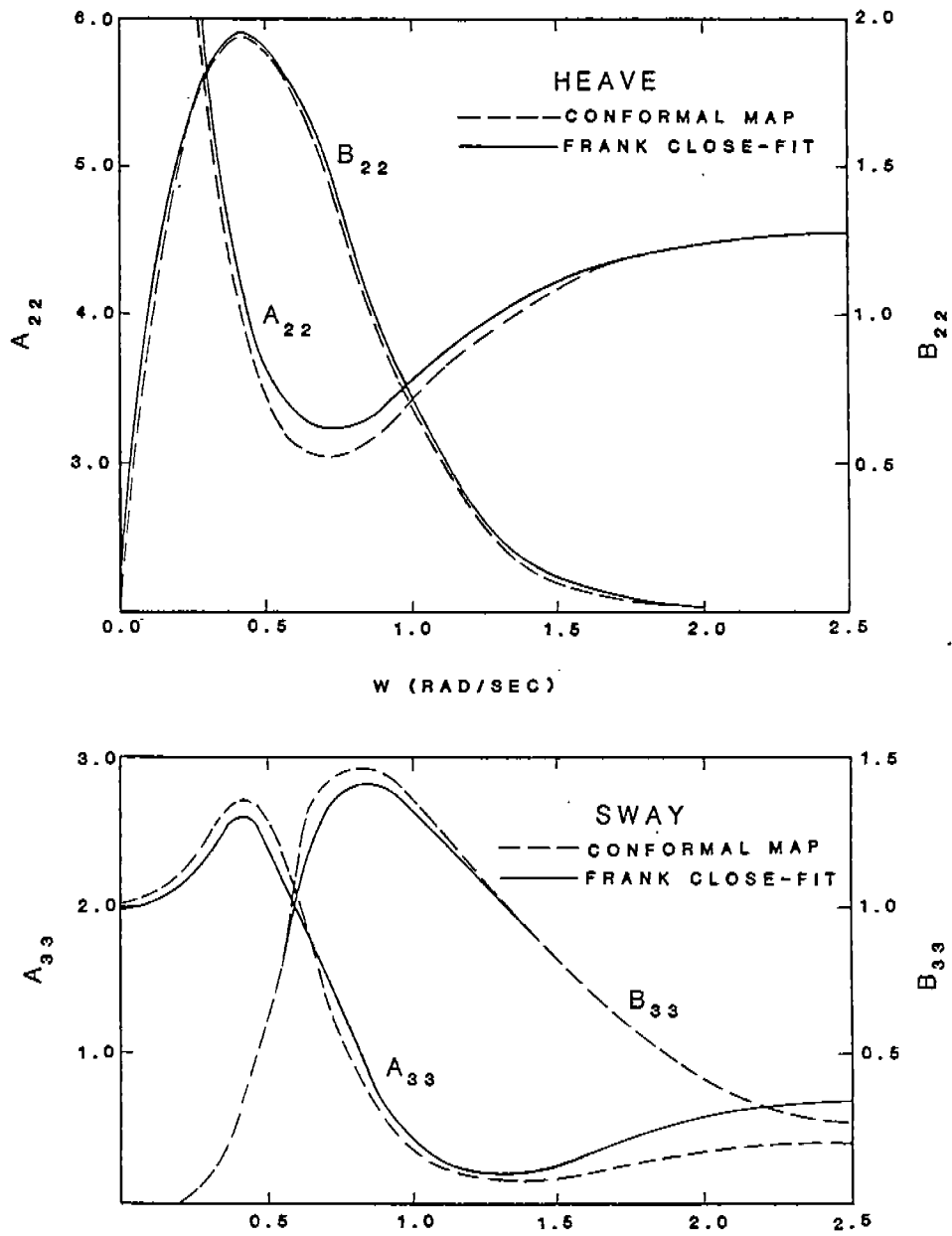


Figure 4 Added mass and damping coefficients. Comparison of HYDREX2 (Frank close-fit) with CGSCORES (Conformal Mapping). SL-7 Containerships, Midships section.

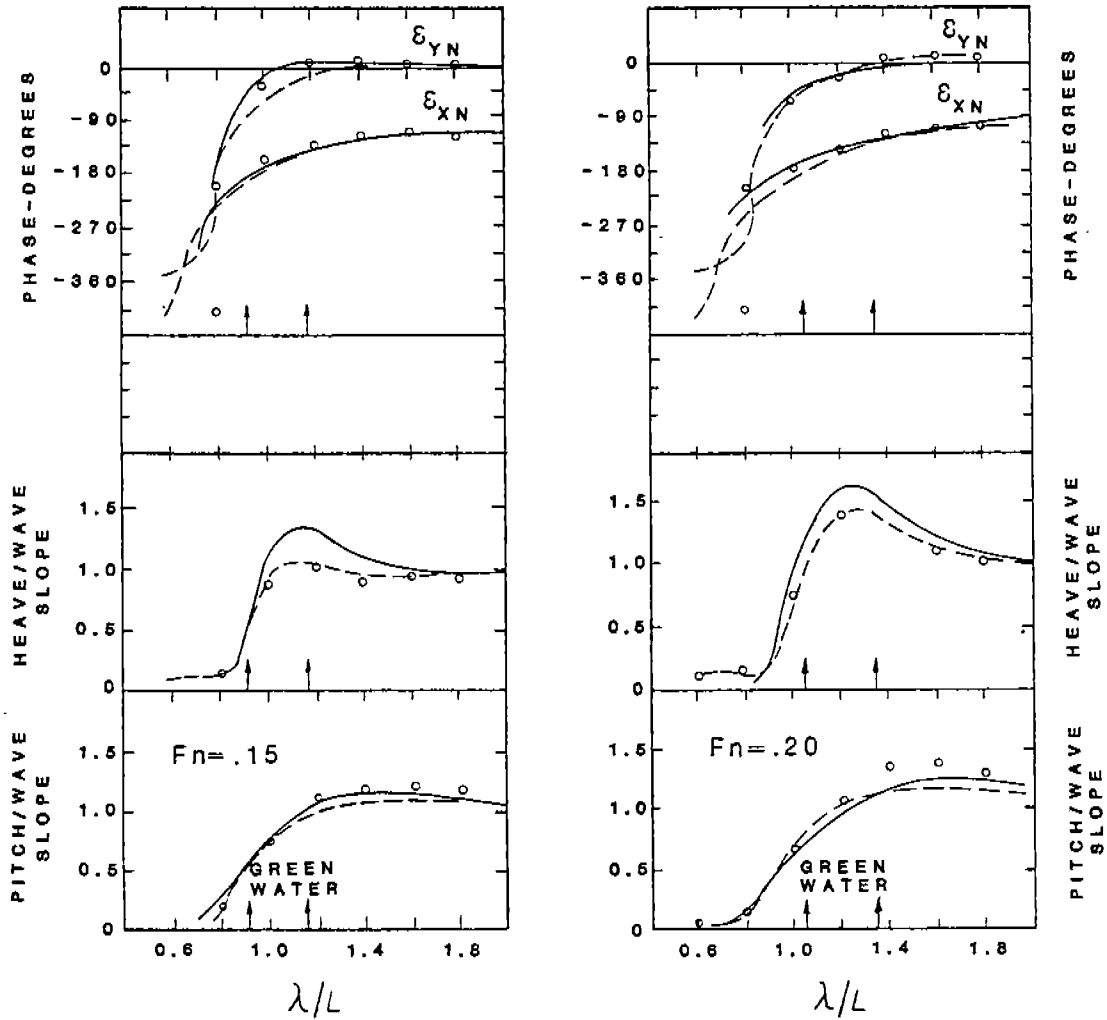


Figure 5 Calculated and measured amplitude and phase characteristics for heave and pitch. Series 60. Block coefficient 0.70. Froude Numbers 0.15 and 0.20.

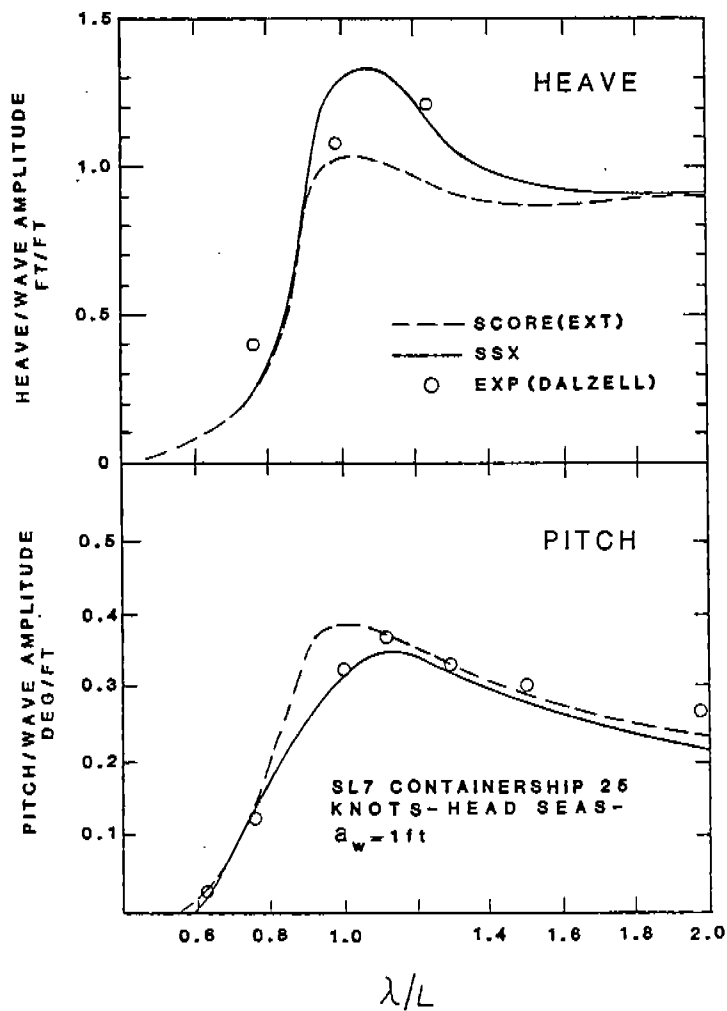


Figure 6 Calculated and measured amplitudes of heave and pitch. SL-7 Containership. 25 knots, head seas.

method, and the broken lines are strip theory results from the SCORES Program [4]. Note that the experimental values for heave at longer wave lengths have not been included because it has been suggested [35] that they were too large and are in error. (The short wavelength experimental values are assumed to be correct.) It is seen that pitch values predicted by SSX agree well with the other values. Heave response, like before, is too high around  $\lambda/L = 1.0$ .

The routine to compute hull girder loads in the present version of SSX is still in a premature stage. The computation of moment arms and sectional inertial forces essentially assumes a lumped mass model. There is no attempt to account for "trapezoidal" sections of the weight curve. Also, weight segments must correspond to the segment between adjacent stations. However, the routine, as written, does provide the means to evaluate the effect of including various forces or different techniques by which to evaluate such forces.

A comparison between computed and experimental midships bending moment amplitudes for the SL-7 containership at 25 knots in a full load condition is shown in Figure 7. The experimental data is from Dalzell [35]; the present theory is represented by the solid line, the broken line shows predictions from SCORES [4]. Agreement is shown to be very good.

Finally, comparison between theory and experiments for midships horizontal and vertical bending moments and pitch angle are shown in Figure 8 for the SL-7 in oblique waves ( $210^\circ$ ). Agreement is satisfactory with the exception of vertical bending moment. Further investigation is needed to identify the causes of disagreement.

### 3.3 IRREGULAR WAVES

A test case was run to check the program's ability to predict motion and loads response in an irregular sea. The SL-7 was subjected to an irregular head sea wave system approximately representing a Bretschneider spectral formulation with  $H_s = 10.8$  ft and  $T_0 = 8$  secs, where  $H_s$  is significant height and  $T_0$  is the period associated with the peak of the sea spectrum. Forward speed was 5 knots. The frequency at which hydrodynamic coefficients used for ship motion-induced forces were to be calculated was set to .8217 - the encounter frequency associated with the peak of the sea spectrum. The sea spectrum was decomposed into ten components. Randomly generated phase angles were used to generate the irregular sea, superimposing the ten regular wave components. A real time simulation of 20 minutes was carried out. Statistics were acquired for heave, pitch and midship's vertical bending moment. Table 1 compares the statistical results of the simulation with predictions from the strip theory frequency domain program developed at MIT and described by Loukakis [36].

As can be seen, the SSX motion values are slightly higher than the MIT program values. The vertical bending moments compare very well. The measured significant RMS wave height from the SSX simulation was close to the desired value of 5.33, providing some assurance that the irregular wave spectrum is correctly being broken down into wave components.



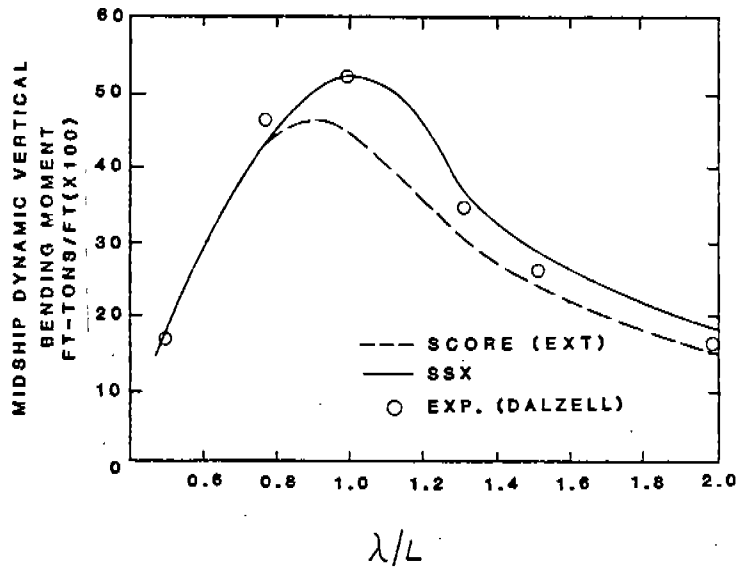


Figure 7 Midship Dynamic Vertical Bending Moment for SL-7 Containership. 25 knots, head seas, full load, regular waves, ampl. = 1 ft.

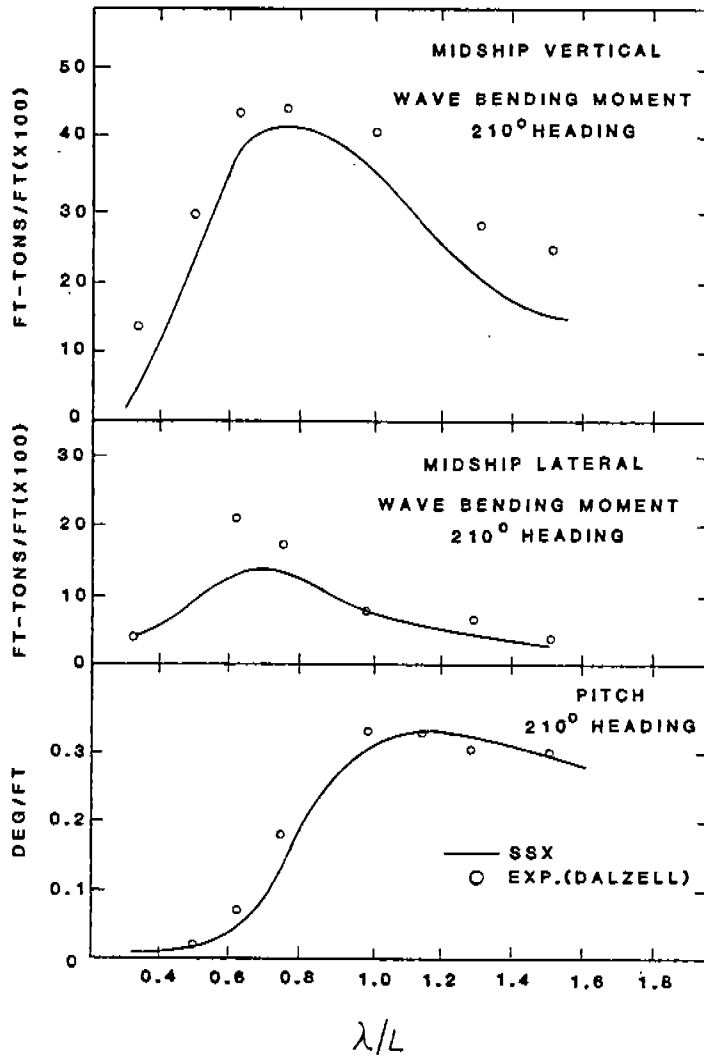


Figure 8 Midship Vertical, Horizontal Wave Bending and Pitch. SL-7 Containership, 25 knots, 210° heading.

Table 1

|                      | SSX   | MITA  |
|----------------------|-------|-------|
| RMS Pitch (deg)      | .2644 | .2606 |
| RMS Heave (ft)       | .5560 | .5308 |
| RMS V.B.M. (ft-tons) | 56844 | 56415 |
| RMS Wave Height      | 5.23  | 5.33  |

One feature of the SSX program is the scheme to average the two most recent zero-crossing periods and derive a "characteristic" frequency at which sectional hydrodynamic coefficients are calculated. This feature was applied to the same test case as before. The results are shown in Table 2.

Table 2

|                       | SSX           |                  |
|-----------------------|---------------|------------------|
|                       | with f (freq) | without f (freq) |
| RMS Pitch (deg)       | .2619         | .2644            |
| RMS Heave (ft)        | .5507         | .5560            |
| RMS V.B.M. (ft.-tons) | 55432         | 56844            |

The effect of invoking the characteristic frequency feature is to reduce the SSX values. The calculated sectional characteristic heave frequency during the course of the simulation generally ranged between  $\omega = .35$  and  $.49$ . This reflects the actual response of the ship and is significantly different from the  $.8217$  value used in the previous simulation. There is still relatively little difference between the RMS values of the two test cases.

### 3.4 NONLINEAR RESPONSE IN LARGER WAVES

As part of many model-test programs, a so-called linearity check is performed where the same test runs are made with increasing wave amplitudes to determine if responses are linearly proportional to wave amplitude. Similar experiments have been performed using SSX. The runs were made at 25 knots in head seas with wave amplitudes of 5, 10 and 15 feet. Added mass and damping coefficients were computed at the mean still water draft. Plots showing peak heave, pitch and midship's vertical bending moment are provided in Figures 9, 10 and 11, respectively.

Note that in Figures 9 and 10, the heave and pitch response has been divided by wave amplitude. In both of these figures, nonlinear behavior is exhibited; that is, normalized response is reduced at higher amplitudes, with the exception of the pitch "bow-down" response. It appears quite linear. In Figure 11, the midship's vertical wave bending moment is shown for various wave lengths and wave heights. Note that in this plot, the response has not been normalized by wave amplitude. In general, response is linear with respect to wave amplitude, with the exception of slight nonlinearities shown for hogging at 15 feet wave amplitude for  $\lambda/L = 0.75, 1.0, 1.25$ .

Also shown in Figure 11 is the effect of including the flare impact force. The broken lines show the bending moment when flare is included. The effect is only measurable in sagging. The values for hogging were so close to the non-flare values that, if drawn, they would be coincident with the lines shown already. (For hogging, a slight reduction in the magnitude of bending moment was the effect of including flare forces.)

The effect of including flare forces on the heave and pitch response is rather minor except around  $\lambda/L = 1$ , as shown in Figures 12 and 13, respectively. The heave and pitch response is shown for a ten-foot regular wave over a range of wave lengths. As shown in Figures 12 and 13, the effect of flare force inclusion is to reduce the amplitudes in both directions (heave up and down, pitch bow-up and bow-down) between  $\lambda/L = 1.0$  and 1.25. It can also be seen in these figures that for this wave amplitude, the SL-7 containership pitches bow-down more than bow-up, and heave-down more than heave-up over the range of wavelengths.

Linearity checks were performed up to regular wave amplitudes of 15 feet because, at 20 feet and above, inconsistent results were sometimes obtained from the program. A check of what was physically occurring at these greater wave amplitudes was performed graphically. Figures 14 and 15 show the motion of the SL-7 containership at 25 knots for  $\lambda/L = 1.5$  and 1.75, respectively. It can be seen that for  $\lambda/L = 1.5$ , bow submergence and emergence are quite extreme. For  $\lambda/L = 1.0$  and  $\lambda/L = 1.25$ , behavior was equally extreme.

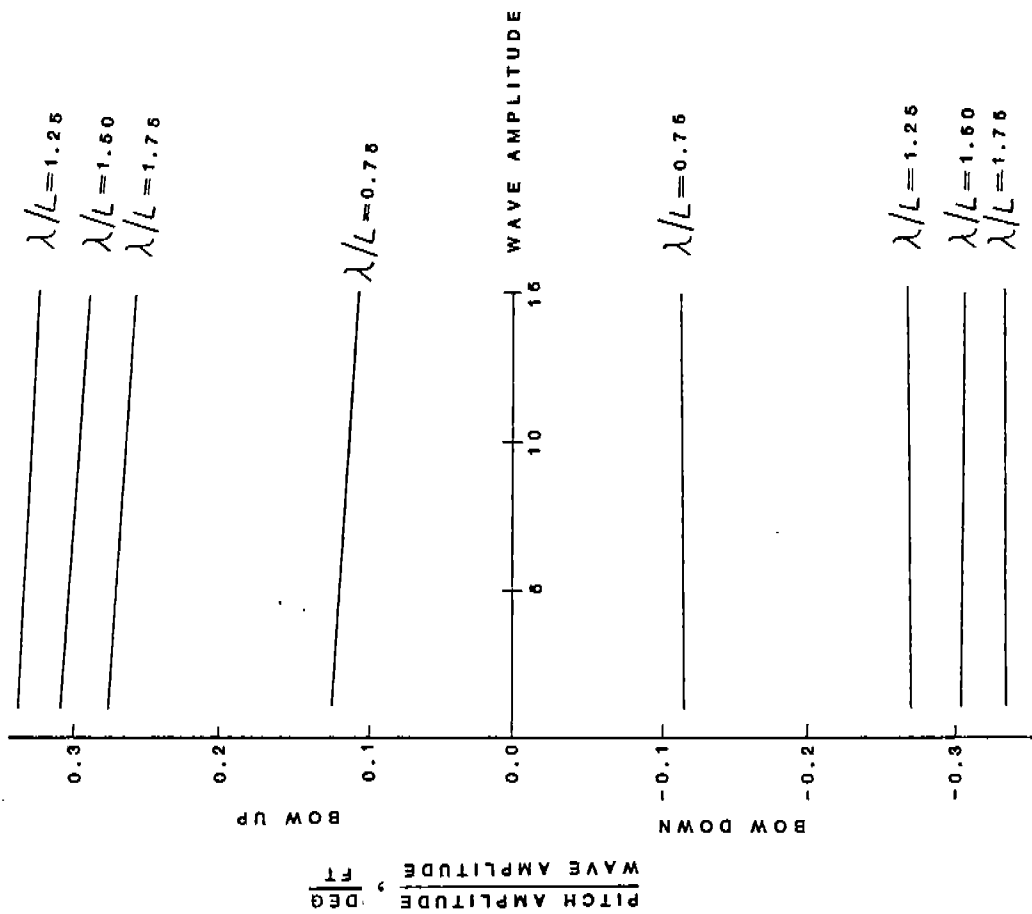


Figure 9 Heave Amplitude/Wave Amplitude vs. Wave Amplitude. SL-7 Containership, 25 knots, full load condition, Head Seas.

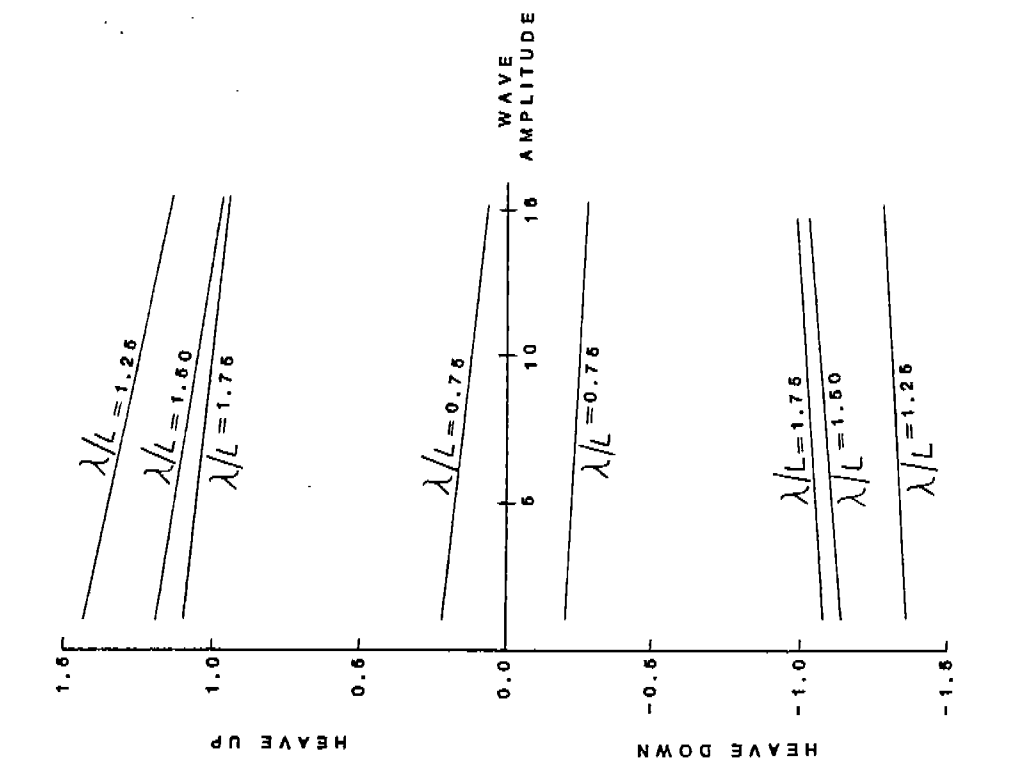


Figure 10 Pitch Amplitude/Wave Amplitude vs. Wave Amplitude. SL-7 Containership, Full Load, Head Seas, 25 knots.

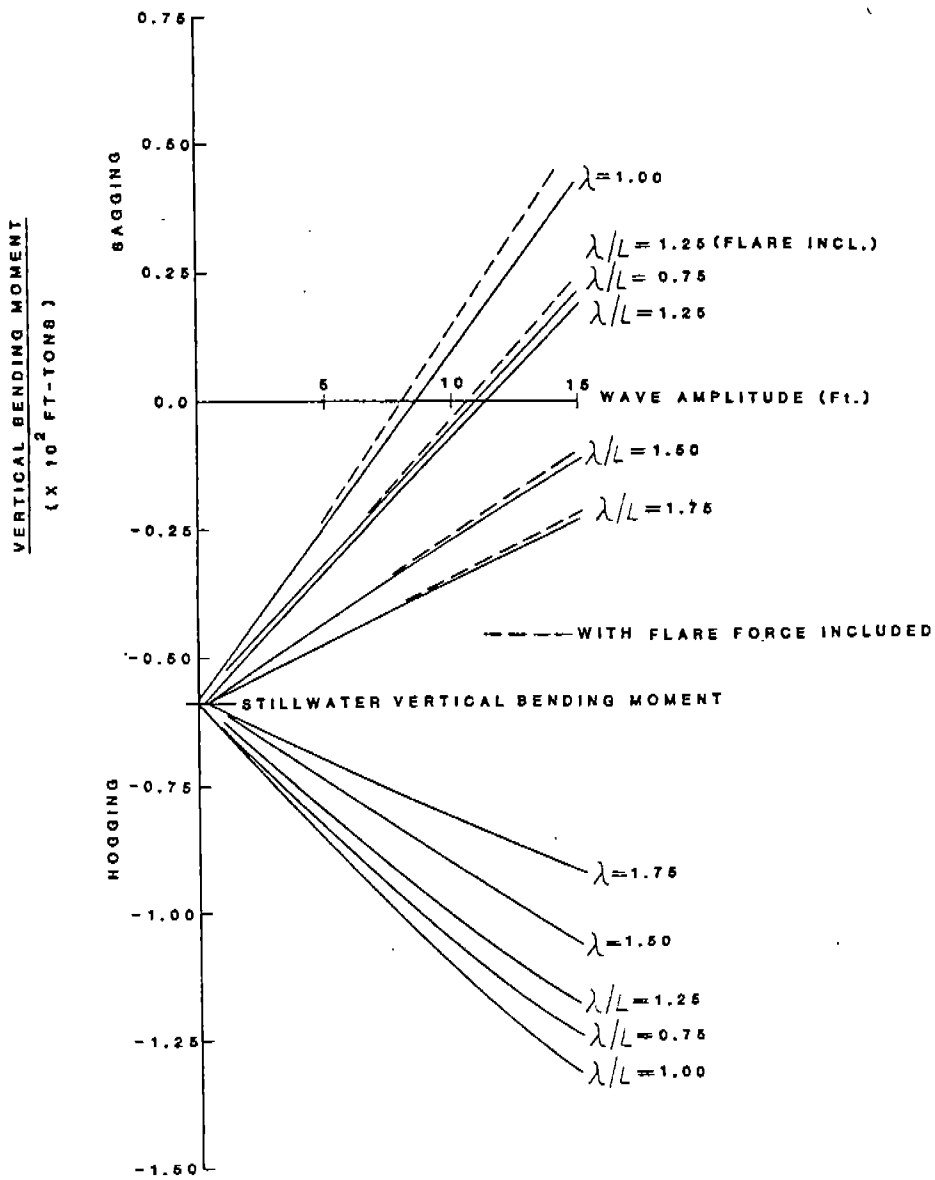


Figure 11 Midship Vertical Bending Moment, SL-7 Containership, 25 knots, head seas (effect of flare on sagging shown).

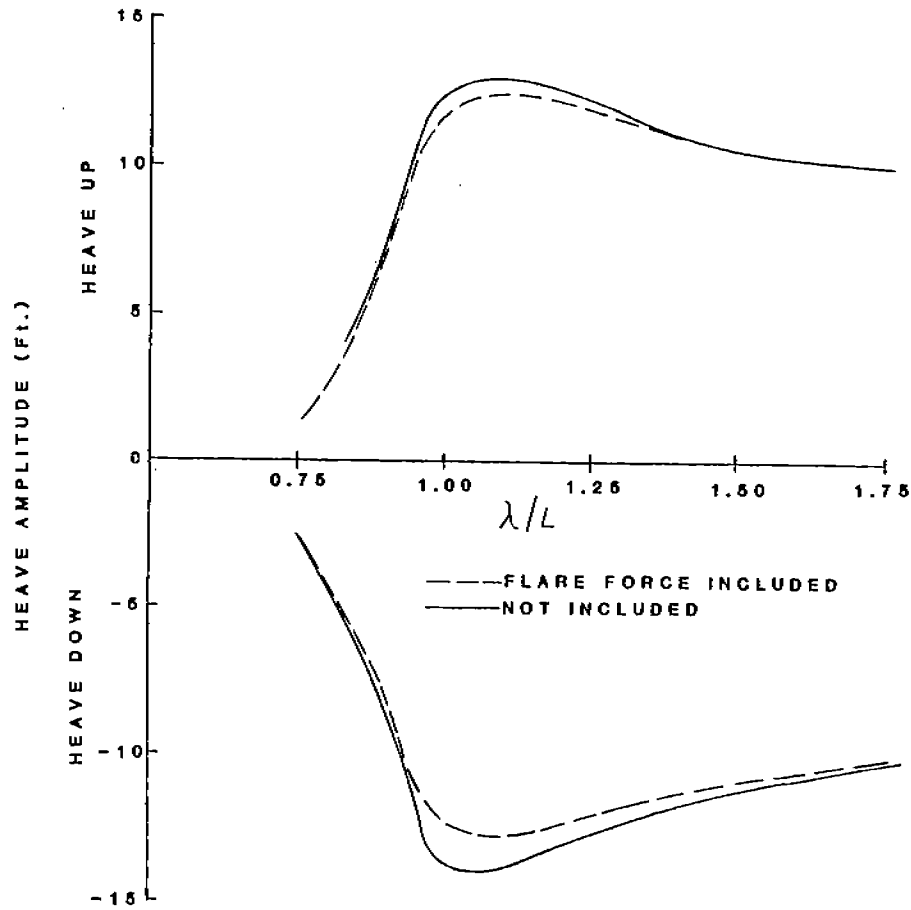


Figure 12 Heave Amplitude vs. Wavelength/Shiplength. Comparison of Theoretical Prediction with and without flare force included

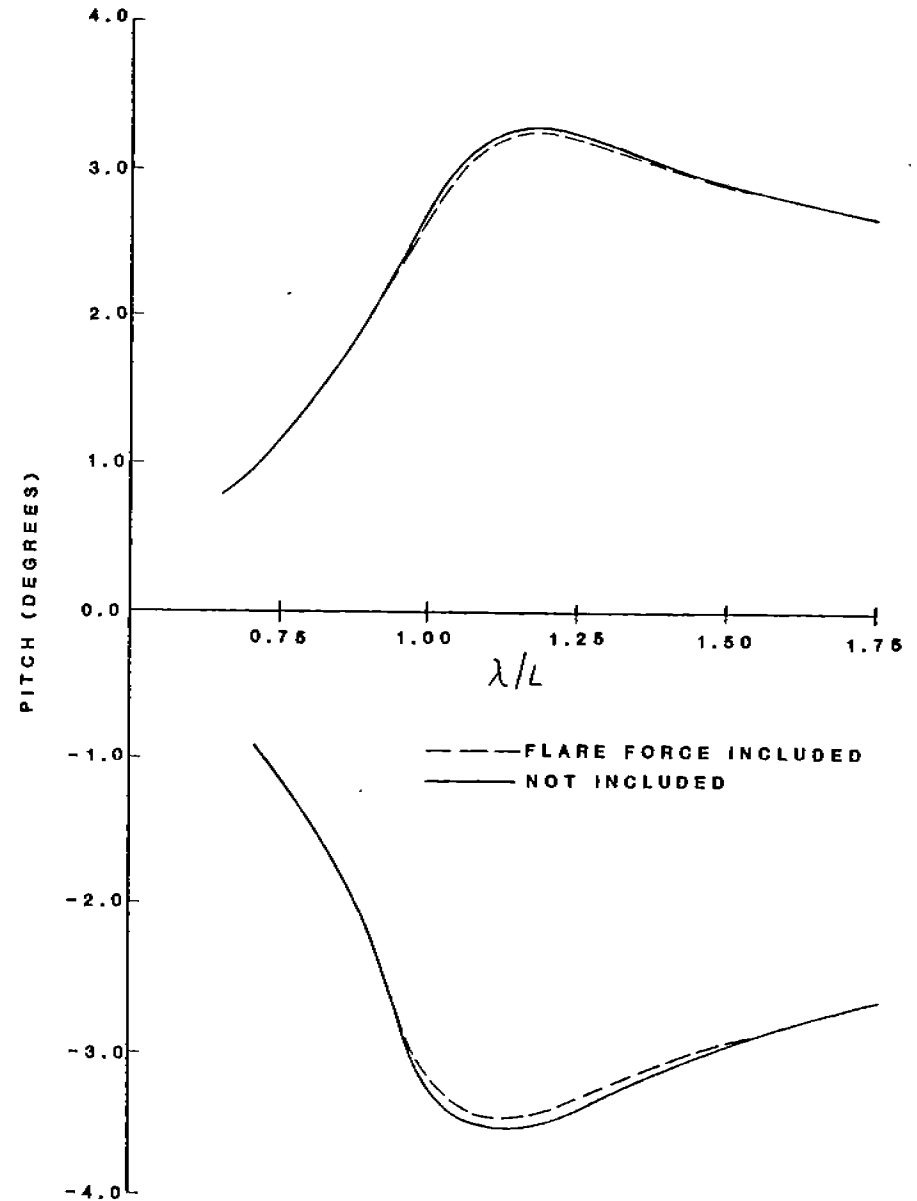


Figure 13 Pitch Amplitude vs. Wavelength/Shiplength. Comparison of Theoretical Prediction with and without flare force included.

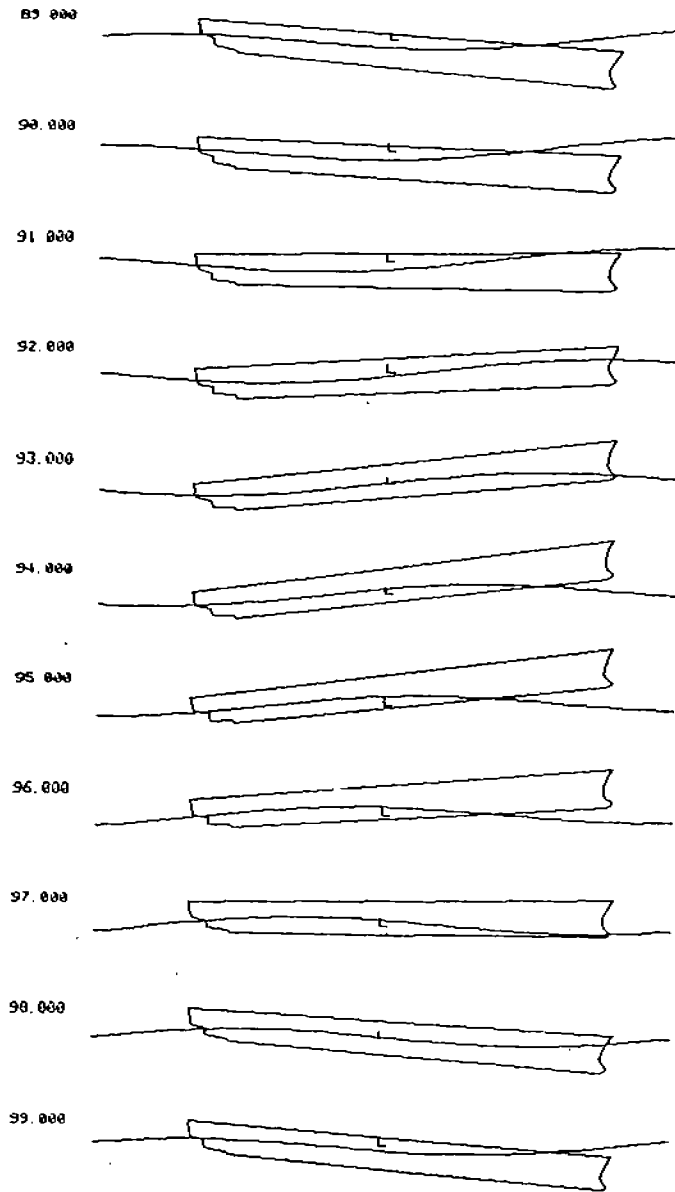


Figure 14 SL-7 Containership in Head Seas at 25 knots. Wavelength/Shiplength = 1.5.

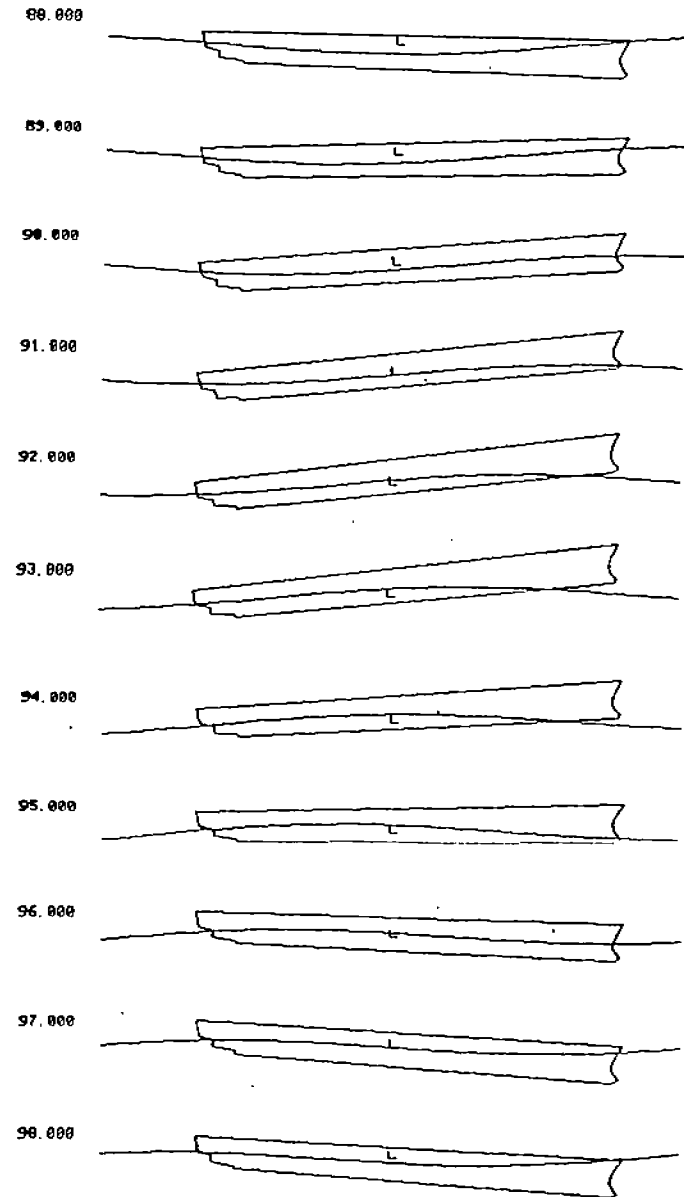


Figure 15 SL-7 Containership in Head Seas at 25 knots. Wave Amplitude = 20 ft, Wavelength/Shiplength = 1.75.

### 3.5 INFLUENCE OF DRAFT-DEPENDENT HYDRODYNAMIC COEFFICIENTS

In order to evaluate the influence of calculating the sectional added mass and damping coefficients at the actual draft (average) at each station for each instant, a series of runs were made at 25 knots in head seas,  $\lambda/L = 2.0$ . Flare forces were not included in these runs. Wave amplitudes were 5, 10 and 20 feet. It was found that within this range of conditions, the use of draft-dependent coefficients had negligible effect on pitch and heave motions and vertical wave bending moment.

### 3.6 FORCE PREDICTION USING THE THREE-DIMENSIONAL APPROACH

The ability of Chapman's method for the prediction of transient motions of a floating body is demonstrated in [5] with the forced oscillation and free oscillation of a two-dimensional rectangle and wedge. Chapman extends the two-dimensional model to three dimensions as reported in [6]. The technique used by Chapman to check the three-dimensional model was to subject a semi-submerged sphere represented by 60 panels to vertical impulse - a unit step in heave velocity. The heave response time history that results is used to derive added mass and damping coefficients. These coefficients are compared with classical results of Havelock [37] and results of Kim [38] with very good correlation.

The basic ability of the three-dimensional method developed here to accurately predict hydrodynamic forces will be verified by two simple tests using a semisubmerged sphere at zero speed. A more comprehensive set of tests is recommended in the future. The two tests presented here should, however, clearly demonstrate that this method has considerable possibilities.

In the first test, the sphere will be subjected to forced vertical oscillations of unit amplitude at various frequencies in otherwise calm water. From the measured pressures, the total vertical time-varying force on the sphere will be calculated, which is then used to derive added mass and damping coefficients. The added mass and damping coefficients can then be compared to the results of Havelock [37].

In the second test, the sphere will be restrained and subjected to regular unidirectional waves of various frequencies and unit amplitude. Wave exciting forces in heave and surge can be measured and compared with analytical predictions given by Garrison [39].

The numerical particulars for both tests are given in Table 3.



The total vertical force was measured over two to three full cycles, depending on the frequency, up to about 20 seconds of simulation time with time steps of 0.2 seconds. The last cycle of the simulation was evaluated to derive added mass and damping coefficients using the method described below.

Table 3  
Numerical Parameters

|            |                              |           |
|------------|------------------------------|-----------|
| NPAN       | = number of panels           | = 72      |
| $L_x$      | = maximum length scale in X  | = 0.25    |
| $l_x$      | = minimum length scale in X  | = 2.5     |
| $L_y$      | = maximum length scale in y  | = 0.25    |
| $l_y$      | = minimum length scale in y  | = 2.5     |
| T          | = maximum time scale         | = 12 sec. |
| g          | = gravitational acceleration | = 1.0     |
| $\rho$     | = fluid density              | = 1.0     |
| a          | = sphere radius              | = 1.0     |
| $\Delta t$ | = time step size             | = 0.1     |

The vertical displacement of the sphere from its mean position is given by

$$y = A \sin \omega t.$$

The linearized equation of motion for the sphere in forced heave is

$$(m + a) \ddot{y} + b \dot{y} + c y = Y_T.$$

where

|       |  |
|-------|--|
| m     | = mass of the cylinder,  |
| a     | = hydrodynamic or "added" mass in heave,                               |
| b     | = damping coefficient against vertical motion,                         |
| c     | = hydrostatic restoring coefficient against vertical displacement, and |
| $Y_T$ | = external driving force.  |

The total hydrodynamic force acting on the body in the vertical direction is

$$Y_h(t) = a \ddot{y} - b \dot{y}.$$

Assuming this hydrodynamic force is a harmonic function in time and can be written as

$$Y_h(t) = \gamma \sin(\omega t - \epsilon),$$

the hydrodynamic coefficients are obtained by equating these two expressions for  $Y_h$ ,

$$a = \frac{\gamma \cos \epsilon}{\omega^2 A}$$

and

$$b = \frac{\gamma \sin \epsilon}{\omega A}.$$

The amplitude  $\gamma$  and phase shift  $\beta$  are obtained by comparing plots of the displacement and dynamic pressure force  $Y_h$  acting on the sphere as functions of time and measuring the shift in phase. A comparison of the results obtained with Havelock's results is shown in Figure 16. The solid line is Havelock's results. The numerical predictions are shown as circles. As can be seen from Figure 16, agreement is excellent.

Figure 17 shows a time sequence of the free surface disturbance induced by the vertical motion of the sphere. The circle represents the mean still waterline of the sphere. The radius of the sphere is 1.0 and the dimensions of the free surface area shown are 8 x 8. The wave number for the sequence is  $k = 1.0$ . The internal standing waves inside of the sphere are shown. (These internal standing waves should produce no net pressure.)

### Wave Excitation Forces on Semisubmerged Sphere

The purpose of the second simulation was to investigate forces on the sphere when subjected to regular waves of different frequencies. Figure 18 shows the comparison between analytical predictions from Garrison [38] for the heave and surge wave exciting forces on a semisubmerged sphere. The agreement is excellent.

Figure 19 shows the diffraction wave system that develops from regular waves ( $h = 1.4$ ) imposed on the sphere. The ambient wave amplitudes have been subtracted out of the free surface plot to more clearly exhibit the diffracted waves.

### Other Tests

Attempts were also made to: (a) test a semisubmerged sphere in forced vertical oscillations at a uniform forward speed; and (b) to conduct forced oscillation tests for ellipsoids with L/B ratios of 8:1 and 4:1. In each of these cases, forces quickly increase to unrealistic levels, clearly indicating

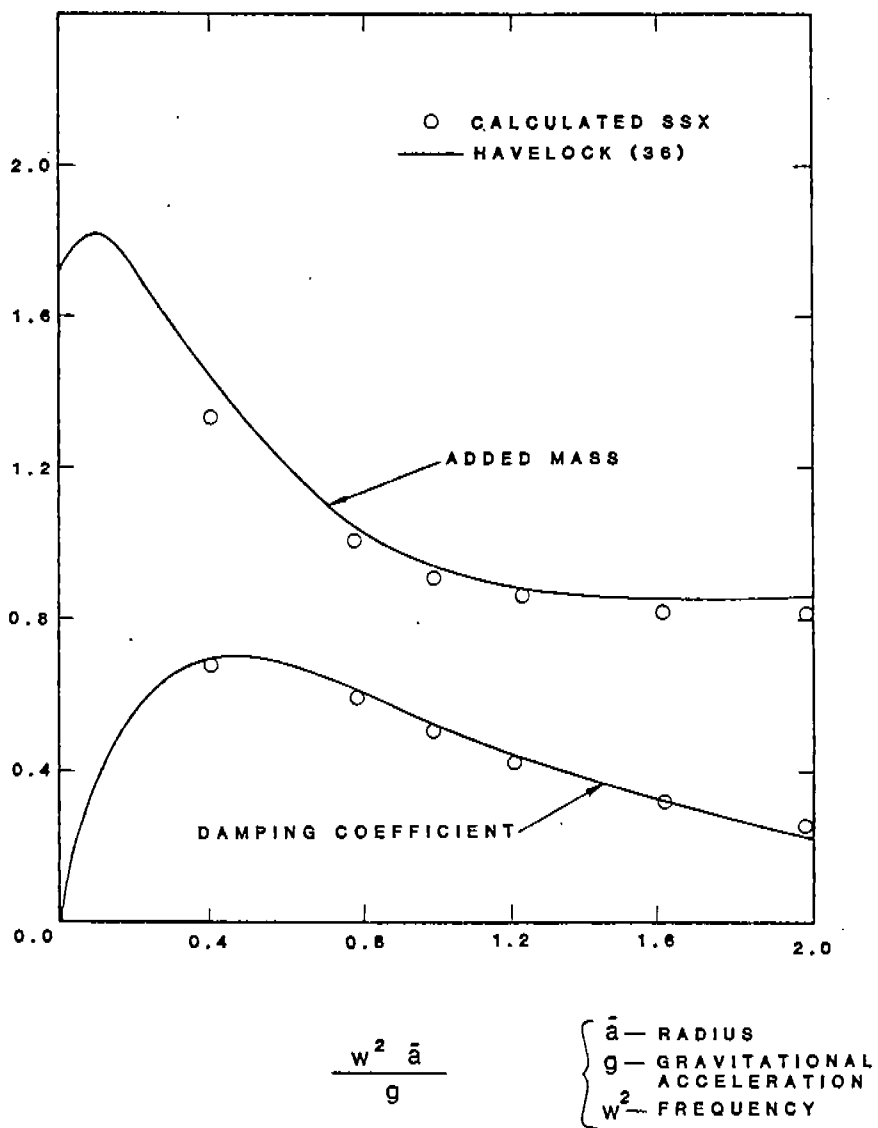


Figure 16 Added Mass and Damping Coefficients for semisubmerged sphere.

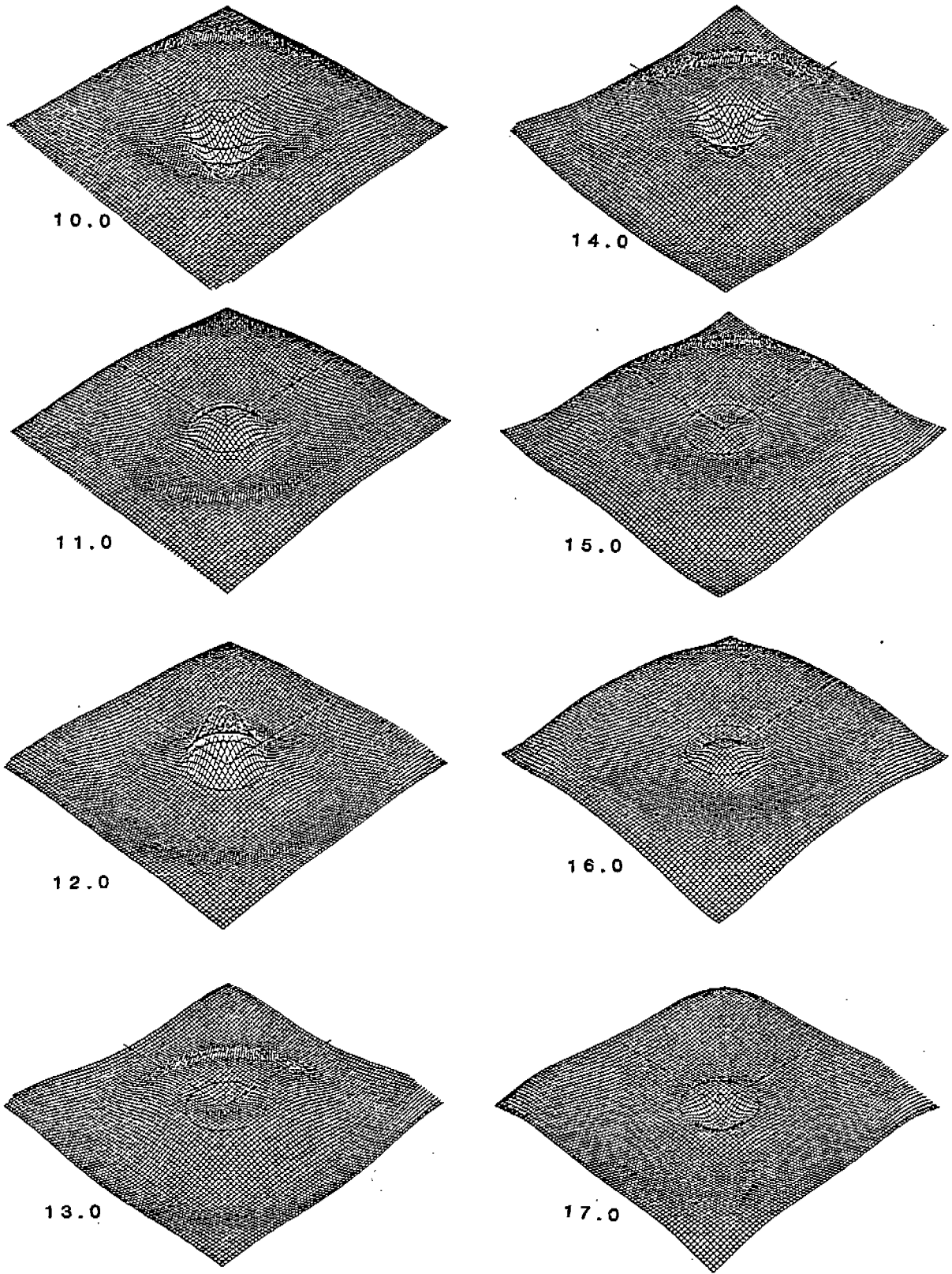


Figure 17 Radiated waves from vertically oscillating sphere at  $k = 1.0$ .  
Time = 10 - 17 secs after startup.

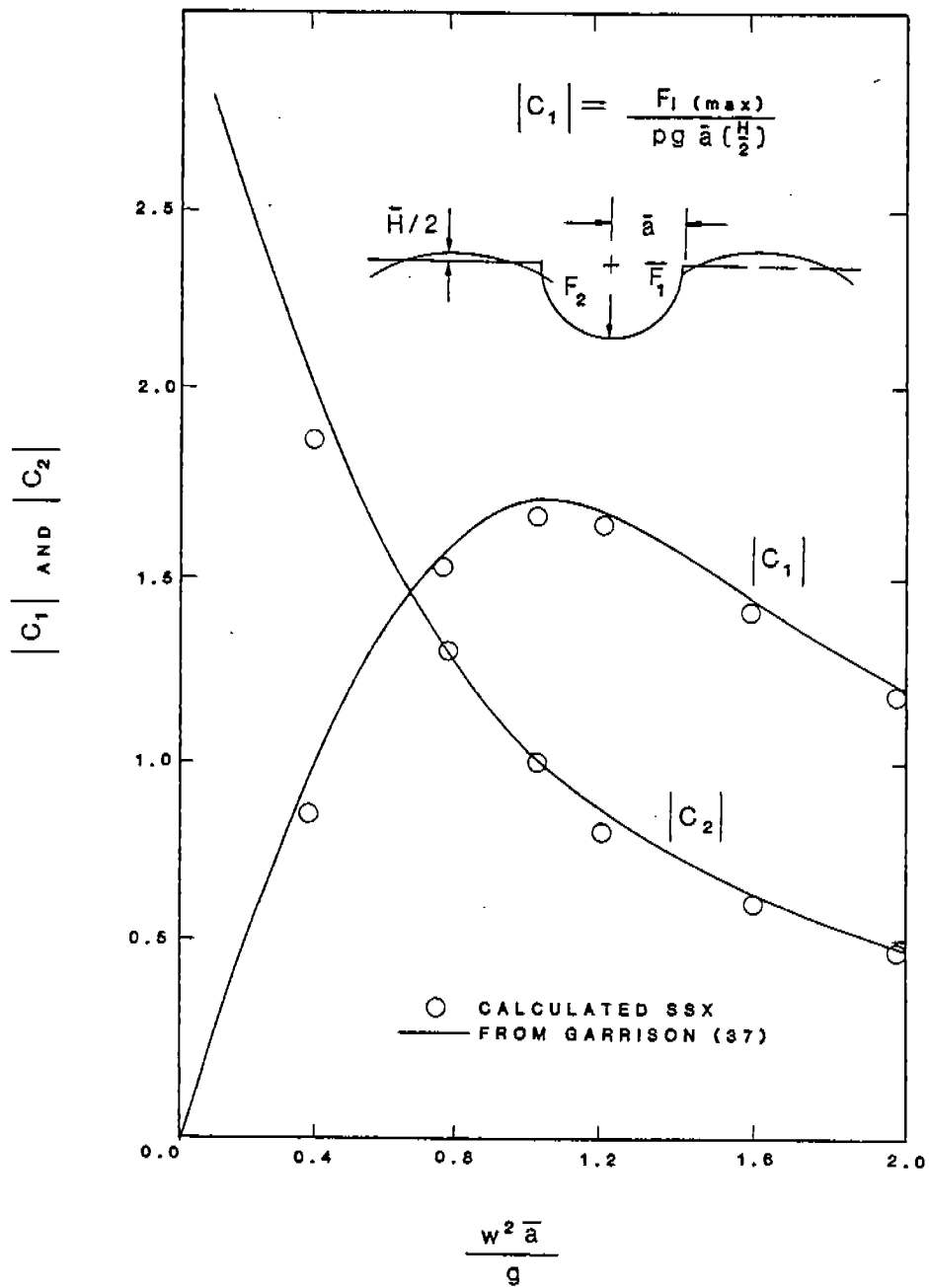


Figure 18 Wave excitation forces for semisubmerged sphere.

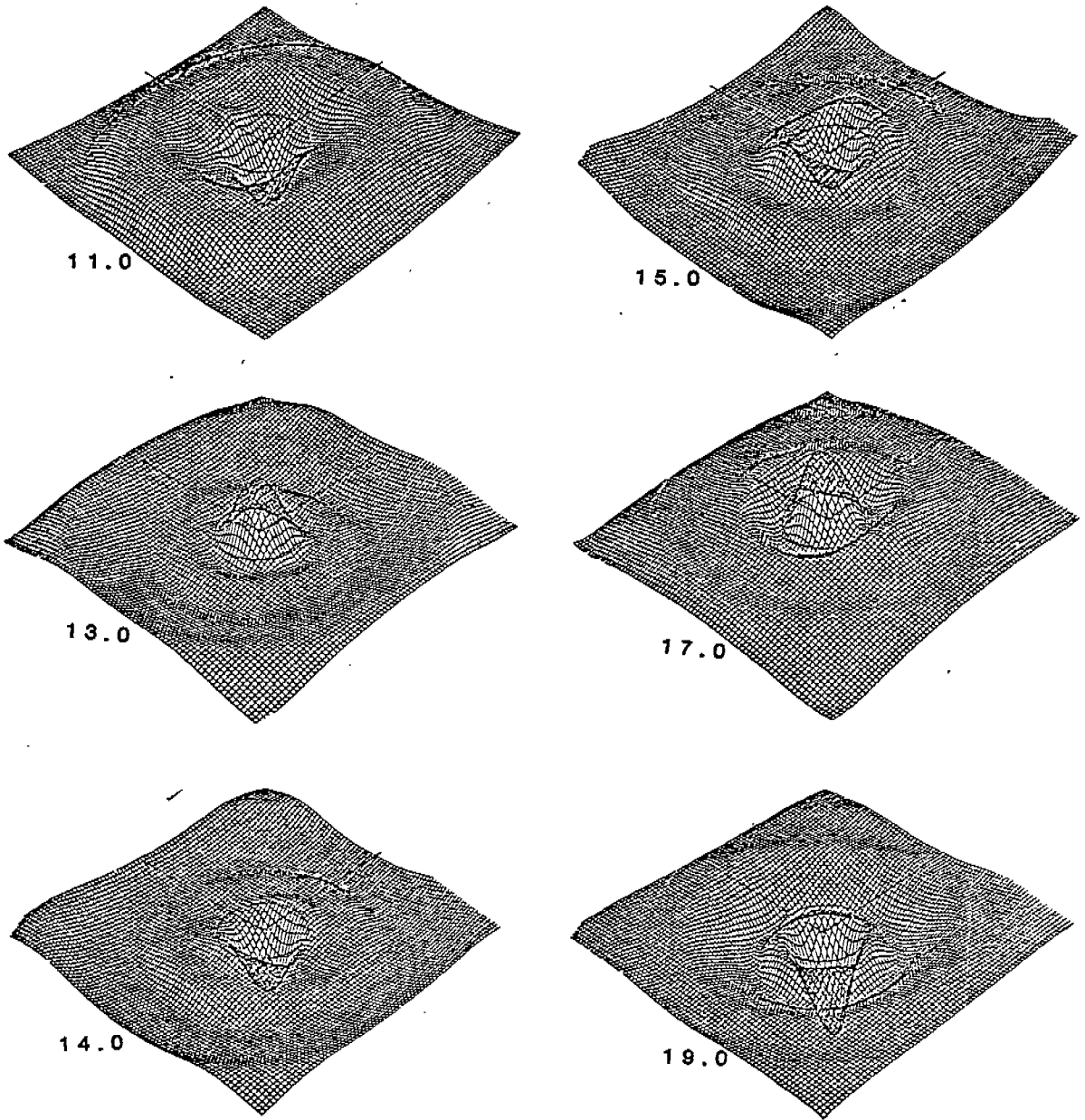


Figure 19     Diffracted waves from semisubmerged sphere,  
 regular unidirectional waves,  $k=1.4$ .  
 $T=11-19$  seconds.

that there is some problem associated with both forward speed and cases with shapes more closely representing a slender ship. The reasons for these "blow-ups" could not be readily identified.

### 3.7 LIMITATIONS, RUN TIMES AND COMPUTATIONAL REQUIREMENTS

#### Theoretical and Numerical Limitations

- (1) The limitations of strip theory generally apply.
- (2) The limitations of linear wave theory apply.
- (3) The limitations of linearized hydrodynamic formulations (linearized free surface, separation of potentials, linearized Bernoulli equation, etc.) apply.
- (4) Strip theory formulations here do not include terms related to the spatial derivative of damping coefficients (as found in "extended" SCORES).
- (5) Flare force is computed using the relative free surface velocity in contrast to using a component of average subsurface wave velocity. Furthermore, free surface distortion effects are not included (diffractions, "pile-up," etc.).
- (6) The ambient wave system for the three-dimensional analysis is formulated for unidirectional waves only.
- (7) The P, E and X source influences matrices are computed only for the mean hull position in the present formulation. Details of the computation are presented in Volume II of this report.
- (8) The use of either a pre-selected or time-varying estimate of added mass and damping coefficients for the ship motion-induced forces in irregular waves is an approximation.
- (9) Pitch angle must not become large enough to cause the intersection of a station plane and the sea surface so that multiple regions or a closed contour in the station plane is defined [3].

#### Run Times

Typical run times for several simulations with strip theory derived forces and three-dimensionally derived forces are shown in Table 4 for a VAX 11/750 without a floating point processor.

It is estimated that reductions in run time by a factor of 5-10 would occur with any of the large mainframe CDC machines; and a reduction by a factor of 15-20 would occur with a supercomputer (CRAY or similar). It should be emphasized that the program is expensive to run, regardless of the type of machine on which it is installed.

Table 4

| SIMULATION OF .....  | VAX 11/750 CPU TIME |
|--|---------------------|
| Motion and Hull Girder Response of 25-station ship in one regular wave. Steady state evaluated after 2 minutes real time with $\Delta t = 0.5$ sec.                    | 1200 CPU seconds    |
| Motion and Hull Girder Response of 25 station ship in irregular wave system of 10 components evaluated after 10 minutes of real time simulation. $\Delta t = 0.5$ sec. | 16 CPU hours        |
| Pressure prediction only of 120 panel body for 15 seconds real time $\Delta t = 0.1$ sec.  | 17 CPU hours        |

#### Computational Requirements

The computer programs were developed on a DEC VAX 11/750. The peak working set size was 250. The memory requirement for the completely integrated program (2D plus 3D) is 650K bytes. When the 3D subroutines are not used and only 2D simulations are used, the memory requirement is 250K bytes.



#### 4.0 CONCLUSIONS AND RECOMMENDATIONS

A numerical method for the simulation of ship motions, hull girder loads and transient three-dimensional hydrodynamic pressures has been developed. The result is a computer program that is capable of evaluating nonlinear responses of surface ships associated with large amplitude motions. The fundamental limitation of the approach is the use of linear wave theory. The restrictions on the severity of the sea conditions that this limitation imposes is not yet known, although it is believed that the method would be appropriate for evaluating motions and hull girder loads in moderately severe sea states.

Furthermore, the three-dimensional technique developed is capable of predicting transient hydrodynamic pressures on time-varying hull shapes moving with arbitrary motions and forward speed. Further effort is required to fully realize these capabilities. An outline of the suggested work is given in the next section.

Although the method presented here and the associated computer program need additional attention to advance them from the investigative and developmental stage at which they now are, some progress has been made toward a more advanced method for ship motion and wave load prediction. There presently exists no other program for the time domain simulation of ship motions and loads in six degrees of freedom for a full range of headings. Nor is there any other program capable of predicting three-dimensional distributed hydrodynamic pressures in the time domain for arbitrary ship motions in waves.

The CAPSIZE program [3,29] and the NSUP program [6] which were used as starting points for the present method have been extended or modified in the following principal ways:

- Actual strip theory formulation used for hydrodynamic forces instead of simplified and approximate methods for added mass, damping and wave exciting forces
- Hull girder loading prediction added, including "flare force" effect
- The addition of the incident wave potential into the three-dimensional formulation
- A number of computational and numerical changes associated with the calculation of two-dimensional hydrodynamic coefficients, the evaluation of two-dimensional hydrodynamic forces, and the evaluation of pressure in the wave above the still water level.

The bulk of the effort in this project was committed to the concept development of a suitable method and the development and testing of the computer code. Because of the particularly time-consuming nature of debugging and testing simulation programs, the extent of the validation effort has been limited to the minimum number of runs needed to verify the capabilities of the

computer programs and evaluate the basic theory and numerical techniques when possible. A full and extensive validation of the methods and codes would require the type of test program typically associated with actual (physical) model testing, with a level of effort representative of a major project in itself. An outline of a simulator test program is provided in Appendix A as a possible basis for the same type of studies reported in SSC-246\* [40] and SSC-271\*\* [41] to validate and further investigate SCORES after its initial development.

### Recommendations

The work presented herein represents an initial level of investigation and development. The numerical methods and computer programs require further testing, validation and improvement. The following list of recommendations indicates areas where further effort is required or desirable in the near-term, mid-term or far-term.

#### Near-Term

1. A comprehensive test and validation program is required, an outline of which is given in Appendix A for the two-dimensional method.
2. The method or program coding for the computation of Fróude-Krylov forces or strip-theory derived forces should be corrected or improved to provide better predictions, specifically heave motion.
3. The numerical methods and coding associated with the three-dimensional technique should be improved. Specifically, the following areas will be investigated and modified or improved:
  - (a) Investigate the method used for computing the source panel velocity array and the source potential influence array. The so-called exact method of Smith and Hess might be improved. More efficient methods might also be incorporated. There may be reason to suspect that the problems with the ellipsoid tests attempted are due to numerical problems in this area.
  - (b) Investigate the theoretical formulations that include some forward speed effects. Investigate the numerical/computational representation of the theory. Check the coding dealing with forward speed aspects. Identify any deficiencies and make the appropriate changes and additions.

\* Kaplan, P., et al, "Theoretical Estimates of Wave Loads on the SL-7 Containership in Regular and Irregular Waves," SSC-246, 1974.

\*\* Kaplan, P., et al, "A Correlation Study of SL-7 Containership Loads and Motions - Model Tests and Computer Simulations," SSC-271, 1977.

- (c) Examine the feasibility and necessity of applying a different free-surface condition interior to the hull. As Chapman suggested in [6], instead of using the linearized free-surface condition over the entire free surface, the surface elevation interior to the hull can be constrained to be uniformly zero by placing surface panels over this region. This would suppress standing waves which are excited in the hull interior as a by-product of the potential flow method of simulation.
- (d) Investigate more efficient ways of computing. For example, if we can assume that motions are sufficiently small, then certain values which are functions, sums and products of the various source influence arrays might be precomputed and stored, rather than computing them inside computational loops over  $k_x$  and  $k_y$  wave numbers for each panel.
- (e) Demonstrate the accuracy and capabilities of the program by comparing numerical results with predictions from theory or other programs for various shapes at zero speed (sphere, ellipsoids; barge shape, ship shape) and for those shapes with forward speed. The basis for these comparisons would be added mass, damping and wave excitation coefficients.
- (f) Carry out sufficient testing to develop an experience base for setting the parameters for the spectral wave representation.
- (g) Investigate and demonstrate the ability of the method to properly predict the distributed hydrodynamic loads on the underwater hull.
- (h) Examine the feasibility and identify the numerical techniques that would allow for the underwater hull form to change with time.

#### Mid-Term

- 4. Appropriate specific methods to predict slamming loads and green water loads should be investigated and incorporated in the present program.
- 5. An advanced statistical processing routine should be incorporated in the program to derive frequency response spectra, as well as probability density functions and cumulative distributions of maxima and minima.
- 6. A method should be investigated and appropriate subroutines incorporated to structurally model the ship hull girder in order to predict transient non-rigid body response.
- 7. Improved coding for the computation of sectional mass distribution properties should be written.
- 8. Coding for the computation of appendage damping should be written.

## Far-Term

9. Linear wave theory was assumed for the present model and is consistent with the linearized free-surface formulations associated with strip theory and the three-dimensional method. It may be worthwhile, however, to pursue the use of a nonlinear wave theory to generate large amplitude deterministic unidirectional waves. Although the nonlinear wave could not be applied, in a strict sense, within the hydrodynamic formulations as they exist in the two- and three-dimensional formulations, they could be accounted for in an approximate way. Hydrostatic and dynamic buoyancy forces could be accounted for "exactly." Ship motion-induced hydrodynamic forces could be approximated as they are in the present version. Diffraction forces could be possibly estimated by using the same method as now exists but representing the nonlinear wave in the vicinity of the ship with a similarly proportioned linear wave. For the transient pressure simulation, the exact nonlinear wave kinematics could be used to prescribe part of the body boundary condition at the center of each panel.
10. The use of the impulse response function and convolution integral to predict ship motion-induced damping forces should be investigated and appropriate subroutine(s) included in the present version. This would be a partial alternative for the "characteristic frequency" scheme now in place.

The efforts listed under each of the above categories can be grouped into the three separate projects with the following approximate level of effort suggested:

|                   |               |
|-------------------|---------------|
| Near-Term Project | 3000 manhours |
| Mid-Term Project  | 2000 manhours |
| Far-Term Project  | 2000 manhours |

The near-term project should be considered a necessary requirement in order that the methods and computer programs initiated here are sufficiently developed and proven for practical use.

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## APPENDIX A

### NUMERICAL SIMULATOR TEST PROGRAM

#### 1.0 OBJECTIVE

The objective is to fully exercise the computer program SSX, using as a test vehicle a ship for which model tests have been carried out for a range of headings, speeds and wave heights, with emphasis on lower speeds and higher wave heights.

#### 2.0 TEST SERIES

##### 2.1 TEST SERIES A - REGULAR WAVE TESTS

The ship is subjected to unit amplitude unidirectional regular waves at five headings, four Froude numbers, and at six frequencies corresponding to various wavelength/ship-length ratios. The critical parameters are:

|   |  |
|---|--|
| <u>Wavelength Parameters</u> ( $\lambda/L$ )<br>at $\beta = 0$ 180° | 0.5, 0.75, 1.0, 1.25, 1.5, 2.0<br>(or equivalent at other headings)  |
| <u>Froude Numbers</u> ( $F_n$ )                                     | 0.0, 0.05, 0.10, 0.15  |
| <u>Headings</u> ( $\beta$ )   | 180, 135, 90, 45, 0  |
| <u>Measurements</u>   | - (a) C.G. Motions<br>(b) Midships Hull Girder Loads<br>(c) Sectional Forces Including<br>Hydrostatic, Froude-Krylov,<br>Wave Exciting and Ship<br>Motion Induced Forces |
| <u>Total Number of Runs</u>   | 120  |

##### 2.2 TEST SERIES B - LINEARITY CHECKS (NO FLARE FORCE)

The ship is subjected to unidirectional regular waves of four different heights, six frequencies, two speeds and five headings. The critical parameters are:

|  |  |
|--|--|
| <u>Wave Height</u> ( $H_w$ )                 | 0.2D, 0.4D, 0.6D, 0.8D<br>(Where D is Ship Depth)                    |
| <u>Wavelength Parameters</u> ( $\lambda/L$ ) | 0.5, 0.75, 1.0, 1.25, 1.5<br>(With the Constraint $H/\lambda < .1$ ) |



|  |  |
|--|--|
| <u>Froude Numbers (<math>F_n</math>)</u> | 0.0, 0.10  |
| <u>Headings</u>                          | 180, 135, 90, 45, 0                                |
| <u>Measurements</u>                      | (a) C.G. Motions<br>(b) Midships Hull Girder Loads |
| <u>Total Number of Runs</u>              | 200  |

### 2.3 TEST SERIES C - LINEARITY CHECK WITH FLARE FORCE INCLUDED

The ship is subjected to unidirectional regular waves in head and following directions, three speeds and five frequencies and four wave heights. The critical parameters are:

|   |  |
|---|--|
| <u>Wavelength Parameters (<math>\lambda/L</math>)</u> | 0.5, 0.75, 1.0, 1.25, 1.5  |
| <u>Froude Numbers (<math>F_n</math>)</u>              | 0.0, 0.10, 0.15  |
| <u>Headings</u>                                       | 180, 0   |
| <u>Wave Heights</u>                                   | 0.2D, 0.4D, 0.6D, 0.8D   |
| <u>Measurements Taken</u>                             | (a) C.G. Motions<br>(b) Midships Hull Girder Loads<br>(c) Sectional Forces |
| <u>Total Number of Runs</u>                           | 120  |

### 2.4 TEST SERIES D - LINEARITY CHECK WITH DRAFT-DEPENDENT HYDRODYNAMIC COEFFICIENT SELECTION

The ship is subjected to unidirectional regular waves of four different heights, six frequencies, two speeds and three headings. The critical parameters are:

|                              |  |
|------------------------------|--|
| <u>Wave Height</u>           | 0.2D, 0.4D, 0.6D, 0.8D   |
| <u>Wavelength Parameters</u> | 0.5, 0.75, 1.0, 1.25, 1.5, 2.0   |
| <u>Froude Numbers</u>        | 0.0, 0.10  |
| <u>Headings</u>              | 180, 135, 90   |
| <u>Measurements</u>          | (a) C.G. Motions<br>(b) Midships Hull Girder Loads<br>(c) Sectional Forces |
| <u>Total Number of Runs</u>  | 144  |

## 2.5 TEST SERIES E - IRREGULAR WAVE TESTS

The ship is subjected to four unidirectional irregular systems with frequency spectra corresponding to a two-parameter Bretschneider formulation, one speed, five headings. The critical parameters are:

|  |   |
|--|---|
| <u>Significant Wave Heights (<math>H_s</math>)</u> | 10, 20, 30, 40 ft.  |
| <u>Spectral Modal Frequencies</u>                  | Most Probable   |
| <u>Headings</u>                                    | 180, 135, 90, 45, 0   |
| <u>Froude Number</u>                               | 0.10  |
| <u>Measurements Taken</u>                          | (a) C.G. Motion Statistics<br>(b) Midship Hull Girder Load Statistics |
| <u>Total Number of Runs</u>                        | 20  |

In Test Series E, the frequency at which the 2-D hydrodynamic coefficients are selected is fixed according to the peak of the response spectra obtained from a linear frequency domain analysis.

## 2.6 TEST SERIES F - IRREGULAR WAVE TESTS WITH DRAFT-DEPENDENT HYDRO-DYNAMIC COEFFICIENT SELECTION

The ship is subjected to three unidirectional wave systems, one speed, three headings. Two-dimensional coefficients are selected as a function of instantaneous sectional draft. The critical parameters are:

|  |   |
|--|---|
| <u>Significant Wave Heights (<math>H_s</math>)</u> | 20, 30, 40  |
| <u>Headings</u>                                    | 180, 135, 90  |
| <u>Froude Number</u>                               | 0.10  |
| <u>Measurements Taken</u>                          | (a) C.G. Motion Statistics<br>(b) Midship Hull Girder Load Statistics |
| <u>Total Number of Runs</u>                        | 6   |

For Test Series F, the ship motion hydrodynamic coefficients are selected as with Test Series E.

## 2.7 TEST SERIES G - IRREGULAR WAVE TESTS (WITH VARYING FREQUENCY)

Same as Test Series D, except the 2-D hydrodynamic coefficients associated with ship motion-induced forces are selected according to the most recent measured zero crossing periods of motion.

|                                     |   |
|-------------------------------------|---|
| <u>Additional Measurement Taken</u> | Automatically - Selected<br>Frequencies Used for Heave,<br>Sway and Roll 2-D Coefficients |
| <u>Total Number of Runs</u>         | 20  |

## 3.0 SCOPE OF WORK

The following table is used to estimate computer costs using a VAX 11/750 with below average rates:

| <u>Test Series</u> | <u>No. Runs</u> | <u>CPU Hours Per Run</u> | <u>Subtotal Hours</u> |
|--------------------|-----------------|--------------------------|-----------------------|
| A                  | 120             | 1                        | 120                   |
| B                  | 200             | 1                        | 200                   |
| C                  | 120             | 1                        | 120                   |
| D                  | 144             | 1                        | 144                   |
| E                  | 20              | 32                       | 640                   |
| F                  | 6               | 32                       | 192                   |
| G                  | 20              | 32                       | 640                   |
|                    |                 |                          | <u>2056 CPU HOURS</u> |

If we assume a rate for CPU time on a commercial VAX 11/750 is at least \$60/CPU Hour, the total computer charge would be at least \$123,360.00. There are also a significant amount of manhours involved in preparing the input data. Analyses and evaluation of the resultant data would also require at least 1000 manhours.

Even if the number of runs could be reduced by half, a full set of validation runs for SSX requires a surprisingly large amount of effort and computational resources.

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NAVAL ARCHITECTS  
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A TIME DOMAIN SIMULATION  
METHOD FOR SHIP MOTION  
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VOLUME II

USER'S MANUAL FOR  
PROGRAMS HYDREX2, HYDREX3 AND SSX

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## 1. INTRODUCTION

Program SSC performs a time-domain simulation of ship motions, hull girder loads and distributed pressures. Two preprocessing programs - HYDREX2 and HYDREX3 - perform certain calculations in preparation for a simulation. Program HYDREX2 computes two-dimensional sectional hydrodynamic coefficients and HYDREX3 calculates several arrays associated with the three-dimensional source representation of the hull. Program input includes a description of the ship's hull by sectional offsets, representation of the hull by quadrilateral or triangular panels, ship geometric and inertial particulars, operational and environmental specifications, and simulation parameters and job options. Output includes a time history plus statistical summaries of heave, pitch, sway, roll, yaw, vertical and horizontal shear and bending moments, torsional moments and dynamic pressures at centers of hull surface panels. The present version of the program contains several features to facilitate experiments with alternative techniques. The program SSX can be run in a batch or interactive mode. The three programs are written in FORTRAN 77 and developed on a DEC VAX 11/750 computer.

This volume (Volume II) provides instructions to the user for performing computations with SSX, HYDREX2 and HYDREX3. A technical description of the model formulation and solution procedures are provided in Volume I.

Program structure and computational overview is first given in Section 2. Program software specifications are described in Section 3. Run procedures are given in Section 4 with input and output variable descriptions. A sample case in Section 5 illustrates code application. Program listings are contained in Appendix A.

## 2. GENERAL STRUCTURE AND COMPUTATIONAL OVERVIEW

A simplified schematic of the basic flow of information is provided in Figure 1. It shows the major sequence of information processing. Three data files are initially required. The basic information file [BIF] contains basic geometric and mass properties of the ship, operational and environmental parameters, and calculation specifications, simulation parameters and job options. The ship offset file [OFF] gives station offset data. The hull panel file [PAN] contains nodal points and connectivity relationships to represent the hull surface by quadrilateral and/or triangular panels.

Program HYDREX2 uses the [BIF] file and [OFF] file to calculate two-dimensional hydrodynamic added mass and damping coefficients using the Frank Close-Fit method. HYDREX2 places the results of its computation in a two-dimensional coefficient file [COF]. It also provides a printout of its results.

Program HYDREX3 uses the [PAN] file to compute panel center coordinates, areas, normals, source density matrix [E], panel pressure matrix [PP] and forward velocity pressure matrix [PX]. The results are placed in a data file [MAT]. HYDREX3 need not be run if it is anticipated that distributed pressures will not be required.

Program SSX uses the [BIF] data file, the [COF] data file, the [MAT] data file (if a pressure distribution is desired), and the [OFF] data file. SSX output includes an input data summary, time domain results and response statistics.

A simplified flow schematic of the SSX program is given in Figure 2. The main program routine SSX initially asks for the names of the [BIF], [OFF], [COF] and [MAT] data files. Subroutine READIN is called to load the necessary variables and arrays using data read in from the four data files.

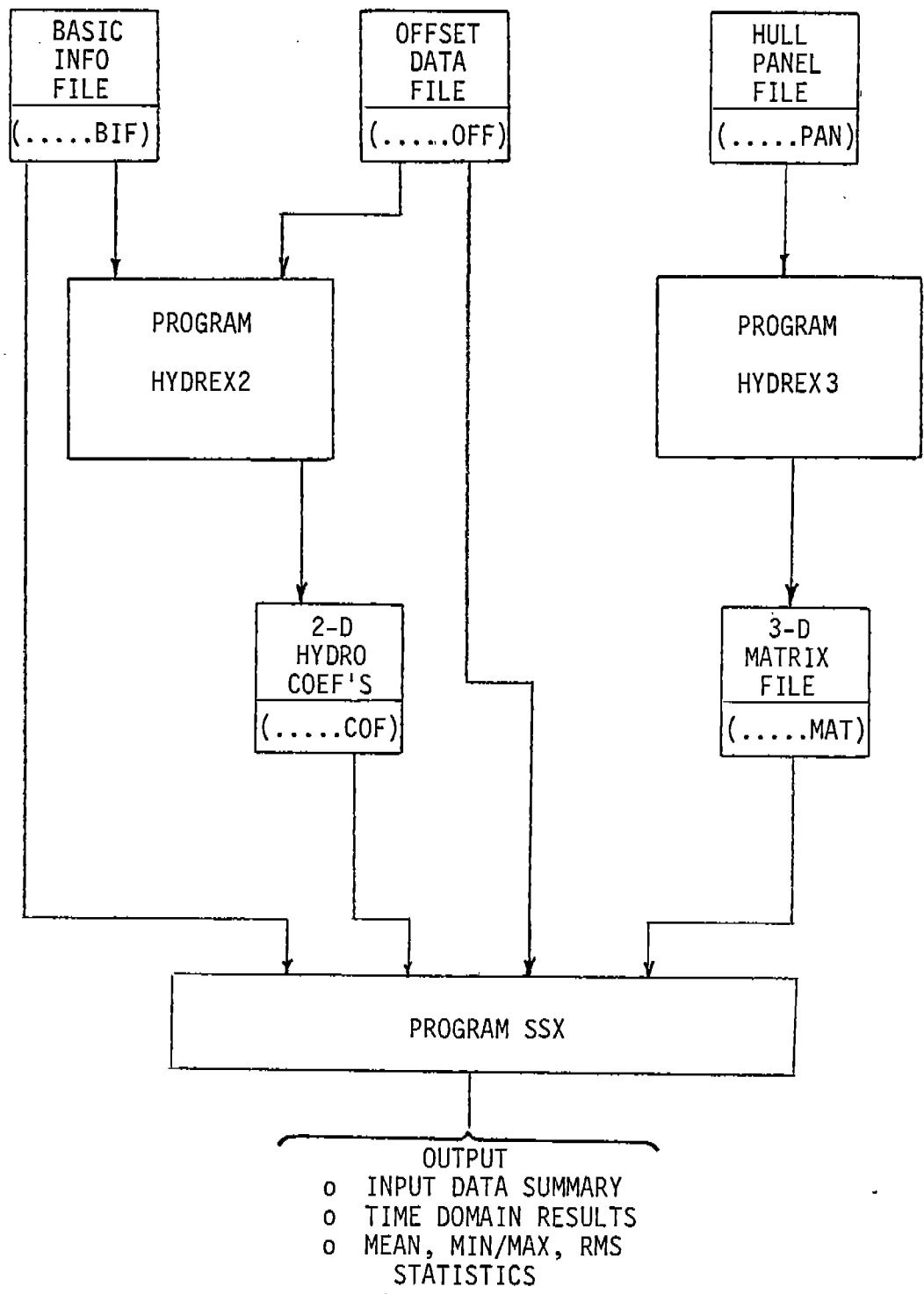


FIGURE 1 - BASIC INFORMATION FLOW



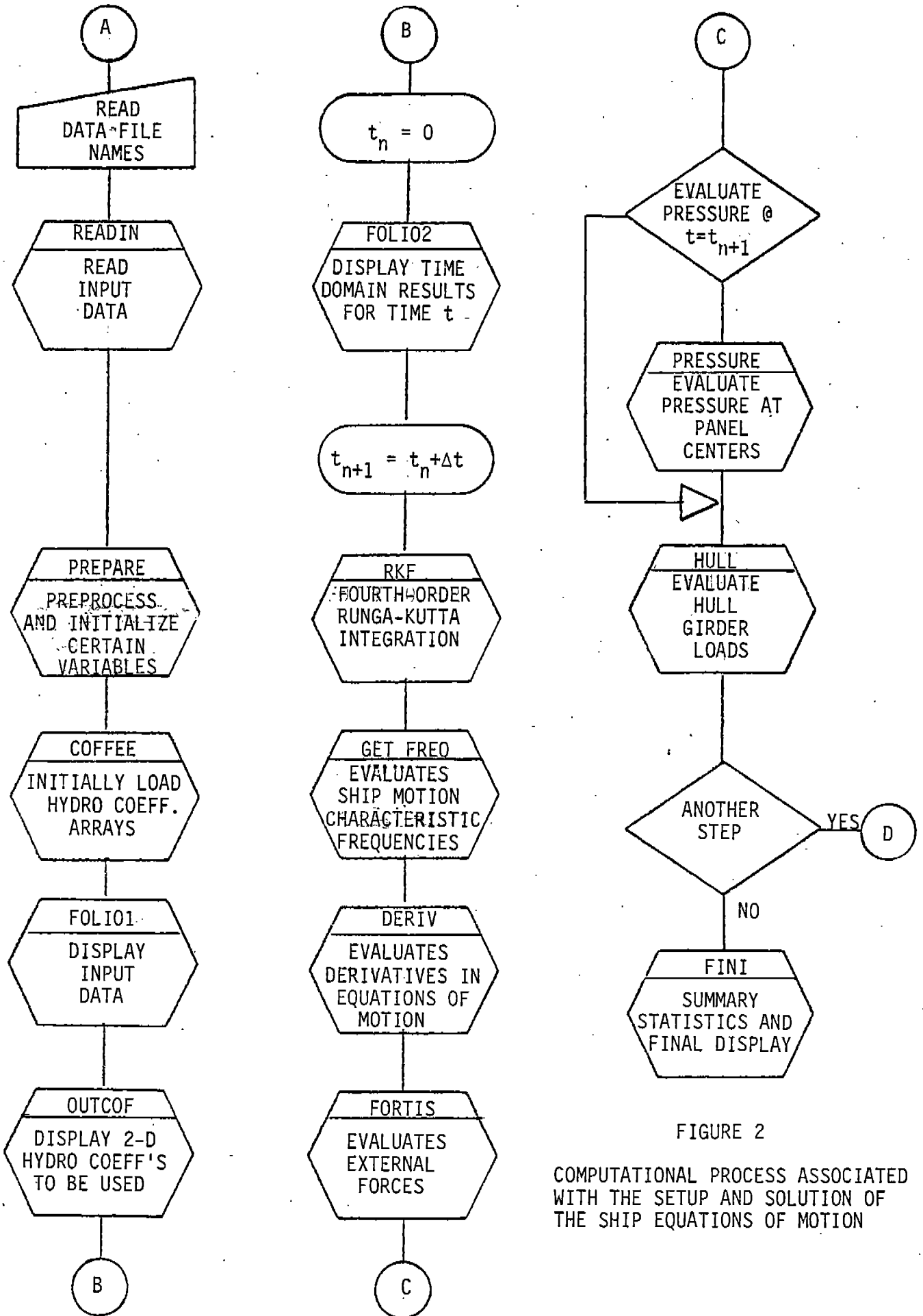


FIGURE 2  
 COMPUTATIONAL PROCESS ASSOCIATED  
 WITH THE SETUP AND SOLUTION OF  
 THE SHIP EQUATIONS OF MOTION

At this point all data has been entered, and subroutine PREPARE is called to initialize certain arrays and perform some computations in preparation for the simulation. Next, subroutine COFFEE is called to load a set of arrays that contain the initial set of two-dimensional hydrodynamic coefficients associated with the mean still water draft, the frequencies of encounter of the wave components and the predicted characteristic frequencies of ship motion.

The program then calls FOLIO1 to display the basic input data and some computed preparatory information. At the option of the user, subroutine OUTCOF can be called to display the initial set of two-dimensional hydrodynamic characteristics.

Now the actual simulation is ready to begin. At each time step, the subroutine FOLIO2 is called to display motion and loads response values for that time. Thus FOLIO2 is initially called at  $t = 0$ . The program then proceeds with the simulation. Within the time step loop in the main program, three subroutines are called - RK4, HULL and PRESSURE. The subroutine RK4 sets up and solves the equations of motions, the subroutine HULL evaluates hull girder loads, and the subroutine PRESSURE evaluates dynamic pressure at the center of each hull surface panel. PRESSURE is only called for a user-selected period of time within the simulation. Both the RK4 and PRESSURE subroutines lead into the large number of subroutines which will be described in the next two paragraphs. At the end of each time step, the time is compared to the user specified stop time. The loop is updated if the stop time has not been reached. If it has been, the simulation stops, and subroutine FINI is called to provide summary statistics.

Figure 3 provides a simplified representation of the computational process associated with the solution of the equations of motion. The subroutine RK4 calls DERIV four times for each time step according to the fourth order Runge-Kutta scheme. The subroutine DERIV sets up the equations of motion in the form of four first order ordinary differential equations. It evaluates the derivatives to be integrated by RK4. The subroutine DERIV calls subroutine FORTIS to evaluate external forces and moments, which include hydrostatic and hydrodynamic forces calculated by subroutine AQUA2D. The

subroutine AQUA2D calls a number of other supporting subroutines and functions. MOCHA evaluates 2-D hydrodynamic coefficients when they are to be a function of sectional draft and/or ship motion characteristic frequencies. KRYLOV evaluates ponential and trigonometric integrals for each section. Functions ETAF and ETABAR evaluate the wave surface elevation.

The computational process associated with hull surface pressure calculations is provided in Figure 4. Subroutine PRESSURE calls ACPTR which computes the pressures and normal acceleration at panel centers induced by the free surface. Next subroutine ZBLACN determines the relative normal acceleration at each panel by subtracting the free surface induced accelerations and incident wave field accelerations from the hull kinematical acceleration. The time derivative of panel source strengths are also evaluated in ZBLACN. Next, subroutine POTB is called to obtain body-induced pressures. The body-induced pressure is added to the free-surface-induced pressure at each panel. The final subroutine is CFSR which advances the free surface representation in preparation for the next time step.

If subroutines to evaluate bottom slamming and green water loads were to be included in later modifications to the program SSX, they should be called from the subroutine AQUA2D. Any force as a function of time, displacement velocity and position on the ship can be evaluated from the information available at any instant within AQUA2D.

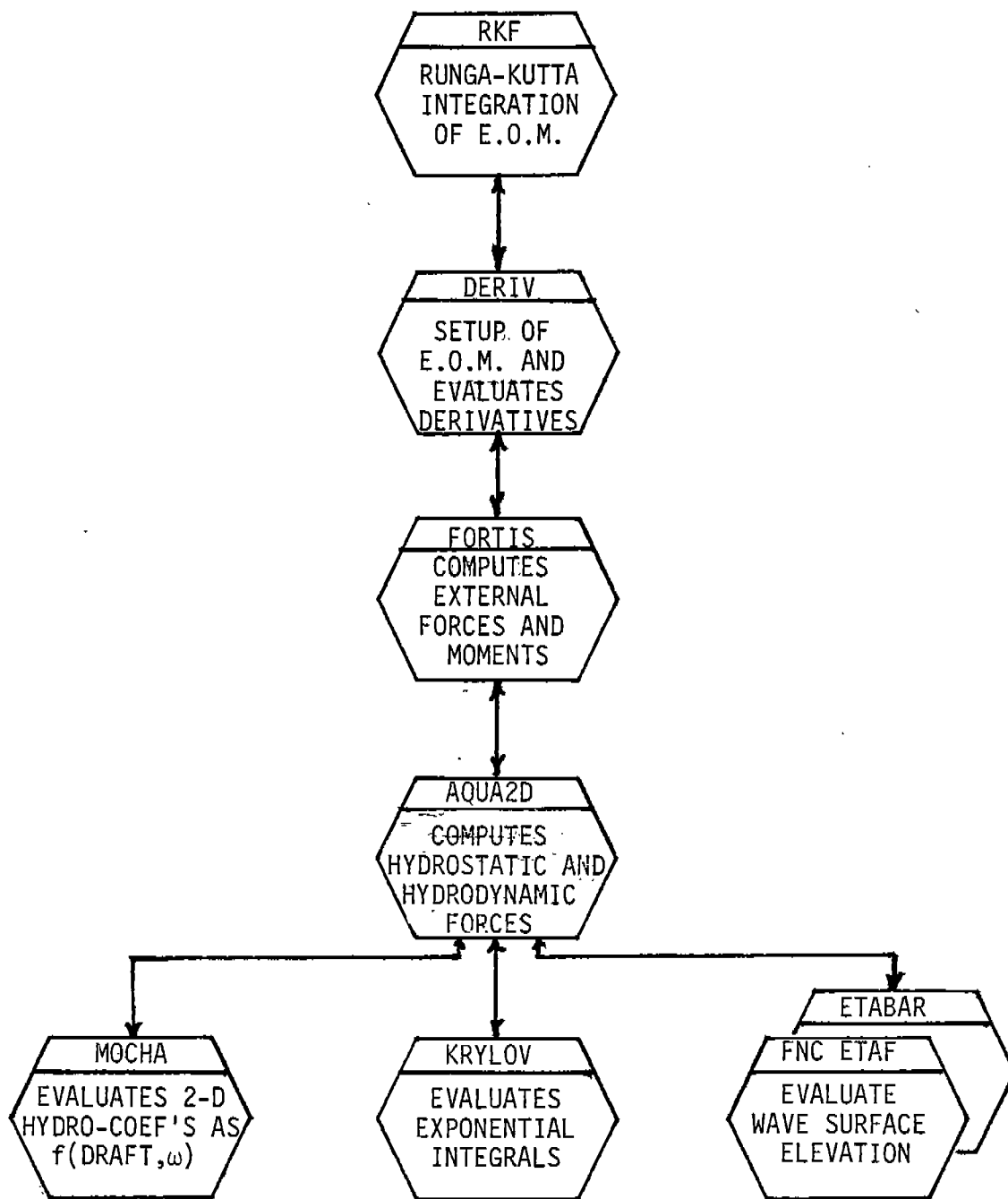


FIGURE 3

COMPUTATIONAL PROCESS ASSOCIATED WITH  
THE SETUP AND SOLUTION OF THE SHIP  
EQUATIONS OF MOTION

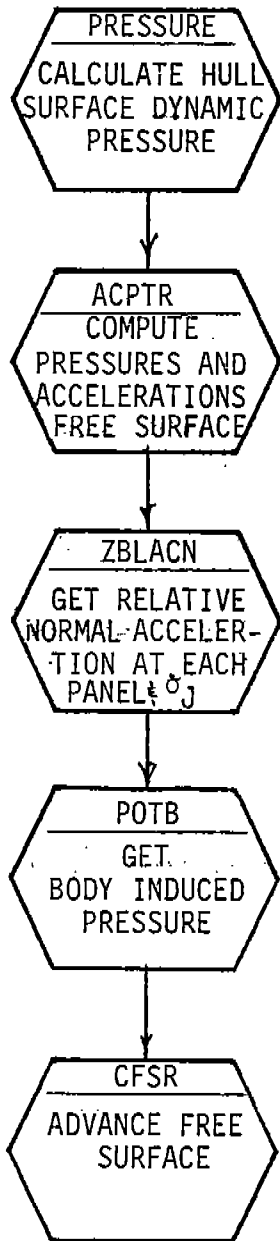


FIGURE 4

COMPUTATIONAL PROCESS ASSOCIATED WITH HULL  
PRESSURE CALCULATIONS

### 3. GENERAL SOFTWARE SPECIFICATIONS

The three codes conform to ANSI X3.9-1978 FORTRAN 77. They were developed on a Digital Equipment Corporation VAX 11/750 computer with a working set size of 250 pages. The only obvious machine depending coding are OPEN statements to open and assign files.

Rather than completely developing new codes, two previously developed and tested codes were used as a basis for the present three codes. The CAPSIZE program written at the University of California, Berkeley by Paul Wood was used as a starting point. The basic variable assignments used in that program were retained, as was the basic computational scheme for the equations of motion. For the three dimensional pressure computations, a program developed by R.B. Chapman entitled NSUP was used as a starting point. Some of the subroutines contained in NSUP and CAPSIZE were adopted directly, while others were completely rewritten. A number of new subroutines were written as well.

## SSX Subroutines

The following is a list of subroutines and functions used in the SSX. Short descriptions are given here and the more complicated subroutines should be sufficiently well documented internally to clearly illustrate the computational process.

- MAIN                    The purpose of the main program is to accept data file names, open the appropriate files, read input data, call several subroutines to prepare data for the simulation, and then execute the time step loop which calls the several subroutines which perform the simulation of motion, hull girder loads and hull dynamic pressures. Calls READIN, SETUP, PREPARE, COFFEE, FOLIO1, OUTCOF, FOLIO2, RK4, HULL, PRESSURE and FINI.
- READIN                 Reads data from the basic information file [BIF], the ship offset file [OFF], the sectional hydrodynamic coefficient file [COF] and the panel pressure matrices file [MAT]. Prints an echo of data read from [BIF]. Called from MAIN.
- PREPARE                Performs computations in preparation for simulation, including (a) from given offsets on the port side, assigns symmetrical offsets on starboard side, (b) recomputes offset coordinates relative to center of mass, (c) consolidates fore and aft profile coordinates into a single pair of arrays, (d) loads certain sectional mass arrays and generalized mass matrix, (e) computes wave numbers, (f) converts initial data in degrees to radians, (g) converts initial linear velocities from ship coordinates to fixed coordinates, and (h) divides all sectional hydrodynamic coefficients by fluid density, (i) initializes special free surface representation for pressure calculations. Calls AFSR. Called from MAIN.

ROTATE                Sets up coordinate system rotation matrices. Called from PREPARE, FORTIS.

FOLI01                Displays ship motion response and hull girder loading response values for each time step. Also accumulates data for post simulation statistical analysis. Called from MAIN.

DERIV                 Evaluates the right hand side of the equations of motion. These values are the ones integrated by the integration scheme. Causes external forces to be calculated, combined with inertial properties to yield translational and angular momentum in the appropriate coordinate systems for solution using Newton's second law. Called from RK4. Calls FORTIS.

FINI                  Prints final execution and response value statistics - including mean, rms, maximum and minimum.

RK4                    Fourth-order Runge-Kutta integration subroutine. Integrates an array YYDOT containing rotational and translational velocities and accelerations to yield array YY which contains new displacements and velocities. Called from MAIN. Calls DERIV.

ERROR                 Subroutine to display a limited amount of diagnostics in the event of a runtime error, principally due to data input errors.

KRYLOV                Evaluates two-dimensional exponential and trigonometric integrals over each station of the ship. Called from AQUA2D.

COFFEE                Performs two-dimensional linear interpolation in the table of sectional hydrodynamic coefficients. Coefficients are selected for the mean design draft, and at encounter



frequencies associated with each wave component and at "characteristic" frequencies of ship motion initially assumed to be the modal frequency of the sea spectrum if irregular waves are imposed.

FXIF Provides an index (IT) and a multiplier (TX) for linear interpolation for draft in the table of sectional hydrodynamic coefficients.

TXIT Provides an index (TX) and a multiplier (IT) for linear interpolation for frequency in the table of sectional hydrodynamic coefficients.

FUNCTION COX Performs the two-way linear interpolation in the table of sectional hydrodynamic coefficients using IF, FX, IT, TX obtained through FXIF and TXIT.

FUNCTION COXDXP Same as COXDX except it uses sectional hydrodynamic coefficients that have not been divided by sectional area, and determines the x-spatial derivative.

FUNCTION ETABAR Provides the wave elevation in fixed coordinate system above a given point of the ship's centerline along hull, stern and transom.

FUNCTION ETAF Provides the coordinate of the water surface in yawed and pitched coordinates given a section number, distance off the centerline and relative displacement of the section.

GETFREQ Evaluates the average of the two most recent zero crossing periods and calculates characteristic frequencies of ship motion in heave, sway and roll. Called from RK4.

OUTCOF Prints the temporary sectional hydrodynamic coefficients currently in arrays in COMMON/COEFFX/. Normally done only at the beginning of the simulation. Called from MAIN.

FOLI01            Displays or prints a summary of basic input data. Called from MAIN.

FOLI02            Displays or prints ship motion and hull girder response. Called from MAIN.

FOLI03            Displays or prints ship hull pressure response. Called from MAIN.

TBAR              Provides the average depth of immersion of a given section.

SPECTRA           Provides the amplitude and frequency of ten wave components derived from a two parameter Bretschneider spectrum. Called from SETUP.

PRESSURE          Provides values of dynamic pressure at the center of each hull panel for a particular instant of time. Called from MAIN. Calls ACPTR, ZBLACN, POTB and CFSR.

ZBLACN            In the pressure computations, applies the specified body accelerations for the six degrees of freedom to compute the resulting normal accelerations at the panel centers. The free-surface induced normal accelerations ACNW(J) are subtracted to obtain the net normal acceleration at each panel center, ACN(J), of the body relative to the fluid. These accelerations must be cancelled by the time derivative of the body source density distribution, ST(K). The ACN(J) vector is multiplied by the E matrix to get the necessary net rate of change of the panel source densities, ST(K). The total source strength densities are accumulated in STOLD(K). Called by PRESSURE.

- ACPTR            In the pressure computations computes free-surface induced accelerations ACNW(J) and pressures PRFS(T) at panel centers. Called by PRESSURE.
- POTB            In the pressure computations this subroutine is used to compute the generalized body-induced pressure BPRES generated by a known source strength distribution and its time derivative. The matrix P(J,K) is multiplied by a vector, ST(K), representing the time derivative of source densities of the panels. Similarly the term proportional to forward speed is computed from the matrix PX and a vector representing the accumulated source densities, STOLD(K). Called by PRESSURE.
- CFSR            In the pressure computations advances free-surface by one time increment. Moves body relative to the free surface. Adds the changes in free-surface elevation induced by the body sources acting over the time increment to the free-surface representation. Second order effects in time are included.
- AFSR            In the pressure computations sets up and initializes the free-surface representation called by PREPARE.

#### HYDREX2 Subroutines

The following is a list of subroutines in HYDREX2.

- MAIN            Accepts data file names, opens the appropriate files, reads input data, loops over six drafts, calling FLOAT and HYDRO within the loops. Also calls INDATA to read in data from files.

FLOAT "Floats" the ship at the specified draft. Creates a set of "wet" offsets to be used for hydrostatic calculations and for the computation of sectional hydrodynamic coefficients.

HYDRO Applies the Frank Close-fit method to "wet" offsets. Adopted from CAPSIZE which adopted it with some modifications from the original NSRDC code. HYDRO calls GIRL, BEER, WINE and STATN.

STATN Revises the station offsets by inserting additional points in order to optimize the hydrodynamic computations and to suppress anomolous behaviour which sometimes occurs at certain "singular frequencies". INSERT is called by STATN as part of this process.

GIRL Computes certain frequency-independent coefficients which are used in the two-dimensional hydrodynamic computations.

BEER Computes the two-dimensional hydrodynamic coefficients for the special cases of zero or infinite frequency. It calls the simultaneous linear equation solver LINEQT.

WINE Performs the computation of the two-dimensional hydrodynamic coefficients for finite nonzero frequencies, calling several subroutines for special operations. WOMEN computes some of the interaction between segments of the section. SONG performs the integration of pressures around the section. ROMEO evaluates the exponential integral with complex argument. JULIET is a simultaneous linear equation solver for certain sets of equations in HYDRO.

## HYDREX3 Subroutines

The following routines are contained in the program HYDREX3.

- MAIN                    Reads input data and calls subroutines EBD and POTST.
- EBD                    This subroutine reads in the (x,y,z) coordinates of each of the four corner points into a set of arrays XPT(N), YPT(N), ZPT(N). Panels are identified by a set of four integers giving the array positions of the four corner points of each panel. Panel areas, normals and center point coordinates are then computed. Finally, the E matrix giving the source time derivative distribution for a set of prescribed normal accelerations is computed. The inverse of E is computed first by the subroutine GE which gives the acceleration induced at any panel center point, J, by a uniformly distributed time derivative of source strength density of unit magnitude acting over any surface of panel, JL. Subroutine MATIN inverts E to obtain the desired form. Simultaneously, the matrix PX which gives the x component of velocity at the center of panel J induced by a source strength of unit magnitude distributed over a panel, JL, is computed.
- POTST                  Calculates the matrix P(N,J) giving the net force or moment for the Jth degree of freedom induced by a unit time derivative of source strength over panel N. Fundamental to this is the need to compute the potential integrated over each panel area due to a uniform source density over every other panel. For panels which are far apart relative to their dimensions this value is, for unit source density, simply proportional to the product of their areas divided by the distance between centers. The method used here is to divide each panel into a large number of small subpanels and then calculate the results numerically, adding the contributions of each subpanel

under the assumption that their separations are large relative to their dimensions.

SELF This subroutine is called POTST to compute diagonal terms in the P matrix.

MATIN The matrix inversion routine used by subroutine EBD to invert matrix E.

GE A subroutine called by EBD to compute the elements of matrix E prior to inversion. It computes the velocity (acceleration) at field point (XF, YF, ZF) induced by a source density (time rate of change of source strength) of value unity distributed uniformly over panel J.

GO A subroutine called by GE.

SOLID Also called by subroutine GE to compute the solid angle of a panel relative to the field point.

PREP Prepares all panels for the GE subroutine. It is called by EDB prior to using GE.

#### 4. RUN PROCEDURES

This section presents the procedures involved with running HYDREX2, HYDREX3 and SSX. It also gives input data description and formats.

##### 4.1 PROGRAM HYDREX2

HYDREX2 calculates two-dimensional sectional added mass and damping coefficients using the Frank Close-fit method. Two data files are required - a [BIF] file which contains basic ship and program run information and the [OFF] file which contains the ship's offsets. The program will create a third file [COF] into which it will write the results of its calculations.

When HYDREX2 is run, the only required input is the following three files names:

| <u>File Name</u>     | <u>Format</u> |
|----------------------|---------------|
| Basic Input File     | (A)*          |
| Ship Offset File     | (A)           |
| 2-D Coefficient File | (A)           |

Descriptions of the [BIF] and [OFF] file are given in the next two sections.

##### Basic Information File [BIF]

The basic information file is used for programs HYDREX3 and SSX as well as HYDREX2. The full [BIF] file will contain more information than that which is described below. However, to run HYDREX2, only the first part of the [BIF] file need be created. The remainder can be created at some other time prior to running SSX. Table 1 below summarizes the input data variables and the associated format.

---

\* (A) format is alphanumeric string variable

TABLE 1

| BASIC INFORMATION FILE (PARTIAL) |                              |             |
|----------------------------------|------------------------------|-------------|
| Data Set No.                     | Variables                    | Line Format |
| 1                                | TITLE                        | (A)         |
| 2                                | TF, TA                       | 6F10.0      |
| 3                                | XCG, YCG                     | 6F10.0      |
| 4                                | (DRAFT (I), I=1,6)           | 6F10.0      |
| 5                                | (OM(I), I=1,12)              | 6F10.0      |
| 6                                | YMAX, ZMAX, WMAX, NWL        | 3F10.0, I15 |
| 7                                | NFWD                         | I5          |
| 8                                | (YFWD(I), XFWD(I), I=1,NFWD) | 2F10.0      |
| 9                                | NAFT                         | I5          |
| 10                               | (YAFT(I), XAFT(I), I=1,NAFT) | 2F10.0      |
| .                                |                              |             |
| .                                |                              |             |

A description of the variables contained in Table 1 is provided below:

- TITLE - Any user-specified alphanumeric string up to 81 characters in length. This is displayed at the beginning of certain pages of output.
- XCG - Distance aft of the forward perpendicular of the ship's center of gravity (ft).
- YCG - Height above baseline of ship's center of gravity (ft)
- TF,TA - Ships forward and after draft at FP and AP.
- DRAFT(I) - An array of six drafts at which hydrodynamic coefficients are calculated. The specified drafts should cover a range beginning at approximately the height above the baseline of the top of the turn of the bilge and ending at the minimum hull depth. One of the drafts should be near the mean design draft for the displacement under consideration.
- OM(I) - An array of twelve frequencies at which hydrodynamic coefficients are calculated.



- YMAX,  
ZMAX - Specifies the maximum desired vertical (YMAX) and horizontal separation between adjacent offset points for calculation of two-dimensional hydrodynamic coefficients. If both YMAX and ZMAX are positive values, interpolated offset points (straight line) will be added before computing coefficients using the method developed by W. Frank (1967).
- WMAX - A "deck" on the interior waterline has been added to the geometry of each station which is surface piercing to avoid "irregular" frequencies. The default is to use only one segment for this, but a positive WMAX will allow multiple segments each with a maximum length of WMAX.
- NWL - Specifies the number of interior waterline segments to be used to avoid "irregular" frequencies. The default is to use one segment, and this is selected if NWL is zero. A negative value (not recommended) will suppress the modification to Frank's procedure which eliminates irregular frequencies.
- NFWD - Number of forward profile points. This is restricted to  $0 < NFWD < 25$ . These points are numbered by the control variable I in a counterclockwise direction when viewed from the starboard side.
- YFWD(I) - Height of point I of the forward profile measured forward of the first station.
- XFWD(I) - Distance of point I of the forward profile measured forward of the first station. If NFWD=1, XFWD(1) is defined to be the forward most point of the submerged hull.
- NAFT - Number of after profile points. This is restricted to  $0 < NAFT < 25$ . These points are numbered by the control variable I in a clockwise direction when viewed from the starboard side.
- YAFT(I) - Height of point I of the after profile.
- XAFT(I) - Distance of point I of the after profile measured aft of the last (MSTA) station. If NAFT=1, XAFT(1) is defined to be the after most point of the submerged hull.

The offset file [OFF] contains the ship's offsets in the SHCP\* format. An actual SHCP data can be used for the [OFF] file. HYDREX2 will read only what it needs. Table 2 below summarizes the input data variables and associated formats.

TABLE 2

| SHIP OFFSET FILE |                                      |                   |
|------------------|--------------------------------------|-------------------|
| Data Set Number  | Variables                            | Line Format       |
| 1                | SHCP Card A <u>or</u> (blank)        | (A)               |
| 2                | SHCP Card B <u>or</u> (blank)        | (A)               |
| 3                | SHCP Card C <u>or</u> space          | F10.3, (A)        |
| 4                | SHCP Card D - STATN, Y1<br>Z1, JTEST | F6.3,<br>2F7.0,I6 |

A description of the variables contained in Table 2 is provided below.

SHCP CARD A - This can be left blank, or if it is an actual SHCP data file, leave it intact.

SHCP CARD B - Blank or actual SHCP Card B

SHCP CARD C or SPACE - This can be the actual SHCP Card C or simply one variable SPACE, where SPACE is the station spacing. It is actually a multiplier of the x-values of the stations given in the next card set.

STATN - The real distance from the station to the F.P. is the product of STATN and SPACE. The STATN values must be the same for all offsets on the same station.

\* Ship Hull Characteristic File (U.S. Navy)

- Y1            -    The half breadth of the offset point. The offsets for each station should be ordered from the bottom toward the uppermost parts of the station.
  
- Z1            -    The height of the offset point.
  
- JTEST        -    The "breakpoint" indicator 77777 is ignored. The last offset on each station is signified by JTEST = 88888. The last offset on the last station is signified by JTEST = 99999. JTEST values other than zero (or blank) 77777, 88888 or 99999 are illegal.

For HYDREX2, the maximum number of offset points per station is 25. However, for SSX, the maximum number of offset points is 12. It is recommended that as many offset point (up to 25) as possible are used for HYDREX2 to increase the accuracy of the close-fit calculations. However, prior to running SSX, the [OFF] file must be edited to delete extra offset points in order to reduce the total number per station down to 12. The number of stations must be limited to 25.

An example of the actual input and output of HYDREX2 will be provided in Section 5 with a sample case illustration.

#### 4.2        PROGRAM HYDREX3

HYDREX3 calculates coordinates of hull surface panel center, areas and normal vectors of panels, body source density influence matrix [E], a body panel pressure influence matrix [PP], and a forward velocity source density influence matrix [PX].

When HYDREX3 is run, the only required input is the following two file names:

| <u>File Name</u>               | <u>Format</u> |
|--------------------------------|---------------|
| Panel Description File [PAN]   | (A)           |
| Matrix and Geometry File [MAT] | (A)           |

Table 3 summarizes the input variables and formats for the data for the panel description file [PAN].

TABLE 3

| PANEL DESCRIPTION FILE |  |        |
|------------------------|--|--------|
| Data Set               | Variables  | Format |
| 1                      | NPT, NPAN  | 4I5    |
| 2                      | XPT(N), YPT(N), ZPT(N)<br>N=1, NPT               | 3F10.0 |
| 3                      | KK(M,1), KK(M,2), KK(M,3),<br>KK(M,4), M=1, NPAN | 4I5    |

The variables and their use are further described below.

NPT - Number of nodal points used to specify coordinates of panel corners. Not to exceed 150.

NPAN - Number of quadrilateral or triangular panels. Not to exceed 120.

XPT(N)  
YPT(N)  
ZPT(N) - The coordinates of the nodal points used as corner points for the panels. The coordinate system used here has Z positive downwards, X positive forward and Y positive starboard. The origin is at midships at the mean design water line.

KK(M,1) - These are four integers which identify which nodal points are used to define the corners of the panels. The convention for the sequencing of the aft panel corners is to go around the panel clockwise when viewed from outside of the ship hull along the panel normal.

### 4.3 PROGRAM SSX

Program SSX performs a time domain simulation of ship motions, calculating hull girder loads and distributed pressures (optional). A prerequisite of executing SSX is to have run HYDREX2 and HYDREX3 in order to perform some "preprocessing". SSX requires four data files.

- (1) Basic Information Data File [BIF]
- (2) Ship Offset Data File [OFF]
- (3) 2-D Hydrodynamic Coefficient Data File [COF]
- (4) 3-D Geometry/Matrix File [MAT]

The program initially asks for the names of these four files in the above order. It then reads the data. If the [MAT] file name is 'NONE', then no pressure computations will be performed, and data sets 21 and 22 in the [BIF] file are skipped. The first eight data groups in the [BIF] file were given in Table 1. The variable and format list for the remainder of the [BIF] file is given in Table 4.

TABLE 4

| BASIC INFORMATION FILE (PARTIAL) |  |        |
|----------------------------------|--|--------|
| Data Set                         | Variables  | Format |
| 9                                | DISPL  | 8F10.0 |
| 10                               | (RADII(I), I=1,6)  | 8F10.0 |
| 11                               | SPEED  | 8F10.0 |
| 12                               | (DAMPL(I), I=1,6)  | 8F10.0 |
| 13                               | (DAMPQ(I), I=1,6)  | 8F10.0 |
| 14                               | IXWAVE<br>IXWAVE = 1 → Include Set 14A<br>IXWAVE ≠ 1 → Include Set 14B | I2     |
| 14A-1                            | NWAVES   | 4F10.0 |
| 14A-2                            | WVAMP(N), WVFRE(N), WVDIR(N),<br>WVPHA(N), N=1, NWAVES                 | 4F10.0 |
| 14B-1                            | H13, PKFRE, HEAD   | 8F10.0 |
| 15                               | (POSIT(I), I=1,6)  | 8F10.0 |
| 16                               | (VELOC(I), I=1,6)  | 8F10.0 |
| 17                               | TSTART, TSTOP, TOUTPT, TSTEP   | 8F10.0 |
| 18A                              | (JOBFO(10), I=1,10)  | 10I1   |
| 18B                              | (JOBPO(10), I=1,10)  | 10I1   |
| ⋮                                |  |        |

TABLE 4 (Cont.)

| BASIC INFORMATION FILE (PARTIAL) |   |        |
|----------------------------------|---|--------|
| Data Set                         | Variables                                   | Format |
| 18C                              | JOBCO                                       | I1     |
| 19                               | NWTSTA                                      | I2     |
| 20                               | SEGWT(I), SEGMOX(I), YBAR(I)<br>I=1, NWTSTA | 3F10.0 |
| 21                               | TPSTART, TPSTOP, TPRAMP                     | 8F10.0 |
| 22                               | BGX, SMX, BGY, SMY, TSCALE                  | 8F10.0 |

The variables and their use are further described below.

DISPL - Ship displacement (long tons). It is best to enter the displacement calculated in HYDREX2 for the design draft and printed in the hydrostatic section of the HYDREX2 printout.

RADII(1) - Radius of gyration for roll,  $\rho_{xx}$ .

RADII(2) - Radius of gyration for yaw,  $\rho_{yy}$ .

RADII(3) - Radius of gyration for pitch,  $\rho_{zz}$ .

RADII(4) - Radius of gyration,  $\rho_{xy}$ . The products of inertia are computed as:

$$I_{xy} = I_{yx} = \rho_{xy} \cdot |\rho_{xy}| \cdot m$$

where  $m$  is the mass of the ship

RADII(5) - Radius of gyration,  $\rho_{xz}$

RADII(6) - Radius of gyration,  $\rho_{yz}$

SPEED - Intended speed of the ship (feet/sec)

DAMPL(1) - Linear surge damping

DAMPL(2) - Linear heave damping

DAMPL(3) - Linear sway damping

DAMPL(4) - Linear roll damping

DAMPL(5) - Linear yaw damping

DAMPL(6) - Linear pitch damping

DAMPQ(1) - Quadratic surge damping

DAMPQ(2) - Quadratic heave damping

DAMPQ(3) - Quadratic sway damping

DAMPQ(4) - Quadratic roll damping

DAMPQ(5) - Quadratic yaw damping

DAMPQ(6) - Quadratic pitch damping



- IXWAVE - Flag to indicate how the user wants to specify the wave system. IXWAVE=1 means the user will specify the amplitude, frequency, direction and phase angle of N WAVE wave components. IXWAVE  $\neq$  1 means the user will specify the significant height and modal frequency to be used for a two-parameter (Bretschneider) unidirectional sea spectral formulation from which 10 wave components will be calculated with random phase angles.
  
- NWAVES - Number of sinusoidal waves. This is restricted to  $0 < \text{NWAVES} < 20$ .
  
- WVAMP(I) - Amplitude of wave component I (ft)
  
- WVFRE(I) - Circular frequency of wave component I
  
- WVPHA(I) - Phase angle in degrees at time equal zero of wave component
  
- H13 - Significant wave height in feet to be used in two-parameter Bretschneider unidirectional sea spectral formulation
  
- PKFRE - Modal frequency in cycles/sec for Bretschneider formulation
  
- POSIT(1) - Initial X-coordinate of mass center
  
- POSIT(2) - Initial Y-coordinate of mass center
  
- POSIT(3) - Initial Z-coordinate of mass center
  
- POSIT(4) - Initial roll angle in degrees
  
- POSIT(5) - Initial yaw angle in degrees
  
- POSIT(6) - Initial pitch angle in degrees
  
- VELOC(1) - Initial speed (ft./sec.)
  
- VELOC(2) - Initial heave velocity

- VELOC(3) - Initial sway velocity
- VELOC(4) - Initial roll rate
- VELOC(5) - Initial yaw rate
- VELOC(6) - Initial pitch rate

The position values are specified with respect to the wave coordinate system fixed on the earth. The velocities are with respect to the ship coordinate system. Positions are in feet; velocities in feet/sec.

- TSTART - Time at which the actual simulation is to start. All forces are multiplied by a ramp function that increases linearly in time from a value of zero at  $t=0$  to one at TSTART. This ramp is used to avoid transients caused by arbitrary initial conditions.
- TSTOP - Time at which the simulation is to end
- TOUTPT - Interval at which the ship position and velocity are to be output.
- TSTEP - Integration time step
- JOBCO - Option for selection of different techniques used to select the sectional hydrodynamic coefficients. The sectional coefficients may be calculated for any combination of fixed or varying draft and frequency dependence, according to the following table:

| <u>JOBCO</u> | <u>Frequency</u> | <u>Draft</u> |
|--------------|------------------|--------------|
| 1            | Fix              | Fix          |
| 2            | Float            | Fix          |
| 3            | Fix              | Float        |
| 4            | Float            | Float        |

The selection of fixed draft means the mean still waterline draft is always used for coefficient selection. Fixed frequency means the encounter frequency associated with the peak of the sea spectrum (for irregular waves) or the frequency of encounter for a regular wave is always used.

Floating frequency means the computed "characteristic" frequencies of motion as the simulation progresses are used. Floating draft means the actual instantaneous mean sectional draft is used at each time step.

- JOBFO(10) - Option array for which terms to include in the calculation of hydrodynamic forces. In the present version only JOBFO(2) and JOBFO(3) are used. If JOBFO(2)=1 then forward speed terms are included. If JOBFO(3)=1 then flare force is included. If JOBFO(4)=1, pressures in the wave above still water are assumed to be hydrostatic.
- JOBPO(10) - Printout option array. Only JOBFO(1) is used in present version. A value of 1 for JOBPO(1) will cause the initially selected values for hydrodynamic coefficients to be printed.
- NWTSTA - Number of weight stations. In present version of SSX, NWTSTA must equal number of hydrodynamic stations or strips.
- SEGWT(I) - Weight of weight station I (long-tons)
- SEGMOX(I) - Sectional roll gyradius (ft)
- YBAR(I) - Distance from section c.g. and waterline, positive up.
- TPZERO - Time at which to begin 3-D hull pressure computations
- TPSTART - Time at which 3-D hull pressure evaluation actually begins. Between time TPZERO and TPSTART, the exciting normal accelerations are multiplied by a ramp function that increases linearly in time from a value of zero at  $t = TPZERO$  to a value of one at  $t = TPSTART$
- TPSTOP - Time at which 3-D hull pressure evaluation ends
- BGX - Maximum longitudinal length scale. Corresponds to  $L_x$  in eq. (72) of Volume I. Try 2.5  $p_{pp}$
- SMX - Minimum longitudinal length scale corresponds to  $l_x$  in eq. (73) of Volume I. Try station spacing of ship

- BGY - Maximum transverse length scale. Corresponds to  $L_y$  in eq. (72) of Volume I. Try 5B
- SMY - Minimum transverse length scale. Correspondence to  $l_y$  in eq. (73) in Volume I. Try 0.1B
- TSCALE - Maximum time scale. Trade off between accuracy and computational effort. Try 0.6 (TPSTOP-TPZERO)

An example of actual input and output is shown in the next section with a sample run.

## 5. SAMPLE CASE

This example is for the SL-7 containership. An example of the [BIF], [OFF] and [PAN] data files are given. The [BIF] file is in free format. An example of the output for HYDREX2, HYDREX3 and SSX is given.

The sample case shows the full load SL-7 containership at 25 knots in head seas in a one-foot regular wave of  $\omega = 0.34$ . The basic information file [BIF] is shown in Figure 5. The offset [OFF] file is provided as Figure 6 and the panel file [PAN] is given as Figure 7. An example of the output of HYDREX2 is shown in Figure 8. An example of output of HYDREX3 is given in Figure 9. An example of output of SSX is given in Figure 10. Pressure data output is shown in Figure 11.

```

SL7 CONTAINERSHIP
328.328
-478.0542.3005
5.10.20.32.8.40.50.
0.0.60.2
9
0.0.
1.59.6
8.25.16.88
18.0.7.04
30.0.
42.1.92
56.7.92
72.17.12
81.75.23.44
6
0.-71.12
6.0.-67.68
10.75.-19.44
23.0.-19.44
64.83.17.92
47760.
47.219.219.0.0.0.
16.89
0.0.0.0.0.0.
0.0.0.0.0.0.
1
1
1.2.180.0.0.0.
0.9.418.0.0.0.0.
16.89.0.0.0.0.0.
30.100.5.5
25
094.5500.37.31.9.5145
294.5500.37.31.9.5145
296.2025.37.31.9.5145
496.2025.37.31.9.5145
1143.8223.37.31.9.5145
1134.9353.37.31.9.5145
1681.7071.37.31.9.5145
2040.0422.37.31.9.5145
2508.4206.37.31.9.5145
2578.8111.37.31.9.5145
3053.3377.37.31.9.5145
3591.3250.37.31.9.5145
3530.8231.37.31.9.5145
3175.7759.37.31.9.5145
3725.5737.37.31.9.5145
3530.6277.37.31.9.5145
2595.3904.37.31.9.5145
3151.4265.37.31.9.5145
2534.0626.37.31.9.5145
1761.6687.37.31.9.5145
1715.7387.37.31.9.5145
715.4792.37.31.9.5145
747.0264.37.31.9.5145
747.0264.37.31.9.5145
0.0.0.0.0.

```

Figure 5. Basic Information File [BIF]







Station 12

Dist. from F.P. -396.23  
 DRAFT (fwd) 32.80  
 DRAFT (aft) 32.80

Area 3031.365  
 Roll Ctr abv WL 0.088

| F eq. | -----HEAVE----- |        | -----SWAY----- |        | -----ROLL----- |        | ---SWAY-ROLL--- |      |
|-------|-----------------|--------|----------------|--------|----------------|--------|-----------------|------|
|       | A22             | B22    | A33            | B33    | A44            | B44    | A34             | B34  |
| 0.00  | 99.0000         | 0.0000 | 2.2896         | 0.0000 | 949.04         | 0.00   | 34.4            | 0.0  |
| 0.10  | 11.6174         | 0.8778 | 1.9201         | 0.0002 | 547.86         | 0.03   | 20.1            | 0.0  |
| 0.20  | 7.2671          | 1.4760 | 2.0412         | 0.0072 | 561.35         | 1.07   | 21.3            | 0.1  |
| 0.30  | 5.1586          | 1.8017 | 2.2434         | 0.0544 | 581.59         | 7.57   | 23.3            | 0.6  |
| 0.40  | 3.9867          | 1.9286 | 2.4304         | 0.2150 | 593.28         | 27.55  | 24.8            | 2.4  |
| 0.50  | 3.3206          | 1.9089 | 2.3837         | 0.5407 | 571.59         | 62.45  | 23.5            | 5.8  |
| 0.60  | 2.9645          | 1.7834 | 1.9918         | 0.9268 | 515.09         | 94.63  | 18.7            | 9.4  |
| 0.70  | 2.8127          | 1.5879 | 1.4567         | 1.1956 | 456.55         | 106.05 | 13.0            | 11.3 |
| 0.90  | 2.8677          | 1.1182 | 0.6952         | 1.3047 | 401.19         | 83.61  | 6.3             | 10.4 |
| 1.30  | 3.3945          | 0.4073 | 0.2841         | 0.9812 | 408.22         | 29.37  | 4.9             | 5.4  |
| 1.60  | 3.6990          | 0.1707 | 0.2848         | 0.7194 | 423.20         | 11.92  | 6.0             | 2.9  |
| 2.20  | 7.4603          | 0.0000 | 0.7945         | 0.0000 | 748.80         | 0.00   | 15.4            | 0.0  |

-----STATION OFFSETS-----

| Height above<br>Baseline | Half<br>BREADth | Submerged Offsets |        |
|--------------------------|-----------------|-------------------|--------|
|                          |                 | -Y-               | -Z-    |
| 0.000                    | 0.000           | -32.800           | 0.000  |
| 0.591                    | 26.303          | -32.209           | 26.303 |
| 0.820                    | 30.060          | -31.980           | 30.060 |
| 1.640                    | 33.962          | -31.160           | 33.962 |
| 3.281                    | 38.298          | -29.519           | 38.298 |
| 4.921                    | 40.610          | -27.879           | 40.610 |
| 6.562                    | 42.778          | -26.238           | 42.778 |
| 8.202                    | 44.310          | -24.598           | 44.310 |
| 9.842                    | 45.524          | -22.958           | 45.524 |
| 11.483                   | 46.478          | -21.317           | 46.478 |
| 13.123                   | 47.345          | -19.677           | 47.345 |
| 16.404                   | 48.703          | -16.396           | 48.703 |
| 19.685                   | 49.715          | -13.115           | 49.715 |
| 26.247                   | 50.929          | -6.553            | 50.929 |
| 29.528                   | 51.305          | -3.272            | 51.305 |
|                          |                 | WATERLINE         | 51.593 |
| 32.808                   | 51.594          |                   |        |
| 39.370                   | 52.027          |                   |        |
| 52.493                   | 52.605          |                   |        |
| 64.304                   | 52.750          |                   |        |

Figure 8. Output of HYDREX2  
 Partial Listing

|    | NX      | NY      | NZ      | XP        | YP       | ZP      | AREA      |
|----|---------|---------|---------|-----------|----------|---------|-----------|
| 1  | 0.0000  | 0.0000  | 1.0000  | 390.0000  | 0.5867   | 30.0000 | 79.2000   |
| 2  | 0.0176  | 0.5404  | 0.8412  | 405.0000  | 3.2150   | 28.5000 | 499.6292  |
| 3  | 0.0264  | 0.7592  | 0.6504  | 405.0000  | 6.8350   | 25.5000 | 355.6513  |
| 4  | 0.0301  | 0.9964  | -0.0797 | 405.0000  | 7.8800   | 21.0000 | 541.9710  |
| 5  | 0.0394  | 0.9341  | -0.3549 | 405.0000  | 6.5000   | 15.0000 | 578.1236  |
| 6  | 0.0525  | 0.9600  | -0.2752 | 405.0000  | 4.5000   | 9.0000  | 562.5251  |
| 7  | 0.0698  | 0.9974  | -0.0166 | 405.0000  | 3.6200   | 4.8000  | 216.5585  |
| 8  | 0.0000  | 0.0000  | 1.0000  | 325.0000  | 1.3000   | 30.0000 | 182.0000  |
| 9  | 0.0120  | 0.4445  | 0.8957  | 325.0000  | 5.6225   | 28.5000 | 472.4277  |
| 10 | 0.0283  | 0.8527  | 0.5216  | 325.0000  | 9.5625   | 25.5000 | 246.2683  |
| 11 | 0.0365  | 0.9992  | 0.0133  | 336.6667  | 10.0800  | 22.0000 | 210.1590  |
| 12 | 0.0694  | 0.9962  | 0.0531  | 325.0000  | 10.7600  | 16.5000 | 632.4223  |
| 13 | 0.1151  | 0.9722  | 0.2037  | 325.0000  | 11.8800  | 7.8000  | 604.7863  |
| 14 | 0.0000  | 0.0000  | 1.0000  | 257.5000  | 2.1200   | 30.0000 | 275.6000  |
| 15 | 0.0158  | 0.3772  | 0.9260  | 257.5000  | 7.9225   | 28.5000 | 516.9807  |
| 16 | 0.0510  | 0.7405  | 0.6701  | 257.5000  | 12.9625  | 25.5000 | 263.3413  |
| 17 |         | 0.9135  | 0.3928  | 257.5000  | 16.9000  | 18.0000 | 853.8872  |
| 18 |         | 0.9205  | 0.3594  | 257.5000  | 21.8225  | 6.0000  | 847.3912  |
| 19 | 0.0000  | 0.0138  | 0.9999  | 180.0000  | 4.5400   | 29.9375 | 817.2777  |
| 20 | 0.0000  | 0.0000  | 0.9424  | 195.0000  | 10.5600  | 28.9167 | 404.9271  |
| 21 | 0.0125  |         | 0.9458  | 165.0000  | 18.8533  | 26.9167 | 799.3557  |
| 22 | 0.0532  |         | 0.7425  | 195.0000  | 20.1067  | 25.0000 | 203.6394  |
| 23 | 0.0786  | -0.8111 | 0.7547  | 180.0000  | 27.4450  | 18.0000 | 1312.5892 |
| 24 | 0.1172  | -0.9156 |         | 180.0000  | 34.0325  | 6.0000  | 1184.2212 |
| 25 | 0.1012  | -0.9948 |         | 67.5000   | 9.8800   | 29.8125 | 2668.0806 |
| 26 | 0.0000  | -0.0189 | 0.0000  | 0.0000    | 27.6267  | 28.7500 | 1157.2611 |
| 27 | 0.0000  | -0.1640 | 0.9800  | 0.0000    | 28.8000  | 26.9167 | 1200.2803 |
| 28 | -0.0131 | -0.6156 | 0.7880  |           | 40.1600  | 25.0000 | 329.9435  |
| 29 | 0.0138  | -0.2842 | 0.9587  |           | 11.9650  | 18.0000 | 1871.9629 |
| 30 | 0.0481  | -0.9936 | 0.1027  | -60.0000  | 11.9650  | 6.0000  | 1657.5626 |
| 31 | 0.0062  | -1.0000 | 0.0000  | -60.0000  |          | 29.7500 | 2376.4260 |
| 32 | 0.0000  | -0.0189 | 0.9998  | -120.0000 |          | 28.2500 | 1457.4741 |
| 33 | -0.0257 | -0.1065 | 0.9940  | -150.0000 | -16.0000 | 0.0000  | 219.3015  |
| 34 | -0.0281 | -0.1878 | 0.9818  | -150.0000 | -37.7000 | 0.0000  | 475.0337  |
| 35 | -0.0249 | -0.2841 | 0.9585  | -120.0000 | -45.5600 |         | 1116.7516 |
| 36 | -0.0814 | -0.9296 | 0.3594  | -150.0000 | -47.8533 |         | 70.0344   |
| 37 | -0.0474 | -0.9514 | 0.3044  | -150.0000 | -51.0933 | 8.0000  | 71.130    |
| 38 | 0.0000  | -0.1361 | 0.9907  | -210.0000 | -14.5600 | 28.0000 |           |
| 39 | -0.1575 | -0.4456 | 0.8813  | -240.0000 | -26.4267 | 22.0000 |           |
| 40 | -0.1307 | -0.9247 | 0.3576  | -210.0000 | -42.5333 | 16.0000 | 580.      |
| 41 | -0.1182 | -0.8364 | 0.5353  | -240.0000 | -42.4000 | 8.0000  | 645.650   |
| 42 | 0.0000  | -0.7894 | 0.6139  | -330.0000 | -4.6667  | 24.0000 | 1026.1578 |
| 43 | -0.1077 | -0.4486 | 0.8872  | -300.0000 | -16.5333 | 18.0000 | 1805.6287 |
| 44 | -0.1705 | -0.7102 | 0.6830  | -330.0000 | -25.0467 | 8.0000  | 760.3083  |
| 45 | 0.1558  | 0.8949  | -0.4183 | -352.5000 | -20.3800 | 13.8333 | 1936.0449 |
| 46 | 0.0130  | -0.7893 | 0.6139  | -382.5000 | -5.1667  | 23.8333 | 769.6833  |
| 47 | -0.1703 | -0.9843 | 0.0462  | -405.0000 | -6.0733  | 15.0000 | 891.4889  |
| 48 | -0.2034 | -0.7057 | 0.6787  | -382.5000 | -14.0867 | 5.1667  | 573.8820  |
| 49 | 0.3141  | 0.9442  | -0.0988 | -405.0000 | -9.4200  | 11.0000 | 929.3218  |
| 50 | -1.0000 | 0.0000  | 0.0000  | -427.5000 | -3.2100  | 11.0000 | 49.7100   |
| 51 | -0.1932 | -0.6290 | 0.7530  | -435.0000 | -3.2100  | 1.1667  | 62.5984   |

Figure 9. Output of HYDREX3.  
Partial Listing

-----Inertial Characteristics-----

|        |          |         |     |         |    |
|--------|----------|---------|-----|---------|----|
| WEIGHT | 47760.00 | L. TONS |     |         |    |
| XCG    | -478.050 | ft      | Kxx | 47.000  | ft |
| YCG    | 42.300   | ft      | Kyy | 219.000 | ft |
| ZCG    | 0.000    | ft      | Kzz | 219.000 | ft |
|        |          |         | Kxy | 0.000   | ft |
|        |          |         | Kxz | 0.000   | ft |
|        |          |         | Kyz | 0.000   | ft |

-----DAMPING COEFFICIENTS-----

(user specified)

|        |            |            |
|--------|------------|------------|
| MOTION | LINEAR     | QUADRATIC  |
| SURGE  | 0.0000E+00 | 0.0000E+00 |
| HEAVE  | 0.0000E+00 | 0.0000E+00 |
| SWAY   | 0.0000E+00 | 0.0000E+00 |
| ROLL   | 0.0000E+00 | 0.0000E+00 |
| YAW    | 0.0000E+00 | 0.0000E+00 |
| PITCH  | 0.0000E+00 | 0.0000E+00 |

-----OPS DATA-----

SPEED 42.23

-----Initial Conditions-----

-----POSITION-----

|       |       |     |
|-------|-------|-----|
| X(CG) | 0.000 | ft  |
| Y(CG) | 9.412 | ft  |
| Z(CG) | 0.000 | ft  |
| ROLL  | 0.000 | deg |
| YAW   | 0.000 | deg |
| PITCH | 0.000 | deg |

-----VELOCITY-----

|       |        |         |
|-------|--------|---------|
| SPEED | 42.230 | ft/sec  |
| HEAVE | 0.000  | ft/sec  |
| SWAY  | 0.000  | ft/sec  |
| ROLL  | 0.000  | deg/sec |
| YAW   | 0.000  | deg/sec |
| PITCH | 0.000  | deg/sec |

-----Run Parameters-----

SIMULATION PARAMETERS

|             |        |
|-------------|--------|
| TIME(START) | 30.00  |
| TIME(END)   | 100.00 |
| OUTPUT DT   | 0.50   |
| INTEG. DT   | 0.50   |

JOB OPTIONS

-----MISCELLANEOUS VALUES-----

|         |           |
|---------|-----------|
| RHO     | 0.000889  |
| G       | 32.169998 |
| JOBFO 1 |           |
| JOBFO   | 1 1 0     |

Figure 10. Part of Output of SSX  
Partial Listing

-----WAVE COMPONENTS-----

| WAVE | AMPLITUDE | FREQUENCY | DIRECTION | PHASE |
|------|-----------|-----------|-----------|-------|
| 1    | 1.00      | 0.34      | 180.00    | 0.00  |

-----SECTIONAL WEIGHT PROPERTIES-----

| SECTION | WEIGHT   | Kxx   | Centroid |
|---------|----------|-------|----------|
| 1       | 94.550   | 37.31 | 9.51     |
| 2       | 294.550  | 37.31 | 9.51     |
| 3       | 296.203  | 37.31 | 9.51     |
| 4       | 496.203  | 37.31 | 9.51     |
| 5       | 1143.822 | 37.31 | 9.51     |
| 6       | 1134.935 | 37.31 | 9.51     |
| 7       | 1681.707 | 37.31 | 9.51     |
| 8       | 2040.042 | 37.31 | 9.51     |
| 9       | 2508.421 | 37.31 | 9.51     |
| 10      | 2578.811 | 37.31 | 9.51     |
| 11      | 3053.338 | 37.31 | 9.51     |
| 12      | 3591.325 | 37.31 | 9.51     |
| 13      | 3530.823 | 37.31 | 9.51     |
| 14      | 3175.776 | 37.31 | 9.51     |
| 15      | 3725.574 | 37.31 | 9.51     |
| 16      | 3530.628 | 37.31 | 9.51     |
| 17      | 2595.390 | 37.31 | 9.51     |
| 18      | 3151.427 | 37.31 | 9.51     |
| 19      | 2534.063 | 37.31 | 9.51     |
| 20      | 1915.669 | 37.31 | 9.51     |
| 21      | 1761.739 | 37.31 | 9.51     |
| 22      | 715.479  | 37.31 | 9.51     |
| 23      | 715.479  | 37.31 | 9.51     |
| 24      | 747.026  | 37.31 | 9.51     |
| 25      | 747.026  | 37.31 | 9.51     |

Figure 10. Part of Output of SSX  
Partial Listing

| TIME      | WAVE     | POSITION OF CENTER OF GRAVITY |          |         |
|-----------|----------|-------------------------------|----------|---------|
|           |          | X                             | Y        | Z       |
| 0. 0000   | 1. 0000  | 0. 0000                       | 0. 0000  | 0. 0000 |
| 0. 5000   | 0. 9702  | 21. 1150                      | 0. 0002  | 0. 0000 |
| 1. 0000   | 0. 8825  | 42. 2301                      | 0. 0016  | 0. 0000 |
| 1. 5000   | 0. 7422  | 63. 3456                      | 0. 0048  | 0. 0000 |
| 2. 0000   | 0. 5576  | 84. 4618                      | 0. 0101  | 0. 0000 |
| 2. 5000   | 0. 3397  | 105. 5792                     | 0. 0169  | 0. 0000 |
| 3. 0000   | 0. 1016  | 126. 6983                     | 0. 0238  | 0. 0000 |
| 3. 5000   | -0. 1426 | 147. 8197                     | 0. 0285  | 0. 0000 |
| 4. 0000   | -0. 3783 | 168. 9435                     | 0. 0284  | 0. 0000 |
| 4. 5000   | -0. 5914 | 190. 0700                     | 0. 0202  | 0. 0000 |
| 5. 0000   | -0. 7693 | 211. 1992                     | 0. 0009  | 0. 0000 |
| 5. 5000   | -0. 9012 | 232. 3307                     | -0. 0322 | 0. 0000 |
| 6. 0000   | -0. 9794 | 253. 4639                     | -0. 0808 | 0. 0000 |
| 6. 5000   | -0. 9991 | 274. 5980                     | -0. 1451 | 0. 0000 |
| 7. 0000   | -0. 9592 | 295. 7319                     | -0. 2235 | 0. 0000 |
| 7. 5000   | -0. 8621 | 316. 8644                     | -0. 3120 | 0. 0000 |
| 8. 0000   | -0. 7135 | 337. 9943                     | -0. 4046 | 0. 0000 |
| 8. 5000   | -0. 5224 | 359. 1200                     | -0. 4931 | 0. 0000 |
| 9. 0000   | -0. 3001 | 380. 2405                     | -0. 5676 | 0. 0000 |
| 88. 0000  | 0. 6254  | 3715. 1992                    | 0. 9302  | 0. 0000 |
| 88. 5000  | 0. 7957  | 3736. 2339                    | 1. 0866  | 0. 0000 |
| 89. 0000  | 0. 9187  | 3757. 2551                    | 1. 1782  | 0. 0000 |
| 89. 5000  | 0. 9870  | 3778. 2671                    | 1. 1995  | 0. 0000 |
| 90. 0000  | 0. 9966  | 3799. 2751                    | 1. 1492  | 0. 0000 |
| 90. 5000  | 0. 9469  | 3820. 2847                    | 1. 0303  | 0. 0000 |
| 91. 0000  | 0. 8409  | 3841. 3008                    | 0. 8500  | 0. 0000 |
| 91. 5000  | 0. 6849  | 3862. 3284                    | 0. 6189  | 0. 0000 |
| 92. 0000  | 0. 4881  | 3883. 3718                    | 0. 3509  | 0. 0000 |
| 92. 5000  | 0. 2621  | 3904. 4341                    | 0. 0621  | 0. 0000 |
| 93. 0000  | 0. 0205  | 3925. 5173                    | -0. 2304 | 0. 0000 |
| 93. 5000  | -0. 2224 | 3946. 6223                    | -0. 5090 | 0. 0000 |
| 94. 0000  | -0. 4521 | 3967. 7488                    | -0. 7573 | 0. 0000 |
| 94. 5000  | -0. 6549 | 3988. 8945                    | -0. 9603 | 0. 0000 |
| 95. 0000  | -0. 8187 | 4010. 0569                    | -1. 1060 | 0. 0000 |
| 95. 5000  | -0. 9335 | 4031. 2317                    | -1. 1858 | 0. 0000 |
| 96. 0000  | -0. 9926 | 4052. 4143                    | -1. 1949 | 0. 0000 |
| 96. 5000  | -0. 9924 | 4073. 5996                    | -1. 1329 | 0. 0000 |
| 97. 0000  | -0. 9328 | 4094. 7822                    | -1. 0034 | 0. 0000 |
| 97. 5000  | -0. 8176 | 4115. 9570                    | -0. 8141 | 0. 0000 |
| 98. 0000  | -0. 6535 | 4137. 1196                    | -0. 5763 | 0. 0000 |
| 98. 5000  | -0. 4504 | 4158. 2656                    | -0. 3041 | 0. 0000 |
| 99. 0000  | -0. 2205 | 4179. 3926                    | -0. 0138 | 0. 0000 |
| 99. 5000  | 0. 0225  | 4200. 4980                    | 0. 2774  | 0. 0000 |
| 100. 0000 | 0. 2640  | 4221. 5815                    | 0. 5521  | 0. 0000 |
| RMS       | 0. 7112  | 304. 5160                     | 0. 8489  | 0. 0000 |
| MEAN      | -0. 0134 | 3704. 3413                    | -0. 0220 | 0. 0000 |
| MAX       | 1. 0000  | 4221. 5815                    | 1. 2009  | 0. 0000 |
| MIN       | -0. 9991 | 0. 0000                       | -1. 1991 | 0. 0000 |

- Figure 10. Left side of 132 character width printout of SSX (partial)

| ROLL   | ROTATIONS IN DEGREES | PITCH   | YAW    | MIDS. VERT   | SPEED   |
|--------|----------------------|---------|--------|--------------|---------|
| 0.0000 | 0.0000               | 0.0000  | 0.0000 | 0.0000E+00   | 42.2300 |
| 0.0000 | 0.0000               | 0.0000  | 0.0000 | -0.57850E+06 | 42.2301 |
| 0.0000 | 0.0000               | -0.0000 | 0.0000 | -0.60541E+06 | 42.2305 |
| 0.0000 | 0.0000               | -0.0008 | 0.0000 | -0.63125E+06 | 42.2315 |
| 0.0000 | 0.0000               | -0.0021 | 0.0000 | -0.65431E+06 | 42.2335 |
| 0.0000 | 0.0000               | -0.0047 | 0.0000 | -0.67303E+06 | 42.2364 |
| 0.0000 | 0.0000               | -0.0088 | 0.0000 | -0.68603E+06 | 42.2403 |
| 0.0000 | 0.0000               | -0.0148 | 0.0000 | -0.69229E+06 | 42.2451 |
| 0.0000 | 0.0000               | -0.0227 | 0.0000 | -0.69116E+06 | 42.2503 |
| 0.0000 | 0.0000               | -0.0323 | 0.0000 | -0.68247E+06 | 42.2557 |
| 0.0000 | 0.0000               | -0.0433 | 0.0000 | -0.66558E+06 | 42.2608 |
| 0.0000 | 0.0000               | -0.0548 | 0.0000 | -0.64439E+06 | 42.2649 |
| 0.0000 | 0.0000               | -0.0654 | 0.0000 | -0.61736E+06 | 42.2676 |
| 0.0000 | 0.0000               | -0.0745 | 0.0000 | -0.58740E+06 | 42.2684 |
| 0.0000 | 0.0000               | -0.0800 | 0.0000 | -0.55682E+06 | 42.2669 |
| 0.0000 | 0.0000               | -0.0806 | 0.0000 | -0.52814E+06 | 42.2628 |
| 0.0000 | 0.0000               | -0.0751 | 0.0000 | -0.50389E+06 | 42.2560 |
| 0.0000 | 0.0000               | -0.0627 | 0.0000 | -0.48633E+06 | 42.2466 |
| 0.0000 | 0.0000               | -0.0427 | 0.0000 | -0.47728E+06 | 42.2349 |
| 0.0000 | 0.0000               | 0.2169  | 0.0000 | -0.62122E+06 | 42.0858 |
| 0.0000 | 0.0000               | 0.1917  | 0.0000 | -0.62237E+06 | 42.0545 |
| 0.0000 | 0.0000               | 0.1554  | 0.0000 | -0.62240E+06 | 42.0315 |
| 0.0000 | 0.0000               | 0.1099  | 0.0000 | -0.62129E+06 | 42.0183 |
| 0.0000 | 0.0000               | 0.0577  | 0.0000 | -0.61914E+06 | 42.0157 |
| 0.0000 | 0.0000               | 0.0021  | 0.0000 | -0.61605E+06 | 42.0238 |
| 0.0000 | 0.0000               | -0.0537 | 0.0000 | -0.61218E+06 | 42.0422 |
| 0.0000 | 0.0000               | -0.1062 | 0.0000 | -0.60782E+06 | 42.0697 |
| 0.0000 | 0.0000               | -0.1523 | 0.0000 | -0.60310E+06 | 42.1104 |
| 0.0000 | 0.0000               | -0.1893 | 0.0000 | -0.59842E+06 | 42.1451 |
| 0.0000 | 0.0000               | -0.2149 | 0.0000 | -0.59400E+06 | 42.1885 |
| 0.0000 | 0.0000               | -0.2275 | 0.0000 | -0.59013E+06 | 42.2318 |
| 0.0000 | 0.0000               | -0.2265 | 0.0000 | -0.58704E+06 | 42.2730 |
| 0.0000 | 0.0000               | -0.2118 | 0.0000 | -0.58496E+06 | 42.3092 |
| 0.0000 | 0.0000               | -0.1846 | 0.0000 | -0.58401E+06 | 42.3385 |
| 0.0000 | 0.0000               | -0.1463 | 0.0000 | -0.58419E+06 | 42.3591 |
| 0.0000 | 0.0000               | -0.0993 | 0.0000 | -0.58556E+06 | 42.3697 |
| 0.0000 | 0.0000               | -0.0465 | 0.0000 | -0.58798E+06 | 42.3697 |
| 0.0000 | 0.0000               | 0.0091  | 0.0000 | -0.59137E+06 | 42.3593 |
| 0.0000 | 0.0000               | 0.0641  | 0.0000 | -0.59538E+06 | 42.3389 |
| 0.0000 | 0.0000               | 0.1154  | 0.0000 | -0.59986E+06 | 42.3093 |
| 0.0000 | 0.0000               | 0.1598  | 0.0000 | -0.60459E+06 | 42.2737 |
| 0.0000 | 0.0000               | 0.1948  | 0.0000 | -0.60915E+06 | 42.2327 |
| 0.0000 | 0.0000               | 0.2183  | 0.0000 | -0.61340E+06 | 42.1891 |
| 0.0000 | 0.0000               | 0.2289  | 0.0000 | -0.61708E+06 | 42.1456 |
| 0.0000 | 0.0000               | 0.1599  | 0.0000 | 0.13537E+05  | 0.1269  |
| 0.0000 | 0.0000               | -0.0053 | 0.0000 | -0.60289E+06 | 42.1984 |
| 0.0000 | 0.0000               | 0.2289  | 0.0000 | 0.0000E+00   | 42.3768 |
| 0.0000 | 0.0000               | -0.2282 | 0.0000 | -0.62252E+06 | 0.0000  |

Figure 10. Right side of 132 character  
 width printout of SSX (partial)

PR SURES

|             |             |             |             |             |
|-------------|-------------|-------------|-------------|-------------|
| 0. 38415968 | 0. 24316381 | 0. 23651379 | 0. 22607508 | 0. 21061255 |
| 0. 19246852 | 0. 17776074 | 1. 14601755 | 1. 16095650 | 1. 15000000 |
| 1. 05791485 | 1. 29189086 | 1. 40567327 | 2. 06866050 |             |
| 2. 15990114 | 2. 33326197 | 2. 64962888 | 3. 00000000 |             |
| 3. 24160838 | 2. 96588731 | 3. 32186317 |             |             |
| 4. 04283953 | 3. 78925920 | 3. 97000000 |             |             |
| 4. 74622965 | 4. 53059387 |             |             |             |
| 5. 02411795 | 5. 23495000 |             |             |             |
| 5. 02339220 |             |             |             |             |
| 5. 83894500 |             |             |             |             |

-----TOTAL PRESSURES-----

|              |              |              |              |             |
|--------------|--------------|--------------|--------------|-------------|
| -1. 46834481 | -0. 83654284 | -0. 78719705 | -1. 71809840 | 0. 00665942 |
| 0. 23063746  | 1. 18806934  | -0. 45587230 | -0. 17283618 | 0. 35000000 |
| 17. 64413261 | 2. 48809695  | 1. 18732226  | 0. 37171781  |             |
| 1. 17200983  | 1. 83491194  | 2. 47403312  | 0. 99945000  |             |
| 1. 39725995  | 1. 88052142  | 2. 80034971  |              |             |
| 1. 43834829  | 1. 74238777  | 2. 93543900  |              |             |
| 2. 78261328  | 2. 71100593  | 3. 00000000  |              |             |
| 56. 71600723 | 3. 11889052  |              |              |             |
| 8. 52794075  | 7. 50000000  |              |              |             |

BY INDUCED PRESSURES

|              |              |              |              |              |
|--------------|--------------|--------------|--------------|--------------|
| -1. 85250449 | -1. 07970667 | -1. 02371085 | -1. 94417346 | -0. 20395313 |
| 0. 03816893  | 1. 01030862  | -1. 60188985 | -1. 33379269 | -0. 83561438 |
| 16. 58621788 | 1. 19620597  | -0. 21835098 | -1. 69694269 | -1. 50451446 |
| -0. 98789138 | -0. 49835002 | -0. 17559581 | -2. 02371120 | -1. 95836079 |
| -1. 84434843 | -1. 08536589 | -0. 52151340 | -0. 05880385 | -2. 24148512 |
| -2. 60449123 | -2. 04687142 | -1. 03688288 | 0. 02363856  | 1. 00506485  |
| -1. 96361649 | -1. 81958807 | -0. 79081982 | -1. 89128637 | 2. 57368040  |
| 51. 69189072 | -2. 11605954 | -1. 65408003 | -1. 66934085 | -1. 67518842 |
| 3. 50454855  | 1. 93148851  | -1. 43030024 | -0. 60449553 | 1. 84978557  |
| 0. 40260637  | -0. 62282950 | -0. 58973241 | -0. 03671409 | 2. 83566427  |
| -0. 41253781 | 0. 83666974  | -0. 10891942 | 0. 55786502  | 0. 00000000  |
| 0. 03906842  | -2. 58687186 | -1. 62076259 | -0. 77843022 | 7. 79187298  |
| -0. 35989502 | -0. 11869927 | -0. 31951666 | -1. 83989894 | -1. 40328050 |
| -0. 73166758 | 11. 49159718 | -0. 16707416 | -0. 29368657 | -1. 92022955 |
| -1. 52942920 | -1. 04738855 | -0. 60960990 | -0. 22613873 | -2. 24740911 |
| -1. 75089848 | -1. 78046894 | -0. 91861057 | -0. 41995618 | 0. 12429215  |
| -2. 62914538 | -2. 34276557 | -1. 72142696 | -1. 22348630 | 1. 35021770  |
| 30. 93821144 | -2. 01162410 | -1. 54072320 | -0. 79989606 | -1. 55071521 |
| 10. 87392330 | 0. 00000000  | -2. 08891344 | -1. 69489515 | -1. 51071918 |
| -1. 30158806 | 2. 16383004  | 1. 18314898  | -1. 51980424 | -0. 68915701 |
| 1. 78912711  | 0. 56204945  | -0. 96726221 | -0. 71865982 | -0. 38488629 |
| 0. 34184885  | -0. 67282838 | -0. 67956263 | -0. 36257684 | 0. 79844165  |
| 0. 00000000  | -0. 03861484 |              |              |              |

- Figure 11. Pressure output of SSX  
(partial listing for one time step) -

APPENDIX A

Program Listings

HYDREX2

HYDREX3

SSX

(Appendix A Contained Under Separate Cover)



APPENDIX A

Program Listings

HYDREX2

HYDREX3

SSX

PROGRAM SSX

```

C CHARACTER*25 OFFIL, BIFIL, COFIL, MATFIL
C CHARACTER*30 TITLE
COMMON/HEAD/TITLE
COMMON/IOFILE/ OFFIL, BIFIL, COFIL, MATFIL
COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF, MAT
INTEGER OUTPUT, BIF, OFF, COF
COMMON /COEFF4 / COEFF4(6, 12, 8, 25), AREAN(25, 6)
COMMON/OPTION/JOBEO, JOBO(10), JOBPO(10)
COMMON/JOBP/JOBP
COMMON /JOB/ IJOB, IFORCE, IAXIS, IWANT
COMMON / / /
1 NWAVES, WVSUM,
2 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
3 WN(20), WNX(20), WNZ(20),
4 CSK(20), CCK(20), CXK(20),
CYK(20), CZK(20), XW(20)
COMMON / / /
1 MSTA, NPROF, NFWD, NAFT, LPTS(25),
2 XOFF(25), YOFF(25,25), ZOFF(25,25),
XPROF(51), YPROF(51)
COMMON /DAMP / DAMPL(6), DAMPG(6)
COMMON /LHS / Y(13)
COMMON /MASS / RHO, G, GAMMA
COMMON /MASS / DISPL, SMASS, XCG, YCG, ZCG,
1 AMX, AMY, AMZ, RADII(6)
COMMON /MASS / PMI(3,3)
COMMON /RESIST/ SPEED
COMMON /TIME / TIME, RAMP
COMMON /TIME / TO, TSTART, TSTOP, TOUTPT, TSTEP, ERR
COMMON/PREXIN/TPSTART, TPSTOP, TPRAMP, BGX, SMX, BGY, SMY, TSCALE
COMMON/DRFT1/ DRAFT1(6)
COMMON /SIGMA / NK, SIGMA(24), SIGMAO, ERRO, OM(12)
COMMON /STATS / NRHS(4), DELTA, HSUM, HMIN, HMAX, TTO, TT1
DATA TO /0.0/
DATA NEGS /12/
LOGICAL WL, ADJUST, INTERACT
DATA RHO /0.00088861607142/
DATA ERRO/1.0E-37/
DATA INPUT/5/, OUTPUT/6/, BIF/1/, OFF/2/, COF/9/
DATA MAT/3/

```

---

C \*\*\* 1.0 Get DATA file names and assign/open files

```

C TYPE 902
C ACCEPT 901, BIFIL
C TYPE 903
C ACCEPT 901, OFFIL
C TYPE 904
C ACCEPT 901, COFIL
C TYPE 905
C ACCEPT 901, MATFIL

```

```

C OPEN(UNIT=BIF, STATUS='OLD', FILE=BIFIL)
C OPEN(UNIT=OFF, STATUS='OLD', FILE=OFFIL)
C OPEN(UNIT=COF, STATUS='OLD', FORM='UNFORMATTED', FILE=COFIL)
C IF(MATFIL.NE.'NONE') OPEN(UNIT=MAT, STATUS='OLD', FORM='UNFOR
1 MATTED', FILE=MATFIL)

```

C \*\*\* 2.0 Read in data from [BIF], [OFF] and [COF] files

```

C CALL READIN

```

C \*\*\* 3.0 Perform preparatory computations

```

C CALL PREPARE

```

```

C CALL COFFEE
C CALL FOLIO1
C IF(JOBPO(1).EQ.1) CALL OUTCOF

```

```

C *** 5.0 Simulation
C
C WRITE (OUTPUT,130)
C TSTEP=-ABS(TSTEP)
C TIME=TO
C TTO=TO
C TT1=TO
110 CALL FOLIO2 !Show time zero condition
C TNEXT=TIME+TOUTPT
C *** Fourth-order Runge-Kutta integration
C CALL RKF (TIME,Y,TNEXT,ERR,NEGS,TSTEP)
C *** Get hull girder loads
C CALL HULL
C *** Get panel pressures
C TP=TIME-TPSTART
C IF (TP. GE. 0. .AND. TP. LE. TPSTART) CALL PRESSURE(TP)
C
C TIME=TNEXT
C IF (TIME. LT. TSTOP) GO TO 110
120 CALL FOLIO2
C CALL FINI
C STOP
C
130 FORMAT (1H1,27X,29HPOSITION OF CENTER OF GRAVITY,
# 9X,20HROTATIONS IN DEGREES,27X,19HHORIZONTAL VELOCITY/
# 4X,4HTIME,8X,4HWAVE,8X,1HX,11X,1HY,11X,1HZ,13X,
# 4HROLL,9X,3HYAW,7X,5HPITCH,6X,6HRUDDER,
# 7X,5HSPEED,5X,9HSWAY RATE/)
C
C 901 FORMAT(A)
C 902 FORMAT(' Name of Basic INPUT File [BIF] ? > '$)
C 903 FORMAT(' Name of Offset File [OFF] ? > '$)
C 904 FORMAT(' Name of Coefficient File [COF] ? > '$)
C 905 FORMAT(' Name of 3-D Matrix File [MAT] ? > '$)
C
END

```

SUBROUTINE READIN

C

```

COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF, MAT
INTEGER OUTPUT, BIF, OFF, COF
INTEGER ECHO
COMMON /SIGMA / NK, SIGMA(24), SIGMAO, ERRO, OM(12)
COMMON/DRFT1/DRAFT1(6)
COMMON/IOFILE/ OFFIL, BIFIL, COFIL, MATFIL
COMMON/HEAD/TITLE
CHARACTER*30 TITLE
COMMON/COEFF4/COEFF4(6, 12, 8, 25), AREAN(25, 6)
COMMON/OPTION/JOBCO, JOBFO(10), JOBPO(10)
COMMON / / NWAVES, WVSUM,
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
2 WN(20), WNX(20), WNZ(20),
3 CSK(20), CCK(20), CXK(20),
4 CYK(20), CZK(20), XW(20)
COMMON / / MSTA, NPROF, NFWD, NAFT, LPTS(25),
1 XOFF(25), YOFF(25,25), ZOFF(25,25),
2 XPROF(51), YPROF(51)
COMMON /DAMP / DAMPL(6), DAMPG(6)
COMMON /LHS / Y(13)
COMMON /MASS / RHO, G, GAMMA
COMMON /MASS / DISPL, SMASS, XCG, YCG, ZCG,
1 AMX, AMY, AMZ, RADII(6)
COMMON /MASS / PMI(3,3)
COMMON /RESIST/ SPEED
COMMON /TIME / TIME, RAMP
COMMON /TIME / TO, TSTART, TSTOP, TOUTPT, TSTEP, ERR
COMMON/SXPROP/ SEGMAS(26), SEGMOX(26), STRMAS(26), STRMOM(26),
* STRMOX(26), XBAR(26), YBAR(26), SEGWT(26), NWTSTA
REAL XAFT(25), YAFT(25), XFWD(25), YFWD(25)
EQUIVALENCE (XFWD(1), XPROF(1)), (YFWD(1), YPROF(1)),
# (XAFT(1), XPROF(26)), (YAFT(1), YPROF(26))
COMMON/PREXIN/TPSTART, TPSTOP, TPRAMP, BGX, SMX, BGY, SMY, TSCALE
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREAP(120),
* ST(120), ACN(120), ACNW(120), AN(120,3), E(120), P(120,6),
* PRFS(120), STOLD(120), PX(120,6)
COMMON/A/NPAN, NPT, GEE, RHOP, NKX, NKY, EYE, DT, TIM, UFWD
COMMON/SOURCE/ EE(120,120)
REAL POSIT(6), VELOC(6)
COMMON/DISPLAY/TARE(6)
EQUIVALENCE (POSIT(1), Y(1)), (VELOC(1), Y(7))
DATA MAXSTA/25/
DATA MAXPTS/25/
DATA MAXFWD/25/
DATA MAXAFT/25/
DATA MAXWVS/20/
DATA TPSTOP/0./
DATA ERRMIN/1.0E-10/
CHARACTER*81 CARDID
DATA DEGREE/0.01745 32925 19943/
DATA G /32.17/
DATA NWL/1/
DATA ECHO/4/
C *** OPEN DATA FILE ECHO PRINTOUT FILE
OPEN(UNIT=ECHO, TYPE='NEW', NAME='ECHO.DAT')
C
C WRITE(ECHO, 197)
C *** TITLE
READ (BIF, 199) TITLE
WRITE(ECHO, 196) TITLE
C *** DRAFT (fwd), DRAFT (aft), long. loc's of DRAFT marks
READ (BIF, 200) TF, TA, XFPERP, XAPERP
WRITE(ECHO, 200) TF, TA, XFPERP, XAPERP
C *** Center of Gravity (XCG aft of FP, YCG above BL)
READ (BIF, 200) XCG, YCG, ZCG
WRITE(ECHO, 200) XCG, YCG, ZCG
C *** Six DRAFTs at which hydro. coeffs are computed
READ(BIF, 200) (DRAFT1(I), I=1,6)
WRITE(ECHO, 200) (DRAFT1(I), I=1,6)
C *** Minimum segment lengths for Frank Close Fit
READ (BIF, 201) YMAX, ZMAX, WMAX, NWL
WRITE(ECHO, 201) YMAX, ZMAX, WMAX, NWL
ADJUST=ZMAX. GT. 0. 0. AND. YMAX. GT. 0. 0

```

```

C *** Number of forward profile points
  READ (BIF,190) NFW
  WRITE(ECHO,190) NFW
  IF (NFW.GT.25) CALL ERROR(15, IDUM, RDUM)
C *** Coordinates of forward profile points
  IF (NFW.GT.0) READ (BIF,430) (YFWD(I), XFWD(I), I=1, NFW)
  WRITE(ECHO,430) (YFWD(I), XFWD(I), I=1, NFW)
C *** Number of aft profile points
  READ (BIF,190) NAFT
  WRITE(ECHO,190) NAFT
  IF (NAFT.GT.25) CALL ERROR(16, IDUM, RDUM)
C *** Coordinates of aft profile points
  IF (NAFT.GT.0) READ (BIF,430) (YAFT(I), XAFT(I), I=1, NAFT)
  WRITE(ECHO,430) (YAFT(I), XAFT(I), I=1, NAFT)
C *** Displacement
  READ(BIF,202) DISPL
  WRITE(ECHO,200) DISPL
C *** Radii of gyration
  READ(BIF,202) RADII
  WRITE(ECHO,200) RADII
C *** Ship speed
  READ(BIF,202) SPEED
  WRITE(ECHO,200) SPEED
C *** Linear Damping Constants
  READ(BIF,202) DAMPL
  WRITE(ECHO,200) DAMPL
C *** Quadratic Damping Constants
  READ(BIF,202) DAMPG
  WRITE(ECHO,200) DAMPG
C *** Number of wave components
  WRITE(ECHO,203) NWAIVES
  READ(BIF,203) NWAIVES
C *** Wave component specifications
  DO 140 J=1, NWAIVES
  READ(BIF,202) WVAMP(J), WVFRE(J), WVDIR(J), WVPHA(J)
  WRITE(ECHO,200) WVAMP(J), WVFRE(J), WVDIR(J), WVPHA(J)
140 CONTINUE
C *** Initial position
  READ(BIF,202) POSIT
  WRITE(ECHO,200) POSIT
C *** Initial velocities
  READ(BIF,202) VELOC
  WRITE(ECHO,200) VELOC
C *** Simulation specification
  READ(BIF,202) TSTART, TSTOP, TOUTPT, TSTEP
  WRITE(ECHO,200) TSTART, TSTOP, TOUTPT, TSTEP
C *** Number of weight stations
  READ(BIF,203) NWTSTA
  WRITE(ECHO,203) NWTSTA
C *** Segment weight, rotational gyradii, centroid
  DO 141 J=1, NWTSTA
  READ(BIF,202) SEGWT(J), SEGMOX(J), YBAR(J)
  WRITE(ECHO,200) SEGWT(J), SEGMOX(J), YBAR(J)
141 CONTINUE
C
  IF(MATFIL.EQ.'NONE') GO TO 1000
  READ(BIF,202) TPSTART, TPSTOP, TPRAMP
  WRITE(ECHO,200) TPSTART, TPSTOP, TPRAMP
  READ(BIF,202) BGX, SMX, BGY, SMY, TSCALE
  WRITE(ECHO,200) BGX, SMX, BGY, SMY, TSCALE

```

```

C-----
C
C
C Section 2.0 - READ OFFSET file
C The offset file can be an actual SHCP DATA File
1000 CONTINUE
  WRITE(ECHO,198)
C
C *** CARD TYPE A
  READ (OFF,410) CARDID
  WRITE(ECHO,410) CARDID
C *** CARD TYPE B
  READ (OFF,410)
C *** CARD TYPE C
  READ (OFF,412) SPACE, ZSCAL, YSCAL, SHIPL, NAPN, KINDO
  WRITE(ECHO,412)SPACE, ZSCAL, YSCAL, SHIPL, NAPN, KINDO
  IF (SPACE.EQ.0.0) SPACE=1.0
  ZSCAL=1.0
  YSCAL=1.0
  MSTA=0
  NFW=0
  NAFT=0

```

```

C      SUBROUTINE AFSR
      INTIIALIZES FREE SURFACE
      COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120), ST(120),
*      ACN(120), ACNW(120), AN(120, 3), E(120), F(120, 6), PRFS(120),
*      STOLD(120), PX(120, 6)
      COMMON/FS/AKZ(100, 100), SS(100, 100), CC(100, 100),
*      DKX(100), DKY(100), AKX(100), AKY(100)
      COMMON/FS1/A(100, 100), B(100, 100), AS(100, 100), BS(100, 100)
      COMMON/A/NPAN, NPT, GEE, RHOP, NKX, NKY, EYE, DT, TIM, UFWD
      COMMON/PREXIN/TPSTART, TPSTOP, TPRAMP, BGX, SMX, BGY, SMY, TSCALE
      COMPLEX A, B, AS, BS, EYE
      DKTT=1. 0/(GEE*TSCALE*TSCALE)
      NBX=0. 5/(BGX*DKTT)
      NBY=0. 5/(BGY*DKTT)
      AMX=1. 0/SMX
      AMY=1. 0/SMY
      N=0
      M=0
      AOLD=0. 0
1      IF(N. GE. NBX) GO TO 2
      N=N+1
      AKX(N)=N*N*DKTT
      AOLD=AKX(N)
      IF(AOLD. GE. AMX) NBX=N
      GO TO 1
2      N=N+1
      AKX(N)=AOLD+1. 0/BGX
      AOLD=AKX(N)
      IF(AOLD. LE. AMX) GO TO 2
      NKX=N
      AKX(NKX+1)=AOLD+1. 0/BGX
      AOLD=0. 00
11     IF(M. GE. NBY) GO TO 22
      M=M+1
      AKY(M)=M*M*DKTT
      AOLD=AKY(M)
      IF(AOLD. GE. AMY) NBY=M
      GO TO 11
22     M=M+1
      AKY(M)=AOLD+1. 0/BGY
      AOLD=AKY(M)
      IF(AOLD. LE. AMY) GO TO 22
      NKY=M
      AKY(M+1)=AKY(M)+1. 0/BGY
      AFF=0. 000
      DO 17 N=1, NKX
      AF=(AKX(N+1)+AKX(N))*0. 50
      DKX(N)=AF-AFF
17     AFF=AF
      BFF=0. 00
      DO 18 N=1, NKY
      BF=(AKY(N+1)+AKY(N))*0. 50
      DKY(N)=BF-BFF
18     BFF=BF
      DO 100 N=1, NKX
      DO 100 M=1, NKY
      A(N, M)=(0. 0, 0. 0)
      B(N, M)=(0. 0, 0. 0)
      AS(N, M)=(0. 0, 0. 0)
      BS(N, M)=(0. 0, 0. 0)
      AKZ(N, M)=SQRT(AKX(N)**2+AKY(M)**2)
      SIG=SQRT(GEE*AKZ(N, M))
      SS(N, M)=SIN(SIG*DT)
      CC(N, M)=COS(SIG*DT)
100    CONTINUE
      RETURN
      END

```

SUBROUTINE PRESSURE(TP)

```

COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF, MAT
INTEGER OUTPUT, BIF, OFF, COF
INTEGER ECHO
COMMON /SIGMA / NK, SIGMA(24), SIGMAO, ERRO, DM(12)
COMMON/DRFT1/DRAFT1(6)
COMMON/IOFILE/ OFFIL, BIFIL, COFIL, MATFIL
COMMON/HEAD/TITLE
CHARACTER*30 TITLE
COMMON/COEFF4/COEFF4(6, 12, 8, 25), AREAN(25, 6)
COMMON / / NWAVES, WVSUM,
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
4 WN(20), WNX(20), WNZ(20),
CSK(20), CCK(20), CXK(20),
2 CYK(20), CZK(20), XW(20)
COMMON / / MSTA, NPROF, NFWD, NAFTA, LPTS(25),
1 XOFF(25), YOFF(25,25), ZOFF(25,25),
2 XPROF(51), YPROF(51)
COMMON /DAMP / DAMPL(6), DAMPG(6)
COMMON /LHS / Y(13)
COMMON /MASS / RHO, G, GAMMA
COMMON /MASS / DISPL, SMASS, XCG, YCG, ZCG,
1 AMX, AMY, AMZ, RADII(6)
COMMON /MASS / PMI(3,3)
COMMON /RESIST/ SPEED
COMMON /TIME / TIME, RAMP
COMMON /TIME / TO, TSTART, TSTOP, TOUTPT, TSTEP, ERR
COMMON/SXPROP/ SEGMAS(26), SEGMOX(26), STRMAS(26), STRMOM(26),
* STRMOX(26), XBAR(26), YBAR(26), SEGWT(26), NWTSTA
REAL YAFT(25), YAFT(25), XFWD(25), YFWD(25)
EQUIVALENCE (XFWD(1), XPROF(1)), (YFWD(1), YPROF(1)),
* (YAFT(1), XPROF(26)), (YAFT(1), YPROF(26))
COMMON/PREXIN/TPSTART, TPSTOP, TPRAMP, BGX, SMX, BGY, SMY, TSCALE
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120),
* ST(120), ACN(120), ACNW(120), AN(120, 3), E(120), P(120, 6),
* PRFS(120), STOLD(120), PX(120, 6)
COMMON/FS/AKZ(100, 100), SS(100, 100), CC(100, 100),
* DKX(100), DKY(100), AKX(100), AKY(100)
COMMON/BD2/XPT(150), YPT(150), ZPT(150), WRF(150),
* WRFR(150), KK(150, 4)
COMMON/A/NPAN, NPT, GEE, RHOP, NKX, NKY, EYE, DT, TIM, UFWD
REAL POSIT(6), VELOC(6)
COMMON/SOURCE/EE(120, 120), BPRES(120), PT(6)
COMMON/DISPLAY/TARE(6)

```

TIM=TP  
UFWD=YY(7)

\*\*\* Get free surface induce component of normal acceleration  
at panel centers ACNW(J) and pressure force array PF.

CALL ACPTR

\*\*\* Get exciting normal acceleration at panel centers

CALL XNA(AXMT, AYMT, AZMT, ARLMT, APMT, AYWMT)

\*\*\* Compute panel source strengths

CALL ZBLACN(AXMT, AYMT, AZMT, ARLMT, APMT, AYWMT)

\*\*\* Compute body induced forces and pressures

CALL POTB

\*\*\* Advance free surface for single time step

CALL CFSR

\*\*\* Forces and moments

CALL PRFR(PT)

RETURN  
END

```

SUBROUTINE ACPTR
C Computes wave induced accelerations and pressures at
C panel centers
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120), ST(120),
* ACN(120), ACNW(120), AN(120, 3), E(120), P(120, 6), PRFS(120),
* STOLD(120), PX(120, 6)
COMMON/FS/AKZ(100, 100), SS(100, 100), CC(100, 100),
* DKX(100), DKY(100), AKX(100), AKY(100)
COMMON/FS1/A(100, 100), B(100, 100), AS(100, 100), BS(100, 100)
COMMON/A/NPAN, NPT, GEE, RHOP, NKX, NKY, EYE, DT, TIM, UFWD
COMPLEX EYE, SC, BX, CX, BY, CY, BYCON, SCON, SK, SKON,
* B1, B2, C1, C2
COMPLEX A, B, AS, BS
DIMENSION BX(100), CX(100), PF(6), DX(100)
DO 1500 J=1, NPAN
AX=AN(J, 1)
AY=AN(J, 2)
AZ=AN(J, 3)
X=XPAN(J)
Y=YPAN(J)
Z=ZPAN(J)
ACT=0.00
PRT=0.00
DVDZ=0.00
DO 16 N=1, NKX
CCXX=AKX(N)*X
S=SIN(CCXX)
C=COS(CCXX)
BX(N)=CMPLX(C, S)*DKX(N)
DX(N)=-AKX(N)*AKX(N)*AX
16 CX(N)=AKX(N)*AX*EYE
DO 163 M=1, NKY
CCYY=AKY(M)*Y
S=SIN(CCYY)
C=COS(CCYY)
BY=CMPLX(C, S)*DKY(M)
BYCON=CONJG(BY)
CY=AKY(M)*AY*EYE
DO 163 N=1, NKX
ARGZ=AKZ(N, M)*Z
DEP=EXP(-ARGZ)
B1=DEP*BY*BX(N)
B2=DEP*BYCON*BX(N)
CZ=AKZ(N, M)*AZ
SC=B1*A(N, M)
SCON=B2*AS(N, M)
SK=B1*B(N, M)
SKON=B2*BS(N, M)
C1=CX(N)+CY-CZ
C2=CX(N)-CY-CZ
ACT=ACT-C1*SC-C2*SCON
DVZ=DVZ+EYE*AKX(N)*(C1*SK+C2*SKON)/SQRT(AKZ(N, M))
PRT=PRT+SC+SCON
-163 CONTINUE
C NORMAL ACCELERATION INDUCED BY FREE SURFACE
DVDZ=DVZ*UFWD*SQRT(GEE)
ACNW(J)=ACT*GEE+DVDZ
C PRESSURE INDUCED BY FREE SURFACE
PRFS(J)=PRT*GEE*RHOP
1500 CONTINUE
60 FORMAT(' FREE SURFACE INDUCE PRESSURES')
1070 FORMAT(1X, 5F16.8)
TYPE 60
TYPE 1070, (PRFS(J), J=1, NPAN)
RETURN
END

```



```

SUBROUTINE CFSR
C *** Advance free surface wave spectra in time
COMMON/BD/XPAN(120),YPAN(120),ZPAN(120),AREA(120),ST(120),
* ACN(120),ACNW(120),AN(120,3),E(120),P(120,6),PRFS(120),
* STOLD(120),PX(120,6)
COMMON/FS/AKZ(100,100),SS(100,100),CC(100,100),
* DKX(100),DKY(100),AKX(100),AKY(100)
COMMON/FS1/A(100,100),B(100,100),AS(100,100),BS(100,100)
COMMON/A/NPAN,NPT,GEE,RHOP,NKX,NKY,EYE,DT,TIM,UFWD
DIMENSION CX(100)
COMPLEX EYE,AT,ATS,CX,CY,CXY1,CXYS,A,B,AS,BS,DFWD
DO 100 N=1,NKX
CXX=AKX(N)*UFWD*DT
S=SIN(CXX)
C=COS(CXX)
DFWD=CMPLX(C,-S)
DO 100 M=1,NKY
CT=CC(N,M)
STT=SS(N,M)
AT=A(N,M)*CT+B(N,M)*STT
ATS=AS(N,M)*CT+BS(N,M)*STT
B(N,M)=B(N,M)*CT-A(N,M)*STT
BS(N,M)=BS(N,M)*CT-AS(N,M)*STT
A(N,M)=AT
AS(N,M)=ATS
C MOVE FREE SURFACE RELATIVE TO BODY WITH FWD SPEED
A(N,M)=A(N,M)*DFWD
B(N,M)=B(N,M)*DFWD
AS(N,M)=AS(N,M)*DFWD
BS(N,M)=BS(N,M)*DFWD
100 CONTINUE
6 CONTINUE
C *** Add effects of source panels acting over one time step
DO 1500 J=1,NPAN
STAR=(ST(J)*0.50*DT+STOLD(J))*DT
C ST IS TIME RATE OF CHANGE OF SOURCE STRENGTH
C STOLD IS SOURCE STRENGTH AT START OF TIME STEP
C STAR IS AVERAGE VALUE OF STRENGTH OVER THE TIME STEP
STAR2=STOLD(J)*DT
STAR=STAR*AREA(J)*0.6366197724
STAR2=STAR2*AREA(J)*0.31831
X=XPAN(J)
IF(JTM.GT.0) X=X-UFWD*DT*.5
Y=YPAN(J)
Z=ZPAN(J)
DO 93 N=1,NKX
CXX=AKX(N)*X
S=SIN(CXX)
C=COS(CXX)
93 CX(N)=CMPLX(C,-S)
DO 94 M=1,NKY
CYY=AKY(M)*Y
S=SIN(CYY)
C=COS(CYY)
CY=CMPLX(C,-S)
DO 94 N=1,NKX
ARGZ=AKZ(N,M)*Z
DEP=EXP(-ARGZ)
CXY1=CX(N)*CY*DEP
CXYS=CX(N)*DEP*CONJG(CY)
A(N,M)=A(N,M)+STAR*CXY1
AS(N,M)=AS(N,M)+STAR*CXYS
C B(N,M) AND BS(N,M) INCREMENTED NEGLECTING CHANGES IN
C SOURCE STRENGTHS
C OVER TIME INTERVAL ARE SECOND ORDER IN DT
B(N,M)=B(N,M)-STAR2*CXY1*SS(N,M)
BS(N,M)=BS(N,M)-STAR2*CXYS*SS(N,M)
94 CONTINUE
1500 CONTINUE
RETURN
END

```

```

SUBROUTINE POTB
C *** Pressures induced by time rate of change of source
C strength of panel J in space fixed coordinates
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120), ST(120),
* ACN(120), ACNW(120), AN(120,3), E(120), P(120,6), PRFS(120),
* STOLD(120), PX(120,6)
COMMON/A/NPAN, NPT, GEE, RHOP, NKX, NKY, EYE, DT, TIM, UFWD
COMMON/SOURCE/EE(120,120), BPRES(120)
COMPLEX EYE
DIMENSION PP(6)
PP(1)=0.00
PP(2)=0.00
PP(3)=0.00
PP(4)=0.00
PP(5)=0.00
PP(6)=0.00
DO 1500 J=1, NPAN
C ST(J) IS TIME RATE OF CHANGE IN HULL FIXED SYSTEM OF SOURCE STRENGTH
C OF PANEL J
C STAV IS AVERAGE SOURCE STRENGTH OVER TIME STEP AT CENTER OF PANEL J
STAV=STOLD(J)+0.5*DT*ST(J)
C TIME DERIVATIVE IN SPACE FIXED SYSTEM
DO 1200 K=1,6
1200 CONTINUE
1500 CONTINUE
PP(K)=PP(K)+(ST(J)*P(J,K)-STAV*UFWD*PX(J,K))*RHOP
DO 1800 J=1, NPAN
BPRES(J)=(ST(J)*P(J,3)-STAV*UFWD*PX(J,3))*RHOP
DENOM=AN(J,3)*AREA(J)
IF(DENOM.EQ.0.) THEN
BPRES(J)=0.
ELSE
BPRES(J)=BPRES(J)/DENOM
ENDIF
1800 CONTINUE
TYPE *, ' BODY INDUCED PRESSURES '
TYPE 1070, (BPRES(J), J=1, NPAN)
1070 FORMAT(1X, 5F16.8)
TYPE *
TYPE 1870
1870 FORMAT(' BODY INDUCED FORCES=' )
TYPE 2020, PP(1), PP(2), PP(3), PP(4), PP(5), PP(6)
2020 FORMAT(3X, 3F20.6)
RETURN
END

```

```

C
C *** SUBROUTINE PRFR(PT)
      Computes forces and moments
      COMMON/BD/XPAN(120),YPAN(120),ZPAN(120),AREA(120),ST(120),
      * ACN(120),ACNW(120),AN(120,3),E(120),P(120,6),PRFS(120),
      * STOLD(120),PX(120,6)
      COMMON/A/NPAN,NPT,GEE,RHOP,NKX,NKY,EYE,DT,TIM,UFWD
      DIMENSION PF(6)
      COMMON/SOURCE/EE(120,120),BPRES(120)
      COMMON/MASS/RHO,G,GAMMA,DISPL,SMASS,XCG,YCG,ZCG
      COMMON / / MSTA,NPROF,NFWD,NAFT,LPTS(25),
      * XOFF(25),YOFF(25,25),ZOFF(25,25),
      * XPROF(51),YPROF(51)
      COMPLEX EYE
      COMMON/DISPLAY/TARE(6)
      X1=0.0
      X2=0.0
      X3=0.0
      X4=0.0
      X5=0.0
      X6=0.0
      XCGAM=XCG-(.5*(XOFF(1)-XOFF(MSTA)))
      DO 725 J=1,NPAN
      TPRES=BPRES(J)+PRFS(J)
      PRS=TPRES(J)*AREA(J)
      FRX=-AN(J,1)*PRS
      FRY=-AN(J,2)*PRS
      FRZ=-AN(J,3)*PRS
      XF=XPAN(J)+XCGAM
      YF=YPAN(J)
      ZF=ZPAN(J)+TARE(2)
      X1=X1+FRX
      X2=X2+FRY
      X3=X3+FRZ
      X4=X4+YF*FRZ-ZF*FRY
      X5=X5+ZF*FRX-XF*FRZ
      X6=X6+XF*FRY-YF*FRX
725 CONTINUE
      TYPE 80
      FORMAT(' PRESSURE INDUCED FORCES---')
      TYPE 40,X1,X2,X3,X4,X5,X6
      PT(1)=X1
      PT(2)=X2
      PT(3)=X3
      PT(4)=X4
      PT(5)=X5
      PT(6)=X6
40 FORMAT(5X,3F15.7)
      RETURN
      END

```

```

SUBROUTINE ZBLACN(ACBX, ACBY, ACBZ, ACBRL, ACBP, ACBYW)
C *** Computes source strengths satisfying body b. c.
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120), ST(120)
*, ACN(120), ACNW(120), AN(120, 3), E(120), P(120, 6), PRFS(120),
* STOLD(120), PX(120, 6)
COMMON/A/NPAN, NPT, GEE, RHOP, NKX, NKY, EYE, DT, TIM, UFWD
COMMON/SOURCE/EE(120, 120), BPRES(120)
COMPLEX EYE
DO 1800 J=1, NPAN
X=XPAN(J)
Y=YPAN(J)
Z=ZPAN(J)
ACX=ACBX+ACBP*Z-ACBYW*Y
ACY=ACBY+ACBYW*X-ACBRL*Z
ACZ=ACBZ+ACBRL*Y-ACBP*X
C *** Dot product of body acc. + nrml free-surface included
C normal acceleration
ACN(J)=-ACNW(J)+ACX*AN(J, 1)+ACY*AN(J, 2)+ACZ*AN(J, 3)
C *** Integrate time derivative for total source strenght
C at start of time step
STOLD(J)=STOLD(J)+DT*ST(J)
ST(J)=0.00
1800 CONTINUE
DO 1500 J=1, NPAN
AC=ACN(J)
DO 1400 K=1, NPAN
ST(K)=ST(K)+EE(J, K)*AC
1400 CONTINUE
1500 CONTINUE
RETURN
END

```

```

C *** CARD TYPE D
C 30 N=1
MSTA=MSTA+1
IF (MSTA.GT.25) CALL ERROR(10, IDUM, RDUM)
READ (OFF, 416) STATNO, Y11, Z1, JTEST
WRITE(ECHO, 417) STATNO, Y11, Z1, JTEST
XOFF(MSTA)=STATNO*SPACE
GO TO 50
40 CONTINUE !loop within each station
N=N+1
IF (N.GT.25) CALL ERROR(11, MSTA, RDUM)
READ (OFF, 416) S, Y11, Z1, JTEST
WRITE(ECHO, 417) S, Y11, Z1, JTEST
IF (S.NE.STATNO) CALL ERROR(12, MSTA, RDUM)
50 YOFF(N, MSTA)=Z1*ZSCAL
ZOFF(N, MSTA)=Y11*YSCAL
IF (JTEST.EQ.0 .OR. JTEST.EQ.77777) GO TO 40
LPTS(MSTA)=N !No. of points- MSTA
IF (N.LT.2) CALL ERROR(13, MSTA, RDUM)
IF (JTEST.EQ.88888) GO TO 30 !Go onto next station
IF (JTEST.NE.99999) CALL ERROR(14, JTEST, RDUM)

```

---

Section 3.0 READ COEFFICIENT FILE

```

C READ (COF) MSTA !Number of Stations
C READ (COF) (QM(I), I=1, 12) !Frequency (rad/sec)
C READ (COF) (DRAFT1(I), I=1, 6) !DRAFTS
DO 300 L=1, MSTA !Station index
DO 300 K=1, 6 !DRAFT index
DO 300 J=1, 12 !Frequency index
300 READ(COF) (COEFF4(K, J, I, L), I=1, 8)
CONTINUE
READ (COF) ((AREAN(L, K), K=1, 6), L=1, MSTA) !Section areas
C DO 688 L=1, MSTA
C TYPE 687, (AREAN(L, K), K=1, 6)
C688 CONTINUE
DO 675 J=1, 6
TARE(J)=POSIT(J)
675 CONTINUE
687 FORMAT(1X, 6F10.2)

```

---

Section 4.0 READ 3D ARRAY FILE [MAT]

```

C IF(MATFIL.EQ.'NONE') RETURN
C READ(MAT) NPT, NPAN
DO 500 I=1, 3
500 READ(MAT) (AN(J, I), J=1, NPAN)
CONTINUE
READ(MAT) (XPAN(J), J=1, NPAN)
READ(MAT) (YPAN(J), J=1, NPAN)
READ(MAT) (ZPAN(J), J=1, NPAN)
READ(MAT) (AREAP(J), J=1, NPAN)
DO 505 K=1, 6
505 READ(MAT) (PX(JL, K), JL=1, NPAN)
CONTINUE
DO 510 J=1, NPAN
510 READ(MAT) (EE(J, K), K=1, NPAN)
CONTINUE
DO 520 K=1, 6
520 READ(MAT) (P(J, K), J=1, NPAN)
CONTINUE
C CLOSE(BIF)
CLOSE(OFF)
CLOSE(COF)
CLOSE(MAT)
CLOSE(4)
RETURN
180 FORMAT (5X, I5)
190 FORMAT (I5)
197 FORMAT(1H1/, 81(1H*)//, ' INPUT DATA ECHO ', T64,

```

```

*PROGRAM SSX'//,B1(1H*)//,33(1H-),
*[BIF] DATA FILE',32(1H-)/)
198 FORMAT(1H1//,B1(1H*)//, ' INPUT DATA ECHO ',T64,
*PROGRAM SSX'//,B1(1H*)//,33(1H-),
*[OFF] DATA FILE',32(1H-)/)
196 FORMAT(1X,A)
199 FORMAT(A)
200 FORMAT (6F10.2)
201 FORMAT (3F10.2,I5)
202 FORMAT (8F10.0)
203 FORMAT (I2)
210 FORMAT (F10.2, I5, 5X, F10.2)

```

```

C
410 FORMAT (A)
412 FORMAT (4F10.3,13X,I2,4X,I1)
414 FORMAT (5X,I5,5X,'INPUT OF SHCP TYPE D OFFSET DATA')
416 FORMAT (F6.3,2F7.0,I6)
417 FORMAT (F7.3,2F10.2,I6)
420 FORMAT (2I5,F10.2)
430 FORMAT (2F10.2)
C
END

```

SUBROUTINE FINI

```

C
COMMON/IO/INPUT,OUTPUT,BIF,OFF,COF
INTEGER OUTPUT,BIF,OFF,COF
COMMON /STATS / NRHS(4), DELTA, HSUM, HMIN, HMAX, TTO, TT1
COMMON /TIME / TIME, RAMP
COMMON /TIME / TO, TSTART, TSTOP, TOUTPT, TSTEP, ERR
COMMON/SUMMARY/ NT,YYBAR(10),YYRMS(10),YYMAX(10),YYMIN(10)
IF (TT1.LE.TTO) GO TO 100
DELTA=TT1-TTO
NRHS(1)=NRHS(1)+1
HMIN=AMIN1(DELTA,HMIN)
HMAX=AMAX1(DELTA,HMAX)
100 HSUM=HSUM/FLOAT(NRHS(1))
C *** Print execution statistics
WRITE (OUTPUT,110) NRHS,DELTA,HMIN,HSUM,HMAX,TSTEP,ERR
C *** Compute response statistics
DO 200 I=1,10
YYBAR(I)=YYBAR(I)/FLOAT(NT)
ARGI=(YYRMS(I)/FLOAT(NT)-YYBAR(I)**2)
IF(ARGI.LT.0) YYRMS(I)=0.
IF(ARGI.GE.0) YYRMS(I)=SQRT(ARGI)
C
YYRMS(I)=SQRT(YYRMS(I)/NT-YYBAR(I)**2)
200 CONTINUE
C *** Print response statistics
WRITE(OUTPUT,201) ' RMS ',YYRMS
WRITE(OUTPUT,201) ' MEAN ',YYBAR
WRITE(OUTPUT,201) ' MAX ',YYMAX
WRITE(OUTPUT,201) ' MIN ',YYMIN
C
RETURN
C
110 FORMAT (76H0*** TIME INCREMENT : FWD NO STE
$P BACK TOTAL/
$ 36H *** NUMBER OF COMPUTATIONS OF RHS : ,4I10/36H *** FINAL V
1ALUE OF TIME INCREMENT : ,1PG13.4/38H *** MINIMUM VALUE OF TIME INC
2REMENT : ,G13.4/38H *** AVERAGE VALUE OF TIME INCREMENT : ,G13.4/38H
3 *** MAXIMUM VALUE OF TIME INCREMENT : ,G13.4/42H *** FLAG VALUE, F
4I, (PLUS 1.0 EXPECTED) : ,G13.4/33H *** FINAL VALUE OF ERROR LIMIT
5: ,G13.4)
C
C
201 FORMAT(A8,4X,4(1X,F11.4),3F12.4,2X,E15.5,2X,F12.4)
C
END

```

## SUBROUTINE PREPARE

```

COMMON /C / C(3,3)
COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF
INTEGER OUTPUT, BIF, OFF, COF
COMMON / / NWAVES, WVSUM,
1 3 4 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
WN(20), WNX(20), WNZ(20),
CSK(20), CCK(20), CXK(20),
2 4 CYK(20), CZK(20), XW(20)
COMMON / / MSTA, NPROF, NFWD, NAFTA, LPTS(25),
1 2 XOFF(25), YOFF(25,25), ZOFF(25,25),
XPROF(51), YPROF(51)
COMMON/COEFF4/COEFF4(6,12,8,25),AREAN(25,6)
COMMON /FORCE / XF, YF, ZF, XM, YM, ZM
REAL FORCE(6)
EQUIVALENCE (FORCE(1),XF)
COMMON /LHS / Y(13)
COMMON /MASS / RHO, G, GAMMA
COMMON /MASS / DISPL, SMASS, XCG, YCG, ZCG,
1 AMX, AMY, AMZ, RADII(6)
COMMON /MASS / PMI(3,3)
COMMON /RESIST/ SPEED
COMMON /TIME / TIME, RAMP
COMMON /TIME / TO, TSTART, TSTOP, TOUTPT, TSTEP, ERR
COMMON /V / VX, VY, VZ
COMMON/PREXIN/TPSTART, TPSTOP, TPRAMP, BGX, SMX, BGY, SMY, TSCALE
REAL XAFT(25), YAFT(25), XFWD(25), YFWD(25)
EQUIVALENCE (XFWD(1),XPROF(1)), (YFWD(1),YPROF(1))
EQUIVALENCE (XAFT(1),XPROF(26)), (YAFT(1),YPROF(26))
EQUIVALENCE (XXFWD,XPROF(51)), (XXAFT,YPROF(51))
COMMON/SXPROP/ SEGMAS(26),SEGMOX(26),STRMAS(26),STRMOM(26),
* STRMOX(26),XBAR(26),YBAR(26),
* SEGWT(26),NWTSTA
DATA MAXSTA/25/
DATA MAXPTS/25/
DATA AMX/O./,AMY/O./,AMZ/O./
DATA RAD /O.01745 32925 19943/
GAMMA=RHO*G
SMASS=DISPL/G
DISPL=DISPL/GAMMA
RAMP=TSTART-TO
AMX=1.0+AMX
AMY=1.0+AMY
AMZ=1.0+AMZ

```

```

C *** From given offsets on port side, assign symmetrical
C offsets on starboard side
DO 10 J=1,MSTA
  N=LPTS(J)
  NP=N
  DO 50 I=1,N
    NP=NP+1
    NM=N-I+1
    YOFF(NP,J)=YOFF(NM,J)
    ZOFF(NP,J)=-ZOFF(NM,J)
50  CONTINUE
60  IF (ZOFF(NP,J).NE.ZOFF(1,J)) GO TO 70
    IF (YOFF(NP,J).EQ.YOFF(1,J)) GO TO 80
70  NP=NP+1
    YOFF(NP,J)=YOFF(1,J)
    ZOFF(NP,J)=ZOFF(1,J)
80  IF (NP.GT.MAXPTS) GO TO 90
    IF (NP.GT.1) GO TO 20
90  CONTINUE
STOP 'ERROR'
20  LPTS(J)=NP
10  CONTINUE
C *** Recompute offset coordinates relative to the center of mass
DO 130 J=1,MSTA
  XOFF(J)=-XOFF(J)-XCG
  N=LPTS(J)
  DO 120 I=1,N
    YOFF(I,J)=YOFF(I,J)-YCG
    ZOFF(I,J)=ZOFF(I,J)-ZCG
120  CONTINUE
130  CONTINUE

```

```

C *** Consolidate fore & aft profile coordinates into a single
C pair of arrays
NPROF=NFW
IF (NFW.EQ.0) GO TO 150
X=XOFF(1)
DO 140 I=1,NFW
  XFWD(I)=XFWD(I)+X
  YFWD(I)=YFWD(I)-YCG
140 CONTINUE
IF (NFW.NE.1) GO TO 150
NPROF=0
XXFWD=XFWD(1)
150 IF (NAFT.EQ.0) GO TO 190
X=XOFF(MSTA)
  XXAFT=X-XAFT(1)
  IF (NAFT.EQ.1) GO TO 190
  J=1
  IF (NPROF.EQ.0) GO TO 160
  IF (XXAFT.NE.XFWD(NPROF)) GO TO 160
  IF (YPROF(NPROF).EQ.(YAFT(1)-YCG)) J=2
160 DO 170 I=J,NAFT
  NPROF=NPROF+1
  XPROF(NPROF)=X-XAFT(I)
  YPROF(NPROF)=YAFT(I)-YCG
170 CONTINUE
  IF (XFWD(1).NE.XFWD(NPROF)) GO TO 180
  IF (YFWD(1).EQ.YFWD(NPROF)) GO TO 190
180 NPROF=NPROF+1
  XFWD(NPROF)=XFWD(1)
  YFWD(NPROF)=YFWD(1)
190 CONTINUE

```

```

C *** This section loads certain strip-associated mass and inertial
C arrays using mass/inertial data from segment-associated arrays
C *** Load strip & segment associated inertial arrays
DO 106 I=1,NWTSTA
  SEGMAS(I)=SEGWT(I)/G
106 CONTINUE
VX=SPEED

```

```

C *** Load the inertia matrix
PMI(1,1)=SMASS*RADII(1)**2
PMI(2,2)=SMASS*RADII(2)**2
PMI(3,3)=SMASS*RADII(3)**2
PMI(2,1)=-ABS(RADII(4))*RADII(4)*SMASS
PMI(1,2)=PMI(2,1)
PMI(3,1)=-ABS(RADII(5))*RADII(5)*SMASS
PMI(1,3)=PMI(3,1)
PMI(2,3)=-ABS(RADII(6))*RADII(6)*SMASS
PMI(3,2)=PMI(2,3)

```

```

C *** Compute wave no's and max wave amplitude
WVSUM=0.0
IF (NWAVES.EQ.0) GO TO 260
DO 250 K=1,NWAVES
  WVPHA(K)=RAD*WVPHA(K)
  WN(K)=WVFRE(K)**2/G
  ARG=WVDIR(K)*RAD
  WNX(K)=WN(K)*COS(ARG)
  WNZ(K)=WN(K)*SIN(ARG)
  WVSUM=WVSUM+ABS(WVAMP(K))
250 CONTINUE
260 CONTINUE

```

```

C *** Convert degrees to radians
DO 270 I=4,6
  Y(I)=Y(I)*RAD
  Y(I+6)=Y(I+6)*RAD
270 CONTINUE

```

```

C *** Convert initial linear velocities from ship coordinates
C fixed coordinates.
CALL ROTATE (Y)
VX=Y(7)
VY=Y(8)
VZ=Y(9)
Y(7)=C(1,1)*VX+C(2,1)*VY+C(3,1)*VZ
Y(8)=C(1,2)*VX+C(2,2)*VY+C(3,2)*VZ
Y(9)=C(1,3)*VX+C(2,3)*VY+C(3,3)*VZ

```



```
C *** Divide all 2-D hydro. coeffs by RHO
DO 110 L=1,MSTA
DO 110 K=1,6
DO 110 J=1,12
DO 110 I=1,8
CDEFF4(K, J, I, L)=CDEFF4(K, J, I, L)/RHO
110 CONTINUE
C *** Initialize the free surface for special pressure
C computations
C IF(TPSTOP.GT.5.) CALL AFSR
C RETURN
C END
```

```

SUBROUTINE COFFEE
CCC
COMMON/MASS/RHO, G, GAMMA
COMMON/MASS/DISPL, SMASS, XCG, YCG, ZCG,
* COMMON/LHS/Y(13)
C5 WAVE PROPERTIES..
COMMON / / NWAVES, WVSUM,
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
2 WN(20), WNX(20), WNZ(20),
3 CSK(20), CCK(20), CXK(20),
4 CYK(20), CZK(20), XW(20)

C5
C5
C5
TABLE OF OFFSETS..
COMMON / / MSTA, NPROF, NFWD, NAFTA, LPTS(25),
1 XOFF(25), YOFF(25,25), ZOFF(25,25),
2 XPROF(51), YPROF(51)
COMMON/RESIST/SPEED
COMMON/SIGMA / NK, SIGMA(24), SIGMAO, ERRO, OM(12)

C
C
COMMON/COEFF4/COEFF4(6, 12, 8, 25), AREAN(25, 6)
COMMON/COEFFX/THAH(25, 21), TSAS(25, 21), TRAR(25, 21), TCCA(25, 21),
* THVH(25, 21), TSVS(25, 21), TRVR(25, 21), TCCV(25, 21),
* THACX(25, 21), THVCX(25, 21), TSACX(25, 21), TSVCX(25, 21),
* TRACX(25, 21), TCACX(25, 21), THVCY(25, 21),
* ITP(25), TXP(25), IFP(21), FXP(21)
COMMON/OPTION/JOBCO, JOBFO(10)
COMMON/SXOMEG/OMEGAX(3)
COMMON/DRFT1/ DRAFT1(6)

C
DIMENSION LIX(21), FMUX(21)
DATA DEGREE/0.01745 32925/

C
CCC
CCC
CCC
CCC
CCC
CCC
(Future revision here... no trim now, but in
next revision, the options should include
specifying trim [Y(6)] in addition to draft
as an initial condition)

TAVG=YCG-Y(2) !Draft
TYPE *, ' TAVG=', TAVG
DO 20 JSTA=1, MSTA !Loop over stations

CALL TXIT(TAVG, IT, TX) !Get index & mult.
ITP(JSTA)=IT
TXP(JSTA)=TX
TYPE *, 'IT, TX', IT, TX

CCC
CCC
DO 10 K=1, NWAVES
FREX=ABS((1.0-COS(WVDIR(K)*DEGREE)*SPEED
*WVFRE(K)/G)*WVFRE(K)) !Encounter freq.
* TYPE *, 'Encounter freq=', FREX

C
C-----
C
NOTE: We are using the wave frequency here-- not the
frequency of encounter.

```

C  
C  
C  
C  
FREX=WVFRE(K)

CALL FXIF(FREX, IF, FX)

!Get index & mult.

IFP(K)=IF  
FXP(K)=FX  
THAH(JSTA, K)=COX(IT, TX, IF, FX, 1, JSTA)  
TSAS(JSTA, K)=COX(IT, TX, IF, FX, 2, JSTA)  
TRAR(JSTA, K)=COX(IT, TX, IF, FX, 3, JSTA)  
TCCA(JSTA, K)=COX(IT, TX, IF, FX, 4, JSTA)

THVH(JSTA, K)=COX(IT, TX, IF, FX, 5, JSTA)  
TSVS(JSTA, K)=COX(IT, TX, IF, FX, 6, JSTA)  
TRVR(JSTA, K)=COX(IT, TX, IF, FX, 7, JSTA)  
TCCV(JSTA, K)=COX(IT, TX, IF, FX, 8, JSTA)

Spatial derivatives in x-direction

THACX(JSTA, K)=COXDXP(IT, TX, IF, FX, 1, JSTA)  
TSACX(JSTA, K)=COXDXP(IT, TX, IF, FX, 2, JSTA)  
TRACX(JSTA, K)=COXDXP(IT, TX, IF, FX, 3, JSTA)  
TCACX(JSTA, K)=COXDXP(IT, TX, IF, FX, 4, JSTA)

THVCX(JSTA, K)=COXDXP(IT, TX, IF, FX, 5, JSTA)  
TSVCX(JSTA, K)=COXDXP(IT, TX, IF, FX, 6, JSTA)  
TRVCX(JSTA, K)=COXDXP(IT, TX, IF, FX, 7, JSTA)  
TCVCX(JSTA, K)=COXDXP(IT, TX, IF, FX, 8, JSTA)

CONTINUE  
CONTINUE

For ship motion related hydro coef's, we will  
us, as a default frequency the frequency associated  
with the first wave [wvfre(1)].

Future revision: the default frequencies should  
be the natural period in heave, sway, and roll.

\* FRSI=ABS((1.0-COS(WVDIR(1)\*DEGREE)\*SPEED  
\*WVFRE(1)/G)\*WVFRE(1)) !Encounter freq.  
OMEGAX(1)=FRSI  
OMEGAX(2)=FRSI  
OMEGAX(3)=FRSI

CALL FXIF(FRSI, IF, FX)

DD 30 JSTA=1, MSTA

FXP(21)=FX  
IFP(21)=IF  
TXP(21)=TX  
ITP(21)=IT

THAH(JSTA, 21)=COX(IT, TX, IF, FX, 1, JSTA)  
TSAS(JSTA, 21)=COX(IT, TX, IF, FX, 2, JSTA)  
TRAR(JSTA, 21)=COX(IT, TX, IF, FX, 3, JSTA)

```

C      TCCA(JSTA, 21)=COX(IT, TX, IF, FX, 4, JSTA)
C
C      THVH(JSTA, 21)=COX(IT, TX, IF, FX, 5, JSTA)
C      TSVS(JSTA, 21)=COX(IT, TX, IF, FX, 6, JSTA)
C      TRVR(JSTA, 21)=COX(IT, TX, IF, FX, 7, JSTA)
C      TCCV(JSTA, 21)=COX(IT, TX, IF, FX, 8, JSTA)
C
C      THACX(JSTA, 21)=COXDXP(IT, TX, IF, FX, 1, JSTA)
C      TSACX(JSTA, 21)=COXDXP(IT, TX, IF, FX, 2, JSTA)
C      THVCX(JSTA, 21)=COXDXP(IT, TX, IF, FX, 5, JSTA)
C      TSVCX(JSTA, 21)=COXDXP(IT, TX, IF, FX, 6, JSTA)
C
30    CONTINUE
      RETURN
      END
      FUNCTION COX(IT, TX, IF, FX, ITYPE, JSTA)
CCC
CCC      This function subprogram returns the 2-D hydrodynamic
CCC      coefficient of a particular TYPE (1 thru 8 where 1 is
CCC      HAH, 2 is SAS, and so on) keyed on ITYPE; for a par-
CCC      ticular station, keyed on ISTA. Provided to the function
CCC      is the draft index (IT) and draft interpolation multi-
CCC      plier, the frequency index (IF) and the frequency inter-
CCC      polation multiplier (FX).
CCC
CCC      This function performs a two-dim linear interpolation
CCC      using four values from the big 2-D coefficient array
CCC      COEFF4( , , , )...
CCC
CCC      COMMON /COEFF4/ COEFF4(6, 12, 8, 25), AREAN(25, 6)
CCC
CCC      A1=COEFF4(IT , IF , ITYPE, JSTA)
CCC      A2=COEFF4(IT+1, IF , ITYPE, JSTA)
CCC      A3=COEFF4(IT , IF+1, ITYPE, JSTA)
CCC      A4=COEFF4(IT+1, IF+1, ITYPE, JSTA)
CCC
CCC      B12=A1+(TX*(A2-A1))
CCC      B34=A3+(TX*(A4-A3))
CCC
CCC      COX=B12+(FX*(B34-B12))
CCC
CCC      RETURN
CCC      END
CCC      SUBROUTINE TXIT(TAVG, IT, TX)
CCC
CCC      This routine determines the draft index (IT)
CCC      and draft interpolation multiplier (TX) given
CCC      a value for draft (TAVG).
CCC
CCC      COMMON/DRFT1/ DRAFT1(6)
CCC
CCC      IF (TAVG.GT. DRAFT1(6)) THEN          !Section immersed
CCC          IT=5
CCC          TX=1.0
CCC      ELSE IF (TAVG.LT. DRAFT1(1)) THEN    !Emerged section
CCC          IT=1
CCC          TX=0.0
CCC      ELSE
CCC          DO 10 JJ=2, 6
CCC          IF(TAVG-DRAFT1(JJ)) 12, 11, 10
CCC          CONTINUE
11

```

```

TX=0.0
IT=JJ
GO TO 13
CONTINUE
12 TX=(TAVG-DRAFT1(JJ-1))/(DRAFT1(JJ)-DRAFT1(JJ-1))
IT=JJ-1
GO TO 13
CONTINUE
10 CONTINUE
13 CONTINUE

```

```

END IF
RETURN
END
SUBROUTINE FXIF(FREX, IF, FX)

```

```

CCC
CCC This routine determines the freq. index (IF)
CCC and freq. interpolation multiplier (FX) given
CCC a value for frequency (FREX).
CCC
COMMON /SIGMA/ NK, SIGMA(24), SIGMAO, ERRO, DM(12)

```

```

CCC
IF (FREX.GT.DM(11)) THEN
IF=11
FX=1.0
ELSE
DO 10 JJ=2,12
11 IF(FREX-DM(JJ)) 12,11,10
CONTINUE
FX=0.0
IF=JJ
12 GO TO 13
CONTINUE
FX=(FREX-DM(JJ-1))/(DM(JJ)-DM(JJ-1))
IF=JJ-1
10 GO TO 13
CONTINUE
13 CONTINUE

```

```

END IF
RETURN
END
FUNCTION COXDX(IT, TX, IF, FX, L, J)

```

```

C
COMMON / / NWAVES, WVSUM,
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
3 WN(20), WNX(20), WNZ(20),
4 CSK(20), CCK(20), CXK(20),
COMMON / / CYK(20), CZK(20), XW(20)
1 COMMON / / MSTA, NPROF, NFWD, NAFT, LPTS(25),
2 XOFF(25), YOFF(25,25), ZOFF(25,25),
XPROF(51), YPROF(51)
COMMON/OPTION/JOBCO, JOBFO(10)
COMMON/ETA/YC, XK(20)
COMMON/TEMPSTA/DXFWD, DXAFT, TSTA(25)

```

```

CCC
C-----
C
IF(J.EG.1) THEN
CF=0.
ELSE
CF=COX(IT, TX, IF, FX, L, J-1)
AF=ARX(IT, TX, J-1)

```

CF=CF\*AF  
END IF

CM=COX(IT, TX, IF, FX, L, J)  
AM=ARX(IT, TX, J)  
CM=CM\*AM

IF(J.EQ.MSTA) THEN  
CA=0.  
ELSE  
CA=COX(IT, TX, IF, FX, L, J+1)  
AA=ARX(IT, TX, J+1)  
CA=CA\*AA  
END IF

C1=(CM-CF)/DXFWD  
C2=(CA-CM)/DXAFT

COXDX=(.5/AM)\*(C1+C2)

RETURN  
END

FUNCTION COXDXP(IT, TX, IF, FX, L, J)

COMMON / / NWAVES, WVSUM,  
WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),  
WN(20), WNX(20), WNZ(20),  
CSK(20), CCK(20), CXK(20),  
CYK(20), CZK(20), XW(20)  
COMMON / / MSTA, NPROF, NFWD, NAFTA, LPTS(25),  
XOFF(25), YOFF(25,25), ZOFF(25,25),  
XPROF(51), YPROF(51)

IF(J.EQ.1) THEN  
CF=0.  
DXFWD=XOFF(J+1)-XOFF(J)  
ELSE  
CF=COX(IT, TX, IF, FX, L, J-1)  
AF=ARX(IT, TX, J-1)  
CF=CF\*AF  
DXFWD=XOFF(J)-XOFF(J-1)  
END IF

CM=COX(IT, TX, IF, FX, L, J)  
AM=ARX(IT, TX, J)  
CM=CM\*AM

IF(J.EQ.MSTA) THEN  
CA=0.  
DXAFT=XOFF(J)-XOFF(J-1)  
ELSE  
CA=COX(IT, TX, IF, FX, L, J+1)  
AA=ARX(IT, TX, J+1)  
CA=CA\*AA  
DXAFT=XOFF(J+1)-XOFF(J)  
END IF

C1=(CM-CF)/DXFWD

```
C      C2=(CA-CM)/DXAFT
C      COXDXP=(.5/AM)*(C1+C2)
      TDX=DXFWD+DXAFT
      DCX=CA-CF
      COXDXP=(DCX/TDX)/AM
C
      RETURN
      END
      FUNCTION ARX(IT, TX, J)
CCC     COMMON /COEFF4/ COEFF4(6, 12, 8, 25), AREAN(25, 6)
CCC
CCC     B2=AREAN(J, IT+1)
CCC     B1=AREAN(J, IT)
CCC     ARX=B1+(TX*(B2-B1))
CCC
      RETURN
      END
```

FUNCTION ETABAR(I)

ETABAR RETURNS THE WAVE ELEVATION (ABSOLUTE COORDINATES) ABOVE  
A POINT IN THE SHIP COORDINATE SYSTEM.

POINT IS ON THE SHIP CENTERPLANE.  
THE COORDINATES OF THE POINT ON THE SHIP ARE..  
( XPROF(I), YPROF(I), -ZCG )

PHYSICAL CONSTANTS..  
COMMON /MASS / RHO, G, GAMMA

SHIP MASS PARAMETERS..  
COMMON /MASS / DISPL, SMASS, XCG, YCG, ZCG,  
1 AMX, AMY, AMZ, RADII(6)

ROTATIONAL INERTIA MATRIX FOR FIXED MASS..  
COMMON /MASS / PMI(3,3)

COMMON /TRIG / CTHETA, STHETA, CPHI, SPHI, CPSI, SPSI

WAVE PROPERTIES..  
COMMON / / NWAVES, WVSUM,  
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),  
2 WN(20), WNX(20), WNZ(20),  
3 CSK(20), CCK(20), CXK(20),  
4 CYK(20), CZK(20), XW(20)

TABLE OF OFFSETS..  
COMMON / / MSTA, NPROF, NFWD, NAFTA, LPTS(25),  
1 XOFF(25), YOFF(25,25), ZOFF(25,25),  
2 XPROF(51), YPROF(51)

ETABAR=0.0  
IF (NWAVES.EQ.0) RETURN  
X1=XPROF(I)  
Y1=YPROF(I)\*CTHETA+ZCG\*STHETA  
Z1=YPROF(I)\*STHETA-ZCG\*CTHETA  
DO 110 K=1,NWAVES  
ARG=(X1-XW(K))\*CXK(K)+Y1\*CYK(K)+Z1\*CZK(K)  
ETABAR=COS(ARG)\*WVAMP(K)+ETABAR

110 CONTINUE  
RETURN

END  
FUNCTION ETAF (ZT)

FIND COORDINATE OF WAVE SURFACE.

WAVE PROPERTIES..  
COMMON / / NWAVES, WVSUM,  
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),  
2 WN(20), WNX(20), WNZ(20),  
3 CSK(20), CCK(20), CXK(20),  
4 CYK(20), CZK(20), XW(20)



```

C5 DATA FOR THE COMPUTATION OF THE WAVE SURFACE ELAVATION.
COMMON /ETA / YC, XK(20)
C5 COMMON /CT / CT(3,3)
C3 A=-YC
B=CT(2,2)
IF (NWAVER.EQ.0) GO TO 120
DO 110 K=1,NWAVER
ARG=ZT*CZK(K)+XK(K)
A=COS(ARG)*WVAMP(K)+A
B=SIN(ARG)*WVAMP(K)*CYK(K)+B
110 CONTINUE
120 ETAF=A/B
RETURN
C
END
FUNCTION ETAY(J,K)
COMMON /TRIG / CTHETA, STHETA, CPHI, SPHI, CPSI, SPSI
COMMON / / NWAVES, WVSUM,
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
2 WN(20), WNX(20), WNZ(20),
3 CSK(20), CCK(20), CXK(20),
4 CYK(20), CZK(20), XW(20)
COMMON / / MSTA, NPROF, NFWO, NAFTA, LPTS(25),
1 XOFF(25), YOFF(25,25), ZOFF(25,25),
2 XPROF(51), YPROF(51)
ETAY=0.
IF (NWAVER.EQ.0) RETURN
X1=XOFF(J)
Y1=0.
Z1=0.
ARG=(X1-XW(K))*CXK(K)+Y1*CYK(K)+Z1*CZK(K)
ETAY=COS(ARG)*WVAMP(K)
RETURN
C
END

```

## SUBROUTINE FOLIO1

```

COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF
INTEGER OUTPUT, BIF, OFF, COF
COMMON /SIGMA / NK, SIGMA(24), SIGMAO, ERRO, OM(12)
COMMON/DRFT1/DRAFT1(6)
COMMON/IOFILE/ OFFIL, BIFIL, COFIL
COMMON/HEAD/TITLE
CHARACTER*30 TITLE
COMMON/COEFF4/COEFF4(6, 12, 8, 25), AREAN(25, 6)
COMMON / / NWAVES, WVSUM,
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
4 WN(20), WNX(20), WNZ(20),
CSK(20), CCK(20), CXK(20),
2 CYK(20), CZK(20), XW(20)
COMMON / / MSTA, NPROF, NFWD, NAFTA, LPTS(25),
1 XOFF(25), YOFF(25,25), ZOFF(25,25),
2 XPROF(51), YPROF(51)
COMMON /DAMP / DAMPL(6), DAMPG(6)
COMMON /LHS / Y(13)
COMMON /MASS / RHO, G, GAMMA
COMMON /MASS / DISPL, SMASS, XCG, YCG, ZCG,
1 AMX, AMY, AMZ, RADII(6)
COMMON /MASS / PMI(3,3)
COMMON /RESIST/ SPEED
COMMON/OPTION/JOBCO, JOBFO(10)
COMMON /TIME / TIME, RAMP
COMMON /TIME / TO, TSTART, TSTOP, TOUTPT, TSTEP, ERR
COMMON/SXPROP/ SEGMA(26), SEGMOX(26), STRMAS(26), STRMOM(26),
* STRMOX(26), XBAR(26), YBAR(26), SEGWT(26), NWTSTA
REAL POSIT(6), VELOC(6)
EQUIVALENCE (POSIT(1), Y(1)), (VELOC(1), Y(7))

```

```

WEIGHT=SMASS*G
WRITE(OUTPUT, 100) TITLE
WRITE(OUTPUT, 110) WEIGHT, RADII(1), RADII(2), XCG, RADII(3),
* YCG, RADII(4), ZCG, RADII(5), RADII(6)
WRITE(OUTPUT, 120) DAMPL(1), DAMPG(1), SPEED,
* (DAMPL(I), DAMPG(I), I=2, 6)
WRITE(OUTPUT, 130) (POSIT(I), VELOC(I), I=1, 6)
WRITE(OUTPUT, 140) TSTART, TSTOP, TOUTPT, TSTEP
WRITE(OUTPUT, 160) RHO, G
WRITE(OUTPUT, 165) JOBCO, (JOBFO(I), I=1, 3)
WRITE(OUTPUT, 100) TITLE
WRITE(OUTPUT, 150)
DO 10 I=1, NWAVES
10 WRITE(OUTPUT, 155) I, WVAMP(I), WVFRE(I), WVDIR(I), WVPHA(I)
CONTINUE
WRITE(OUTPUT, 170)
DO 20 I=1, NWTSTA
20 WRITE(OUTPUT, 175) I, SEGWT(I), SEGMOX(I), YBAR(I)
CONTINUE
WRITE(OUTPUT, 100) TITLE

```

```

100 FORMAT(1H1, 80(1H=) //, 1X, A, T55, 'PROGRAM SSX (Version 1.0) //,
*B1(1H=) //)
110 FORMAT(2B(1H-), 'Inertial Characteristics', 29(1H-)//,
* T48, 9(1H-), 'GYRADII', 8(1H-)//,
*T5, 'WEIGHT', F15. 2, 2X, 'L. TONS', T48, 'Kxx', 4X, F10. 3, 5X, 'ft' //,
* T48, 'Kyy', 4X, F10. 3, 5X, 'ft' //,
*T5, 'XCG', 10X, F10. 3, 4X, 'ft', T48, 'Kzz', 4X, F10. 3, 5X, 'ft' //,

```

```

*T5, 'YCG', 10X, F10. 3, 4X, 'ft', T48, 'Kxy', 4X, F10. 3, 5X, 'ft',
*T5, 'ZCG', 10X, F10. 3, 4X, 'ft', T48, 'Kxz', 4X, F10. 3, 5X, 'ft',
*
120 FORMAT(T5, 8(1H-), 'DAMPING COEFFICIENTS', 8(1H-), T48,
* '-----OPS DATA-----',
*13X, '(user specified)',
*T5, 'MOTION', 7X, 'LINEAR', 7X, 'QUADRATIC',
*T5, '-----', 4X, '-----', 5X, '-----',
*T5, 'SURGE', 2(5X, E10. 4), T48, 'SPEED', 5X, F6. 2/,
*T5, 'HEAVE', 2(5X, E10. 4)/,
*T5, 'SWAY', 2(5X, E10. 4)/,
*T5, 'ROLL', 2(5X, E10. 4)/,
*T5, 'YAW', 2(5X, E10. 4)/,
*T5, 'PITCH', 2(5X, E10. 4)/)
130 FORMAT(32(1H-), 'Initial Conditions', 31(1H-)/,
*T5, 11(1H-), 'POSITION', 11(1H-), T48, 10(1H-), 'VELOCITY', 10(1H-)/,
*T5, 'X(CG)', 11X, F10. 3, 2X, 'ft', T48, 'SPEED', 5X, F10. 3, 2X, 'ft/sec',
*T5, 'Y(CG)', 11X, F10. 3, 2X, 'ft', T48, 'HEAVE', 5X, F10. 3, 2X, 'ft/sec',
*T5, 'Z(CG)', 11X, F10. 3, 2X, 'ft', T48, 'SWAY', 5X, F10. 3, 2X, 'ft/sec',
*T5, 'ROLL', 11X, F10. 3, 1X, 'deg', T48, 'ROLL', 6X, F10. 3, 2X, 'deg/sec',
*T5, 'YAW', 11X, F10. 3, 1X, 'deg', T48, 'YAW', 7X, F10. 3, 2X, 'deg/sec',
*T5, 'PITCH', 11X, F10. 3, 1X, 'deg', T48, 'PITCH', 5X, F10. 3, 2X,
*'deg/sec')
140 FORMAT(34(1H-), 'Run Parameters', 33(1H-)/,
*T5, 'SIMULATION PARAMETERS', T48, 'JOB OPTIONS',
*T5, 21(1H-), T48, 11(1H-)/,
*T5, 'TIME(START)', 3X, F8. 2/,
*T5, 'TIME(END)', 5X, F8. 2/,
*T5, 'OUTPUT DT', 5X, F8. 2/,
*T5, 'INTEG. DT', 5X, F8. 2/, 81(1H-)/)
150 FORMAT(33(1H-), 'WAVE COMPONENTS', 33(1H-)/,
*T5, 'WAVE', 5X, 'AMPLITUDE', 5X, 'FREQUENCY', 5X, 'DIRECTION',
*5X, 'PHASE',
*T5, '-----', 5X, '-----', 5X, '-----', 5X, '-----',
*5X, '-----')
155 FORMAT(T7, I2, T16, F6. 2, T30, F6. 2, T44, F7. 2, T55, F7. 2)
160 FORMAT(/31(1H-), 'MISCELLANEOUS VALUES', 30(1H-)/,
*T5, 'RHO', 5X, F10. 6/, T5, 'G', 5X, F10. 6)
165 FORMAT(T5, 'JOBID', I2/, T5, 'JOBFO', 4I4/)
170 FORMAT(27(1H-), 'SECTIONAL WEIGHT PROPERTIES', 27(1H-)/,
*T5, 'SECTION', 5X, 'WEIGHT', 5X, 'Kxx', 5X, 'Centroid',
*T5, '-----', 5X, '-----', 5X, '-----', 5X, '-----')
175 FORMAT(T5, 3X, I2, T13, F10. 3, T26, F8. 2, T39, F8. 2)
RETURN
END

```

SUBROUTINE RKF (TIME, Y, TNEXT, ERR, NEQS, STEP)

RUNGE-KUTTA INTEGRATION FOR CAPSIZE.

REAL Y(1)  
REAL YY(13), YA(13), YB(13)

REAL YC(13)  
DATA YC(13)/0.0/

DATA YY(13)/0.0/  
DATA YA(13)/0.0/  
DATA YB(13)/0.0/

STEP=ABS(STEP)  
NSTEP=(TNEXT-TIME)/STEP+0.5  
IF (NSTEP.LT.1) NSTEP=1  
TSTEP=(TNEXT-TIME)/FLOAT(NSTEP)  
HALF=TSTEP/2.0  
FACT=TSTEP/6.0  
110 CALL DERIV (TIME, Y, YA)  
DO 120 I=1, NEQS  
    YY(I)=HALF\*YA(I)+Y(I)  
120 CONTINUE  
    TIME=HALF+TIME  
    CALL DERIV (TIME, YY, YB)  
    DO 130 I=1, NEQS  
        YA(I)=2.0\*YB(I)+YA(I)  
        YY(I)=HALF\*YB(I)+Y(I)  
130 CONTINUE  
    CALL DERIV (TIME, YY, YB)  
    DO 140 I=1, NEQS  
        YA(I)=2.0\*YB(I)+YA(I)  
        YY(I)=TSTEP\*YB(I)+Y(I)  
140 CONTINUE  
    TIME=HALF+TIME  
    CALL DERIV (TIME, YY, YB)  
    DO 150 I=1, NEQS  
        Y(I)=(YA(I)+YB(I))\*FACT+Y(I)  
150 CONTINUE

This call to DERIV is to get actual acceleration for the new TIME. By calling RHS this time we are loading the acceleration array ACC( ) in COMMON ACCEL with correct accelerations, rather than accel. computed as dictated by the fourth-order RUNGA-KUTTA scheme.

CALL DERIV(TIME, Y, YC)  
CALL GETFREQ

IF (TIME+HALF.LT.TNEXT) GO TO 110  
TIME=TNEXT  
RETURN  
END

SUBROUTINE DERIV (TT, YY, YYDOT)  
Evaluates derivatives in equations of motion

```

TT  -- TIME, THE INDEPENDENT VARIABLE
YY  -- DEPENDENT VARIABLE ARRAY.
      YY(1) = X-COORDINATE OF MASS CENTER
      YY(2) = Y-COORDINATE OF MASS CENTER
      YY(3) = Z-COORDINATE OF MASS CENTER
      YY(4) = THETA ROTATION (ROLL ANGLE)
      YY(5) = PHI  ROTATION (YAW ANGLE)
      YY(6) = PSI  ROTATION (PITCH ANGLE)
      YY(7) = X-COMPONENT OF LINEAR VELOCITY
      YY(8) = Y-COMPONENT OF LINEAR VELOCITY
      YY(9) = Z-COMPONENT OF LINEAR VELOCITY
      YY(10) = ANGULAR VELOCITY ABOUT X-AXIS (ROLL RATE)
      YY(11) = ANGULAR VELOCITY ABOUT Y-AXIS (YAW RATE)
      YY(12) = ANGULAR VELOCITY ABOUT Z-AXIS (PITCH RATE)
      YY(13) = TIME INTEGRAL OF YAW ANGLE (EVALUATED ONLY
              WHEN REQUIRED BY THE AUTOPILOT)

```

YYDOT -- DERIVATIVES OF YY.

TT AND YY ARE SUPPLIED TO THIS SUBROUTINE WHENEVER THE EQUATION SOLVING ROUTINE REQUIRES THE VALUES OF THE DERIVATIVE OF YY--YYDOT. YYDOT IS COMPUTED BY EVALUATING THE RIGHT HAND SIDE OF THE EQUATIONS OF MOTION FOR THE GIVEN TT AND YY AND THE PRE-DEFINED GEOMETRY OF THE SHIP AND EQUATIONS OF THE WAVES.

```

REAL YY(13), YYDOT(13)
COMMON /A / A(6,6)
COMMON /C / C(3,3)
COMMON /CT / CT(3,3)
COMMON /FORCE / XF, YF, ZF, XM, YM, ZM
REAL FORCE(6)
EQUIVALENCE (FORCE(1), XF)
COMMON /H / H(3)
COMMON /MASS / RHO, G, GAMMA
COMMON /MASS / DISPL, SMASS, XCG, YCG, ZCG,
1 COMMON /MASS / PMI(3,3)
COMMON /STATS / NRHS(4), DELTA, HSUM, HMIN, HMAX, TTO, TT1
COMMON /TRIG / CTHETA, STHETA, CPHI, SPHI, CPSI, SPSI
COMMON /Z / SCR(6)
COMMON/ACCEL/ACC(6)
NRHS(4)=NRHS(4)+1
IF (TT.GT.TT1) GO TO 90
IF (TT.EQ.TT1) GO TO 80
C *** Previous time step rejected
NRHS(3)=NRHS(3)+1
TTO=AMIN1(TT, TTO)
TT1=TTO
GO TO 100
C *** Previous time step repeated
80 NRHS(2)=NRHS(2)+1
GO TO 100
C
C *** Previous time step accepted
90 DELTA=TT1-TTO
TTO=TT1
TT1=TT

```

```

IF (DELTA.LE.0.0) GO TO 100
NRHS(1)=NRHS(1)+1
HSUM=HSUM+DELTA
HMIN=AMIN1(HMIN,DELTA)
HMAX=AMAX1(HMAX,DELTA)
100 CONTINUE
C *** Compute forces on ship
CALL FORTIS (TT,YY)
C *** Set the values of the derivatives
C *** Set up the H vector (angular momentum)
H(1)=PMI(1,1)*YY(10)+PMI(1,2)*YY(11)+PMI(1,3)*YY(12)
H(2)=PMI(2,1)*YY(10)+PMI(2,2)*YY(11)+PMI(2,3)*YY(12)
H(3)=PMI(3,1)*YY(10)+PMI(3,2)*YY(11)+PMI(3,3)*YY(12)
C *** Set derivatives of position and rotation
DO 110 I=1,3
  YYDOT(I)=YY(I+6)
110 CONTINUE
YYDOT(5)=(YY(11)*CTHETA-YY(12)*STHETA)/CPSI
YYDOT(4)=YY(10)-YYDOT(5)*SPSI
YYDOT(6)=YY(11)*STHETA+YY(12)*CTHETA
C *** Solve for derivatives of linear and angular velocity
YYDOT(7)=XF/SMASS
YYDOT(8)=YF
YYDOT(9)=ZF
YYDOT(10)=XM-YY(11)*H(3)+YY(12)*H(2)
YYDOT(11)=YM-YY(12)*H(1)+YY(10)*H(3)
YYDOT(12)=ZM-YY(10)*H(2)+YY(11)*H(1)
DET=0.0
L=LNEGF(6,5,1,A(2,2),YYDOT(8),DET,SCR)
IF (L.NE.1) STOP 1
C *** Load the acceleration array ACC [ship coordinates]
ACC(1)=YYDOT(7)
ACC(2)=YYDOT(8)
ACC(3)=YYDOT(9)
ACC(4)=YYDOT(10)
ACC(5)=YYDOT(11)
ACC(6)=YYDOT(12)
C *** Resolve linear accelerations into fixed coordinates.
SCR(1)=YYDOT(7)
SCR(2)=YYDOT(8)
SCR(3)=YYDOT(9)
YYDOT(7)=C(1,1)*SCR(1)+C(2,1)*SCR(2)+C(3,1)*SCR(3)
YYDOT(8)=C(1,2)*SCR(1)+C(2,2)*SCR(2)+C(3,2)*SCR(3)
YYDOT(9)=C(1,3)*SCR(1)+C(2,3)*SCR(2)+C(3,3)*SCR(3)
YYDOT(13)=YY(5)
RETURN
END

```

## SUBROUTINE ROTATE (YY)

```
C
C1 SET UP COORDINATE ROTATION MATRICES CT AND C .
C1 RESOLVE VELOCITY INTO SHIP COORDINATES.
C1
REAL YY(13)
COMMON /C / C(3,3)
COMMON /CT / CT(3,3)
COMMON /TRIG / CTHETA, STHETA, CPHI, SPHI, CPSI, SPSI
COMMON /V / VX, VY, VZ
CTHETA=COS(YY(4))
STHETA=SIN(YY(4))
CPHI=COS(YY(5))
SPHI=SIN(YY(5))
CPSI=COS(YY(6))
SPSI=SIN(YY(6))
C3
C3 SET UP THE CT MATRIX..
C3
CT(1,1)=CPHI*CPSI
CT(1,2)=SPSI
CT(1,3)=-SPHI*CPSI
CT(2,1)=-CPHI*SPSI
CT(2,2)=CPSI
CT(2,3)=SPHI*SPSI
CT(3,1)=SPHI
CT(3,2)=0
CT(3,3)=CPHI
C3
C3 SET UP THE C MATRIX..
C3
C(1,1)=CT(1,1)
C(1,2)=CT(1,2)
C(1,3)=CT(1,3)
C(2,1)=CTHETA*CT(2,1)+STHETA*CT(3,1)
C(2,2)=CTHETA*CT(2,2)
C(2,3)=CTHETA*CT(2,3)+STHETA*CT(3,3)
C(3,1)=CTHETA*CT(3,1)-STHETA*CT(2,1)
C(3,2)=-STHETA*CT(2,2)
C(3,3)=CTHETA*CT(3,3)-STHETA*CT(2,3)
C3
C3 RESOLVE THE LINEAR VELOCITY INTO SHIP COORDINATES..
C3
VX=C(1,1)*YY(7)+C(1,2)*YY(8)+C(1,3)*YY(9)
VY=C(2,1)*YY(7)+C(2,2)*YY(8)+C(2,3)*YY(9)
VZ=C(3,1)*YY(7)+C(3,2)*YY(8)+C(3,3)*YY(9)
C3
C RETURN
C
END
```





AR=ACC(4)

AHIST(1,1)=AH  
AHIST(2,1)=AS  
AHIST(3,1)=AR

200 CONTINUE

TIMTK(1)=TIME

IF(TIME. LE. RAMP) GO TO 400

-----  
SECTION 3.0 - Find peaks and troughs, evaluate freq.  
[Note: Here we are evaluating frequency  
using only the last quarter of a "cycle".  
If we wanted to use a half cycle, we  
would get out of the GOTO310 loop when  
KNT=3 instead of KNT=2 and change  
the 'TQ=...' line to  
TQ=TPK(3)-TPK(1) followed by  
OMEGAX(I)=2.\*PI/(2.\*TQ)  
-----

DO 300 I=1,3

L=0

KNT=0

TSIGN=1.

310 CONTINUE

L=L+1

IF(L.EQ.30) GO TO 325

TSIGN2=TSIGN

DIF=AHIST(I,L)-AHIST(I,L+1)

TSIGN=SIGN(1.,DIF)

IF(L.EQ.1) GO TO 310

IF(TSIGN2.EQ.TSIGN) GO TO 310

KNT=KNT+1

TPK(KNT)=TIMTK(L)

IF(KNT.EQ.2) GO TO 320

GO TO 310

320 CONTINUE

TQ=TPK(1)-TPK(2)

OMEGAX(I)=6.2831853/(4.\*TQ)

GO TO 300

325 OMEGAX(I)=0.

300 CONTINUE

400 CONTINUE

RETURN

END

CCC

SUBROUTINE OUTCOF

COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF  
INTEGER OUTPUT, BIF, OFF, COF  
COMMON/MASS/RHO, G, GAMMA  
COMMON/MASS/DISPL, SMASS, XCG, YCG, ZCG,  
\* AMX, AMY, AMZ, RADII(6)

COMMON/LHS/Y(13)  
COMMON / / NWAVES, WVSUM,  
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),  
2 WN(20), WNX(20), WNZ(20),  
3 CSK(20), CCK(20), CXK(20),  
4 CYK(20), CZK(20), XW(20)  
COMMON / / MSTA, NPROF, NFWD, NAFT, LPTS(25),  
1 XOFF(25), YOFF(25,25), ZOFF(25,25),  
2 XPROF(51), YPROF(51)  
COMMON/RESIST/SPEED  
COMMON/SIGMA / NK, SIGMA(24), SIGMA0, ERRO, OM(12)

C  
C

COMMON/COEFF4/COEFF4(6, 12, 8, 25), AREAN(25, 6)  
COMMON/COEFFX/THAH(25, 21), TSAS(25, 21), TRAR(25, 21), TCCA(25, 21),  
\* THVH(25, 21), TSVS(25, 21), TRVR(25, 21), TCCV(25, 21),  
\* THACX(25, 21), THVCX(25, 21), TSACX(25, 21), TSVCX(25, 21),  
\* TRACX(25, 21), TCACX(25, 21), THVCY(25, 21),  
\* ITP(25), TXP(25), IFP(21), FXP(21)  
COMMON/OPTION/JOBCO, JOBFO(10)  
COMMON/SXOMEG/OMEGAX(3)  
COMMON/DRFT1/ DRAFT1(6)

C

DATA DEGREE/0.01745 32925/

C  
C

-----  
WRITE(OUTPUT, 2)

C

WRITE(OUTPUT, 3)  
DO 10 J=1, MSTA  
WRITE(OUTPUT, 1) J, THAH(J, 21), TSAS(J, 21), TRAR(J, 21), TCCA(J, 21)  
CONTINUE

10

C

WRITE(OUTPUT, 4)  
DO 20 J=1, MSTA  
WRITE(OUTPUT, 1) J, THVH(J, 21), TSVS(J, 21), TRVR(J, 21), TCCV(J, 21)  
CONTINUE

20

C

WRITE(OUTPUT, 5)  
DO 30 J=1, MSTA  
WRITE(OUTPUT, 1) J, THACX(J, 21), TSACX(J, 21), THVCX(J, 21),  
\* TSVCX(J, 21)  
CONTINUE

30

C

DO 40 K=1, NWAVES  
WRITE(OUTPUT, 6) K  
WRITE(OUTPUT, 3)  
DO 110 J=1, MSTA  
WRITE(OUTPUT, 1) J, THAH(J, K), TSAS(J, K), TRAR(J, K), TCCA(J, K)  
CONTINUE

110

C

WRITE(OUTPUT, 4)  
DO 120 J=1, MSTA  
WRITE(OUTPUT, 1) J, THVH(J, K), TSVS(J, K), TRVR(J, K), TCCV(J, K)

120  
C

CONTINUE

WRITE(OUTPUT, 5)

DO 130 J=1, MSTA

WRITE(OUTPUT, 1) J, THACX(J, K), TSACX(J, K), THVCX(J, K), TSVCX(J, K)

130

CONTINUE

40

CONTINUE

1

FORMAT(1X, I2, 5X, 4E15. 5)

2

FORMAT(/'\*\*\*\*\*TEMPORARY HYDRODYNAMIC COEFFICIENTS\*\*\*\*\*'/)

3

FORMAT(1X, 'STA', 4X, 'HAH', 12X, 'SAS', 12X, 'RAR', 12X, 'CCA'//)

4

FORMAT(1X, 'STA', 4X, 'HVB', 12X, 'SVS', 12X, 'RVR', 12X, 'CCV'//)

5

FORMAT(1X, 'STA', 4X, 'HACX', 11X, 'SACX', 11X, 'HVCX', 11X, 'SVCX', //)

6

FORMAT(/1X, 'WAVE NO. - ', I2//)

RETURN

END

```

SUBROUTINE FOLIO2
COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF
INTEGER OUTPUT, BIF, OFF, COF
COMMON /LHS / Y(13)
COMMON /TIME / TIME, RAMP
COMMON /TIME / TO, TSTART, TSTOP, TOUTPT, TSTEP, ERR
COMMON / / NWAVES, WVSUM,
1
2
3
4
WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
WN(20), WNX(20), WNZ(20),
CSK(20), CCK(20), CXK(20),
CYK(20), CZK(20), XW(20)
COMMON/SUMMARY/ NT, YYBAR(10), YYRMS(10), YYMAX(10), YYMIN(10)
COMMON/LOADS/ SIN2(25), SIN3(25), SIN4(25), SIN5(25), SIN6(25),
1
2
V2(25), V3(25), V4(25), V5(25), V6(25),
RED(6, 25)
COMMON/DISPLAY/TARE(10)
REAL YS(10)
EQUIVALENCE (ETA, YS(1))
DATA RAD /0.01745 32925 19943/
ETA=0.0
IF (NWAVES.EQ.0) GO TO 120
C *** Wave amplitude @ c.g.
DO 110 K=1, NWAVES
ARG=Y(1)*WNX(K)-Y(3)*WNZ(K)-TIME*WVFRE(K)+WVPHA(K)
ETA=COS(ARG)*WVAMP(K)+ETA
110 CONTINUE
C *** Position vector
120 DO 130 I=1, 3
YS(I+1)=Y(I)
YS(I+4)=Y(I+3)/RAD
130 CONTINUE
C *** Display relative heave -- subtract c.g. height
C above baseline to bring mean value to zero.
YS(3)=YS(3)-TARE(2)
C *** Speed and sway rate
CPHI=COS(Y(5))
SPHI=SIN(Y(5))
YS(8)=0.
!NO RUDDER
YS(8)=V6(14) !BENDING MOMENT AT STATION J=14
YS(9)=Y(7)*CPHI - Y(9)*SPHI
YS(10)=Y(9)*CPHI+ Y(7)*SPHI
C *** Load mean and rms information
TINIT=.75*TSTOP
IF(TIME.LE.TINIT) GO TO 199
NT=NT+1
IF(NT.NE.1) GO TO 198
DO 184 I=1, 10
YYMAX(I)=0.0
YYMIN(I)=0.0
YYBAR(I)=0.
YYRMS(I)=0.
184 CONTINUE
C
198 CONTINUE
C
DO 185 I=1, 10
IF(YS(I).GT.YYMAX(I)) YYMAX(I)=YS(I)
IF(YS(I).LT.YYMIN(I)) YYMIN(I)=YS(I)
C
YYBAR(I)=YYBAR(I)+YS(I)
YYRMS(I)=YYRMS(I)+YS(I)**2

185 CONTINUE
C
199 CONTINUE
C *** Write to history file
WRITE(10) YS(1), YS(3), YS(7), YS(5), YS(6), YS(4)
C *** Print YS vector
WRITE (OUTPUT, 190) TIME, (YS(I), I=1, 9)
C
RETURN
C
190 FORMAT(5(1X, F11. 4), 3F12. 4, 2X, E15. 5, 2X, F12. 4)
END

```

SUBROUTINE FORTIS (TT, YY)  
TT -- TIME, THE INDEPENDENT VARIABLE

YY -- DEPENDENT VARIABLE ARRAY.  
YY(1) = X-COORDINATE OF MASS CENTER  
YY(2) = Y-COORDINATE OF MASS CENTER  
YY(3) = Z-COORDINATE OF MASS CENTER  
YY(4) = THETA ROTATION (ROLL ANGLE)  
YY(5) = PHI ROTATION (YAW ANGLE)  
YY(6) = PSI ROTATION (PITCH ANGLE)  
YY(7) = X-COMPONENT OF LINEAR VELOCITY  
YY(8) = Y-COMPONENT OF LINEAR VELOCITY  
YY(9) = Z-COMPONENT OF LINEAR VELOCITY  
YY(10) = ANGULAR VELOCITY ABOUT X-AXIS (ROLL RATE)  
YY(11) = ANGULAR VELOCITY ABOUT Y-AXIS (YAW RATE)  
YY(12) = ANGULAR VELOCITY ABOUT Z-AXIS (PITCH RATE)

REAL YY(12)  
COMMON /C / C(3,3)  
COMMON /DAMP / DAMPL(6), DAMPG(6)  
COMMON /FORCE / XF, YF, ZF, XM, YM, ZM  
REAL FORCE(6)  
EQUIVALENCE (FORCE(1), XF)  
COMMON /MASS / RHO, G, GAMMA  
COMMON /MASS / DISPL, SMASS, XCG, YCG, ZCG,  
1 AMX, AMY, AMZ, RADII(6)  
COMMON /MASS / PMI(3,3)  
COMMON /STATS / NRHS(4), DELTA, HSUM, HMIN, HMAX, TTO, TT1  
COMMON /TIME / TIME, RAMP  
COMMON /TIME / TO, TSTART, TSTOP, TOUTPT, TSTEP, ERR  
COMMON /TRIG / CTHETA, STHETA, CPHI, SPHI, CPSI, SPSI  
COMMON /V / VX, VY, VZ

C \*\*\* Load rotational transform matrices - transform velocity  
C into ship coordinates

C \*\*\* Initialize force vector  
DO 110 I=1,6

FORCE(I)=0.0  
110 CONTINUE

C \*\*\* Ensure constant speed for version SSX  
VX=SPEED

C \*\*\* Compute hydrodynamic and inertial forces on ship  
CALL AQUA2D (TT, YY)

C \*\*\* Add damping forces/moments in ship coordinate system

XF=XF-(DAMPL(1)+ABS(VX)\*DAMPQ(1))\*VX  
YF=YF-(DAMPL(2)+ABS(VY)\*DAMPQ(2))\*VY  
ZF=ZF-(DAMPL(3)+ABS(VZ)\*DAMPQ(3))\*VZ  
XM=XM-(DAMPL(4)+ABS(YY(10))\*DAMPQ(4))\*YY(10)  
YM=YM-(DAMPL(5)+ABS(YY(11))\*DAMPQ(5))\*YY(11)  
ZM=ZM-(DAMPL(6)+ABS(YY(12))\*DAMPQ(6))\*YY(12)

C \*\*\* Fixed added mass coefficients - not normally used except  
C perhaps for surge.

XF=XF/AMX  
YF=YF/AMY  
ZF=ZF/AMZ

C \*\*\* Ramp function applied to force  
IF (TT.GT.TSTART) GO TO 130  
IF (RAMP.EQ.0.0) GO TO 130  
R=(TT-TO)/RAMP

DO 120 I=1,6  
FORCE(I)=FORCE(I)\*R

120 CONTINUE  
130 CONTINUE

C  
RETURN  
END

```

SUBROUTINE SPECTRA(PFRQ, TRWH13, DMGBEG, DMGEND, WAMP, PER)
DIMENSION WAMP(10), PER(10), OMEGA(10)
DIMENSION DMG2(101), SPEC(101)
IC1=-8
IC2=0
IC3=-7
IC4=-1
IC5=-9
IC6=1
IW=0
DMG2(1)=DMGBEG
DMG2(101)=DMGEND
1 DELFRQ=(DMG2(101)-DMG2(1))/101.
DO 5 I=2, 101
J=I-1
5 DMG2(I)=DMG2(1)+J*DELFRQ
DO 6 I=1, 101
6 DMG2(I)=DMG2(I)/(2.*3.14159)
7 DO 10 I=1, 101
10 SPEC(I)=5.*(0.25*TRWH13)**2*PFRQ*(DMG2(I)*PFRQ)**(-5)*EXP(
1 (-5./4.)*(DMG2(I)*PFRQ)**(-4))
ODD=0.
EVEN=0.
DO 20 I=2, 50, 2
20 EVEN=EVEN+SPEC(I)
DO 30 I=3, 99, 2
30 ODD=ODD+SPEC(I)
TAREA=(DMG2(2)-DMG2(1))/3.*(SPEC(1)+4.*EVEN+2.*ODD+SPEC(101))
40 IC1=IC1+10
IC2=IC2+10
IC3=IC3+10
IC4=IC4+10
IC5=IC5+10
IC6=IC6+10
ODD=0.
EVEN=0.
IW=IW+1
DO 50 I=IC1, IC2, 2
50 EVEN=EVEN+SPEC(I)
DO 60 I=IC3, IC4, 2
60 ODD=ODD+SPEC(I)
1 WAMP(IW)=(DMG2(2)-DMG2(1))/3.*(SPEC(IC5)+4.*EVEN+2.*
ODD+SPEC(IC6))
WAMP(IW)=1.414*SQRT(WAMP(IW))
PER(IW)=(DMG2(IC6)+DMG2(IC5))/2.
OMEGA(IW)=2.*3.14159*PER(IW)
PER(IW)=1./PER(IW)
IF(IW.LT.10) GO TO 40
PRINT 500, TRWH13, PFRQ, OMEGA, PER, WAMP
RMS=4.*SQRT(TAREA)
PRINT 1010, TAREA, RMS
500 FORMAT(15X, 'H1/3=', F10.2, 2X, 'PEAK FREQ=', F10.4, '//, 2X,
1 'DMG', 3X, 10F6.3, '//, 2X, 'PER', 3X, 10F6.2, '//, 2X, 'WAMP', 2X,
2 10F6.2)
1010 FORMAT(2X, 'TOTAL AREA=', F10.2, 5X, 'RMS=', F10.2)
RETURN
END
SUBROUTINE LINEAR(X, Y, TRWPER, Y2)
DIMENSION X(50), Y(50), Y2(10)
DIMENSION TRWPER(10)
J=0

```

```

140 RA=SQRT((X-XF)**2+(Y-YF)**2+(Z-ZF)**2)
FRA=ARM/RA
1380 RB=SQRT((X-XF)**2+(Y-YF)**2+(Z+ZF)**2)
FRA=FRA-ARM/RB
FRX=-AN(MJ, 1)*FRA
FRY=-AN(MJ, 2)*FRA
FRZ=-AN(MJ, 3)*FRA
P1=P1+FRX
P2=P2+FRY
P3=P3+FRZ
P4=P4+YF*FRZ-ZF*FRY
P5=P5+ZF*FRX-XF*FRZ
P6=P6+XF*FRY-YF*FRX
PBB(NJ, MJ)=PBB(NJ, MJ)+FRA*ARN
138 CONTINUE
P(NJ, 1)=P(NJ, 1)+P1*ARN
P(NJ, 2)=P(NJ, 2)+P2*ARN
P(NJ, 3)=P(NJ, 3)+P3*ARN
P(NJ, 4)=P(NJ, 4)+P4*ARN
P(NJ, 5)=P(NJ, 5)+P5*ARN
P(NJ, 6)=P(NJ, 6)+P6*ARN
128 CONTINUE
127 CONTINUE
DO 554 KQ=1, 6
554 WRITE(3) (P(NJ, KQ), NJ=1, NPAN)
RETURN
END

```

```
TOPI=2.*3.14159
DO 20 II=1,10
IF(II.GT.1) GO TO 69
50 J=J+1
69 IF(TOPI/TRWPER(II)-X(J))112,111,110
110 GOTO 50
111 Y2(II)=Y(J)
112 GOTO 20
20 1 Y2(II)=Y(J-1)+(Y(J)-Y(J-1))/(X(J)-X(J-1))*
(TOPI/TRWPER(II)-X(J-1))
CONTINUE
RETURN
END
```



```

SUBROUTINE TBAR(TT, YY)
CCC
EXTERNAL ETAF
CCC
COMMON/MASS/RHO, G, GAMMA
COMMON/MASS/DISPL, SMASS, XCG, YCG, ZCG,
* COMMON / AMX, AMY, AMZ, RADII(6)
1 / NWAVES, WVSUM,
4 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
1 WN(20), WNX(20), WNZ(20),
4 CSK(20), CCK(20), CXK(20),
1 CYK(20), CZK(20), XW(20)
COMMON / / MSTA, NPROF, NFWD, NAFTA, LPTS(25),
1 XOFF(25), YOFF(25,25), ZOFF(25,25),
2 XPROF(51), YPROF(51)
COMMON/ETA/YC, XK(20)
COMMON/CT/CT(3,3)
COMMON/TEMPSTA/DXFWD, DXAFT, TSTA(25)
COMMON/DISPLAY/TARE(6)
REAL YY(1)

DATA DEGREE/0.01745 32925/
SECTION 1.0 Compute average draft at each station
TSTA(J).

DO 2 K=1, NWAVES
CXK(K)=CT(1,1)*WNX(K)-CT(1,3)*WN(K)
XW(K)=(TT*WVFRE(K)-WVPHA(K)+YY(3)*WNZ(K)-YY(1)*WNX(K))/CXK(K)
CONTINUE

DO 5 J=1, MSTA
XXSTA=XOFF(J)
IF(NWAVES.EQ.0) GO TO 11
DO 10 K=1, NWAVES
XK(K)=(XXSTA-XW(K))*CXK(K)
CONTINUE
10 YC=CT(1,2)*XXSTA+YY(2)
11 YKEEL=YCG-TARE(2)-YC
ETA1=ETAF(0.)
TSTA(J)=ETA1-YKEEL
CONTINUE

5
RETURN
END

```



```

TX=TXP(J)
IF=IFP(K)
FX=FXP(K)
END IF

THAH(J,K)=COX(IT, TX, IF, FX, 1, J)
TSAS(J,K)=COX(IT, TX, IF, FX, 2, J)
TRAR(J,K)=COX(IT, TX, IF, FX, 3, J)
TCCA(J,K)=COX(IT, TX, IF, FX, 4, J)

THVH(J,K)=COX(IT, TX, IF, FX, 5, J)
TSVS(J,K)=COX(IT, TX, IF, FX, 6, J)
TRVR(J,K)=COX(IT, TX, IF, FX, 7, J)
TCCV(J,K)=COX(IT, TX, IF, FX, 8, J)

```

```

Spatial derivatives in x-direction

```

```

THACX(J,K)=COXDX(IT, TX, IF, FX, 1, J)
TSACX(J,K)=COXDX(IT, TX, IF, FX, 2, J)
TRACX(J,K)=COXDX(IT, TX, IF, FX, 3, J)
TCACX(J,K)=COXDX(IT, TX, IF, FX, 4, J)

```

```

CONTINUE

```

```

SECTION 3.0 - This section computes coefficients
              for ship motion related force
              computations, using OMEGAX( ) as
              the frequency at which computations
              are performed. OMEGAX(1) - HEAVE,
              OMEGAX(2) - SWAY, OMEGAX(3) - ROLL.
              These are computed in Subroutine
              GETFREQ.

```

```

DO 20 J=1, MSTA

```

```

IF(JOBCO.EQ.2) THEN
  IF=IFP(21)
  FX=FXP(21)
  IFH=IF
  IFS=IF
  IFR=IF
  FXH=FX
  FXS=FX
  FXR=FX
  CALL TXIT(TSTA(J), IT, TX)
ELSE IF(JOBCO.EQ.3) THEN
  IT=ITP(J)
  TX=TXP(J)
  CALL FXIF(OMEGAX(1), IFH, FXH)
  CALL FXIF(OMEGAX(2), IFS, FXS)
  CALL FXIF(OMEGAX(3), IFR, FXR)
ELSE IF(JOBCO.EQ.4) THEN
  CALL TXIT(TSTA(J), IT, TX)
  CALL FXIF(OMEGAX(1), IFH, FXH)
  CALL FXIF(OMEGAX(2), IFS, FXS)
  CALL FXIF(OMEGAX(3), IFR, FXR)

```

```

END IF

```

THAH(J, 21)=COX(IT, TX, IFH, FXH, 1, J)  
TSAS(J, 21)=COX(IT, TX, IFS, FXS, 2, J)  
TRAR(J, 21)=COX(IT, TX, IFR, FXR, 3, J)  
TCCA(J, 21)=COX(IT, TX, IFS, FXS, 4, J)

THVH(J, 21)=COX(IT, TX, IFH, FXH, 5, J)  
TSVS(J, 21)=COX(IT, TX, IFS, FXS, 6, J)  
TRVR(J, 21)=COX(IT, TX, IFR, FXR, 7, J)  
TCCV(J, 21)=COX(IT, TX, IFS, FXS, 8, J)

Spatial Derivatives in x-directin

THACX(J, 21)=COXDX(IT, TX, IFH, FXH, 1, J)  
TSACX(J, 21)=COXDX(IT, TX, IFS, FXS, 2, J)  
THVCX(J, 21)=COXDX(IT, TX, IFH, FXH, 5, J)  
TSVCX(J, 21)=COXDX(IT, TX, IFS, FXS, 6, J)

CONTINUE

[Note: Vertical derivative of added mass computed  
in SUBROUTINE FROUDE or COFFEE.]

RETURN  
END

SUBROUTINE AQUA2D (TT,YY)

CALCULATES SHIP-MOTION INDUCED FORCES, WAVE INDUCED FORCES,  
AND OTHER HYDRODYNAMIC FORCES ON THE SHIP.

| SECTION | DESCRIPTION   |
|---------|---|
| 1.0     | INITIALIZATION                                      |
| 2.0     | FIND WET OFFSET POINTS/ENDS OF HULL                 |
| 3.0     | INITIALIZATION                                      |
| 4.0     | CALCULATE VARIOUS 2-D INTEGRALS/GEOMETRIC INFO. FOR |
| 5.0     | CALCULATE FORCES THAT ARE SHIP-MOTION RELATED       |
| 6.0     | CALCULATE FORCES THAT ARE WAVE-INDUCED              |
| 7.0     | CALCULATE OTHER FORCES                              |
| 8.0     | INTEGRATE FORCES OVER LENGTH                        |
| 9.0     | ADD UP FORCES AND LOAD FORCE ARRAY                  |
| 10.0    | LOAD MASS/INERTIA MATRIX A(I,J)                     |

NOTE: The hydrodynamic coefficients were divided by sectional area in COEFS part of this program. They were also divided by RHO in the subroutine. Thus, quantities are multiplied by DVOL when they would normally be multiplied by DX, and there are several places where a coefficient is divided by G operated to make sure that all the units are consistent from force to force. At any point in the force calculations, one must multiply by GAMMA to obtain force in lbs.

```

REAL YY(1)
COMMON /A / A(6,6)
COMMON /AM / YFYA, ZFZA, XMZA, XMXR, XMYR,
1 YMZA, YMYR, ZMYA, ZMZR
COMMON/RESIST/SPEED
REAL VT(6), AM(9)
EQUIVALENCE (AM(1), YFYA)
COMMON /C / C(3,3)
COMMON /CT / CT(3,3)
COMMON /ETA / YC, XK(20)
COMMON /FORCE / XF, YF, ZF, XM, YM, ZM
REAL FORCE(6)
EQUIVALENCE (FORCE(1), XF)
COMMON /MASS / RHO, G, GAMMA
COMMON /MASS / DISPL, SMASS, XCG, YCG, ZCG,
1 AMX, AMY, AMZ, RADII(6)
COMMON /MASS / PMI(3,3)
COMMON /TRIG / CTHETA, STHETA, CPHI, SPHI, CPSI, SPSI
COMMON /V / VX, VY, VZ
COMMON /
1 NWAVES, WVSUM,
2 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
3 WN(20), WNX(20), WNZ(20),
4 CSK(20), CCK(20), CXK(20),
5 CYK(20), CZK(20), XW(20)
COMMON /
1 MSTA, NPROF, NFWD, NAFT, LPTS(25),
2 XOFF(25), YOFF(25,25), ZOFF(25,25),
3 XPROF(51), YPROF(51)
COMMON /
1 INPTS(25), IPDINT(25), YWET(25,25), ETA(25,25),
2 ZLAST(25,25), ZWET(25,25), ZNEXT(25,25)

```

```

COMMON / / ZCO(20), ZSO(20), ZCY(20), ZSY(20),
1 ZCZ(20), ZSZ(20)
COMMON / / DZK(20)
COMMON / / JSAVE(20), COSJ(20), SINJ(20),
$ COSJJ(20), SINJJ(20),
1 DWC(20), DWS(20), XCI(20), XSI(20)
COMMON /AREA / AREA, YMOM, ZMOM
EQUIVALENCE (FWD, XPROF(51)), (AFT, YPROF(51))
COMMON/SXFOR/ FBUOY(6, 25), FDAMP(6, 25), FADMA(6, 25), FWVEX(6, 25),
* UDAMP(6, 25), UADMA(6, 25), UWVEX(6, 25), FFLAR(6, 25)
COMMON/SXGDOM/AREAS(25), DXS(25), DX1S(25), DX2S(25),
* YMOMS(25), ZMOMS(25), ISTA, LSTA
COMMON/SXOMEG/OMEGAX(3)
COMMON/TEMPSTA/DXFWD, DXAFT, TSTA(25)
COMMON/OPTION/JOBCD, JOBFO(10)
DIMENSION COLD(25), CNEW(25), TOLD(25), TNEW(25)
DIMENSION TTOLD(25), TTNEW(25)
REAL ZEROF(1200)
EQUIVALENCE (ZEROF(1), FBUOY(1, 1))
COMMON/FCOMP/XFB, YFB, ZFB, XMB, YMB, ZMB,
1 XFD, YFD, ZFD, XMD, YMD, ZMD,
3 XFX, YFX, ZFX, XMX, YMX, ZMX,
4 XFN, YFN, ZFN, XMN, YMN, ZMN,
5 XFU, YFU, ZFU, XMU, YMU, ZMU,
YFF, ZMF
REAL ZEROC(32)
EQUIVALENCE (ZEROC(1), XFB)
COMMON/CDEFFX/THAH(25, 21), TSAS(25, 21), TRAR(25, 21), TCCA(25, 21),
* THVH(25, 21), TSVS(25, 21), TRVR(25, 21), TCCV(25, 21),
* THACX(25, 21), THVCX(25, 21), TSACX(25, 21), TSVCX(25, 21),
* TRACX(25, 21), TCACX(25, 21), THVCY(25, 21),
* ITP(25), TXP(25), IFP(21), FXP(21)

```

```

C *** Initialization
IF (NWAVER.EQ.0) GO TO 120

```

```

-----
SECTION 1.0 - INITIATION

```

```

Project wave numbers into calculation coordinate system
(yaw and pitch but not roll). Compute x-coord of wave
crest.

```

```

-----
DO 110 K=1,NWAVER
CSK(K)=CT(1,2)*WN(K)
CCK(K)=CT(2,2)*WN(K)
CXK(K)=CT(1,1)*WNX(K)-CT(1,3)*WNZ(K)
CYK(K)=CT(2,1)*WNX(K)-CT(2,3)*WNZ(K)
CZK(K)=CT(3,1)*WNX(K)-CT(3,3)*WNZ(K)
DZK(K)=2.0/CZK(K)
XW(K)=(TT*WVFRE(K)-WVPHA(K)+YY(3)*WNZ(K)-YY(1)*WNX(K))/CXK(K)
JSAVE(K)=0

```

```

110 CONTINUE
120 CONTINUE

```

```

-----
1.1 Zero the acceleration force coefficients AM(I) and
zero the sectional force arrays in common SXFOR.
-----

```

```

DO 130 I=1,9
  AM(I)=0.0
130 CONTINUE
C*
DO 131 I=1,1200
  ZERDF(I)=0.0 ! Equivalenced with COMMON/SXFDR/
131 CONTINUE
C*
C
DO 800 I = 1,32
  ZEROC(I)=0. !ZEROC is EQUIVALENCed with COMMON/FCOMP/
800 CONTINUE
C
C*

```

---

```

1.2 Load COMMON/COEFFX/ if JOBCO=2,3 or 4

```

---

```

CALL TBAR(TT,YY) !Get average draft at every sta.
TYPE 1000, (TSTA(J), J=1, MSTA)
1000 FORMAT(1X,25F5.1)
C DO 1000 J=1, MSTA
C TSTA(J)=32.8
C1000 CONTINUE
C
C IF(JOBCO. NE. 1. AND. TT. GT. 0. ) CALL MDCHA
C
C ISTA=0

```

---

```

SECTION 2.0 - FIND WET OFFSET POINTS AND HULL ENDS

```

---

```

FIND WET OFFSET POINTS.

```

```

ETA(L) = Y-COORDINATE OF WAVE SURFACE,
YWET(L) = Y-COORDINATE OF WET OFFSET POINT.
ZWET(L) = Z-COORDINATE OF WET OFFSET POINT.
ZLAST(L) = Z-COORDINATE OF EITHER THE PREVIOUS WET OFFSET
POINT OR THE Z-COORDINATE OF THE WATER SURFACE,
ZNEXT(L) = Z-COORDINATE OF EITHER THE NEXT WET OFFSET POINT
OR THE Z-COORDINATE OF THE WATER SURFACE.

```

```

KEY INDICATES THE STATUS OF THE SEARCH:

```

```

KEY = 1 -- THE FIRST OFFSET POINT IS BEING TESTED,
KEY = 2 -- THE LAST OFFSET POINT WAS WET,
KEY = 3 -- THE LAST OFFSET POINT WAS DRY (ETA COMPUTED),
KEY = 4 -- THE LAST OFFSET POINT WAS DRY AND ETAMAX WAS
USED (ETA WAS NOT COMPUTED).

```

---

```

DO 260 J=1, MSTA
KEY=1

```

```

I=1
L=0
LWET=0
N=LPTS(J)
XXSTA=XOFF(J)
IF (NWAVER.EQ.0) GO TO 150
DO 140 K=1,NWAVER
  XK(K)=(XXSTA-XW(K))*CXK(K)
140 CONTINUE
150 YC=CT(1,2)*XXSTA+YY(2)
  ETAMAX=(WVSUM-YC)/CT(2,2)
160 Y1=YOFF(I,J)*CTHETA-ZOFF(I,J)*STHETA
  Z1=ZOFF(I,J)*CTHETA+YOFF(I,J)*STHETA
  IF (KEY.LT.3) GO TO 170

C5 THE PREVIOUS OFFSET POINT WAS DRY. AN ESTIMATE OF ETA WILL BE
C5 USED TO SEE IF THIS POINT CAN BE WET.
C5
  IF (Y1.LT.ETAMAX) GO TO 170
  KEY=4
  GO TO 230

C4 COMPUTE THE Y-COORDINATE OF THE WATER SURFACE..
C4
C4
170 ETA1=ETAF(Z1)
  IF (KEY.GT.1) GO TO 180

C5 SAVE INITIAL CALCULATIONS FOR USE WITH FINAL OFFSET POINT.
C5 THE INITIAL AND FINAL OFFSETS REFER TO THE SAME POINT.
C5
  EFIRST=ETA1'
  YFIRST=Y1
  YO=Y1
  ZFIRST=Z1
  ZO=Z1

C4 TEST THE POSITION OF THE CURRENT OFFSET POINT RELATIVE TO THE
C4 WATER SURFACE. THE IF TEST IS SATISFIED IF THE POINT IS
C4 BELOW THE WATER SURFACE.
C4
180 IF (Y1.LT.ETA1) GO TO (212,210,200),KEY

C5 THE CURRENT OFFSET POINT IS DRY. IF KEY=2, THE PREVIOUS POINT
C5 WAS WET, AND THE Z-COORDINATE OF THE INTERSECTION OF THE HULL
C5 AND THE WATER SURFACE MUST BE FOUND BY INTERPOLATION.
C5
  IF (KEY.NE.2) GO TO 181
  ZNEXT(L,J)=(Z1-ZO)/(Y1-ETA1+EY)*EY+ZO
  LWET=L
181 KEY=3
  GO TO 220

C5 IF KEY = 4, THE ETA VALUE FOR THE LAST OFFSET POINT WAS NOT
C5 COMPUTED. COMPUTE ETA..
C5
190 EY=ETAF(ZO)-YO

C5 THE CURRENT OFFSET POINT IS WET, AND THE PREVIOUS POINT IS DRY.
C5 INTERPOLATE TO FIND THE INTERSECTION OF THE HULL AND THE WATER
C5 SURFACE..
C5

```



```

200      ZO=ZO+(Z1-ZO)/(Y1-ETA1+EY)*EY
        GO TO 211
C4
C4      A WET OFFSET POINT HAS BEEN LOCATED.  STORE THE REQUIRED
C4      COORDINATE VALUES.
C4
210      ZNEXT(L,J)=Z1
211      IF (I.EQ.N) GO TO 240
212      L=L+1
        ETA(L,J)=ETA1
        YWET(L,J)=Y1
        ZWET(L,J)=Z1
        ZLAST(L,J)=ZO
        KEY=2
C5
C5      SAVE THE COMPUTED COORDINATE OF THE WATER SURFACE.
C5
220      EY=ETA1-Y1
C5
C5      SAVE THE COORDINATES OF THE OFFSET POINT.
C5
230      YO=Y1
        ZO=Z1
C4
C4      INCREMENT THE INDEX OF THE OFFSET POINT FOR THIS STATION.  IF
C4      THE INDEX DOES NOT REFER TO THE LAST POINT, REPEAT THE LOOP.
C4
        I=I+1
        IF (I.LT.N) GO TO 160
        IF (I.GT.N) GO TO 250
C5
C5      FOR THE LAST OFFSET POINT, THE VALUES OF ETA1, X1, AND Y1 ARE
C5      THE SAME AS FOR THE FIRST OFFSET POINT.  REPEAT THE LOOP USING
C5      THESE VALUES.
C5
        ETA1=EFIRST
        Y1=YFIRST
        Z1=ZFIRST
        GO TO 180
C4
C4      A COMPLETE SEARCH OVER ALL THE OFFSET POINTS FOR THE CURRENT
C4      STATION HAS OBTAINED L WET OFFSET POINTS.  RESET ZLAST(1)
C4      IF THE LAST (AND ALSO THE FIRST) OFFSET POINT IS WET.
C4
240      ZLAST(1,J)=ZO
250      INPTS(J)=L
        IF (L.EQ.0) GO TO 260
        IF (LWET.EQ.L) LWET=0
        IPOINT(J)=LWET+1
        LSTA=J
        IF (ISTA.EQ.0) ISTA=J
C3
260      CONTINUE
        IF (ISTA.EQ.0) GO TO 520
C3
C3      FIND ENDS OF WETTED HULL.
C3
C3      XXF      = X-COORDINATE FORWARD
C3      XXA      = X-COORDINATE AFT
C3
        XXF=XOFF(ISTA)

```







FBUOY(6, J) = CT(2, 2)\*ADX1 - CT(1, 2)\*YDX

Section 5.2 - Damping force

Get damping coefficients . . .

This section will be changed to reflect the dependency on frequency of ship motion.

HVH = COEFF(1, 2, J) / G  
SVS = COEFF(2, 2, J) / G  
RVR = COEFF(3, 2, J) / G  
CCV = COEFF(4, 2, J) / G

HVH = THVH(J, 21) / G  
SVS = TSVS(J, 21) / G  
RVR = TRVR(J, 21) / G  
CCV = TCCV(J, 21) / G

HAH = THAH(J, 21)  
SAS = TSAS(J, 21)  
RAR = TRAR(J, 21)  
CCA = TCCA(J, 21)

VS = VT(5)\*ADX1 - VT(3)\*DVOL  
VR = VT(4)\*DVOL  
FDAMP(1, J) = 0.  
FDAMP(2, J) = (VT(4)\*ZDX - VT(2)\*DVOL - VT(6)\*ADX1)\*HVH  
FDAMP(3, J) = VS\*SVS - VR\*CCV  
FDAMP(4, J) = VS\*CCV - VR\*RVR  
FDAMP(5, J) = (VT(3)\*ADX1 - VT(5)\*ADX2)\*SVS + VT(4)\*ADX1\*CCV  
FDAMP(6, J) = (VT(4)\*ZMOM\*DX1 - VT(2)\*ADX1 - VT(6)\*ADX2)\*HVH

Section 5.3 - Speed terms

5.3.1 - Speed terms associated with damping

UDAMP(1, J) = 0.

UDAMP(2, J) = (2. \*SPEED\*YY(6))\*HVH\*DVOL  
UDAMP(3, J) = -(2. \*SPEED\*YY(5))\*SVS\*DVOL  
UDAMP(4, J) = 0.  
UDAMP(5, J) = (2. \*SPEED\*YY(5))\*SVS\*ADX1  
UDAMP(6, J) = (2. \*SPEED\*YY(6))\*HVH\*ADX1

\*\*\* Extra speed terms from extended strip (X SCORES)

U2TEMP = (YY(2)\*DVOL + YY(6)\*ADX1 + SPEED\*DVOL\*  
\* (VT(6)/OMEGAX(1)\*\*2)) \* (THVCX(J, 21)/G)\*  
\* SPEED

UDAMP(2, J) = UDAMP(2, J) + U2TEMP  
UDAMP(6, J) = UDAMP(6, J) + U2TEMP\*(ADX1/DVOL)

5.3.2 Speed terms associated with added mass

UADMA(1, J) = 0.  
UA21 = (2. \*SPEED\*VT(6)) \* (HAH/G)\*DVOL



END IF

C 721 CONTINUE !Skip to here when TT=TTOLD

-----  
C If there are no waves, skip over the wave loop -  
C [Do 500] and go to 510 (the other side of the section  
C loop).  
C IF (NWAVER.EQ.0) GO TO 510  
-----

SECTION 6.0 - WAVE-INDUCED FORCES  
-----

IF (LSAVE.NE.0) GO TO 450  
IF (DXFWD.LT.SMALL) GO TO 441  
DDXFWD=1.0/DXFWD  
GO TO 460  
441 DDXFWD=0.0  
GO TO 460  
450 DDXFWD=DDXAFT  
460 IF (DXAFT.LT.SMALL) GO TO 461  
DDXAFT=1.0/DXAFT  
GO TO 470  
461 DDXAFT=0.0  
C 470 LSAVE=LSTA-J  
IF (LSAVE.NE.0) LSAVE=INPTS(J+1)

-----  
C Loop over waves (frequency index K)  
-----

DO 500 K=1,NWAVER

-----  
C Section 6.1 - Compute integration multipliers...  
C DXCOS, DXSIN, DX1COS, DX1SIN.  
-----

XWC=XW(K)  
XWSTA=XXSTA-XWC  
SQSTA=XWSTA\*\*2  
XWAFT=XXAFT-XWC  
SQAFT=XWAFT\*\*2  
XWFWD=XXFWD-XWC  
SQFWD=XWFWD\*\*2  
DXWAFT=XWAFT\*DDXAFT  
DXWFWD=XWFWD\*DDXFWD  
WCOSB=CXK(K)

C IF (JSAVE(K).EQ.0) GO TO 472  
COSSTA=COSJ(K)  
SINSTA=SINJ(K)  
GO TO 473  
472 CONTINUE  
ARGSTA=WCOSB\*XWSTA  
COSSTA=COS(ARGSTA)  
SINSTA=SIN(ARGSTA)  
473 CONTINUE

```

C      IF (ABS(WCOSB*DX).GT.0.01) GO TO 480
C
C      DXCOS=DX*COSSTA
C      DXSIN=DX*SINSTA
C      DX1COS=DX1*COSSTA
C      DX1SIN=DX1*SINSTA
C      JSAVE(K)=0
C      GO TO 490
C
C 480   WCOSB2=2.0/WCOSB
C       XCSTA=COSSTA*XWSTA
C       XSSTA=SINSTA*XWSTA
C
C       IF (JSAVE(K).EQ.0) GO TO 481
C       COSFWD=COSJJ(K)
C       SINFWD=SINJJ(K)
C       DWCFWD=DWC(K)
C       DWSFWD=DWS(K)
C       XCIFWD=XCI(K)
C       XSIFWD=XSI(K)
C       GO TO 482
C
C 481   ARGFWD=WCOSB*XWFW
C       COSFWD=COS(ARGFWD)
C       SINFWD=SIN(ARGFWD)
C       DWCFWD=(COSFWD-COSSTA)/WCOSB
C       DWSFWD=(SINFWD-SINSTA)/WCOSB
C       XCIFWD=(SINFWD*XWFW-XSSTA+DWCFWD)*DDXFWD
C       XSIFWD=(XCSTA-COSFWD*XWFW+DWSFWD)*DDXFWD
C
C 482   ARGAFT=WCOSB*XWAFT
C       COSAFT=COS(ARGAFT)
C       SINAFT=SIN(ARGAFT)
C       DWCAFT=(COSSTA-COSAFT)/WCOSB
C       DWSAFT=(SINSTA-SINAFT)/WCOSB
C       XCIAFT=(XSSTA-SINAFT*XWAFT+DWCAFT)*DDXAFT
C       XSIAFT=(COSAFT*XWAFT-XCSTA+DWSAFT)*DDXAFT
C
C       DXCOS=(XCIAFT-XCIFWD)/WCOSB-DXWAFT*DWSAFT+DXWFW*DWSFWD
C       DXSIN=(XSIAFT-XSIFWD)/WCOSB+DXWAFT*DWCAFT-DXWFW*DWCFWD
C       DX1COS=XWC*DXCOS+((SQSTA*SINSTA-SGAFT*SINAFT)*DDXAFT+(SQSTA*
1      SINSTA-SGFWD*SINFWD)*DDXFWD+(XSIFWD-XSIAFT)*WCOSB2+XWFW*XC
2      FWD-XWAFT*XCIAFT)/WCOSB
C       DX1SIN=XWC*DXSIN+((SGFWD*COSFWD-SQSTA*COSSTA)*DDXFWD+(SGAFT*
1      COSAFT-SQSTA*COSSTA)*DDXAFT+(XCIAFT-XCIFWD)*WCOSB2+XWFW*XS
2      FWD-XWAFT*XSIAFT)/WCOSB
C
C       JSAVE(K)=LSAVE
C       IF (LSAVE.EQ.0) GO TO 490
C       COSJ(K)=COSAFT
C       SINJ(K)=SINAFT
C       COSJJ(K)=COSSTA
C       SINJJ(K)=SINSTA
C       DWC(K)=DWCAFT
C       DWS(K)=DWSAFT
C       XCI(K)=XCIAFT
C       XSI(K)=XSIAFT
C
C 490   CONTINUE
C       EXPSTA=EXP(WN(K)*YC)*WVAMP(K)

```



DXCOS=DXCOS\*EXPSTA  
DXSIN=DXSIN\*EXPSTA  
DX1COS=DX1COS\*EXPSTA  
DX1SIN=DX1SIN\*EXPSTA

Section 6.2 - Compute exciting forces (wave dynamic  
pressure plus diffraction force)

HVH=COEFF(1,2,J)/WVFRE(K)  
SVS=COEFF(2,2,J)/WVFRE(K)  
RVR=COEFF(3,2,J)/WVFRE(K)  
CCV=COEFF(4,2,J)/WVFRE(K)

HAH=THAH(J,K)  
HAH=HAH+1.  
SAS=TSAS(J,K)  
SAS=SAS+1.  
RAR=TRAR(J,K)  
CCA=TCCA(J,K)

HVH=THVH(J,K)/WVFRE(K)  
SVS=TSVS(J,K)/WVFRE(K)  
RVR=TRVR(J,K)/WVFRE(K)  
CCV=TCCV(J,K)/WVFRE(K)

AH1=CYK(K)\*ZSO(K)-CCK(K)\*ZCO(K)  
AH2=CCK(K)\*ZSO(K)+CYK(K)\*ZCO(K)  
AS1=CZK(K)\*ZSO(K)  
AS2=CZK(K)\*ZCO(K)  
AR1=CZK(K)\*ZSY(K)+CCK(K)\*ZCZ(K)-CYK(K)\*ZSZ(K)  
AR2=CZK(K)\*ZCY(K)-CCK(K)\*ZSZ(K)-CYK(K)\*ZCZ(K)  
1 FWVEX(1,J)=(CXK(K)\*ZSO(K)-CSK(K)\*ZCO(K))\*DXCOS  
+(CXK(K)\*ZCO(K)+CSK(K)\*ZSO(K))\*DXSIN+FWVEX(1,J)  
YF1=AH1\*HAH+AH2\*HVH  
YF2=AH2\*HAH-AH1\*HVH  
FWVEX(2,J)=YF1\*DXCOS+YF2\*DXSIN+FWVEX(2,J)  
ZF1=AS1\*SAS+AS2\*SVS+(AS1\*CCA+AS2\*CCV)\*WN(K)  
ZF2=AS2\*SAS-AS1\*SVS+(AS2\*CCA-AS1\*CCV)\*WN(K)  
FWVEX(3,J)=ZF1\*DXCOS+ZF2\*DXSIN+FWVEX(3,J)  
1 FWVEX(4,J)=(AR1+(AS1\*RAR+AS2\*RVR)\*WN(K)+AS1\*CCA+  
2 AS2\*CCV)\*DXCOS+(AR2+(AS2\*RAR-AS1\*RVR)\*WN(K)+AS2\*CCA  
-AS1\*CCV)\*DXSIN+FWVEX(4,J)  
1 FWVEX(5,J)=(CXK(K)\*ZSZ(K)-CSK(K)\*ZCZ(K))\*DXCOS+  
2 (CXK(K)\*ZCZ(K)+CSK(K)\*ZSZ(K))\*DXSIN-ZF1\*DX1COS-ZF2  
\*DX1SIN+FWVEX(5,J)  
1 FWVEX(6,J)=(CSK(K)\*ZCY(K)-CXK(K)\*ZSY(K))\*DXCOS-  
2 (CSK(K)\*ZSY(K)+CXK(K)  
\*ZCY(K))\*DXSIN+YF1\*DX1COS+YF2\*DX1SIN+FWVEX(6,J)

-----  
6.3 Speed dependent wave-exciting terms  
-----

DADX=SPEED\*THACX(J,K)/WVFRE(K) !Turns acc.'s into Vel's  
UWVEX(2,J)=AH1\*DADX\*DXSIN-AH2\*DADX\*DXCOS+UWVEX(2,J)  
Another speed term - spatial derivative of damping in x  
DCDX=SPEED\*(THVCX(J,K)/G)\*DVOL\*WVFRE(K)/OMEGAX(1)  
UWVEX(2,J)=UWVEX(2,J)+DCDX\*ETAY(J,K)  
UZF1=AS1\*TSACX(J,K)+AS2\*TSVCX(J,K)  
UZF2=AS2\*TSACX(J,K)-AS1\*TSVCX(J,K)  
UWVEX(3,J)=(-UZF1\*DXSIN+UZF2\*DXCOS)\*(SPEED/WVFRE(K))  
+UWVEX(3,J)  
1  
UWVEX(4,J)=0. ! Modify this  
UWVEX(5,J)=(-UZF1\*DX1SIN+UZF2\*DX1COS)\*(SPEED/WVFRE(K))  
+ UWVEX(5,J)  
1  
UWVEX(6,J)=AH1\*DADX\*DX1SIN-AH2\*DADX\*DX1COS  
DCDX1=SPEED\*(THVCX(J,K)/G)\*ADX1\*WVFRE(K)/OMEGAX(1)  
UWVEX(6,J)=DCDX1\*ETAY(J,K)+UWVEX(6,J)

500 CONTINUE  
510 CONTINUE

-----  
SECTION 7.0 - OTHER FORCES  
-----

Force resulting from acceleration caused by the rotating  
coordinates.

YA=VT(6)\*VT(1)-VT(4)\*VT(3)  
YA=YA/G  
ZA=VT(4)\*VT(2)-VT(5)\*VT(1)  
ZA=ZA/G  
XFN=0.  
YFN=YFYA\*YA  
ZFN=ZFZA\*ZA  
XMN=XMZA\*ZA  
YMN=YMZA\*ZA  
ZMN=ZMYA\*YA

-----  
SECTION 8.0 - INTEGRATE FORCES ALONG LENGTH OF HULL  
-----

Section 8.1 - Integrate buoyancy force  
-----

DO 801 J=ISTA,LSTA  
XFB=FBUOY(1,J)+XFB  
YFB=FBUOY(2,J)+YFB  
ZFB=FBUOY(3,J)+ZFB  
XMB=FBUOY(4,J)+XMB

YMB=FBUOY(5,J)+YMB  
ZMB=FBUOY(6,J)+ZMB  
CONTINUE

801

-----  
Section 8.2 - Integrate damping force  
-----

DO 802 J=ISTA,LSTA  
XFD=FDAMP(1,J)+XFD  
YFD=FDAMP(2,J)+YFD  
ZFD=FDAMP(3,J)+ZFD  
XMD=FDAMP(4,J)+XMD  
YMD=FDAMP(5,J)+YMD  
ZMD=FDAMP(6,J)+ZMD  
CONTINUE

802

-----  
Section 8.3 - Integrate exciting force  
-----

DO 803 J=ISTA,LSTA  
XFX=FWVEX(1,J)+XFX  
YFX=FWVEX(2,J)+YFX  
ZFX=FWVEX(3,J)+ZFX  
XMX=FWVEX(4,J)+XMX  
YMX=FWVEX(5,J)+YMX  
ZMX=FWVEX(6,J)+ZMX  
CONTINUE

803

-----  
Section 8.4 - Integrate Speed Related forces  
-----

IF(JOBFO(2).NE.1) GO TO 814  
DO 804 J=ISTA,LSTA  
XFU=UDAMP(1,J)+UADMA(1,J)+UWVEX(1,J)+XFU  
YFU=UDAMP(2,J)+UADMA(2,J)+UWVEX(2,J)+YFU  
ZFU=UDAMP(3,J)+UADMA(3,J)+UWVEX(3,J)+ZFU  
XMU=UDAMP(4,J)+UADMA(4,J)+UWVEX(4,J)+XMU  
YMU=UDAMP(5,J)+UADMA(5,J)+UWVEX(5,J)+YMU  
ZMU=UDAMP(6,J)+UADMA(6,J)+UWVEX(6,J)+ZMU

804

CONTINUE

!Go directly here to skip over

814

-----  
Section 8.5 - Integrate flare force  
-----

IF(JOBFO(3).NE.1) GO TO 815  
DO 805 J=ISTA,LSTA  
YFF=FFLAR(2,J)+YFF  
ZMF=FFLAR(6,J)+ZMF

805

CONTINUE

!Directly to here to skip over

815

C4

SET UP MATRIX OF FORCE COEFFICIENTS FOR FORCES AND ACCELERATIONS

C4 IN SHIP COORDINATE DIRECTIONS..

C4 520 CONTINUE

C4

-----  
SECTION 9.0 - TOTAL FORCES

Add up forces. Then resolve forces and moments into the ship offset coordinate system.

-----  
XFT=XFB+XFD+XFX+XFN+XFW+XFU  
YFT=YFB+YFD+YFX+YFN+YFW+YFU+YFF  
ZFT=ZFB+ZFD+ZFX+ZFN+ZFU  
XMT=XMB+XMD+XMX+XMN+XMU  
YMT=YMB+YMD+YMX+YMN+YMU  
ZMT=ZMB+ZMD+ZMX+ZMN+ZMU+ZMF

C  
XF=XFT\*GAMMA  
YF=YFT\*GAMMA  
TEMP=YF  
ZF=ZFT\*GAMMA  
YF=YF\*CTHETA+ZF\*STHETA  
ZF=ZF\*CTHETA-TEMP\*STHETA  
XM=XMT\*GAMMA  
TEMP=YMT  
YM=(YMT\*CTHETA+ZMT\*STHETA)\*GAMMA  
ZM=(ZMT\*CTHETA-TEMP\*STHETA)\*GAMMA

-----  
SECTION 10.0 - LOAD MASS/INERTIA MATRIX A(I,J)

-----  
A(2,4)=0.0  
DO 530 I=1,9  
AM(I)=RHO\*AM(I)  
530 CONTINUE  
CC=CTHETA\*\*2  
CS=CTHETA\*STHETA  
SS=STHETA\*\*2  
A(2,2)=YFYA\*CC+ZFZA\*SS+SMASS  
A(2,3)=(ZFZA-YFYA)\*CS  
A(3,2)=A(2,3)  
A(2,5)=ZMYA\*CS  
A(2,6)=ZMYA\*CC  
A(3,3)=ZFZA\*CC+YFYA\*SS+SMASS  
A(3,4)=XMZA\*CTHETA  
A(4,3)=A(3,4)  
A(4,2)=XMZA\*STHETA  
A(4,4)=XMXR+PMI(1,1)  
A(4,5)=XMYR\*CTHETA+PMI(1,2)  
A(5,4)=A(4,5)  
A(4,6)=PMI(1,3)-XMYR\*STHETA  
A(6,4)=A(4,6)  
A(5,2)=(ZMYA+YMZA)\*CS

```
A(3,6)=-A(5,2)
A(6,3)=A(3,6)
A(5,3)=YMZA*CC-ZMYA*SS
A(3,5)=A(5,3)
A(5,5)=YMYR*CC+ZMZR*SS+PMI(2,2)
A(5,6)=(ZMZR-YMYR)*CS+PMI(2,3)
A(6,5)=A(5,6)
A(6,2)=ZMYA*CC-YMZA*SS
A(6,6)=ZMZR*CC+YMYR*SS+PMI(3,3)
RETURN
```

C

```
END
```

SUBROUTINE HULL  
This subroutine calculates hull girder loads

```
COMMON /C / C(3,3)
COMMON /IO/INPUT, OUTPUT, BIF, OFF, COF
INTEGER OUTPUT, BIF, OFF, COF
COMMON / / NWAVES, WVSUM,
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
4 WN(20), WNX(20), WNZ(20),
CSK(20), CCK(20), CXK(20),
CYK(20), CZK(20), XW(20)
COMMON / / MSTA, NPROF, NFWD, NAFTA, LPTS(25),
1 XOFF(25), YOFF(25,25), ZOFF(25,25),
2 XPROF(51), YPROF(51)
COMMON /COEFFX/THAH(25,21), TSAS(25,21), TRAR(25,21), TCCA(25,21),
* THVH(25,21), TSVS(25,21), TRVR(25,21), TCCV(25,21),
* THACX(25,21), THVCX(25,21), TSACX(25,21), TSVCX(25,21),
* TRACX(25,21), TCACX(25,21), THVCY(25,21),
* ITP(25), TXP(25), IFP(21), FXP(21)
COMMON /FORCE / XF, YF, ZF, XM, YM, ZM
REAL FORCE(6)
EQUIVALENCE (FORCE(1), XF)
COMMON /LHS / Y(13)
COMMON /MASS / RHO, G, GAMMA
COMMON /MASS / DISPL, SMASS, XCG, YCG, ZCG,
1 AMX, AMY, AMZ, RADII(6)
COMMON /MASS / PMI(3,3)
COMMON /RESIST/SPEED
COMMON /V / VX, VY, VZ
COMMON /SXPROP/ SEGMAS(26), SEGMOX(26), STRMAS(26), STRMDM(26),
* STRMOX(26), XBAR(26), YBAR(26), SEGWT(26), NWTSTA
COMMON /SXFOR/ FBUOY(6,25), FDAMP(6,25), FADMA(6,25), FWVEX(6,25),
* UDAMP(6,25), UADMA(6,25), UWVEX(6,25), FFLAR(6,25)
COMMON /SXGEO/ AREAS(25), DXS(25), DX1S(25), DX2S(25),
* YMOMS(25), ZMOMS(25), ISTA, LSTA
COMMON /ACCEL/ACC(6)
COMMON /LOADS/ SIN2(25), SIN3(25), SIN4(25), SIN5(25), SIN6(25),
1 V2(25), V3(25), V4(25), V5(25), V6(25),
2 RED(6,25)
COMMON /OPTION/JOBCO, JOBFO(10)
```

SECTION 1.0 GET SECTIONAL INERTIA/MOMENT OF INERTIAS

```
DO 90 J=1, MSTA
XBAR(J)=XOFF(J)
STRMAS(J)=SEGMAS(J)
SIN2(J)=0.
SIN3(J)=0.
SIN4(J)=0.
SIN5(J)=0.
SIN6(J)=0.
DO 91 I=1,6
RED(I, J)=0.
CONTINUE
CONTINUE
```

```
DO 100 J=1, MSTA
DSTRPI=XOFF(J)
DO 101 JJ=1, J
```

```
C
TVACCL=ACC(2)+XBAR(JJ)*ACC(6)
THACCL=ACC(3)-XBAR(JJ)*ACC(5)+YBAR(JJ)*ACC(4)
C-----NOTE!!! We are adding in weight of the section
C by including gravitational acceleration. This
C is different from standard strip theory load
C computations. (They compute wave induce dynamic
C loads) We are computing total loads at each time
C step... We have the TOTAL HYDROSTATIC RESTORING
C force not just the change due to unit wave/motion
C [see eq. 70 STF METHOD/TRANS SNAME '70]
```

```
GVACCL=-G*COS(Y(4))
GHACCL=G*SIN(Y(4))
```

```
HACCL=-THACCL+GHACCL
VACCL=-TVACCL+GVACCL
```

```
C
C
C SIN2(J)=STRMAS(JJ)*VACCL+SIN2(J)
C SIN3(J)=STRMAS(JJ)*HACCL+SIN3(J)
C SIN4(J)=STRMOX(JJ)*ACC(4)-((STRMAS(JJ)*YBAR(JJ))*
1 (ACC(3)+XBAR(JJ)*ACC(5)))+SIN4(J)
C DSTRPI=XOFF(J)-(.5*DXS(J))
C ZETMX=XBAR(JJ)-DSTRPI
C SIN5(J)=ZETMX*STRMAS(JJ)*HACCL+SIN5(J)
C SIN6(J)=ZETMX*STRMAS(JJ)*VACCL+SIN6(J)
```

```
C 101 CONTINUE
```

```
C-----
C-----
C 100 CONTINUE
```

```
C-----
C SECTION 2.0 GET ADDED MASS FORCES FADMA( , )
C-----
```

```
C
C DO 200 J=1,MSTA
C DO 200 I=1,6
C FADMA(I,J)=0.
200 CONTINUE
```

```
C DO 201 J=ISTA,LSTA
```

```
C
C AREA=AREAS(J)
C DX=DXS(J)
C DX1=DX1S(J)
C DX2=DX2S(J)
C YMOM=YMOMS(J)
C ZMOM=ZMOMS(J)
```

```
C
C DVOL=AREA*DX
C ADX1=AREA*DX1
C ADX2=AREA*DX2
C YDX=YMOM*DX
C ZDX=ZMOM*DX
```





```
FBARM=FBUOY(I, JJ)*ARMX
FAARM=FADMA(I, JJ)*ARMX
FUARM=(UADMA(I, JJ)+UDAMP(I, JJ)+UWVEX(I, JJ))*ARMX
FFARM=FFLAR(I, JJ)*ARMX
```

C

```
REDSUM=-FBARM-FDARM-FAARM-FWARM+REDSUM
IF(JOBFO(2).EQ.1) REDSUM=REDSUM-FUARM
IF(JOBFO(3).EQ.1) REDSUM=REDSUM-FFARM
```

312

```
CONTINUE
```

C

```
RED(IX, J)=REDSUM*GAMMA
```

C

311

```
CONTINUE
```

310

```
CONTINUE
```

C

C

C

C

```
DO 305 J=1, MSTA
V2(J)=SIN2(J)-RED(2, J)
V3(J)=SIN3(J)-RED(3, J)
V4(J)=SIN4(J)-RED(4, J)
V5(J)=SIN5(J)-RED(5, J)
V6(J)=SIN6(J)-RED(6, J)
```

C

305

```
CONTINUE
```

C

```
C-----OUTPUT FORCES FOR EACH SECTION
```

```
RETURN
```

69

```
FORMAT(1X, I2, 1X, 6F15.4)
END
```

## SUBROUTINE KRYLOV (J)

```

C
COMMON / / N WAVES, WVSUM,
1 WVAMP(20), WVFRE(20), WVDIR(20), WVPHA(20),
3 WN(20), WNX(20), WNZ(20),
4 CSK(20), CCK(20), CXK(20),
COMMON / / MYK(20), CZK(20), XW(20)
1 COMMON / / MSTA, NPROF, NFWD, NAFTA, LPTS(25),
2 XOFF(25), YOFF(25,25), ZOFF(25,25),
3 XPROF(51), YPROF(51)
COMMON / / INPTS(25), IPOINT(25), YWET(25,25), ETA(25,25),
1 ZLAST(25,25), ZWET(25,25), ZNEXT(25,25)
COMMON / / ZCO(20), ZSO(20), ZCY(20), ZSY(20),
1 ZCZ(20), ZSZ(20)
COMMON / / DZK(20)
COMMON / / JSAVE(20), COSJ(20), SINJ(20),
$ COSJJ(20), SINJJ(20),
1 DWC(20), DWS(20), XCI(20), XSI(20)
COMMON / / ISAVE(20), CSI(20), SNI(20),
$ CSII(20), SNII(20),
1 DKC(20), DKS(20), ZCI(20), ZSI(20)
COMMON /AREA / AREA, YMOM, ZMOM
AREA=0.0
YMOM=0.0
ZMOM=0.0
KSAVE=0
DO 110 K=1, N WAVES
    ZCO(K)=0.0
    ZSO(K)=0.0
    ZCY(K)=0.0
    ZSY(K)=0.0
    ZCZ(K)=0.0
    ZSZ(K)=0.0
    ISAVE(K)=0
110 CONTINUE
    LI=INPTS(J)
    II=IPOINT(J)
    LAST=II-1
111 DO 190 I=II, LI
        ETA1=ETA(I, J)
        Y1=YWET(I, J)
        Z0=ZLAST(I, J)
        Z1=ZWET(I, J)
        Z2=ZNEXT(I, J)

C3
        DZ0=Z1-Z0
        DZ2=Z2-Z1
        DZ=(Z2-Z0)/2.0
        ZA=Z1-DZ0/3.0
        ZB=Z1+DZ2/3.0
        DZ1=(DZ2*ZB+DZ0*ZA)/2.0
        AREA=AREA+(ETA1-Y1)*DZ
        ZMOM=ZMOM+(ETA1-Y1)*DZ1
        YMOM=(ETA1**2-Y1**2)/2.0*DZ+YMOM
        IF (N WAVES. EQ. 0) GO TO 190

C
        IF (KSAVE. EQ. 0) GO TO 112
        SQ0=SQ1
        SQ1=SQ2
        DDZ0=DDZ2

```

```

DZZO=DZZZ
GO TO 130
112 SGO=Z0**2
SQ1=Z1**2
IF (DZ0.EQ.0.0) GO TO 120
IF ((DZ2/DZ0).GT.1.0E+06) GO TO 120
DDZO=1.0/DZ0
DZZO=Z0*DDZO
GO TO 130
120 DDZO=0.0
DZZO=0.0
130 IF (DZ2.EQ.0.0) GO TO 140
IF ((DZ0/DZ2).GT.1.0E+06) GO TO 140
DDZ2=1.0/DZ2
DZZ2=Z2*DDZ2
GO TO 150
140 DDZ2=0.0
DZZ2=0.0
150 SQ2=Z2**2
KSAVE=0
IF (I.EQ.LAST) GO TO 151
IN=I
IF (IN.EQ.LI) IN=0
IF (Z1.EQ.ZLAST(IN+1,J) .AND. Z2.EQ.ZWET(IN+1,J)) KSAVE=1
151 CONTINUE
IF(ETA1.GT.0. .AND. Y1.GT.0. ) ETA1=Y1
DO 180 K=1,NWAVES
  CK=CCK(K)
  YK=CYK(K)
  ZK=CZK(K)
  EE=EXP(CK*ETA1)
  EY=EXP(CK*Y1)
  ARG=YK*ETA1
  CE=COS(ARG)
  SE=SIN(ARG)
  ARG=YK*Y1
  CY=COS(ARG)
  SY=SIN(ARG)
  D=CK**2+YK**2
  CK=CK/D
  YK=YK/D
  COSE=(CK*CE+YK*SE)*EE
  SINE=(CK*SE-YK*CE)*EE
  COSY=(CK*CY+YK*SY)*EY
  SINY=(CK*SY-YK*CY)*EY
  YCO=COSE-COSY
  YSO=SINE-SINY
  YCY=ETA1*COSE-Y1*COSY-CK*YCO-YK*YSO
  YSY=ETA1*SINE-Y1*SINY-CK*YSO+YK*YCO
C
  IF (ISAVE(K).EQ.0) GO TO 152
  COS1=CSI(K)
  SIN1=SNI(K)
  GO TO 153
152 ARG1=ZK*Z1
  COS1=COS(ARG1)
  SIN1=SIN(ARG1)
153 CONTINUE
IF (ABS(ZK*DZ).GT.0.01) GO TO 160
C
DZCOS=DZ*COS1

```

```
DZSIN=DZ*SIN1
DZ1COS=DZ1*COS1
DZ1SIN=DZ1*SIN1
ISAVE(K)=0
GO TO 170
```

C 160

```
ZK2=DZK(K)
ZCOS1=COS1*Z1
ZSIN1=SIN1*Z1
```

C

```
IF (ISAVE(K).EQ.0) GO TO 161
COS0=CSII(K)
SIN0=SNII(K)
DKCO=DKC(K)
DKSO=DKS(K)
ZCIO=ZCI(K)
ZSIO=ZSI(K)
GO TO 162
```

C

161

```
ARGO=ZK*ZO
COS0=COS(ARGO)
SIN0=SIN(ARGO)
DKCO=(COS1-COS0)/ZK
DKSO=(SIN1-SIN0)/ZK
ZCIO=(ZSIN1-SIN0*ZO+DKCO)*DDZO
ZSIO=(COS0*ZO-ZCOS1+DKSO)*DDZO
```

C

162

```
ARG2=ZK*Z2
COS2=COS(ARG2)
SIN2=SIN(ARG2)
DKC2=(COS2-COS1)/ZK
DKS2=(SIN2-SIN1)/ZK
ZCI2=(SIN2*Z2-ZSIN1+DKC2)*DDZ2
ZSI2=(ZCOS1-COS2*Z2+DKS2)*DDZ2
```

C

1

1

```
DZCOS=(ZCIO-ZCI2)/ZK-DZZO*DKSO+DZZ2*DKS2
DZSIN=(ZSIO-ZSI2)/ZK+DZZO*DKCO-DZZ2*DKC2
DZ1COS=((SQ1*SIN1-SQ0*SIN0)*DDZO+(SQ1*SIN1-SQ2*SIN2)*DDZ2+(Z
SIO-ZSIO)*ZK2+Z2*ZCI2-ZO*ZCIO)/ZK
DZ1SIN=((SQ2*COS2-SQ1*COS1)*DDZ2+(SQ0*COS0-SQ1*COS1)*DDZO+(Z
CIO-ZCI2)*ZK2+Z2*ZSI2-ZO*ZSIO)/ZK
```

C

```
ISAVE(K)=KSAVE
IF (KSAVE.EQ.0) GO TO 170
CSI(K)=COS2
SNI(K)=SIN2
CSII(K)=COS1
SNII(K)=SIN1
DKC(K)=DKC2
DKS(K)=DKS2
ZCI(K)=ZCI2
ZSI(K)=ZSI2
```

C

170

C

```
CONTINUE
```

```
ZCO(K)=ZCO(K)+DZCOS*YCO-DZSIN*YSO
ZSO(K)=ZSO(K)+DZCOS*YSO+DZSIN*YCO
ZCY(K)=ZCY(K)+DZCOS*YCY-DZSIN*YSY
ZSY(K)=ZSY(K)+DZCOS*YSY+DZSIN*YCY
ZCZ(K)=ZCZ(K)+DZ1COS*YCO-DZ1SIN*YSO
ZSZ(K)=ZSZ(K)+DZ1COS*YSO+DZ1SIN*YCO
```

```

180 CONTINUE
190 CONTINUE
   IF (LI.EQ.LAST) RETURN
   LI=LAST
   II=1
   GO TO 111

C
  END
  FUNCTION LNEGF(M,N,N1,A,B,DTRMNT,Z)
C.. SOLVES SIMULTANEOUS LINEAR EQUATIONS BY GAUSSIAN REDUCTION.
C.. FORTRAN IV EQUIVALENT OF LNEGS.
  REAL A(M,M),B(M,M),Z(M),DTRMNT,RMAX,RNEXT,W,DOV
  NM1=N-1
  DO 200 J=1,NM1
    J1=J+1
C.. FIND ELEMENT OF COL J, ROWS J-N, WHICH HAS MAX ABSOLUTE VALUE.
    LMAX=J
    RMAX=ABS(A(J,J))
    DO 110 K=J1,N
      RNEXT=ABS(A(K,J))
      IF (RMAX.GE.RNEXT) GO TO 110
      RMAX=RNEXT
      LMAX=K
110 CONTINUE
    IF (LMAX.NE.J) GO TO 120
C.. MAX ELEMENT IN COLUMN IS ON DIAGONAL
    IF (A(J,J)) 150,290,150
C.. MAX ELEMENT IS NOT ON DIAGONAL. EXCHANGE ROWS J AND LMAX.
120 DO 130 L=J,N
      W=A(J,L)
      A(J,L)=A(LMAX,L)
130 A(LMAX,L)=W
      DO 140 L=1,N1
        W=B(J,L)
        B(J,L)=B(LMAX,L)
140 B(LMAX,L)=W
      DTRMNT=-DTRMNT
C.. ZERO COLUMN J BELOW THE DIAGONAL.
150 Z(J)=1./A(J,J)
      DO 190 K=J1,N
        IF (A(K,J)) 160,190,160
160 W=-Z(J)*A(K,J)
        DO 170 L=J1,N
          A(K,L)=W*A(J,L)+A(K,L)
170 DO 180 L=1,N1
          B(K,L)=W*B(J,L)+B(K,L)
180 CONTINUE
190 CONTINUE
200 CONTINUE
    IF (A(N,N)) 210,290,210
210 Z(N)=1./A(N,N)
C.. OBTAIN SOLUTION BY BACK SUBSTITUTION.
    DO 220 L=1,N1
      B(N,L)=Z(N)*B(N,L)
220 DO 250 K=1,NM1
      J=N-K
      J1=J+1
      DO 240 L=1,N1
        W=0.
        DO 230 I=J1,N
          W=A(J,I)*B(I,L)+W
230 B(J,L)=(B(J,L)-W)*Z(J)
240

```

```
250 CONTINUE
C. . EVALUATE DETERMINANT.
    IF (DTRMNT) 260, 280, 260
260 DO 270 J=1, N
270 DTRMNT=DTRMNT*A(J, J)
280 LNEGF=1
    RETURN
C. . SINGULAR MATRIX, SET ERROR FLAG.
290 LNEGF=2
    DTRMNT=0.
    RETURN
C
    END
```

PROGRAM HYDREX2

PROGRAM HYDREX2

```

C CHARACTER*25 OFFIL,BIFIL,COFIL
C CHARACTER*30 TITLE
COMMON/HEAD/TITLE
COMMON/IOFILE/ OFFIL,BIFIL,COFIL
COMMON/IO/INPUT,OUTPUT,BIF,OFF,COF
INTEGER OUTPUT,BIF,OFF,COF
COMMON /COEFF4 / COEFF4(6,12,8,25),AREAN(25,6)
COMMON/SHIP/ISTA,LSTA,ISWL,LSWL,TF,TA,XCG,YCG,DISPL
COMMON /DRFT12/ DRAFT(6,2),IDRAFT
COMMON /SIGMA / NK, SIGMA(24), SIGMAO, ERRO, DM(12)
COMMON /U / RHO, G
COMMON/GEOMETRY/MSTA,LPTS(25),YOFF(25,25),NAFT,XAFT(25),
* YAFT(25),NFWD,XFWD(25),YFWD(25),XOFF(25),
* ZOFF(25,25),XPERP,XAPERP,SHIPL,SHIPB,SHIPT,
* Y1(21,25),ZWL(25),WL(25),INPTS(25),XWLF,XWLA,XXF,
* XXA,TAN,NON,NOE,NWL,CR,XXFWD,XXSTA,XXAFT,DX,DX1,
* DX2,Z2(21),Y2(21),ZZ(20),YY(20),SNE(20),CSE(20),
* DEL(20),ROL(20),ADJUST,WMAX,YMAX,ZMAX,AREA,VERT
LOGICAL WL,ADJUST
DATA RHO /0.00088861607142/
DATA ERRO/1.0E-37/
DATA DM/ 0.000, 0.200, 0.400, 0.600, 0.800, 1.000,
1 1.200, 1.600, 2.000, 2.400, 2.800, 9.999/
C DATA DM/0.0,0.0992,.1718,.243,.3137,.3842,.45469,
C * .525,.59534,.49611,.7358,9.999/
DATA DM/0.0,0.099,0.2,0.3,0.4,0.5,0.6,0.7,0.9,
* 1.3,1.6,2.2/
DATA INPUT/5/,OUTPUT/6/,BIF/1/,OFF/2/,COF/3/

```

---

\*\*\* 1.0 Get DATA file names and assign/open files

```

TYPE 902
ACCEPT 901,BIFIL
TYPE 903
ACCEPT 901,OFFIL
TYPE 904
ACCEPT 901,COFIL

```

```

OPEN(UNIT=BIF,STATUS='OLD',FILE=BIFIL)
OPEN(UNIT=OFF,STATUS='OLD',FILE=OFFIL)
OPEN(UNIT=COF,STATUS='NEW',FORM='UNFORMATTED',FILE=COFIL)

```

\*\*\* 2.0 Read in data from two files

```

CALL INDATA
CLOSE(BIF)
CLOSE(OFF)
DO 10 J=1,6
DRAFT(J,2)=DRAFT(J,1) !Aft draft = fwd draft
10 CONTINUE

```

\*\*\* 3.0 Float ship at indexed draft then compute added mass and damping coefficients using Frank Close Fit.

```

DO 20 IDRAFT=1,6
CALL FLOAT
CALL HYDRO
20 CONTINUE

```

\*\*\* 4.0 Write to unformatted coefficient file containing



added mass and damping coefficients for each station  
at six drafts and twelve frequencies.

```
WRITE (COF) MSTA !Number of Stations
WRITE (COF) (OM(I), I=1,12) !Frequency (rad/sec)
WRITE (COF) (DRAFT(I,1), I=1,6) !Drafts
DO 30 L=1, MSTA !Station index
DO 30 K=1,6 !Draft index
DO 30 J=1,12 !Frequency index
WRITE (COF) (COEFF4(K, J, I, L), I=1,8)
30 CONTINUE
WRITE (COF) ((AREAN(L, K), K=1,6), L=1, MSTA) !Section areas
STOP
```

```
901 FORMAT(A)
902 FORMAT(' Name of Basic INPUT File [BIF] ? > '$)
903 FORMAT(' Name of Offset File [OFF] ? > '$)
904 FORMAT(' Name of Coefficient File [COF] ? > '$)
```

```
END
SUBROUTINE HYDRO
```

CALCULATION OF HYDRODYNAMIC FORCE COEFFICIENTS FOR THE SHIP.

```
COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF
INTEGER OUTPUT, BIF, OFF, COF
COMMON/HEAD/TITLE
CHARACTER*30 TITLE
COMMON /COEFF4/ COEFF4(6, 12, 8, 25), AREAN(25, 6)
COMMON/DRFT12/DRAFT(6, 2), IDRAFT
COMMON/SHIP/ISTA, LSTA, ISWL, LSWL, TF, TA, XCG, YCG, DISPL
COMMON /SIGMA / NK, SIGMA(24), SIGMA0, ERRO, OM(12)
COMMON /U / RHO, G
COMMON/GEOMETRY/MSTA, LPTS(25), YOFF(25, 25), NAFT, XAFT(25),
* YAFT(25), NFWD, XFWD(25), YFWD(25), XOFF(25),
* ZOFF(25, 25), XPERP, XAPERP, SHIPL, SHIPB, SHIPT,
* Y1(21, 25), ZWL(25), WL(25), INPTS(25), XWLF, XWLA, XXF,
* XXA, TAN, NON, NOE, NWL, CR, XXFWD, XXSTA, XXAFT, DX, DX1,
* DX2, Z2(21), Y2(21), ZZ(20), YY(20), SNE(20), CSE(20),
* DEL(20), ROL(20), ADJUST, WMAX, YMAX, ZMAX, AREA, VERT
LOGICAL ADJUST, WL
COMMON / HA1, SA1, RA1, CA1, HV1, SV1, RV1, CV1,
1 RHO2, RSIG, WN, W1, W2, ERR, XRI, YRI, EJT
COMMON //HAO(24), SAO(24), RAO(24), CAO(24),
1 HVO(24), SVO(24), RVO(24), CVO(24)
REAL FA(24, 8)
EQUIVALENCE (FA(1, 1), HAO(1))
COMMON / BLOGP(20, 20), YLOGP(20, 20),
1 BLOGM(20, 20), YLOGM(20, 20)
REAL COEFF0(8), COEFF1(8)
EQUIVALENCE (COEFF1(1), HA1)
DATA TPI /6.2831853072/
NODES=NWL
ERR=ERRO
RHO2=RHO*2.0
XXSTA=XXF
XWO=XWLF
VLO=0.0
VOL1=0.0
VOLV=0.0
WPO=0.0
```

WP1=0.0  
WP2=0.0  
WPT=0.0

CR IS THE CENTER OF ROLL. ROLL IS ASSUMED TO BE ABOUT THE HORIZONTAL AXIS THROUGH THE GIVEN CENTER OF GRAVITY.

CR=YCG-DRAFT(IDRAFT,1)-(XFPERP-XCG)\*TAN  
STRIP CALCULATION PROCEDURE FOR EACH WET STATION..

DO 240 J=ISTA, LSTA  
XXFWD=XXSTA  
XXSTA=XOFF(J)  
IF (J.EQ.LSTA) GO TO 122  
XXAFT=XOFF(J+1)  
XW2=XXAFT  
IF (J.EQ.LSWL) XW2=XWLA  
GO TO 124  
122 XXAFT=XXA  
XW2=XWLA  
124 X1=XOFF(J)-XCG

INTEGRATE OVER WATERPLANE OF THE SHIP..

IF (J.LT.ISWL) GO TO 130  
DXFWD=XWO-XXSTA  
XWO=XXSTA  
IF (.NOT.WL(J)) GO TO 130  
DXAFT=XWO-XW2  
DX=DXFWD+DXAFT  
A=X1-DXAFT/3.0  
B=X1+DXFWD/3.0  
XW2=DXFWD\*B\*B+DXAFT\*A\*A+(DXFWD\*\*3+DXAFT\*\*3)/18.0  
XW1=DXFWD\*B+DXAFT\*A  
ZW=ZWL(J)  
DZW=ZW\*DX  
WPO=WPO+DZW  
WP1=WP1+ZW\*XW1  
WP2=WP2+ZW\*XW2  
WPT=WPT+DZW\*ZW\*\*2

SET UP STATION GEOMETRY..

130 NON=INPTS(J)  
IF (NON.EQ.0) GO TO 140  
NWL=0  
IF (WL(J)) NWL=NODES  
CALL STATN (ZOFF(1,J), Y1(1,J), ZWL(J), INPTS(J))  
IF (NON.LE.0) GO TO 140

INTEGRATE OVER SUBMERGED VOLUME OF THE SHIP..

DXFWD=XXFWD-XXSTA  
DXAFT=XXSTA-XXAFT  
DX=(DXFWD+DXAFT)/2.0  
VOL0=VOL0+AREA\*DX  
A=X1-DXAFT/3.0  
B=X1+DXFWD/3.0  
DX2=(DXFWD\*B\*B+DXAFT\*A\*A)/2.0+(DXFWD\*\*3+DXAFT\*\*3)/36.0  
DX1=(DXFWD\*B+DXAFT\*A)/2.0  
VOL1=VOL1+AREA\*DX1  
VOLV=VOLV+VERT\*DX

```

C 140 CONTINUE
C DO 150 I=1,8
C   CDEFFO(I)=0.0
C 150 CONTINUE
C CALL GIRL
C
C DO 210 K=1,12
C   ESIG=DM(K)
C   RSIG=ESIG*RHO2
C   WN=ESIG*ESIG/G
C *** Zero (K=1) or Infinite (K=12) Frequency Computations
C   IF (K.EQ.1.OR.K.EQ.12) THEN
C     CALL BEER(K)
C   ELSE
C *** Non zero and non-infinite frequency computations
C     W1=TPI/WN
C     W2=2.0/WN
C     CALL WINE (K)
C   ENDIF
C
C-----
C 180 CONTINUE
C DO 190 I=1,8
C   COEFF4(IDRAFT,K,I,J)=COEFF1(I)/AREA
C 190 CONTINUE
C
C   AREAN(J, IDRAFT)=AREA
C   HAO(K)=HA1
C   SAO(K)=SA1
C   RAO(K)=RA1
C   CAO(K)=CA1
C   HVO(K)=HV1
C   SVO(K)=SV1
C   RVO(K)=RV1
C   CVO(K)=CV1
C 210 CONTINUE
C-----
C PRINTOUT Offset and Hydrodynamic Coefficient Info.
C WRITE (OUTPUT,300) TITLE
C WRITE(OUTPUT,301) J, XOFF(J), AREA, DRAFT(IDRAFT,1), CR,
C * DRAFT(IDRAFT,2)
C WRITE (OUTPUT,302)
C WRITE (OUTPUT,310) (DM(K), HAO(K), HVO(K), SAO(K), SVO(K), RAO(K),
C 1 RVO(K), CAO(K), CVO(K), K=1,12)
C *** Offset Information
C WRITE(OUTPUT,705)
C WRITE(OUTPUT,706)
C N=LPTS(J)
C M=0
C IF (J.LT.ISTA) GO TO 670
C IF (J.GT.LSTA) GO TO 670
C M=INPTS(J)
C IF (M.NE.0) WRITE(OUTPUT,710) (YOFF(I,J), ZOFF(I,J), Y1(I,J), ZOFF
C 1 (I,J), I=1,M)

```

```

IF (WL(J)) WRITE (OUTPUT,720) ZWL(J)
670 M=M+1
IF (M.LE.N) WRITE (OUTPUT,730) (YOFF(I,J),ZOFF(I,J),I=M,N)
C
220 CONTINUE
240 CONTINUE
GAMMA=RHO*G
DISPL=VOLO*GAMMA
C
WPT=WPT/3.0
YFYTO=WPO*GAMMA
YFZRO=WP1*GAMMA
XWA=0.0
IF (WPO.NE.0.0) XWA=WP1/WPO
XCB=VOL1/VOLO
YCB=VOLV/VOLO
XBM=WPT/VOLO
ZBM=(WP2-WP1*XWA)/VOLO
XGM=XBM+YCB
ZGM=ZBM+YCB
XWA=XWA+XCG
XCB=XCB+XCG
YCB=YCB+YCG
XMXRO=XGM*DISPL
ZMZRO=ZGM*DISPL
WRITE (OUTPUT,270) TITLE
WRITE (OUTPUT,281) DRAFT(IDRAFT,1),DRAFT(IDRAFT,2)
WRITE(OUTPUT,290) WPO,XWA,WP1,WP2,WPT
WRITE(OUTPUT,291) DISPL,VOLO,XCB,YCB,VOL1,VOLV
WRITE(OUTPUT,292) ZBM,XBM,ZGM,XGM
WRITE(OUTPUT,293) XMXRO,ZMZRO,YFYTO,YFZRO
WRITE(OUTPUT,294) XXF,XXA
WRITE (OUTPUT,280) XCG,YCG
C *** OUTPUT OF NOTES
WRITE(OUTPUT,420)
IF (ADJUST) WRITE(OUTPUT,430) ZMAX,YMAX
IF (.NOT.ADJUST) WRITE (OUTPUT,440)
IF (NWL.GT.0) GO TO 110
GO TO 120
110 WRITE (OUTPUT,460) NWL
IF (WMAX.GT.0.0) WRITE (OUTPUT,470) WMAX
120 CONTINUE
C
RETURN
270 FORMAT (1H1/,81(1H=)/,1X,A30,'HYDROSTATIC COEFFICIENTS'
*,T64,'PROGRAM HYDREX',/,81(1H=)/)
280 FORMAT (//81(1H-)//,' NOTE: All moments are about',
*'center of gravity' /,7X,'XCG=',F11.3,' YCG=',F11.3,/)
281 FORMAT(T5,' DRAFT (fwd) =',F8.2,/,T5,' DRAFT (aft) =',F8.2/)
290 FORMAT(T10,'----- WATERPLANE -----',/)
* T10,'Area',,T40,F13.2,T60,' units',/,/
* T10,'LCF',,T40,F13.2,T60,' units',/,/
* T10,'1st Long. Moment',,T40,E13.7,T60,' units',/,/
* T10,'2nd Long. Moment',,T40,E13.7,T60,' units',/,/
* T10,'2nd Transv. Moment',,T40,E13.7,T60,' units',/,/
291 FORMAT(T10,'----- VOLUME -----',/)
* T10,'Displacement',,T40,F13.2,T60,' units',/,/
* T10,'Volume of Displacement',,T40,E13.7,T60,' units',/,/
* T10,'LCB',,T40,F13.2,T60,' units',/,/
* T10,'VCB',,T40,F13.2,T60,' units',/,/
* T10,'Long. Moment',,T40,E13.7,T60,' units',/,/

```

```

* T10, 'Vert. Moment', T40, E13.7, T60, ' units', //)
292 FORMAT(T10, '-----METACENTRIC HEIGHTS', //)
* T10, 'BM (longitudinal)', T40, F13.2, T60, ' units', //)
* T10, 'BM (transverse)', T40, F13.2, T60, ' units', //)
* T10, 'GM (longitudinal)', T40, F13.2, T60, ' units', //)
* T10, 'GM (transverse)', T40, F13.2, T60, ' units', //)
293 FORMAT(T10, '-----HYDROSTATIC FORCES', //)
* T10, 'Roll Restoring Moment', T40, E13.7, T60, ' units', //)
* T10, 'Pitch Restoring Moment', T40, E13.7, T60, ' units', //)
* T10, 'Heave Restoring Force', T40, E13.7, T60, ' units', //)
* T10, 'Pitch Induced Heave Force', T40, E13.7, T60, ' units', //)
294 FORMAT(/, T5, 'LWL begins at', F10.2, /
*, T5, 'LWL ends at', F10.2, //)
300 FORMAT(1H1/, 81(1H=), /, 1X, A30, 'ADDED MASS/DAMPING COEFFICIENTS'
*, T66, 'PROGRAM HYDREX', /, 81(1H=))
301 FORMAT(T36, 'Station', I2/, T36, '-----', /,
*, T5, 'Dist. from F.P.', F8.2, T50, 'Area', F11.3, /
*, T5, 'DRAFT (fwd)', F8.2, T50, 'Roll Ctr abv WL', F11.3/
*, T5, 'DRAFT (aft)', F8.2, //)
302 FORMAT(10X, '-----HEAVE-----',
*, 4X, '-----SWAY-----', 4X, '-----ROLL-----',
*, 4X, '---SWAY-ROLL---', /, 14X, 'A22', 5X, 'B22',
*, 7X, 'A33', 5X, 'B33', 8X, 'A44', 6X, 'B44', 6X, 'A34', 6X, 'B34', /
*, 2X, 'Freq.', //)
309 FORMAT ((1X, OPF22.4, 3(5X, 1P2E10.2)))
310 FORMAT(1X, F5.2, 3X, F8.4, F8.4, 2X, F8.4, F8.4,
*, 2X, F9.2, F9.2, F9.1, F9.1)

```

```

C
420 FORMAT (1H1/, 33(1H*), ' N O T E S ', 33(1H*))
430 FORMAT(T5, 'Generate additional offset points.', /,
*, T5, 'Maximum segment height=', F10.3/,
*, T5, 'Maximum segment width =', F10.3/)
440 FORMAT(/T5, 'Use segments as defined by table of offsets. ')
450 FORMAT(T5, 'No internal free-surface segments are used. ')
460 FORMAT(T5, I3, 2X, 'nodes for internal free-surface. ')
470 FORMAT(5X, 'Add internal nodes if surface segment lengths',
*, ' exceed', F10.3)
706 FORMAT (6X, 'Height above', 5X, 'Half', 15X,
*, 'Submerged Offsets', /, 10X, 'Baseline', 5X, 'BREADth',
*, 14X, '-Y-', 9X, '-Z-'//)
705 FORMAT (///// , 33(1H-), 'STATION OFFSETS', 32(1H-), //)
710 FORMAT (6X, 2F12.3, F18.3, F12.3)
720 FORMAT (39X, 9HWATERLINE, F12.3)
730 FORMAT (6X, 2F12.3)

```

```

END
SUBROUTINE INDATA

```

```

C
COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF
INTEGER OUTPUT, BIF, OFF, COF
COMMON /SIGMA / NK, SIGMA(24), SIGMAO, ERRO, OM(12)
COMMON / DRFT12/ DRAFT(6,2), IDRAFT
COMMON/IOFILE/ OFFIL, BIFIL, COFIL
COMMON/HEAD/TITLE
CHARACTER*30 TITLE
COMMON /U / RHO, G
COMMON/SHIP/ISTA, LSTA, ISWL, LSWL, TF, TA, XCG, YCG, DISPL
COMMON/GEOMETRY/MSTA, LPTS(25), YOFF(25,25), NAFT, XAFT(25),
*, YAFT(25), NFWD, XFWD(25), YFWD(25), XOFF(25),
*, ZOFF(25,25), XPERP, XAPERP, SHIPL, SHIPB, SHIPT,
*, Y1(21,25), ZWL(25), WL(25), INPTS(25), XWLF, XWLA, XXF,
*, XXA, TAN, NON, NOE, NWL, CR, XXFWD, XXSTA, XXAFT, DX, DX1,

```

```
* DX2, Z2(21), Y2(21), ZZ(20), YY(20), SNE(20), CSE(20),
* DEL(20), ROL(20), ADJUST, WMAX, YMAX, ZMAX, AREA, VERT
LOGICAL ADJUST, WL
```

```
C CHARACTER*81 CARDID
DATA DEGREE/0.01745 32925 19943/
DATA G /32.17/
DATA NWL/1/
```

```
C WRITE(OUTPUT, 197)
```

```
C *** TITLE
```

```
READ (BIF, 199) TITLE
WRITE(OUTPUT, 196) TITLE
```

```
C *** DRAFT (fwd), DRAFT (aft), long. loc's of DRAFT marks
```

```
READ (BIF, 200) TF, TA, XFPERP, XAPERP
WRITE(OUTPUT, 200) TF, TA, XFPERP, XAPERP
```

```
C *** Center of Gravity (XCG aft of FP, YCG above BL)
```

```
READ (BIF, 200) XCG, YCG, ZCG
WRITE(OUTPUT, 200) XCG, YCG, ZCG
```

```
C *** Six DRAFTs at which hydro. coeffs are computed
```

```
READ(BIF, 200) (DRAFT(I, 1), I=1, 6)
WRITE(OUTPUT, 200) (DRAFT(I, 1), I=1, 6)
```

```
C *** Minimum segment lengths for Frank Close Fit
```

```
READ (BIF, 201) YMAX, ZMAX, WMAX, NWL
WRITE(OUTPUT, 201) YMAX, ZMAX, WMAX, NWL
ADJUST=ZMAX. GT. 0. 0. AND. YMAX. GT. 0. 0
```

```
C *** Number of forewate profile points
```

```
READ (BIF, 190) NFWD
WRITE(OUTPUT, 190) NFWD
IF (NFWD. GT. 25) CALL ERROR(15, IDUM, RDUM)
```

```
C *** Coordinates of forward profile points
```

```
IF (NFWD. GT. 0) READ (BIF, 430) (YFWD(I), XFWD(I), I=1, NFWD)
WRITE(OUTPUT, 430) (YFWD(I), XFWD(I), I=1, NFWD)
```

```
C *** Number of aft profile points
```

```
READ (BIF, 190) NAFT
WRITE(OUTPUT, 190) NAFT
IF (NAFT. GT. 25) CALL ERROR(16, IDUM, RDUM)
```

```
C *** Coordinates of aft profile points
```

```
IF (NAFT. GT. 0) READ (BIF, 430) (YAFT(I), XAFT(I), I=1, NAFT)
WRITE(OUTPUT, 430) (YAFT(I), XAFT(I), I=1, NAFT)
```

```
-----
Section 2.0 - READ OFFSET file
The offset file can be an actual SHCP DATA File
```

```
WRITE(OUTPUT, 198)
```

```
C *** CARD TYPE A
```

```
READ (OFF, 410) CARDID
WRITE(OUTPUT, 410) CARDID
```

```
C *** CARD TYPE B
```

```
READ (OFF, 410)
```

```
C *** CARD TYPE C
```

```
READ (OFF, 412) SPACE, ZSCAL, YSCAL, SHIPL, NAPN, KINDO
WRITE(OUTPUT, 412) SPACE, ZSCAL, YSCAL, SHIPL, NAPN, KINDO
IF (SPACE. EQ. 0. 0) SPACE=1. 0
ZSCAL=1. 0
YSCAL=1. 0
MSTA=0
```

NFWD=0  
NAFT=0

C  
C

\*\*\* CARD TYPE D

```
30 N=1
MSTA=MSTA+1
IF (MSTA.GT.25) CALL ERROR(10, IDUM, RDUM)
READ (OFF, 416) STATNO, Y11, Z1, JTEST
WRITE(OUTPUT, 417) STATNO, Y11, Z1, JTEST
XOFF(MSTA)=STATNO*SPACE
GO TO 50
40 CONTINUE                                !loop within each station
N=N+1
IF (N.GT.25) CALL ERROR(11, MSTA, RDUM)
READ (OFF, 416) S, Y11, Z1, JTEST
WRITE(OUTPUT, 417) S, Y11, Z1, JTEST
IF (S.NE.STATNO) CALL ERROR(12, MSTA, RDUM)
50 YOFF(N, MSTA)=Z1*ZSCAL
ZOFF(N, MSTA)=Y11*YSCAL
IF (JTEST.EQ.0 .OR. JTEST.EQ.77777) GO TO 40
LPTS(MSTA)=N                                !No. of points- MSTA
IF (N.LT.2) CALL ERROR(13, MSTA, RDUM)
IF (JTEST.EQ.88888) GO TO 30                !Go onto next station
IF (JTEST.NE.99999) CALL ERROR(14, JTEST, RDUM)
```

C  
C

```
DO 220 J=1, MSTA
XOFF(J)=-XOFF(J)
220 CONTINUE
IF (NFWD.EQ.0) GO TO 240
X=XOFF(1)
DO 230 I=1, NFWD
XFWD(I)=XFWD(I)+X
230 CONTINUE
240 CONTINUE
X=XOFF(MSTA)
DO 250 I=1, NAFT
XAFT(I)=X-XAFT(I)
250 CONTINUE
RETURN
180 FORMAT (5X, I5)
190 FORMAT (I5)
197 FORMAT(1H1/, 81(1H*)//, ' INPUT DATA ECHO ', T64,
*'PROGRAM HYDREX'//, 81(1H*)//, 33(1H-),
*'[BIF] DATA FILE', 32(1H-)//)
198 FORMAT(1H1/, 81(1H*)//, ' INPUT DATA ECHO ', T64,
*'PROGRAM HYDREX'//, 81(1H*)//, 33(1H-),
*'[OFF] DATA FILE', 32(1H-)//)
196 FORMAT(1X, A)
199 FORMAT(A)
200 FORMAT (6F10.2)
201 FORMAT (3F10.2, I5)
210 FORMAT (F10.2, I5, 5X, F10.2)
```

C

```
410 FORMAT (A)
412 FORMAT (4F10.3, 13X, I2, 4X, I1)
414 FORMAT (5X, I5, 5X, 'INPUT OF SHCP TYPE D OFFSET DATA')
416 FORMAT (F6.3, 2F7.0, I6)
417 FORMAT (F7.3, 2F10.2, I6)
420 FORMAT (2I5, F10.2)
430 FORMAT (2F10.2)
```

C

```

END
SUBROUTINE INSERT(A1,A2,J1,L1,L2)
Purpose: Inserts array A2 into array A1 at location J1.
REAL A1(1), A2(1)
IF (L1.LT.J1) GO TO 120
M=L1+L2
I=L1
K=L1-J1+1
DO 110 J=1,K
  A1(M)=A1(I)
  M=M-1
  I=I-1
110 CONTINUE
120 I=J1-1
DO 130 K=1,L2
  M=K+I
  A1(M)=A2(K)
130 CONTINUE
RETURN
END
SUBROUTINE STATN (Z, Y, ZW, NPTS)

```

CCCCCCCCCCCC

CALCULATION OF DATA CONCERNING STATION GEOMETRY.  
 REVISION OF OFFSETS FOR GOOD RESULTS MAY BE PERFORMED.

NON = NUMBER OF CALCULATED MIDPOINTS.  
 NWL = NUMBER OF WATERLINE MIDPOINTS.  
 Z = HORIZONTAL COORDINATE OF SEGMENT ENDPOINT.  
 Y = VERTICAL COORDINATE OF SEGMENT ENDPOINT.  
 ZZ = HORIZONTAL COORDINATE OF SEGMENT MIDPOINT.  
 YY = VERTICAL COORDINATE OF SEGMENT MIDPOINT.  
 SNE = HORIZONTAL COMPONENT OF UNIT NORMAL TO SEGMENT.  
 CSE = VERTICAL COMPONENT OF UNIT NORMAL.  
 DEL = LENGTH OF SEGMENT.  
 ROL = MOMENT OF UNIT NORMAL ABOUT CENTER OF ROLL (CG).

```

REAL Y(1), Z(1)
COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF
INTEGER OUTPUT, BIF, OFF, COF
COMMON/GEOMETRY/MSTA, LPTS(25), YOFF(25, 25), NAFT, XAFT(25),
* YAFT(25), NFWD, XFWD(25), YFWD(25), XOFF(25),
* ZOFF(25, 25), XFPERP, XAPERP, SHIPL, SHIPB, SHIPT,
* Y1(21, 25), ZWL(25), WL(25), INPTS(25), XWLF, XWLA, XXF,
* XXA, TAN, NON, NOE, NWL, CR, XXFWD, XXSTA, XAFT, DX, DX1,
* DX2, Z2(21), Y2(21), ZZ(20), YY(20), SNE(20), CSE(20),
* DEL(20), ROL(20), ADJUST, WMAX, YMAX, ZMAX, AREA, VERT
COMMON / / HA1, SA1, RA1, CA1, HV1, SV1, RV1, CV1,
1 RHO2, RSIG, WN, W1, W2, ERR, XRI, YRI, EJT
COMMON / / HAO(24), SAO(24), RAO(24), CAO(24),
1 HVO(24), SVO(24), RVO(24), CVO(24)
COMMON // ZNEW(20), YNEW(20), ZZNEW(20), YYNEW(20), CNEW(20), SNEW(20),
1DNEW(20), RNEW(20), MORE(20), I, K, ZO, YO, M1, MTOT, ZINT, YINT, NUMBER, M, N1
2, I1, JOB, ZS, YS, D, C, S, P, G, NUT
LOGICAL ADJUST
DATA LIMIT/20/
DATA MAXPTS/20/
AREA=0.0
VERT=0.0
ZO=0.0
YO=Y(1)

```



```
MTOT=0
K=1
I1=K
IF (Z(1).EQ.0.0) I1=2
IF (I1.GT.NPTS) GO TO 130
```

```
CCC
CALCULATION LOOP FOR SUBMERGED OFFSET POINTS..
```

```
DO 120 I=I1,NPTS
  ZINT=Z(I)-ZO
  YINT=Y(I)-YO
  D=SQRT(ZINT*ZINT+YINT*YINT)
  IF (D.EQ.0.0) GO TO 120
  IF (.NOT.ADJUST) GO TO 110
```

```
CCC
CODE INSTRUCTIONS FOR THE ADDITION OF POINTS..
```

```
NUMBER=MAXO(IABS(IFIX(ZINT/ZMAX)), IABS(IFIX(YINT/YMAX)))
MORE(K)=NUMBER
MTOT=MTOT+NUMBER
```

```
CCC
CONTINUE STANDARD PROCEDURE..
```

```
110  ZS=ZO+Z(I)
     YS=YO+Y(I)
     AREA=AREA+YINT*ZS
     VERT=VERT+YINT*(ZO*(YO-CR+YINT/3.0)+Z(I)*(YO-CR+YINT/1.5))
     C=ZINT/D
     S=YINT/D
     CSE(K)=C
     SNE(K)=S
     DEL(K)=D
     ZZ(K)=0.5*ZS
     YY(K)=0.5*YS
     ROL(K)=(CR-YY(K))*S-ZZ(K)*C
     Z2(K)=ZO
     Y2(K)=YO
     ZO=Z(I)
     YO=Y(I)
     K=K+1
```

```
120 CONTINUE
```

```
CCC
END OF CALCULATION LOOP FOR SUBMERGED POINTS.
```

```
130  Z2(K)=ZO
     Y2(K)=YO
     NON=K-1
```

```
CCC
ADD UPPERMOST SEGMENT..
```

```
C
IF (NWL.NE.0) GO TO 150
SECTION IS SUBMERGED.
IF (ZO.EQ.0.0) GO TO 200
IF (.NOT.ADJUST) GO TO 140
```

```
CCC
CODE INSTRUCTIONS FOR THE ADDITION OF POINTS..
```

```
C
NUMBER=IABS(IFIX(ZO/ZMAX))
MORE(K)=NUMBER
MTOT=MTOT+NUMBER
```

C C CONTINUE STANDARD PROCEDURE. .

140 DEL(K)=ABS(ZO)  
CSE(K)=-1.0  
SNE(K)=0.0  
ZZ(K)=0.5\*ZO  
YY(K)=YO  
ROL(K)=ZZ(K)  
NON=K  
K=K+1  
Z2(K)=0.0  
Y2(K)=YO  
GO TO 200

C C C ADD SEGMENT UP TO WATERLINE. .

150 ZINT=ZW-ZO  
YINT=-YO  
IF (NWL.LT.0) NWL=0  
IF (ZW.LE.0.0) NWL=0  
D=SQRT(ZINT\*ZINT+YINT\*YINT)  
IF (D.EQ.0.0) GO TO 170  
IF (.NOT.ADJUST) GO TO 160

C C C CODE INSTRUCTIONS FOR THE ADDITION OF POINTS. .

NUMBER=MAXO(IABS(IFIX(ZINT/ZMAX)), IABS(IFIX(YINT/YMAX)))  
MORE(K)=NUMBER  
MTOT=MTOT+NUMBER

C C C CONTINUE STANDARD PROCEDURE. .

160 ZS=ZO+ZW  
YS=YO  
AREA=AREA+YINT\*ZS  
VERT=VERT-YINT\*(ZO\*(CR+YINT/1.5)+ZW\*(CR+YINT/3.0))  
C=ZINT/D  
S=YINT/D  
CSE(K)=C  
SNE(K)=S  
DEL(K)=D  
ZZ(K)=0.5\*ZS  
YY(K)=0.5\*YS  
ROL(K)=(CR-YY(K))\*S-ZZ(K)\*C  
NON=K  
K=K+1  
Z2(K)=ZW  
Y2(K)=0.0

C C C ADD DECK AT WATERLINE. .

170 CONTINUE  
IF (NWL.EQ.0) GO TO 200  
ZINT=ZW/FLOAT(NWL)  
IF (WMAX.EQ.0.0) GO TO 180  
IF (ZINT.LT.WMAX) GO TO 180  
NWL=IFIX(ZW/WMAX)+1  
ZINT=ZW/FLOAT(NWL)

180 D=ZINT\*0.5  
DO 190 I=1, NWL  
DEL(K)=ZINT

C

```

SUBROUTINE POTST
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120), ST(120),
* ACN(120), ACNW(120), AN(120,3), E(120), P(120,6), PRFS(120),
* STOLD(120), PX(120,6)
COMMON/BD2/XPT(150), YPT(150), ZPT(150), WRF(150), WRFR(150),
* KK(150,4)
COMMON/A/NPAN, NPT, GEE, RHO, NKX, NKY, EYE, DT, TIM, UFWD
COMPLEX A, B, EYE
DIMENSION XPSL(3,4), XPSLR(3,4), PBB(120,120)
COMMON/PTST/ARE4(200,4), X4(200,4), Y4(200,4), Z4(200,4)
* , SEL(200,4)
DO 1500 J=1, NPAN
  AREA4(J,4)=-1.0
  JT=4
  IF(KK(J,4).EQ.0) JT=3
  DO 1500 JJ=1, JT
    J2=1
    IF(JJ.LT.JT) J2=JJ+1
    KF=KK(J, JJ)
    KG=KK(J, J2)
    X4(J, JJ)=(XPT(KF)+XPT(KG)+XPAN(J))/3.0
    Y4(J, JJ)=(YPT(KF)+YPT(KG)+YPAN(J))/3.0
    Z4(J, JJ)=(ZPT(KF)+ZPT(KG)+ZPAN(J))/3.0
    AF=XPT(KF)-XPAN(J)
    BF=YPT(KF)-YPAN(J)
    CF=ZPT(KF)-ZPAN(J)
    AG=XPT(KG)-XPAN(J)
    BG=YPT(KG)-YPAN(J)
    CG=ZPT(KG)-ZPAN(J)
    CALL SELF(AF, BF, CF, AG, BG, CG, FEE)
    SEL(J, JJ)=FEE
    CR=AF*BG-BF*AG
    AR=BF*CG-CF*BG
    BR=CF*AG-AF*CG
    ARE4(J, JJ)=0.5*SQRT(AR*AR+BR*BR+CR*CR)
1500 CONTINUE
DO 127 NJ=1, NPAN
1277 DO 1277 MJ=1, NPAN
  PBB(NJ, MJ)=0.00
  P(NJ, 1)=0.00
  P(NJ, 2)=0.00
  P(NJ, 3)=0.00
  P(NJ, 4)=0.00
  P(NJ, 5)=0.00
  P(NJ, 6)=0.00
  DO 128 NK=1, 4
  ARN=ARE4(NJ, NK)
  IF(ARN.LT.0.0) GO TO 128
  P1=0.0
  P2=0.0
  P3=0.0
  P4=0.0
  P5=0.0
  P6=0.0
  X=X4(NJ, NK)
  Y=Y4(NJ, NK)
  Z=Z4(NJ, NK)
  DO 138 MJ=1, NPAN
  DO 138 MK=1, 4
  XF=X4(MJ, MK)
  YF=Y4(MJ, MK)
  ZF=Z4(MJ, MK)
  ARM=ARE4(MJ, MK)
  IF(ARM.LT.0.00) GO TO 138
  IF(NJ.NE.MJ) GO TO 140
  IF(MK.NE.NK) GO TO 140
  FRA=SEL(MJ, MK)/ARM
  GO TO 1380

```

```
CSE(K)=-1.0
SNE(K)=0.0
YY(K)=0.0
ZZ(K)=Z2(K)-D
ROL(K)=ZZ(K)
NON=K
K=K+1
Y2(K)=0.0
Z2(K)=Z2(K-1)-ZINT
```

```
190 CONTINUE
Z2(K)=0.0
NON=K-1
```

```
END OF FIRST PASS. ADD ADDITIONAL SEGMENTS IF REQUIRED..
```

```
200 IF (NON.GT.LIMIT) GO TO 290
IF (.NOT.ADJUST) GO TO 280
IF (NON.GT.MAXPTS) GO TO 280
IF (MTOT.EQ.0) GO TO 280
MTOT=MTOT+NON
M1=NON-NWL
IF (MTOT.LE.MAXPTS) GO TO 230
```

```
DECREASE MORE UNTIL MTOT IS EQUAL MAXPTS ..
```

```
210 DO 220 K=1,M1
IF (MORE(K).LE.0) GO TO 220
MORE(K)=MORE(K)-1
MTOT=MTOT-1
IF (MTOT.LE.MAXPTS) GO TO 230
```

```
220 CONTINUE
GO TO 210
```

```
INSERT ADDITIONAL SEGMENTS AS INDICATED BY MORE ..
```

```
230 I=1
DO 270 M=1,M1
I1=I+1
NUMBER=MORE(M)
IF (NUMBER.LE.0) GO TO 260
N1=NUMBER+1
```

```
ZO=Z2(I)
YO=Y2(I)
ZINT=Z2(I1)-ZO
YINT=Y2(I1)-YO
ZINT=ZINT/FLOAT(2*N1)
YINT=YINT/FLOAT(2*N1)
D=DEL(I)/FLOAT(N1)
C=CSE(I)
S=SNE(I)
```

```
DO 240 K=1,N1
P=(K-1)*2
Q=P+1.0
ZNEW(K)=ZO+P*ZINT
YNEW(K)=YO+P*YINT
ZZNEW(K)=ZO+Q*ZINT
YYNEW(K)=YO+Q*YINT
RNEW(K)=(CR-YYNEW(K))*S-ZZNEW(K)*C
```

```
240 CONTINUE
DO 250 K=1,NUMBER
DNEW(K)=D
```

```

      CNEW(K)=C
250   SNEW(K)=S
      ZZ(I)=ZZNEW(N1)
      YY(I)=YYNEW(N1)
      ROL(I)=RNEW(N1)
      DEL(I)=D
      NUT=NON+1
      CALL INSERT (Z2, ZNEW(2), I1, NUT, NUMBER)
      CALL INSERT (Y2, YNEW(2), I1, NUT, NUMBER)
      CALL INSERT (ZZ, ZZNEW, I, NON, NUMBER)
      CALL INSERT (YY, YYNEW, I, NON, NUMBER)
      CALL INSERT (DEL, DNEW, I, NON, NUMBER)
      CALL INSERT (ROL, RNEW, I, NON, NUMBER)
      CALL INSERT (CSE, CNEW, I, NON, NUMBER)
      CALL INSERT (SNE, SNEW, I, NON, NUMBER)
260   I=I1+NUMBER
      NON=NON+NUMBER
270  CONTINUE
280  NOE=NON+NON
C
      RETURN
C
      ERROR DIAGNOSTICS..
C
      TOO MANY WET SEGMENTS..
C
290  WRITE (OUTPUT,300) XXSTA
      STOP 7
C
300  FORMAT(' More than 20 wet segments for station at X=',F13.5)
C
      END
      SUBROUTINE FLOAT
C
      THIS SUBROUTINE APPLIES THE GIVEN DRAFT TO THE ORIGINAL TABLE
      OF OFFSETS.
      COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF
      INTEGER OUTPUT, BIF, OFF, COF
C
      COMMON/SHIP/ISTA, LSTA, ISWL, LSWL, TF, TA, XCG, YCG, DISPL
      COMMON/DRFT12/ DRAFT(6,2), IDRAFT
      COMMON/GEOMETRY/MSTA, LPTS(25), YOFF(25,25), NAFT, XAFT(25),
*      YAFT(25), NFWD, XFWD(25), YFWD(25), XOFF(25),
*      ZOFF(25,25), XFPERP, XAPERP, SHIPL, SHIPB, SHIPT,
*      Y1(21,25), ZWL(25), WL(25), INPTS(25), XWLF, XWLA, XXF,
*      XXA, TAN, NON, NOE, NWL, CR, XXFWD, XXSTA, XXAFT, DX, DX1,
*      DX2, Z2(21), Y2(21), ZZ(20), YY(20), SNE(20), CSE(20),
*      DEL(20), ROL(20), ADJUST, WMAX, YMAX, ZMAX, AREA, VERT
      LOGICAL WL, ADJUST
C
      PLACE SHIP AT GIVEN DRAFT..
C
      -----
      TF=DRAFT(IDRAFT,1)
      TA=DRAFT(IDRAFT,2)
      -----
C
      IF (XFPERP.NE.XAPERP) GO TO 130

```

```

XFPERP=XOFF(1)
XAPERP=XOFF(MSTA)
IF (XFPERP.NE.XAPERP) GO TO 130
TAN=0.0
IF (TF.EQ.TA) GO TO 140
WRITE(OUTPUT,570) XFPERP,TF,TA
STOP 5
130 TAN=(TA-TF)/(XFPERP-XAPERP)
140 ISTA=0
ISWL=0
LSWL=0
XX=XOFF(1)
DO 190 J=1,MSTA
  IF (XX.GE.XOFF(J)) GO TO 142
  WRITE (OUTPUT,575) J,XOFF(J),XX
  STOP 6

  TJ IS DRAFT OF STATION J.
142 XX=XOFF(J)
  TJ=(XFPERP-XX)*TAN+TF
  YNEXT=YOFF(1,J)-TJ
  N=0
  IF (YNEXT.GT.0.0) GO TO 152
  LSTA=J
  IF (ISTA.EQ.0) ISTA=J
  Y1(1,J)=YNEXT
  N=LPTS(J)
  DO 150 I=2,N
    YNEXT=YOFF(I,J)-TJ
    IF (YNEXT.GT.0.0) GO TO 160
    Y1(I,J)=YNEXT
150 CONTINUE
  I=N+1
  IF (YNEXT.EQ.0.0) GO TO 170

  SECTION IS NOT SURFACE PIERCING..
152 INPTS(J)=N
  WL(J)=.FALSE.
  GO TO 190

  SECTION IS SURFACE PIERCING.  FIND WATERLINE COORDINATES..
160 YO=YOFF(I-1,J)-TJ
  IF (YO.EQ.0.0) GO TO 170
  INPTS(J)=I-1
  ZO=ZOFF(I-1,J)
  ZWL(J)=ZO-YO*(ZOFF(I,J)-ZO)/(YNEXT-YO)
  GO TO 180
170 INPTS(J)=I-2
  ZWL(J)=ZOFF(I-1,J)
180 WL(J)=.TRUE.
  LSWL=J
  IF (ISWL.EQ.0) ISWL=J
190 CONTINUE
  IF (ISTA.NE.0) GO TO 200
  WRITE (OUTPUT,580)
  STOP 77

```



```

C      INTERPOLATE FOR WATERLINE.
C
      XO=XAFT(J-1)
      XX=(XX-XO)/(YO-YNEXT)*YO+XO
267    IF (XX.GE. XWLA) GO TO 269
      XWLA=XX
C
268    XXA=AMIN1(XX, XXA)
269    YO=YNEXT
270    CONTINUE
290    IF (ISWL.EQ.0) ISWL=26
C
420    FORMAT (32HOS T A T I O N   G E O M E T R Y)
480    FORMAT (22H0      DRAFT FWD (AT X =,F10.3,3H) =,F10.3/5X,17HDRAFT AF
      1T (AT X =,F10.3,3H) =,F10.3)
570    FORMAT (61H0*** TWO DRAFTS SPECIFIED, AND LENGTH BETWEEN PERPS. IS
      1 ZERO./31H *** BOTH PERPENDICULARS AT X =,F12.4/16H *** DRAFT FWD
      2=,F12.4/16H *** DRAFT AFT =,F12.4)
575    FORMAT (12H0*** STATION, I3, 5H (X =,F12.4,18H) IS OUT OF ORDER./29H
      1 *** PREVIOUS STATION HAS X =,F12.4)
580    FORMAT (25H0*** SHIP IS ABOVE WATER.)
C
      END
      SUBROUTINE ERROR(NO, IDUM, RDUM)
      COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF
      INTEGER OUTPUT, BIF, OFF, COF
      WRITE(OUTPUT,10) NO
      WRITE(OUTPUT,11) IDUM, RDUM
10     FORMAT(' STOPPED DUE TO ERROR NO. ', I2, '/')
11     FORMAT(1X, I3, 5X, F10.3)
      STOP
      END
      SUBROUTINE BEER (K)
C
C1     TWO-DIMENSIONAL HYDRODYNAMIC CALCULATION FOR THE SPECIAL CASE
C1     OF ZERO OR INFINITE FREQUENCY.
C1
C5
      COMMON/GEOMETRY/MSTA, LPTS(25), YOFF(25,25), NAFT, XAFT(25),
      *   YAFT(25), NFWD, XFWD(25), YFWD(25), XOFF(25),
      *   ZOFF(25,25), XFPERP, XAPERP, SHIPL, SHIPB, SHIPT,
      *   Y1(21,25), ZWL(25), WL(25), INPTS(25), XWLF, XWLA, XXF,
      *   XXA, TAN, NON, NOE, NWL, CR, XXFWD, XXSTA, XXAFT, DX, DX1,
      *   DX2, Z2(21), Y2(21), ZZ(20), YY(20), SNE(20), CSE(20),
      *   DEL(20), ROL(20), ADJUST, WMAX, YMAX, ZMAX, AREA, VERT
      COMMON // HAH, SAS, RAR, CCA, HVH, SVS, RVR, CCV, RHO2, RSIG, WN, W1, W2, ERR, Z
      1RI, YRI, EJT
      COMMON / / HAO(24,8), BLOGP(20,20), YLOGP(20,20), BLOGM(20
      1,20), YLOGM(20,20), CONH(40), CONR(40,2), CROLL(40,40), CHEAV(40,40), HE
      2AVI(20,20), HEAVT(20,20), ROLLI(20,20), ROLLT(20,20), EJI, CZRI, CZLI, SZ
      3RI, SZLI, RARI, RALI, RBRI, RBLI, CLI, CRI, SLI, SRI, I, IPESQ, J, NJ
C5
C5     INPUT AND OUTPUT LOGICAL UNITS.
      COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF
      INTEGER OUTPUT, BIF, OFF, COF
C5
C5     OUTPUT LISTING PAGE HEADING DATA.
C5
      HAH=0.0
      SAS=0.0
      RAR=0.0

```



CCA=0.0  
HVH=0.0  
SVS=0.0  
RVR=0.0  
CCV=0.0  
NN=NON-NWL

C3  
C3  
C3

IF (WN.NE.0.) GO TO 130  
ZERO FREQUENCY CASE.

DO 120 I=1, NN

XM1=ZZ(I)-Z2(1)  
XP1=ZZ(I)+Z2(1)  
YP1=YY(I)+Y2(1)  
FCR1=.5\*ALOG(XM1\*\*2+YP1\*\*2)  
FCL1=.5\*ALOG(XP1\*\*2+YP1\*\*2)  
ACR1=ATAN2(YP1, XM1)  
ACL1=ATAN2(YP1, XP1)

DO 110 J=1, NN

XM2=ZZ(I)-Z2(J+1)  
XP2=ZZ(I)+Z2(J+1)  
YP2=YY(I)+Y2(J+1)  
FCR2=.5\*ALOG(XM2\*\*2+YP2\*\*2)  
FCL2=.5\*ALOG(XP2\*\*2+YP2\*\*2)  
ACR2=ATAN2(YP2, XM2)  
ACL2=ATAN2(YP2, XP2)

SIMJ=SNE(I)\*CSE(J)-SNE(J)\*CSE(I)  
CIMJ=CSE(I)\*CSE(J)+SNE(I)\*SNE(J)  
SIPJ=SNE(I)\*CSE(J)+SNE(J)\*CSE(I)  
CIPJ=CSE(I)\*CSE(J)-SNE(I)\*SNE(J)

DCNR=SIPJ\*(FCR1-FCR2)+CIPJ\*(ACR1-ACR2)

1 PCR=CSE(J)\*(XM1\*FCR1-YP1\*ACR1-XM1-XM2\*FCR2+YP2\*ACR2+XM2)+SNE  
(J)\*(YP2\*FCR2+XM2\*ACR2+YP1-YP1\*FCR1-XM1\*ACR1-YP2)

DCNL=SIMJ\*(FCL2-FCL1)+CIMJ\*(ACL2-ACL1)

1 PCL=CSE(J)\*(XP2\*FCL2-YP2\*ACL2-XP2-XP1\*FCL1+YP1\*ACL1+XP1)+SNE  
(J)\*(YP2\*FCL2+XP2\*ACL2-YP2-YP1\*FCL1-XP1\*ACL1+YP1)

CROLL(I, J)=BLOGM(J, I)+2.0\*(DCNR-DCNL)

CHEAV(I, J)=BLOGP(J, I)+2.0\*(DCNR+DCNL)

ROLLT(I, J)=-YLOGM(J, I)-2.0\*(PCR-PCL)

HEAVT(I, J)=-YLOGP(J, I)-2.0\*(PCR+PCL)

IF (J.EQ.NN) GO TO 110

XM1=XM2

XP1=XP2

YP1=YP2

FCR1=FCR2

FCL1=FCL2

ACR1=ACR2

ACL1=ACL2

110 CONTINUE

120 CONTINUE

GO TO 160

C3  
C3  
C3

INFINITE FREQUENCY CASE.

130 CONTINUE

DO 150 I=1, NN

DO 140 J=1, NN

CROLL(I, J)=BLOGM(J, I)

CHEAV(I, J)=BLOGP(J, I)

```

        ROLLT(I, J)=-YLOGM(J, I)
        HEAVT(I, J)=-YLOGP(J, I)
140   CONTINUE
150   CONTINUE
C3
C3   SOLUTION FOR EITHER THE ZERO OR INFINITE FREQUENCY CASE..
C3
160   CONTINUE
      DO 170 I=1, NN
        CONH(I)=CSE(I)
        CONR(I, 1)=-SNE(I)
        CONR(I, 2)=ROL(I)
170   CONTINUE
      IT=LNEGT(40, NN, 1, CHEAV, CONH, ERR, HEAVI)
      IF (IT.EQ.0) GO TO 180
      IF (WN.EQ.0.0) WRITE (OUTPUT, 230)
      IF (WN.NE.0.0) WRITE (OUTPUT, 240)
      WRITE (OUTPUT, 270) XXSTA
      IF (IT.NE.0) GO TO 190
180   CONTINUE
      IT=LNEGT(40, NN, 2, CROLL, CONR, ERR, ROLLI)
      IF (IT.EQ.0) GO TO 200
      IF (WN.EQ.0.0) WRITE (OUTPUT, 250)
      IF (WN.NE.0.0) WRITE (OUTPUT, 260)
      WRITE (OUTPUT, 270) XXSTA
      IF (IT.EQ.1) GO TO 200
190   WRITE (OUTPUT, 280)
      STOP 10
C3
C3   EVALUATE VELOCITY POTENTIALS AND FORCE COEFFICIENTS..
C3
200   DO 220 I=1, NN
        PAH=0.0
        PAS=0.0
        PAR=0.0
        DO 210 J=1, NN
          PAH=PAH+CONH(J)*HEAVT(J, I)
          PAS=PAS+CONR(J, 1)*ROLLT(J, I)
          PAR=PAR+CONR(J, 2)*ROLLT(J, I)
210   CONTINUE
C6
C6   THE PRESSURES IN PHASE WITH THE SINUSOIDAL DISPLACEMENT ARE..
C6
C6   HEAVE  -- PAH = PAH*RHO*ESIG*ESIG
C6   SWAY   -- PAS = PAS*RHO*ESIG*ESIG
C6   ROLL   -- PAR = PAR*RHO*ESIG*ESIG
C6
C6   THE ACCELERATION COMPONENTS OF THE FORCE ARE EQUAL
C6   IN MAGNITUDE TO THE ABOVE, BUT HAVE THE OPPOSITE SIGN.
C6
      DDD=DEL(I)
      DCI=CSE(I)*DDD
      DSI=-SNE(I)*DDD
      DFR=ROL(I)*DDD
C6
C6   INTEGRATION TO OBTAIN FORCE ACCELERATION COEFFICIENTS..
C6
      HAH=HAH+PAH*DCI
      SAS=SAS+PAS*DSI
      RAR=RAR+PAR*DFR
      CCA=PAR*DSI+PAS*DFR+CCA

```

C6  
220 CONTINUE  
HAH=HAH\*RHO2  
SAS=SAS\*RHO2  
RAR=RAR\*RHO2  
CCA=CCA\*RHO2/2.0  
IF(WN.EQ.0.) HAH=99. ! INFINITE AT ZERO FREQ  
RETURN

C  
230 FORMAT (43H \*\*\* HEAVE MATRIX, ZERO ENCOUNTER FREQUENCY)  
240 FORMAT (37H \*\*\* HEAVE MATRIX, INFINITE FREQUENCY)  
250 FORMAT (47H \*\*\* SWAY-ROLL MATRIX, ZERO ENCOUNTER FREQUENCY)  
260 FORMAT (41H \*\*\* SWAY-ROLL MATRIX, INFINITE FREQUENCY)  
270 FORMAT (31H \*\*\* COEFFS. FOR STATION AT X =, F13.5)  
280 FORMAT (26H \*\*\* EXECUTION TERMINATED.)

C  
END  
SUBROUTINE GIRL

C  
C1  
C1  
C1  
C6  
C6  
C6  
C5  
CALCULATION OF FREQUENCY INDEPENDENT TERMS TO BE USED IN THE  
TWO-DIMENSIONAL HYDRODYNAMIC CALCULATIONS.

THIS SUBROUTINE IS CALLED ONCE FOR EACH STATION OF THE SHIP  
WHEN THE HYDRODYNAMIC COEFFICIENTS ARE BEING GENERATED.

COMMON/GEOMETRY/MSTA, LPTS(25), YOFF(25,25), NAFT, XAFT(25),  
\* YAFT(25), NFWD, XFWD(25), YFWD(25), XOFF(25),  
\* ZOFF(25,25), XFPERP, XAPERP, SHIPL, SHIPB, SHIPT,  
\* Y1(21,25), ZWL(25), WL(25), INPTS(25), XWLF, XWLA, XXF,  
\* XXA, TAN, NON, NOE, NWL, CR, XXFWD, XXSTA, XXAFT, DX, DX1,  
\* DX2, Z2(21), Y2(21), ZZ(20), YY(20), SNE(20), CSE(20),  
\* DEL(20), RDL(20), ADJUST, WMAX, YMAX, ZMAX, AREA, VERT  
COMMON // HA1, SA1, RA1, CA1, HV1, SV1, RV1, CV1, RHO2, RSIG, WN, W1, W2, ERR, Z  
1RI, YRI, EJT  
COMMON / / HAO(24,8), BLOGP(20,20), YLOGP(20,20), BLOGM(20,  
120), YLOGM(20,20), I, J, ACLI, ACLT, ACRI, ACRT, APLI, APLT, APRI, APRT, CIMJ,  
2CIPJ, DCNL, DCNR, DPNL, DPNR, FCLI, FCLT, FCRI, FCRT, FPLI, FPLT, FPRI, FPRT, P  
3CL, PCR, PPL, PPR, SIMJ, SIPJ, ZMI, ZMT, ZPI, ZPT, YMI, YMT, YPI, YPT  
DATA PIN/-3.14159265358979/  
DATA TPI/6.28318530717958/  
DO 160 I=1, NON  
ZMI=ZZ(I)  
ZPI=ZMI  
YMI=YY(I)-Y2(1)  
YPI=YY(I)+Y2(1)  
FPRI=ALOG(ZMI\*ZMI+YMI\*YMI)/2.0  
FPLI=FPRI  
FCRI=ALOG(ZMI\*ZMI+YPI\*YPI)/2.0  
FCLI=FCRI  
APRI=ATAN2(YMI, ZMI)  
APLI=APRI  
ACRI=ATAN2(YPI, ZMI)  
ACLI=ACRI  
DO 150 J=1, NON  
J1=J+1  
YMT=YY(I)-Y2(J1)  
YPT=YY(I)+Y2(J1)  
ZMT=ZZ(I)-Z2(J1)  
ZPT=ZZ(I)+Z2(J1)  
C  
CALCULATE ANGLES (MEASURED OUTSIDE SECTION)..

```

APRT=ATAN2(YMT, ZMT)
IF (ZMT.GE.0.0) GO TO 130
IF (J1.GT.1) GO TO 110
IF (YMT.LT.0.0) APRT=APRT+TPI
GO TO 120
110 IF (YMT.GE.0.0) APRT=APRT-TPI
120 IF (YPT.LT.0.0) GO TO 130
ACRT=PIN
GO TO 140
130 ACRT=ATAN2(YPT, ZMT)
140 ACLT=ATAN2(YPT, ZPT)
APLT=ATAN2(YMT, ZPT)
FPRT=ALOG(ZMT*ZMT+YMT*YMT)/2.0
FPLT=ALOG(ZPT*ZPT+YMT*YMT)/2.0
FCRT=ALOG(ZMT*ZMT+YPT*YPT)/2.0
FCLT=ALOG(ZPT*ZPT+YPT*YPT)/2.0
SIMJ=SNE(I)*CSE(J)-SNE(J)*CSE(I)
CIMJ=CSE(I)*CSE(J)+SNE(I)*SNE(J)
SIPJ=SNE(I)*CSE(J)+SNE(J)*CSE(I)
CIPJ=CSE(I)*CSE(J)-SNE(I)*SNE(J)
DPNR=SIMJ*(FPRI-FPRT)+CIMJ*(APRI-APRT)
1 PPR=CSE(J)*(ZMI*FPRI-YMI*APRI-ZMI-ZMT*FPRT+YMT*APRT+ZMT)+SNE
(J)*(YMI*FPRI+ZMI*APRI-YMI-YMT*FPRT-ZMT*APRT+YMT)
1 DPNL=SIPJ*(FPLT-FPLI)+CIPJ*(APLT-APLI)
1 PPL=CSE(J)*(ZPT*FPLT-YMT*APLT-ZPT-ZPI*FPLI+YMI*APLI+ZPI)+SNE
(J)*(YMI*FPLI+ZPI*APLI+YMT-YMT*FPLT-ZPT*APLT-YMI)
1 DCNR=SIPJ*(FCRI-FCRT)+CIPJ*(ACRI-ACRT)
1 PCR=CSE(J)*(ZMI*FCRI-YPI*ACRI-ZMI-ZMT*FCRT+YPT*ACRT+ZMT)+SNE
(J)*(YPT*FCRT+ZMT*ACRT+YPI-YPI*FCRI-ZMI*ACRI-YPT)
1 DCNL=SIMJ*(FCLT-FCLI)+CIMJ*(ACLT-ACLI)
1 PCL=CSE(J)*(ZPT*FCLT-YPT*ACLT-ZPT-ZPI*FCLI+YPI*ACLI+ZPI)+SNE
(J)*(YPT*FCLT+ZPT*ACLT-YPT-YPI*FCLI-ZPI*ACLI+YPI)
BLOGP(J, I)=DPNR+DPNL-DCNR-DCNL
YLOGP(J, I)=PPR+PPL-PCR-PCL
BLOGM(J, I)=DPNR-PPR-DCNR+DCNL
YLOGM(J, I)=PPR-PPL-PCR+PCL
IF (J.EQ.NON) GO TO 150
FPRI=FPRT
FPLI=FPLT
FCRI=FCRT
FCLI=FCLT
APRI=APRT
APLI=APLT
ACRI=ACRT
ACLI=ACLT
ZMI=ZMT
YMI=YMT
ZPI=ZPT
YPI=YPT
150 CONTINUE
160 CONTINUE
RETURN

```

```

END
SUBROUTINE WINE (K)

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```

C
C1 TWO-DIMENSIONAL HYDRODYNAMIC CALCULATION FOR NON-ZERO FREQUENCIES.
C1
C6 THIS SUBROUTINE IS CALLED FOR EACH STATION AND ALL NON-ZERO
C6 FREQUENCIES WHEN THE HYDRODYNAMIC COEFFICIENTS ARE BEING
C6 GENERATED.

```

C6  
C5  
C5

INPUT AND OUTPUT LOGICAL UNITS.  
COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF  
INTEGER OUTPUT, BIF, OFF, COF

C5  
C5

```
COMMON/GEOMETRY/MSTA, LPTS(25), YOFF(25, 25), NAFT, XAFT(25),
* YAFT(25), NFWD, XFWD(25), YFWD(25), XOFF(25),
* ZOFF(25, 25), XFPERP, XAPERP, SHIPL, SHIPB, SHIPT,
* Y1(21, 25), ZWL(25), WL(25), INPTS(25), XWLF, XWLA, XXF,
* XXA, TAN, NON, NOE, NWL, CR, XXFWD, XXSTA, XAFT, DX, DX1,
* DX2, Z2(21), Y2(21), ZZ(20), YY(20), SNE(20), CSE(20),
* DEL(20), ROL(20), ADJUST, WMAX, YMAX, ZMAX, AREA, VERT
COMMON // HAH, SAS, RAR, CCA, HVH, SVS, RVR, CCV, RH02, RSIG, WN, W1, W2, ERR, Z
1RI, YRI, EJT
COMMON / / HAO(24, 8), BLOGP(20, 20), YLOGP(20, 20), BLOGM(20
1, 20), YLOGM(20, 20), CONH(40), CONR(40, 2), CROLL(40, 40), CHEAV(40, 40), HE
2AVI(20, 20), HEAVT(20, 20), ROLLI(20, 20), ROLLT(20, 20), EJI, CZRI, CZLI, SZ
SRI, SZLI, RARI, RALI, RBRI, RBLI, CLI, CRI, SLI, SRI, I, IPESO, NI
DO 110 I=1, NON
NI=NON+1
CONH(I)=0.0
CONR(I, 1)=0.0
CONR(I, 2)=0.0
CONH(NI)=CSE(I)
CONR(NI, 1)=-SNE(I)
CONR(NI, 2)=ROL(I)
ZRI=WN*ZZ(I)
YRI=-WN*(YY(I)+Y2(1))
EJT=EXP(-YRI)
EJI=EJT
CALL ROMED (CZRI, SZRI, RARI, RBRI, CRI, SRI)
CZLI=CZRI
SZLI=SZRI
RALI=RARI
RBLI=RBRI
CLI=CRI
SLI=SRI
CALL WOMEN (I, BLOGP(1, I), YLOGP(1, I), BLOGM(1, I), YLOGM(1, I), CHEAV
1 (1, I), CROLL(1, I), HEAVI(1, I), HEAVT(1, I), ROLLI(1, I), ROLLT(1, I))
110 CONTINUE
IF (NWL.EQ.0) GO TO 130
I=NOE-NWL+1
DO 120 I=I, NOE
CONH(I)=0.0
CONR(I, 1)=0.0
CONR(I, 2)=0.0
120 CONTINUE
130 IT=JULIET(1, CHEAV, CONH)
IF (IT.EQ.0) GO TO 140
WRITE (OUTPUT, 180) K, WN, XXSTA
IF (IT.NE.1) GO TO 150
140 IT=JULIET(2, CROLL, CONR)
IF (IT.EQ.0) GO TO 160
WRITE (OUTPUT, 190) K, WN, XXSTA
IF (IT.EQ.1) GO TO 160
150 WRITE (OUTPUT, 200)
STOP 11
C
160 HAH=0.0
```

SAS=0.0  
RAR=0.0  
CCA=0.0  
HVH=0.0  
SVS=0.0  
RVR=0.0  
CCV=0.0

C9  
C9  
C9  
C9  
C9  
C9

SLIGHT INCREASE IN SPEED IF THE FINAL INTEGRATION AVOIDS THE  
INTERIOR SURFACE SEGMENTS.  
NI = NON - NWL  
DO \*\*\* I=1,NI

DO 170 I=1,NON  
CALL SONG (HEAVI(1, I), HEAVT(1, I), ROLLI(1, I), ROLLT(1, I), I)

170 CONTINUE

C6  
C6  
C6  
C6  
C6  
C6  
C6  
C6  
C6  
C6

FORCE COEFFICIENTS.  
FORCE IS THAT WHICH MUST BE APPLIED TO THE CYLINDER (PER UNIT  
LENGTH) TO CAUSE SINUSOIDAL OSCILLATIONS AT THE GIVEN FREQUENCY  
AND UNIT AMPLITUDE.  
COEFFICIENTS ARE THE PARTIAL DERIVATIVES OF THE FORCE BY THE  
ACCELERATION OR VELOCITY COMPONENT OF THE GIVEN MOTION.

ACCELERATION TERMS.

HAH=HAH\*RHO2  
SAS=SAS\*RHO2  
RAR=RAR\*RHO2  
CCA=CCA\*RHO2/2.0

C6  
C6  
C6

VELOCITY TERMS.

HVH=HVH\*RSIG  
SVS=SVS\*RSIG  
RVR=RVR\*RSIG  
CCV=CCV\*RSIG/2.0

C6

RETURN

C3

180 FORMAT (36H \*\*\* HEAVE MATRIX, FREQUENCY INDEX =, I3, 15H, WAVE NUMBE  
1R =, 1PE13.5/31H \*\*\* COEFFS. FOR STATION AT X =, OPF13.5)  
190 FORMAT (40H \*\*\* SWAY-ROLL MATRIX, FREQUENCY INDEX =, I3, 15H, WAVE N  
1UMBER =, 1PE13.5/31H \*\*\* COEFFS. FOR STATION AT X =, OPF13.5)  
200 FORMAT (26H \*\*\* EXECUTION TERMINATED.)

C

END

SUBROUTINE WOMEN

1 (I, BLOGP, YLOGP, BLOGM, YLOGM, CHI, CRI, HII, HTI, RII, RTI)

C

REAL BLOGP(1), BLOGM(1), YLOGP(1), YLOGM(1), CHI(1), HII(1), RII(1), CRI(1), HTI(1), RTI(1)

C5

COMMON/GEOMETRY/MSTA, LPTS(25), YOFF(25, 25), NAFT, XAFT(25),  
\* YAFT(25), NFWD, XFWD(25), YFWD(25), XOFF(25),  
\* ZOFF(25, 25), XPERP, XAPERP, SHIPL, SHIPB, SHIPT,  
\* Y1(21, 25), ZWL(25), WL(25), INPTS(25), XWLF, XWLA, XXF,  
\* XXA, TAN, NON, NOE, NWL, CR, XXFWD, XXSTA, XAFT, DX, DX1,  
\* DX2, Z2(21), Y2(21), ZZ(20), YY(20), SNE(20), CSE(20),  
\* DEL(20), ROL(20), ADJUST, WMAX, YMAX, ZMAX, AREA, VERT  
COMMON // HA1, SA1, RA1, CA1, HV1, SV1, RV1, CV1, RHO2, RSIG, WN, W1, UU, ERR, X

```

1RT, YRT, EJT
COMMON / HAO(24, 8), BP(20, 20), YP(20, 20), BM(20, 20), YM(2
10, 20), CONH(40), CONR(40, 2), CROLL(40, 40), CHEAV(40, 40), HEAVI(20, 20), H
2EAVT(20, 20), ROLLI(20, 20), ROLLT(20, 20), EJI, CZRI, CZLI, SZRI, SZLI, RARI
3, RALI, RBRI, RBLI, CLI, CRG, SLI, SRI
DATA TPI/6.28318530717958/
YYI=YY(I)
ZZI=ZZ(I)
SI=SNE(I)
CI=CSE(I)
DO 110 J=1, NON
  XRT=WN*(ZZI-Z2(J+1))
  YRT=-WN*(YYI+Y2(J+1))
  EJT=EXP(-YRT)
  CALL ROMED (CZRT, SZRT, RART, RBRT, CRT, SRT)
  XRT=WN*(ZZI+Z2(J+1))
  CALL ROMED (CZLT, SZLT, RALT, RBLT, CLT, SLT)
  CJ=CSE(J)
  SJ=SNE(J)
  SSS=SI*CJ
  TTT=SJ*CI
  UUU=CI*CJ
  VVV=SI*SJ
  CIPJ=UUU-VVV
  SIPJ=SSS+TTT
  SIMJ=SSS-TTT
  CIMJ=UUU+VVV
  SSS=SIMJ*(CLI-CLT)-CIMJ*(SLI-SLT)
  TTT=SIPJ*(CRG-CRT)-CIPJ*(SRI-SRT)
  UUU=SJ*(RALI-RALT)+CJ*(RBLT-RBLI)
  VVV=SJ*(RARI-RART)+CJ*(RBRI-RBRT)
  WWW=EJT*(SZRT*CIPJ-CZRT*SIPJ)-EJI*(SZRI*CIPJ-CZRI*SIPJ)
  RRR=EJT*(SZLT*CIMJ-CZLT*SIMJ)-EJI*(SZLI*CIMJ-CZLI*SIMJ)
  QQQ=EJI*(SZRI*CJ-CZRI*SJ)-EJT*(SZRT*CJ-CZRT*SJ)
  PPP=EJI*(SZLI*CJ+CZLI*SJ)-EJT*(SZLT*CJ+CZLT*SJ)
  CHI(J)=BLOGP(J)+2.0*(TTT-SSS)
  CRI(J)=BLOGM(J)+2.0*(TTT+SSS)
  HII(J)=YLOGP(J)+UU*(VVV+UUU)
  RII(J)=YLOGM(J)+UU*(VVV-UUU)
  CHN(NJ) = CHI(J)
  CRN(NJ) = CRI(J)
  NJ=NON+J
  CHI(NJ)=TPI*(WWW-RRR)
  CRI(NJ)=TPI*(WWW+RRR)
  HTI(J)=W1*(QQQ-PPP)
  RTI(J)=W1*(QQQ+PPP)
  CHN(J) = -CHI(NJ)
  CRN(J) = -CRI(NJ)
  IF (J.EQ.NON) GO TO 110
  EJI=EJT
  CRG=CRT
  SRI=SRT
  CLI=CLT
  SLI=SLT
  RARI=RART
  RBRI=RBRT
  RALI=RALT
  RBLI=RBLT
  CZRI=CZRT
  SZRI=SZRT
  CZLI=CZLT

```

SZLI=SZLT  
110 CONTINUE  
RETURN

END  
SUBROUTINE SONG (HAI, HOT, RAI, ROT, I)

DIMENSION HAI(1), HOT(1), RAI(1), ROT(1), PP(6)

COMMON/GEOMETRY/MSTA, LPTS(25), YOFF(25, 25), NAFT, XAFT(25),  
\* YAFT(25), NFWD, XFWD(25), YFWD(25), XOFF(25),  
\* ZOFF(25, 25), XPERP, XAPERP, SHIPL, SHIPB, SHIPT,  
\* Y1(21, 25), ZWL(25), WL(25), INPTS(25), XWLF, XWLA, XXF,  
\* XXA, TAN, NON, NOE, NWL, CR, XXFWD, XXSTA, XXAFT, DX, DX1,  
\* DX2, Z2(21), Y2(21), ZZ(20), YY(20), SNE(20), CSE(20),  
\* DEL(20), ROL(20), ADJUST, WMAX, YMAX, ZMAX, AREA, VERT  
COMMON // HAH, SAS, RAR, CCA, HVH, SVS, RVR, CCV, RHO2, RSIG, WN, W1, W2, ERR, Z  
1RI, YRI, EJT  
COMMON / / HAO(24, 8), BLOGP(20, 20), YLOGP(20, 20), BLOGM(20  
1, 20), YLOGM(20, 20), CONH(40), CONR(40, 2)  
PAH=0.0  
PAS=0.0  
PAR=0.0  
PVH=0.0  
PVS=0.0  
PVR=0.0  
DO 110 J=1, NON  
NJ=NON+J  
PAH=PAH+CONH(J)\*HOT(J)-CONH(NJ)\*HAI(J)  
PAS=PAS+CONR(J, 1)\*ROT(J)-CONR(NJ, 1)\*RAI(J)  
PAR=PAR+CONR(J, 2)\*ROT(J)-CONR(NJ, 2)\*RAI(J)  
PVH=PVH+CONH(J)\*HAI(J)+CONH(NJ)\*HOT(J)  
PVS=PVS+CONR(J, 1)\*RAI(J)+CONR(NJ, 1)\*ROT(J)  
PVR=PVR+CONR(J, 2)\*RAI(J)+CONR(NJ, 2)\*ROT(J)

110 CONTINUE  
DDD=DEL(I)  
DCI=CSE(I)\*DDD  
DSI=-SNE(I)\*DDD  
DFR=ROL(I)\*DDD

THE PRESSURES ON THIS SEGMENT OF THE CYLINDER MAY BE CALCULATED.  
THE PRESSURES IN PHASE WITH THE SINUSOIDAL DISPLACEMENT ARE..

HEAVE -- PAH = PAH\*RHO\*ESIG\*ESIG  
SWAY -- PAS = PAS\*RHO\*ESIG\*ESIG  
ROLL -- PAR = PAR\*RHO\*ESIG\*ESIG

OF COURSE THE ACCELERATION COMPONENTS OF THE FORCE ARE EQUAL  
IN MAGNITUDE TO THE ABOVE, BUT HAVE THE OPPOSITE SIGN.

THE PRESSURES IN PHASE WITH THE SINUSOIDAL VELOCITY ARE..

HEAVE -- PVH = PVH\*RHO\*ESIG\*ESIG  
SWAY -- PVS = PVS\*RHO\*ESIG\*ESIG  
ROLL -- PVR = PVR\*RHO\*ESIG\*ESIG

INTEGRATION TO OBTAIN FORCE ACCELERATION COEFFICIENTS..

HAH=HAH+PAH\*DCI



SAS=SAS+PAS\*DSI  
RAR=RAR+PAR\*DFR

C6

CCA=PAR\*DSI+PAS\*DFR+CCA  
INTEGRATION TO OBTAIN FORCE VELOCITY COEFFICIENTS..

C6  
C6

HVH=HVH+PVH\*DCI  
SVS=SVS+PVS\*DSI  
RVR=RVR+PVR\*DFR  
CCV=PVR\*DSI+PVS\*DFR+CCV

C6  
C6  
C6  
C6  
C6

AT THIS POINT THE PRESSURES REQUIRE THE MODIFICATIONS NOTED  
ABOVE TO GIVE THE DIMENSIONAL VALUES.  
THE INTEGRATIONS OF THE PRESSURES ARE COMPLETED ELSEWHERE  
IN THE PROGRAM TO GIVE DIMENSIONAL FORCE COEFFICIENTS.

RETURN

C

END  
SUBROUTINE ROMEO (C, S, RA, RB, CIN, SON)

C

EXPONENTIAL INTEGRAL WITH COMPLEX ARGUMENT.

C1

C1

C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

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C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

C2

THE ARGUMENT IS SUPPLIED THROUGH BLANK COMMON AS THE VARIABLES  
X AND Y.

PARAMETERS AND VARIABLES..

X --- REAL PART OF ARGUMENT  
Y --- IMAGINARY PART OF ARGUMENT  
E --- EXP( -Y )  
C --- COS( X )  
S --- SIN( X )  
CIN --- REAL RESULT  
SON --- IMAGINARY RESULT  
RA --- ALOG( X\*\*2 + Y\*\*2 )/2.0 - CIN  
RB --- ATAN( X/Y ) - PI/2.0 + SON

COMMON/GEOMETRY/MSTA, LPTS(25), YOFF(25, 25), NAFT, XAFT(25),  
\* YAFT(25), NFWD, XFWD(25), YFWD(25), XOFF(25),  
\* ZOFF(25, 25), XFPERP, XAPERP, SHIPL, SHIPB, SHIPT,  
\* Y1(21, 25), ZWL(25), WL(25), INPTS(25), XWLF, XWLA, XXF,  
\* XXA, TAN, NON, NOE, NWL, CR, XXFWD, XXSTA, XXAFT, DX, DX1,  
\* DX2, Z2(21), Y2(21), ZZ(20), YY(20), SNE(20), CSE(20),  
\* DEL(20), ROL(20), ADJUST, WMAX, YMAX, ZMAX, AREA, VERT  
COMMON / / HA1, SA1, RA1, CA1, HV1, SV1, RV1, CV1,  
1 RH02, RSIG, WN, W1, W2, ERR, X, Y, E

C5

C5

INPUT AND OUTPUT LOGICAL UNITS.  
COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF  
INTEGER OUTPUT, BIF, OFF, COF  
DATA GAMMA /0.5772 15664 90153 28606 06512/  
DATA HALFPI/1.5707 96326 79489 66192 31322/

C

AT=ATAN2(X, Y)  
ARG=AT-HALFPI  
C=COS(X)  
S=SIN(X)  
R=X\*X+Y\*Y  
AL=0.5\*ALOG(R)  
TEST=0.00001

```

IF (R.LT.1.) GO TO 130
TEST=.1*TEST
IF (R.LT.2.) GO TO 130
TEST=.1*TEST
IF (R.LT.4.) GO TO 130
TEST=.1*TEST
130 SUMC=GAMMA+AL+Y
SUMS=AT+X
TC=Y
TS=X
COX=1.
DO 140 K=2,501
TO=TC
FACT=COX/FLOAT(K)**2
COX=K
TC=FACT*(Y*TC-X*TS)
TS=FACT*(Y*TS+X*TO)
SUMC=SUMC+TC
SUMS=SUMS+TS
IF ((ABS(TC)+ABS(TS)).LE.TEST) GO TO 150
140 CONTINUE
WRITE (OUTPUT,190) XXSTA,X,Y,WN
STOP 12
150 CIN=E*(C*SUMC+S*SUMS)
SON=E*(S*SUMC-C*SUMS)
RA=AL-CIN
RB=ARG+SON
RETURN
C
190 FORMAT (59H0*** NON-CONVERGENT EXPONENTIAL INTEGRAL FOR STATION AT
1 X =,F13.5/22H *** PARAMETERS -- X =,1PE13.5,5H, Y =,E13.3,15H, WA
2VE NUMBER =,E13.5)
C
END
FUNCTION JULIET (N1,A,B)
C
C.. SOLVES SIMULTANEOUS LINEAR EQUATIONS BY GAUSSIAN REDUCTION.
C.. FOR THE SPECIALIZED MATRICES IN THE SUBROUTINE WINE.
C
REAL A(40,1),B(40,1)
C2
C2 THE A MATRIX MUST BE DIMENSIONED WITH EXACTLY 40 ROWS AND
C2 AT LEAST 40 COLUMNS. THE B MATRIX MUST ALSO BE DIMENSIONED
C2 WITH EXACTLY 40 ROWS AND AT LEAST N1 COLUMNS.
C5
C5 INPUT AND OUTPUT LOGICAL UNITS..
COMMON/IO/INPUT,OUTPUT,BIF,OFF,COF
INTEGER OUTPUT,BIF,OFF,COF
C5
C5 COMMON/GEOMETRY/MSTA,LPTS(25),YOFF(25,25),NAFT,XAFT(25),
* YAFT(25),NFWD,XFWD(25),YFWD(25),XOFF(25),
* ZOFF(25,25),XPERP,XAPERP,SHIPL,SHIPB,SHIPT,
* Y1(21,25),ZWL(25),WL(25),INPTS(25),XWLF,XWLA,XXF,
* XXA,TAN,NON,NOE,NWL,CR,XXFWD,XXSTA,XXAFT,DX,DX1,
* DX2,Z2(21),Y2(21),ZZ(20),YY(20),SNE(20),CSE(20),
* DEL(20),ROL(20),ADJUST,WMAX,YMAX,ZMAX,AREA,VERT
COMMON // HA1,SA1,RA1,CA1,HV1,SV1,RV1,CV1,RHO2,RSIG,WN,W1,W2,ERR,Z
1RI,YRI,EJT
C

```

```
C      COMMON / / HAO(24, 8), BLDGP(20, 20, 4), CONH(40, 3), CROLL(40  
1, 40, 2), HEAVI(20, 20, 4), EJI, CZRI, CZLI, SZRI, SZLI, RARI, RALI, RBRI, RBLI,  
2CLI, CRG, SLI, SRI, Z(40)
```

```
C      N2=N0E/2  
D=1.0
```

```
C      C. . COMPLETE THE MATRIX A.
```

```
C      DO 120 J=1, N2  
        L=N2+J  
        DO 110 I=1, N2  
          K=N2+I  
          A(I, L)=-A(K, J)  
          A(K, L)=A(I, J)  
110     CONTINUE  
120     CONTINUE
```

```
C      NM1=N0E-1
```

```
C      C. . SOLVE  $AT \cdot X = B$  FOR  $X$ , WHERE  $AT$  IS THE TRANSPOSE OF THE  
C      MATRIX  $A$ . STORE THE  $X$  VECTOR(S) IN  $B$ .
```

```
C      DO 210 J=1, NM1  
        J1=J+1
```

```
C      C. . FIND ELEMENT OF ROW J, COLS J--N, WHICH HAS MAX ABSOLUTE VALUE.
```

```
C      LMAX=J  
      RMAX=ABS(A(J, J))  
      DO 130 K=J1, N0E  
        RNEXT=ABS(A(J, K))  
        IF (RMAX. GE. RNEXT) GO TO 130  
        RMAX=RNEXT  
        LMAX=K  
130     CONTINUE  
      IF (LMAX. NE. J) GO TO 140
```

```
C      C. . MAX ELEMENT IS ON DIAGONAL.
```

```
C      IF (A(J, J)) 170, 260, 170
```

```
C      C. . MAX ELEMENT IS NOT ON DIAGONAL.
```

```
C      C. . EXCHANGE COLUMNS J AND LMAX.
```

```
C      140     DO 150 L=J, N0E  
              W=A(L, J)  
              A(L, J)=A(L, LMAX)  
              A(L, LMAX)=W  
150     CONTINUE
```

```
C      C. . EXCHANGE ROWS J AND LMAX.
```

```
C      DO 160 L=1, N1  
        W=B(J, L)  
        B(J, L)=B(LMAX, L)  
        B(LMAX, L)=W  
160     CONTINUE  
D=-D
```

C. . ZERO ROW J TO RIGHT OF DIAGONAL.

```
C
170 D=A(J,J)*D
    V=1.0/A(J,J)
    Z(J)=V
    DO 200 K=J1,NOE
      IF (A(J,K).EQ.0.0) GO TO 200
      W=-V*A(J,K)
      DO 180 L=J1,NOE
        A(L,K)=W*A(L,J)+A(L,K)
180 CONTINUE
    DO 190 L=1,N1
      B(K,L)=W*B(J,L)+B(K,L)
190 CONTINUE
200 CONTINUE
210 CONTINUE
    D=A(NOE,NOE)*D
    IF (A(NOE,NOE).EQ.0.0) GO TO 260
    Z(NOE)=1.0/A(NOE,NOE)
```

C. . OBTAIN SOLUTION BY BACK SUBSTITUTION.

```
C
    DO 220 L=1,N1
      B(NOE,L)=Z(NOE)*B(NOE,L)
220 CONTINUE
    DO 250 K=1,NM1
      J=NOE-K
      J1=J+1
      DO 240 L=1,N1
        W=0.0
        DO 230 I=J1,NOE
          W=A(I,J)*B(I,L)+W
230 CONTINUE
        B(J,L)=(B(J,L)-W)*Z(J)
240 CONTINUE
250 CONTINUE
    IF (ABS(D).LT.ERR) GO TO 270
```

C JULIET=0

C. . NO PROBLEMS DURING THIS EXECUTION.

C RETURN

C. . SINGULAR MATRIX--MAXIMUM ELEMENT IN ROW IS ZERO.

```
C
260 JULIET=3
    WRITE (OUTPUT,280)
    RETURN
```

C. . ABSOLUTE DETERMINANT VALUE LESS THAN ERROR VALUE.

```
C
270 JULIET=1
    WRITE (OUTPUT,290) D,ERR
    RETURN
```

```
C
280 FORMAT (20H0*** SINGULAR MATRIX)
290 FORMAT (18H0*** DETERMINANT =,1PE13.5,24H, ERROR SPECIFICATION =,
1E13.5)
```

C END

```

FUNCTION LNEGT(M, N, N1, A, B, ERROR , Z)
C
C.. SOLVES SIMULTANEOUS LINEAR EQUATIONS BY GAUSSIAN REDUCTION.
C.. SOLVES   A*X=B   FOR   X , AND STORES THE   X   VECTOR(S) IN   B .
C
REAL A(M, M), B(M, M) , Z(M), ERROR , RMAX, RNEXT, W
C
C5
C5 INPUT AND OUTPUT LOGICAL UNITS.
COMMON/IO/INPUT, OUTPUT, BIF, OFF, COF
INTEGER OUTPUT, BIF, OFF, COF
C5
C
D=1.0
NM1=N-1
IF (NM1.EQ.0) GO TO 210
DO 200 J=1, NM1
  J1=J+1
C
C.. FIND ELEMENT OF COL J, ROWS J-N, WHICH HAS MAX ABSOLUTE VALUE.
C
LMAX=J
RMAX=ABS(A(J, J))
DO 110 K=J1, N
  RNEXT=ABS(A(K, J))
  IF (RMAX.GE. RNEXT) GO TO 110
  RMAX=RNEXT
  LMAX=K
110 CONTINUE
IF (LMAX.NE. J) GO TO 120
C
C.. MAX ELEMENT IN COLUMN IS ON DIAGONAL
C
IF (A(J, J)) 150, 270, 150
C
C.. MAX ELEMENT IS NOT ON DIAGONAL. EXCHANGE ROWS J AND LMAX.
C
120 DO 130 L=J, N
  W=A(J, L)
  A(J, L)=A(LMAX, L)
  A(LMAX, L)=W
130 CONTINUE
DO 140 L=1, N1
  W=B(J, L)
  B(J, L)=B(LMAX, L)
  B(LMAX, L)=W
140 CONTINUE
D=-D
C
C.. ZERO COLUMN J BELOW THE DIAGONAL.
C
150 D=A(J, J)*D
Z(J)=1.0/A(J, J)
DO 190 K=J1, N
  IF (A(K, J)) 160, 190, 160
160 W=-Z(J)*A(K, J)
DO 170 L=J1, N
  A(K, L)=W*A(J, L)+A(K, L)
170 CONTINUE
DO 180 L=1, N1
  B(K, L)=W*B(J, L)+B(K, L)

```

```

180 CONTINUE
190 CONTINUE
200 CONTINUE
210 D=A(N,N)*D
      IF (A(N,N).EQ.0.0) GO TO 270
      Z(N)=1./A(N,N)
C
C . . . OBTAIN SOLUTION BY BACK SUBSTITUTION.
C
      DO 220 L=1,N1
        B(N,L)=Z(N)*B(N,L)
220 CONTINUE
      IF (NM1.EQ.0) GO TO 260
      DO 250 K=1,NM1
        J=N-K
        J1=J+1
        DO 240 L=1,N1
          W=0.
          DO 230 I=J1,N
            W=A(J,I)*B(I,L)+W
230 CONTINUE
            B(J,L)=(B(J,L)-W)*Z(J)
240 CONTINUE
250 CONTINUE
260 LNEGT=0
      IF (ABS(D).GE.ERROR) RETURN
      LNEGT=1
      WRITE (OUTPUT,280) D,ERROR
      RETURN
C
C . . . SINGULAR MATRIX--MAXIMUM ELEMENT IN COLUMN IS ZERO.
C
270 LNEGT=3
      WRITE (OUTPUT,290)
      RETURN
C
280 FORMAT (18H0*** DETERMINANT =,1PE13.5,24H, ERROR SPECIFICATION =,
1E13.5)
290 FORMAT (20H0*** SINGULAR MATRIX)
C
      END

```

PROGRAM HYDREX3

PROGRAM HYDREX3

C

```

CHARACTER*25 PANFIL, MATFIL
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120),
* ST(120), ACN(120), ACNW(120), AN(120, 3), E(120), P(120, 6),
* PRFS(120), STOLD(120), PX(120, 6)
COMMON/FS/AKZ(100, 100), SS(100, 100), CC(100, 100),
* DKX(100), DKY(100), AKX(100), AKY(100)
COMMON/BD2/XPT(150), YPT(150), ZPT(150), WRF(150),
* WRFR(150), KK(150, 4)
COMMON/A/NPAN, NPT, GEE, RHO, NKX, NKY, EYE, DT, TIM, UFWD
DIMENSION PF(6), PB(6), PT(6)
COMMON/WAVEX/OMEGA
COMMON/BP/BPRES(120), TPRES(120)
COMPLEX EYE
EYE=(0.0, 1.0)

```

C

```

-----
TYPE 1
ACCEPT 4, PANFIL
TYPE 2
ACCEPT 4, MATFIL
OPEN(UNIT=2, FILE=PANFIL, TYPE='OLD')
OPEN(UNIT=3, FILE=MATFIL, FORM='UNFORMATTED', TYPE='NEW')
OPEN(UNIT=99, FILE='X.DAT', FORM='UNFORMATTED', TYPE='NEW')
CALL EBD
CALL POTST
WRITE(6, 6)
WRITE(6, 7) J, AN(J, 1), AN(J, 2), AN(J, 3),
* XPAN(J), YPAN(J), ZPAN(J), AREA(J)
150 * CONTINUE
STOP
1 FORMAT(' Input name of [PAN] file > '$)
2 FORMAT(' Input name of [MAT] file > '$)
4 FORMAT(A)
6 * FORMAT('J', 9X, 'NX', 9X, 'NY', 9X, 'NZ', 9X, 'XP', 9X,
* 'YP', 9X, 'ZP', 9X, 'AREA')
7 FORMAT(1X, I5, 7F11.4)
END

```



```

SUBROUTINE PREP(J)
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120), ST(120),
* ACN(120), ACNW(120), AN(120,3), E(120), P(120,6), PRFS(120),
* STOLD(120), PX(120,6)
COMMON/BD2/XPT(150), YPT(150), ZPT(150), WRF(150), WRFR(150),
* KK(150,4)
COMMON/ARE/RR(500), XZJ(200), YXJ(200), ZYJ(200)
ZYT=0.0
YXT=0.00
XZT=0.00
J4=J*4
JT=4
IF(KK(J,4).EQ.0) JT=3
DO 20 JJ=1, JT
J4=J4+1
J2=1
IF(JJ.LT.JT) J2=JJ+1
KF=KK(J, JJ)
KG=KK(J, J2)
AG=XPT(KG)
BG=YPT(KG)
CG=ZPT(KG)
AF=XPT(KF)
BF=YPT(KF)
CF=ZPT(KF)
R=SQRT((AF-AG)**2+(BF-BG)**2+(CF-CG)**2)
XT=AF-XPAN(J)
YT=BF-YPAN(J)
ZT=CF-ZPAN(J)
ANX=(AF-AG)/R
ANY=(BF-BG)/R
ANZ=(CF-CG)/R
DOT=ANX*XT+ANY*YT+ANZ*ZT
XT=XT-DOT*ANX
YT=YT-DOT*ANY
ZT=ZT-DOT*ANZ
ZYT=ZYT+ZT*ANY-ANZ*YT
YXT=YXT+YT*ANX-ANY*XT
XZT=XZT+XT*ANZ-ANX*ZT
RR(J4)=R
CONTINUE
XZJ(J)=SIGN(AN(J,2), XZT)
YXJ(J)=SIGN(AN(J,3), YXT)
ZYJ(J)=SIGN(AN(J,1), ZYT)
RETURN
END

```

20

```

SUBROUTINE SELF(AF, BF, CF, AG, BG, CG, FEE)
REAL LB21, LA21
ASQ=AF*AF+BF*BF+CF*CF
BSQ=AG*AG+BG*BG+CG*CG
ADB=AF*AG+BF*BG+CF*CG
ADB2=ADB+ADB
ASAS=(AF*BG-BF*AG)**2+(CF*BG-BF*CG)**2+(AF*CG-BF*AG)**2
FF=0.00
DO 15 MK=1, 10
DO 15 NK=1, MK
LA21=FLOAT(NK-MK)
A2SQ=ASQ*LA21*LA21
DO 15 ML=1, 11-MK
DO 15 NL=1, 11-ML
LB21=FLOAT(NL-ML)
IF(LA21.NE.0.) GO TO 5
IF(LB21.LT.0.) GO TO 5
GO TO 15
5 R=SQRT(A2SQ+ADB2*LA21*LB21+BSQ*LB21*LB21)
FF=FF+1.0/R
15 CONTINUE
FEE=FF*ASAS*0.002
RETURN
END

```

5

15

```

SUBROUTINE SOLID(XPN,G,NSIDE)
DIMENSION CS(4),SN(4),Z(4),XPN(3,4)
G=-6.283185308
ACR12=XPN(1,1)*XPN(1,2)+XPN(2,1)*XPN(2,2)+XPN(3,1)*XPN(3,2)
ACR13=XPN(1,1)*XPN(1,3)+XPN(2,1)*XPN(2,3)+XPN(3,1)*XPN(3,3)
ACR23=XPN(1,2)*XPN(1,3)+XPN(2,2)*XPN(2,3)+XPN(3,2)*XPN(3,3)
IF(NSIDE.EQ.4) GO TO 40
G=-3.141592659
CS(1)=ACR23-ACR13*ACR12
CS(2)=ACR13-ACR12*ACR23
CS(3)=ACR12-ACR23*ACR13
SN(1)=XPN(1,1)*(XPN(2,2)*XPN(3,3)-XPN(3,2)*XPN(2,3))+
+ XPN(2,1)*(XPN(3,2)*XPN(1,3)-XPN(1,2)*XPN(3,3))+
+ XPN(3,1)*(XPN(1,2)*XPN(2,3)-XPN(2,2)*XPN(1,3))
SN(2)=SN(1)
SN(3)=SN(1)
SN(4)=0.
GO TO 50
40 ACR14=XPN(1,1)*XPN(1,4)+XPN(2,1)*XPN(2,4)+XPN(3,1)*XPN(3,4)
ACR24=XPN(1,2)*XPN(1,4)+XPN(2,2)*XPN(2,4)+XPN(3,2)*XPN(3,4)
ACR34=XPN(1,3)*XPN(1,4)+XPN(2,3)*XPN(2,4)+XPN(3,3)*XPN(3,4)
CS(1)=ACR24-ACR14*ACR12
CS(2)=ACR13-ACR23*ACR12
CS(3)=ACR24-ACR34*ACR23
CS(4)=ACR13-ACR34*ACR14
B241=XPN(2,2)*XPN(3,4)-XPN(3,2)*XPN(2,4)
B242=XPN(3,2)*XPN(1,4)-XPN(1,2)*XPN(3,4)
B243=XPN(1,2)*XPN(2,4)-XPN(2,2)*XPN(1,4)
B131=XPN(2,1)*XPN(3,3)-XPN(3,1)*XPN(2,3)
B132=XPN(3,1)*XPN(1,3)-XPN(1,1)*XPN(3,3)
B133=XPN(1,1)*XPN(2,3)-XPN(2,1)*XPN(1,3)
SN(1)=XPN(1,1)*B241+XPN(2,1)*B242+XPN(3,1)*B243
SN(2)=-XPN(1,2)*B131+XPN(2,2)*B132+XPN(3,2)*B133
SN(3)=-XPN(1,3)*B241+XPN(2,3)*B242+XPN(3,3)*B243
SN(4)=XPN(1,4)*B131+XPN(2,4)*B132+XPN(3,4)*B133
50 CONTINUE
D TYPE 8844,SN(1),CS(1)
D TYPE 8844,SN(2),CS(2)
D TYPE 8844,SN(3),CS(3)
D TYPE 8844,SN(4),CS(4)
8844 FORMAT(' SN,CS=',2F15.8)
SUM=SN(1)+SN(2)+SN(3)+SN(4)
IF(ABS(SUM).GT.0.01) GO TO 25
IF(ABS(CS(1)).GT.ABS(SN(1))) GO TO 25
IF(ABS(CS(2)).GT.ABS(SN(2))) GO TO 25
IF(ABS(CS(3)).GT.ABS(SN(3))) GO TO 25
1090 G=SUM*.25
IF(NSIDE.EQ.3) G=SUM*.1666666667
RETURN
25 ST=SN(NSIDE)
DO 30 I=1,NSIDE
IF((ABS(CS(I)).LT.9E-8).AND.(ABS(SN(I)).LT..9E-05))
+ GO TO 1090
IF(ST*SN(I).LT.0.) GO TO 1090
ST=SN(I)
C2=CS(I)/SQRT(SN(I)**2+CS(I)**2)
G=G+ACOS(C2)
30 CONTINUE
RETURN
END

```

```

SUBROUTINE EBD
C INITIALIZE PANELS AND COMPUTE BODY MATRIX
C
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120), ST(120),
* ACN(120), ACNW(120), AN(120, 3), E(120), P(120, 6), PRFS(120),
* STOLD(120), PX(120, 6)
COMMON/BD2/XPT(150), YPT(150), ZPT(150), WRF(150), WRFR(150),
* KK(150, 4)
COMMON/A/NPAN, NPT, GEE, RHO, NKX, NKY, EYE, DT, TIM, UFWD
DIMENSION EP(120), EPP(120)
COMPLEX EYE
C READ IN BODY PANEL PARAMETERS
101 FORMAT(4I5)
100 FORMAT(3F10.0)
103 FORMAT(3F10.2)
104 FORMAT(I3)
C NUMBER OF POINTS AND PANELS
READ(2, 101) NPT, NPAN
TYPE 101, NPT, NPAN
C COORDIANTES OF POINTS
READ(2, 100) (XPT(N), YPT(N), ZPT(N), N=1, NPT)
DO 7777 N=1, NPT
TYPE 103, XPT(N), YPT(N), ZPT(N)
7777 CONTINUE
C DEFINE CORNER PINTS OF EACH PANEL
READ(2, 101) (KK(N, 1), KK(N, 2), KK(N, 3), KK(N, 4), N=1, NPAN)
C COMPUTE PANEL AREAS
DO 150 J=1, NPAN
K1=KK(J, 1)
K2=KK(J, 2)
K3=KK(J, 3)
K4=KK(J, 4)
IF(K4.EQ.0) GO TO 8
XPAN(J)=(XPT(K1)+XPT(K2)+XPT(K3)+XPT(K4))*0.25
YPAN(J)=(YPT(K1)+YPT(K2)+YPT(K3)+YPT(K4))*0.25
ZPAN(J)=(ZPT(K1)+ZPT(K2)+ZPT(K3)+ZPT(K4))*0.25
GO TO 9
C TRIANGULAR PANELS
8 XPAN(J)=(XPT(K1)+XPT(K2)+XPT(K3))/3.0
YPAN(J)=(YPT(K1)+YPT(K2)+YPT(K3))/3.0
ZPAN(J)=(ZPT(K1)+ZPT(K2)+ZPT(K3))/3.0
K4=K3
9 XA=XPT(K3)-XPT(K1)
XB=XPT(K4)-XPT(K2)
YA=YPT(K3)-YPT(K1)
YB=YPT(K4)-YPT(K2)
ZA=ZPT(K3)-ZPT(K1)
ZB=ZPT(K4)-ZPT(K2)
C COMPUTE PANEL AREAS
AZ=XA*YB-YA*XB
AX=YA*ZB-ZA*YB
AY=ZA*XB-XA*ZB
ARE=SQRT(AX*AX+AY*AY+AZ*AZ)
AREA(J)=ARE*0.50
AN(J, 1)=-AX/ARE
AN(J, 2)=-AY/ARE
AN(J, 3)=-AZ/ARE
150 CONTINUE
808 FORMAT(1X, I5, 5F11.4)
DO 1308 J=1, NPAN
JJJ=J
CALL PREP(JJJ)
ST(J)=0.00
DO 1308 K=1, 6
PX(J, K)=0.00
1308 CONTINUE
DO 308 J=1, NPAN
JJJ=J
AX=-AN(J, 1)
AY=-AN(J, 2)
AZ=-AN(J, 3)
XF=XPAN(J)
YF=YPAN(J)
ZF=ZPAN(J)
DO 157 L=1, NPT
WRF(L)=SQRT((XPT(L)-XF)**2+(YPT(L)-YF)**2+(ZPT(L)-ZF)**2)
157 WRFR(L)=SQRT((XPT(L)-XF)**2+(YPT(L)-YF)**2+(ZPT(L)+ZF)**2)

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```

DO 500 I=1,3
500 WRITE(3) (AN(J, I), J=1, NPAN)
CONTINUE
WRITE(3) (XPAN(J), J=1, NPAN)
WRITE(3) (YPAN(J), J=1, NPAN)
WRITE(3) (ZPAN(J), J=1, NPAN)
WRITE(3) (AREA(J), J=1, NPAN)
DO 309 JL=1, NPAN
JLJ=JL
CALL GE(XF, YF, ZF, JLJ, VX, VY, VZ, VXR, VYR, VZR, JJJ)
VX=VX+VXR
VY=VY+VYR
VZ=VZ+VZR
C COMPUTE NORMAL VELOCITY AT PANEL J DUE TO PANEL JL
E(JL)=AX*VX+AY*VY+AZ*VZ
C INCREMENT PX MATRIX
FR1=-AREA(J)*VX*AN(J, 1)
FR2=-AREA(J)*VX*AN(J, 2)
FR3=-AREA(J)*VX*AN(J, 3)
PX(JL, 1)=PX(JL, 1)+FR1
PX(JL, 2)=PX(JL, 2)+FR2
PX(JL, 3)=PX(JL, 3)+FR3
PX(JL, 4)=PX(JL, 4)+YF*FR3-ZF*FR2
PX(JL, 5)=PX(JL, 5)+ZF*FR1-XF*FR3
PX(JL, 6)=PX(JL, 6)+XF*FR2-YF*FR1
309 CONTINUE
WRITE(99) (E(JL), JL=1, NPAN)
308 CONTINUE
DO 2424 K=1, 6
2424 WRITE(3) (PX(JL, K), JL=1, NPAN)
CLOSE (UNIT=99)
C INVERT E MATRIX
CALL MATIN(NPAN)
RETURN
END

```

```

SUBROUTINE MATIN(NPAN)
C INVERST MATRIX
DIMENSION E(120, 120), BB(120), EST(120)
OPEN(UNIT=99, FILE='SCR', FORM='UNFORMATTED', TYPE='OLD')
DO 120 J=1, NPAN
120 READ(99) (E(J, I), I=1, NPAN)
DO 130 J=1, NPAN
DO 11 MM=1, NPAN
11 EST(MM)=0.00
BB(MM)=0.00
BB(J)=1.0
EST(J)=1.0/E(J, J)
DO 17 NIT=1, 6
DO 17 K=1, NPAN
B=BB(K)
DO 15 I=1, NPAN
15 IF(I.NE.K) B=B-E(K, I)*EST(I)
EST(K)=B/E(K, K)
17 CONTINUE
WRITE(3) (EST(K), K=1, NPAN)
130 CONTINUE
RETURN
END

```

C

```

SUBROUTINE GE(XF, YF, ZF, J, V1, V2, V3, V1R, V2R, V3R, NBT)
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120), ST(120),
* ACN(120), ACNW(120), AN(120, 3), E(120), P(120, 6), PRFS(120),
* STOLD(120), PX(120, 6)
COMMON/BD2/XPT(150), YPT(150), ZPT(150), WRF(150), WRFR(150)
* , KK(150, 4)
COMMON/ARE/RR(500), XZJ(200), YXJ(200), ZYJ(200)
DIMENSION XSA(3, 4), XFA(3), XSAR(3, 4)
J4=J*4
V1=0.00
V2=0.00
V3=0.00
V1R=0.00
V2R=0.00
V3R=0.00
XNJ=AN(J, 1)
YNJ=AN(J, 2)
ZNJ=AN(J, 3)
NSIDE=4
IF(KK(J, 4).EQ.0) NSIDE=3
DO 20 JJ=1, NSIDE
J2=1
IF(JJ.LT.NSIDE) J2=JJ+1
J4=J4+1
KF=KK(J, JJ)
AF=XPT(KF)
BF=YPT(KF)
CF=ZPT(KF)
R=RR(J4)
KG=KK(J, J2)
ANX=(AF-XPT(KG))/R
ANY=(BF-YPT(KG))/R
ANZ=(CF-ZPT(KG))/R
A=AF-XF
B=BF-YF
C=CF-ZF
TX=XZJ(J)*ANZ-YXJ(J)*ANY
TY=YXJ(J)*ANX-ZYJ(J)*ANZ
TZ=ZYJ(J)*ANY-XZJ(J)*ANX
EX1=A*ANX+B*ANY+C*ANZ
CALL GO(EX1, R, FF, WRF(KF), WRF(KG))
V1=V1+FF*TX
V2=V2+FF*TY
V3=V3+FF*TZ
XSA(1, JJ)=-A/WRF(KF)
XSA(2, JJ)=-B/WRF(KF)
XSA(3, JJ)=-C/WRF(KF)
EX1R=EX1+2.0*ZF*ANZ
CR=-CF-ZF
CALL GO(EX1R, R, FR, WRFR(KF), WRFR(KG))
V1R=V1R-FR*TX
V2R=V2R-FR*TY
V3R=V3R+FR*TZ
XSAR(1, JJ)=-A/WRFR(KF)
XSAR(2, JJ)=-B/WRFR(KF)
XSAR(3, JJ)=CR/WRFR(KF)
20 CONTINUE
G=6.283185307
IF(J.EQ.NBT) GO TO B4
CALL SOLID(XSA, G, NSIDE)
AGG=A*XNJ+B*YNJ+C*ZNJ
G=-SIGN(G, AGG)
84 CONTINUE
CALL SOLID(XSAR, GR, NSIDE)
AGGR=A*XNJ+B*YNJ-CR*ZNJ
GR=SIGN(GR, AGGR)
85 CONTINUE
7371 FORMAT(' G, GR=', 2F15.5)
V1=V1+XNJ*G
V2=V2+YNJ*G
V3=V3+ZNJ*G
V1R=V1R+XNJ*GR
V2R=V2R+YNJ*GR
V3R=V3R-ZNJ*GR
5590 FORMAT(' V1, V2, V3=', 3F15.5)
5591 FORMAT(' V1R, V2R, V3R=', 3F15.5)
RETURN
END

```

C

```
SUBROUTINE POTST
COMMON/BD/XPAN(120), YPAN(120), ZPAN(120), AREA(120), ST(120),
* ACN(120), ACNW(120), AN(120, 3), E(120), P(120, 6), PRFS(120),
* STOLD(120), PX(120, 6)
COMMON/BD2/XPT(150), YPT(150), ZPT(150), WRF(150), WRFR(150),
* KK(150, 4)
COMMON/A/NPAN, NPT, GEE, RHO, NKX, NKY, EYE, DT, TIM, UFWD
COMPLEX A, B, EYE
DIMENSION XPSL(3, 4), XPSLR(3, 4), PBB(120, 120)
COMMON/PTST/ARE4(200, 4), X4(200, 4), Y4(200, 4), Z4(200, 4)
* , SEL(200, 4)
DO 1500 J=1, NPAN
ARE4(J, 4)=-1.0
JT=4
IF(KK(J, 4).EQ.0) JT=3
DO 1500 JJ=1, JT
J2=1
IF(JJ.LT. JT) J2=JJ+1
KF=KK(J, JJ)
KG=KK(J, J2)
X4(J, JJ)=(XPT(KF)+XPT(KG)+XPAN(J))/3.0
Y4(J, JJ)=(YPT(KF)+YPT(KG)+YPAN(J))/3.0
Z4(J, JJ)=(ZPT(KF)+ZPT(KG)+ZPAN(J))/3.0
AF=XPT(KF)-XPAN(J)
BF=YPT(KF)-YPAN(J)
CF=ZPT(KF)-ZPAN(J)
AG=XPT(KG)-XPAN(J)
BG=YPT(KG)-YPAN(J)
CG=ZPT(KG)-ZPAN(J)
CALL SELF(AF, BF, CF, AG, BG, CG, FEE)
SEL(J, JJ)=FEE
CR=AF*BG-BF*AG
AR=BF*CG-CF*BG
BR=CF*AG-AF*CG
ARE4(J, JJ)=0.5*SQRT(AR*AR+BR*BR+CR*CR)
1500 CONTINUE
DO 1277 NJ=1, NPAN
DO 1277 MJ=1, NPAN
1277 PBB(NJ, MJ)=0.00
P(NJ, 1)=0.00
P(NJ, 2)=0.00
P(NJ, 3)=0.00
P(NJ, 4)=0.00
P(NJ, 5)=0.00
P(NJ, 6)=0.00
DO 128 NK=1, 4
ARN=ARE4(NJ, NK)
IF(ARN.LT.0.0) GO TO 128
P1=0.0
P2=0.0
P3=0.0
P4=0.0
P5=0.0
P6=0.0
X=X4(NJ, NK)
Y=Y4(NJ, NK)
Z=Z4(NJ, NK)
DO 138 MJ=1, NPAN
DO 138 MK=1, 4
XF=X4(MJ, MK)
YF=Y4(MJ, MK)
ZF=Z4(MJ, MK)
ARM=ARE4(MJ, MK)
IF(ARM.LT.0.00) GO TO 138
IF(NJ.NE. MJ) GO TO 140
IF(MK.NE. NK) GO TO 140
FRA=SEL(MJ, MK)/ARM
GO TO 1380
```

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